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**MID-ATLANTIC**

**TECHNICAL REFERENCE MANUAL**

**VERSION 6.0**

Final

May 2016

**About NEEP & the Regional EM&V Forum**



NEEP was founded in 1996 as a non-profit whose mission is to serve the Northeast and Mid-Atlantic to accelerate energy efficiency in the building sector through public policy, program strategies and education. Our vision is that the region will fully embrace energy efficiency as a cornerstone of sustainable energy policy to help achieve a cleaner environment and a more reliable and affordable energy system.

The Regional Evaluation, Measurement and Verification Forum (EM&V Forum or Forum) is a project facilitated by Northeast Energy Efficiency Partnerships, Inc. (NEEP). The Forum’s purpose is to provide a framework for the development and use of common and/or consistent protocols to measure, verify, track, and report energy efficiency and other demand resource savings, costs, and emission impacts to support the role and credibility of these resources in current and emerging energy and environmental policies and markets in the Northeast, New York, and the Mid-Atlantic region.

**About Shelter Analytics**

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**Shelter Analytics, LLC** is dedicated to promoting energy efficiency through planning and integrated design concepts in buildings and businesses. We combine our experience and integrity with innovative approaches to support and improve best-practice methods from planning through implementation.

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**MID-ATLANTIC**

**TECHNICAL REFERENCE MANUAL**

**VERSION 6.0**

**A Project of the Regional Evaluation, Measurement and Verification Forum**

May 2016

October 2010

**Prepared by Shelter Analytics**

**Facilitated and Managed by Northeast Energy Efficiency Partnerships**

**Table of Contents**

[The Regional EM&V Forum 8](#_Toc449959816)

[Acknowledgements 8](#_Toc449959817)

[Subcommittee for the Mid-Atlantic TRM 8](#_Toc449959818)

[Introduction 10](#_Toc449959819)

[**Context** 11](#_Toc449959820)

[**Approach** 11](#_Toc449959821)

[**Task 1: Prioritization/Measure Selection.** 12](#_Toc449959822)

[**Task 2: Development of Deemed Impacts.** 12](#_Toc449959823)

[**Task 3: Development of Recommendations for Update.** 14](#_Toc449959824)

[**Task 4: Delivery of Draft and Final Product.** 14](#_Toc449959825)

[**Use of the TRM** 14](#_Toc449959826)

[**TRM Update History** 19](#_Toc449959827)

[RESIDENTIAL MARKET SECTOR 20](#_Toc449959828)

[*Lighting End Use* 20](#_Toc449959829)

[**General Purpose CFL Screw base, Residential\*** 20](#_Toc449959830)

[**Specialty CFLs, Residential\*** 29](#_Toc449959831)

[**Hardwired CFL Fixtures (Interior)\*** 38](#_Toc449959832)

[**Hardwired CFL Fixtures (Exterior)\*** 46](#_Toc449959833)

[**Solid State Lighting (LED) Recessed Downlight Luminaire\*** 50](#_Toc449959834)

[**ENERGY STAR Integrated Screw Based SSL (LED) Lamp\*** 58](#_Toc449959835)

[*Refrigeration End Use* 70](#_Toc449959836)

[**Freezer** 70](#_Toc449959837)

[**Refrigerator** 76](#_Toc449959838)

[**Refrigerator Early Replacement** 82](#_Toc449959839)

[**Refrigerator and Freezer Early Retirement** 86](#_Toc449959840)

[*Heating Ventilation and Air Conditioning (HVAC) End Use* 91](#_Toc449959841)

[**Central Furnace Efficient Fan Motor** 91](#_Toc449959842)

[**Room Air Conditioner\*** 93](#_Toc449959843)

[**ENERGY STAR Central A/C\*** 97](#_Toc449959844)

[**Air Source Heat Pump\*** 103](#_Toc449959845)

[**Duct Sealing** 111](#_Toc449959846)

[**Ductless Mini-Split Heat Pump\*** 124](#_Toc449959847)

[**HE Gas Boiler** 134](#_Toc449959848)

[**Condensing Furnace (gas)\*** 137](#_Toc449959849)

[**Programmable Thermostat** 140](#_Toc449959850)

[**Room Air Conditioner Early Replacement** 142](#_Toc449959851)

[**Room Air Conditioner Early Retirement / Recycling** 146](#_Toc449959852)

[**Boiler Pipe Insulation** 149](#_Toc449959853)

[**Boiler Reset Controls\*** 152](#_Toc449959854)

[**Ground Source Heat Pumps** 155](#_Toc449959855)

[**High Efficiency Bathroom Exhaust Fan\*** 163](#_Toc449959856)

[**ENERGY STAR Ceiling Fan** 165](#_Toc449959857)

[*Domestic Hot Water (DHW) End Use* 170](#_Toc449959858)

[**Low Flow Shower Head** 170](#_Toc449959859)

[**Faucet Aerators** 175](#_Toc449959860)

[**Domestic Hot Water Tank Wrap** 180](#_Toc449959861)

[**DHW pipe insulation** 184](#_Toc449959862)

[**High Efficiency Gas Water Heater** 187](#_Toc449959863)

[**Heat Pump Domestic Water Heater** 191](#_Toc449959864)

[**Thermostatic Restrictor Shower Valve** 199](#_Toc449959865)

[**Water Heater Temperature Setback\*** 205](#_Toc449959866)

[*Appliance End Use* 209](#_Toc449959867)

[**Clothes Washer** 209](#_Toc449959868)

[**Clothes Washer Early Replacement** 219](#_Toc449959869)

[**Dehumidifier** 229](#_Toc449959870)

[**ENERGY STAR Air Purifier/Cleaner** 233](#_Toc449959871)

[**Clothes Dryer** 236](#_Toc449959872)

[**Dishwasher** 241](#_Toc449959873)

[*Shell Savings End Use* 247](#_Toc449959874)

[**Air sealing\*** 247](#_Toc449959875)

[**Attic/ceiling/roof insulation\*** 255](#_Toc449959876)

[**Efficient Windows - Energy Star Time of sale** 261](#_Toc449959877)

[**Crawl Space Insulation and Encapsulation\*\*** 263](#_Toc449959878)

[*Pool Pump End Use* 272](#_Toc449959879)

[**Pool pump-two speed** 272](#_Toc449959880)

[**Pool pump-variable speed\*** 275](#_Toc449959881)

[*Plug Load End Use* 278](#_Toc449959882)

[**Advanced Power Strip\*** 278](#_Toc449959883)

[*Retail Products Program* 281](#_Toc449959884)

[**ENERGY STAR +50% Soundbar\*\*** 281](#_Toc449959885)

[**Freezer\*\*** 283](#_Toc449959886)

[**Clothes Dryer\*\*** 286](#_Toc449959887)

[**ENERGY STAR Air Cleaner\*\*** 289](#_Toc449959888)

[**Room Air Conditioners (Upstream)\*\*** 291](#_Toc449959889)

[COMMERCIAL & INDUSTRIAL MARKET SECTOR 295](#_Toc449959890)

[*Lighting End Use* 295](#_Toc449959891)

[**General Purpose CFL Screw base, Retail – Commercial\*** 295](#_Toc449959892)

[**Four-Foot Fluorescent T8 Replacement Lamps\*** 301](#_Toc449959893)

[**T5 Lighting** 309](#_Toc449959894)

[**LED Exit Sign** 313](#_Toc449959895)

[**Solid State Lighting (LED) Recessed Downlight Luminaire** 317](#_Toc449959896)

[**Delamping** 322](#_Toc449959897)

[**Occupancy Sensor – Wall-, Fixture-, or Remote-Mounted** 325](#_Toc449959898)

[**Daylight Dimming Control** 329](#_Toc449959899)

[**Advanced Lighting Design – Commercial** 333](#_Toc449959900)

[**LED Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Lighting Luminaires and Retrofit Kits** 348](#_Toc449959901)

[**LED High-Bay Luminaires and Retrofit Kits** 353](#_Toc449959902)

[**LED 1x4, 2x2, and 2x4 Luminaires and Retrofit Kits** 357](#_Toc449959903)

[**LED Parking Garage/Canopy Luminaires and Retrofit Kits** 361](#_Toc449959904)

[**ENERGY STAR Integrated Screw Based SSL (LED) Lamp – Commercial\*** 366](#_Toc449959905)

[**LED Refrigerated Case Lighting** 375](#_Toc449959906)

[**Exterior LED Flood and Spot Luminaires** 379](#_Toc449959907)

[**LED Four-Foot Linear Replacement Lamps** 382](#_Toc449959908)

[*Heating Ventilation and Air Conditioning (HVAC) End Use* 387](#_Toc449959909)

[**Unitary HVAC Systems\*** 387](#_Toc449959910)

[**Ductless Mini-Split Heat Pump (DMSHP)** 400](#_Toc449959911)

[**Variable Frequency Drive (VFD) for HVAC\*** 407](#_Toc449959912)

[**Electric Chillers** 414](#_Toc449959913)

[**Gas Boiler** 423](#_Toc449959914)

[**Gas Furnace** 427](#_Toc449959915)

[**Dual Enthalpy Economizer** 430](#_Toc449959916)

[**AC Tune-Up** 433](#_Toc449959917)

[*Refrigeration End Use* 436](#_Toc449959918)

[**ENERGY STAR Commercial Freezers** 436](#_Toc449959919)

[**ENERGY STAR Commercial Refrigerator** 439](#_Toc449959920)

[**Night Covers for Refrigerated Cases** 442](#_Toc449959921)

[**Anti-Sweat Heater Controls\*** 445](#_Toc449959922)

[**Evaporator Fan Electronically-Commutated Motor (ECM) Retrofit\*\*** 448](#_Toc449959923)

[**Evaporator Fan Motor Controls\*\*** 451](#_Toc449959924)

[*Hot Water End Use* 454](#_Toc449959925)

[**C&I Heat Pump Water Heater** 454](#_Toc449959926)

[**Pre-Rinse Spray Valves** 457](#_Toc449959927)

[*Appliance End Use* 460](#_Toc449959928)

[**Commercial Clothes Washer\*\*** 460](#_Toc449959929)

[*Plug Load End Use* 466](#_Toc449959930)

[**Advanced Power Strip** 466](#_Toc449959931)

[*Commercial Kitchen Equipment End Use* 468](#_Toc449959932)

[**Commercial Fryers\*** 468](#_Toc449959933)

[**Commercial Steam Cookers** 472](#_Toc449959934)

[**Commercial Hot Food Holding Cabinets** 477](#_Toc449959935)

[**Commercial Griddles** 480](#_Toc449959936)

[**Commercial Convection Ovens** 484](#_Toc449959937)

[**Commercial Combination Ovens** 488](#_Toc449959938)

[**A.** **Supporting Calculation Work Sheets** 494](#_Toc449959939)

[**B.** **Recommendation for Process and Schedule for Maintenance and Update of TRM Contents** 495](#_Toc449959940)

[C. Description of Unique Measure Codes 502](#_Toc449959941)

[D. Commercial & Industrial Lighting Operating Hours, Coincidence Factors, and Waste Heat Factors 504](#_Toc449959942)

\* Measure was updated for this version of the TRM

\*\*Measure is newly added to this version of the TRM PREFACE

## The Regional EM&V Forum

The Regional EM&V Forum is a project managed and facilitated by Northeast Energy Efficiency Partnerships, Inc. The Forum’s purpose is to provide a framework for the development and use of common and/or consistent protocols to measure, verify, track and report energy efficiency and other demand resource savings, costs and emission impacts to support the role and credibility of these resources in current and emerging energy and environmental policies and markets in the Northeast and the Mid-Atlantic region. For more information, see http: [www.neep.org/emv-forum](http://www.neep.org/emv-forum).

## Acknowledgements

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## Subcommittee for the Mid-Atlantic TRM

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**Introduction**

This Technical Reference Manual is the outcome of a project conducted for the Regional Evaluation, Measurement and Verification Forum (‘the EMV Forum’) sponsored by Maryland, Delaware and the District of Columbia. The intent of the project was to develop and document in detail common assumptions for approximately thirty prescriptive residential and commercial/industrial electric energy efficiency measures savings. For each measure, the TRM includes either specific deemed values or algorithms[[1]](#footnote-1) for calculating:

* Gross annual electric energy savings;
* Gross electric summer coincident peak demand savings;
* Gross annual fossil fuel energy savings (for electric efficiency measures that also save fossil fuels, and for certain measures that can save electricity or fossil fuels);
* Other resource savings if appropriate (e.g. water savings, O&M impacts);
* Incremental costs; and
* Measure lives.

The TRM is intended to be easy to use and to serve a wide range of important users and functions, including:

* **Utilities and efficiency Program Administrators** – for cost-effectiveness screening and program planning, tracking, and reporting.
* **Regulatory entities, independent program evaluators, and other parties** – for evaluating the performance of efficiency programs relative to statutory goals and facilitating planning and portfolio review; and
* **Markets, such as PJM’s Reliability Pricing Model (its wholesale capacity market) and future carbon markets** – for valuing efficiency resources.

The TRM is intended to be a flexible and living document. To that end, NEEP, the project sponsors and the TRM authors all expect it to be periodically updated with additional measures, modifications to characterizations of existing measures and even removal of some measures when they are no longer relevant to regional efficiency programs. Initial recommendations for a process by which updates could occur are provided in Appendix B.

**Context**

The Forum initiated this project as a benefit to both the Mid-Atlantic States and the overall Forum Region, for the following reasons:

* To improve the credibility and comparability of energy efficiency resources to support state and regional energy, climate change and other environmental policy goals;
* To remove barriers to the participation of energy efficiency resources in regional markets by making EM&V practices and savings assumptions more transparent, understandable and accessible;
* To reduce the cost of EM&V activities by leveraging resources across the region for studies of common interest (where a need for such studies has been identified); and
* To inform the potential development of national EM&V protocols.

This is the sixth generation (fifth update) document that has been prepared for the Mid-Atlantic sponsors, and one of few in the country to serve a multi-jurisdictional audience. For definitions of many energy efficiency terms and acronyms included in the TRM, users of this TRM may want to refer to the EMV Forum Glossary available at:

<http://neep.org/emv-forum/forum-products-and-guidelines>. For measures which were common to both Forum projects (specifically residential and commercial lighting measures, residential central and commercial unitary air conditioning, and variable frequency drives) this TRM used the results of those Forum projects.

It is also recognized that programs mature over time and more evaluation and market-research data have become available over the past few years. In addition, efficiency programs in the region are not identical and either the availability or the results of existing baseline studies and other sources of information can differ across organizations and jurisdictions. Also, different budgets and policy objectives exist, and states may have different EM&V requirements and practices. Given these considerations, the contents of this TRM reflect the consensus agreement and best judgment of project sponsors, managers, and consultants on information that was most useful and appropriate to include within the time, resource, and information constraints of the study.

**Approach**

This section briefly identifies and describes the process used to develop the TRM. In addition, it provides an overview of some of the considerations and decisions involved in the development of estimates for the many parameters. The development of this TRM required a balance of effectiveness, functionality, and relevance with available sources and research costs.

It is helpful to keep in mind that each measure characterization has numerous components, including retrofit scenario, baseline consumption, annual energy savings, coincident peak demand savings, useful life, and incremental cost.

Thus, the project needed to research and develop literally hundreds of unique assumptions. It is further helpful to keep in mind that because the project served a multijurisdictional audience, it required data requests, review, and consensus decision-making by a subcommittee comprised of project sponsors and other stakeholders (see the end of this Introduction for a list of subcommittee members). The subcommittee was responsible for review and approval of the products generated in each of the tasks needed to complete the project.

Development of the TRM consisted of the following tasks:

**Task 1: Prioritization/Measure Selection.**

By design, this TRM focuses on priority prescriptive measures, due to a combination of project resource constraints and the recognition that typically 10 - 20% of a portfolio of efficiency measures (such as CFLs, T8s or super-T8s, some cooling measures, efficient water heaters) likely account for the large majority (90% or more) of future savings claims from prescriptive measures (i.e., those measures effectively characterized by pre-determined incentive and deemed savings values or algorithms ).).

Measures are selected on the basis of projected or expected savings from program data by measure type provided by Baltimore Gas and Electric, expert judgment, and review of other relevant criteria available from regulatory filings and the region’s Program Administrators. Note that some of the measures are variations on other measures (e.g. two different efficiency tiers for room air conditioners). Because gas measures were not common to all sponsors, these are not priority measures, but there is consensus that gas measures are appropriate to include. For those measures where fossil fuel savings occur in addition to electricity savings (for example the clothes washer measure), or where either electric or fossil fuel savings could be realized depending on the heating fuel used (for example domestic hot water conservation measures), appropriate MMBtu savings have been provided.

**Task 2: Development of Deemed Impacts.**

Development of the contents of the TRM proceeds in two stages. The first stage is research, analysis, and critical review of available information to inform the range of assumptions considered for each parameter and each measure included in the TRM. This is based on a comparative study of many secondary sources including existing TRMs from other jurisdictions, local, primary research and data, and information that was developed for the EMV Forum Product “A2” (Common Methods Project).

The comparative analysis itself is not always as straightforward as it might initially seem because the measures and specific variables included in different jurisdictions’ TRMs are sometimes a little different from each other – in efficiency levels promoted, capacity levels considered, the design of program mechanisms for promoting the measures and various other factors. Thus, the comparative analysis of many assumptions requires calibration to common underlying assumptions. Wherever possible, such underlying assumptions – particularly for region-specific issues such as climate, codes and key baseline issues –are derived from the mid-Atlantic region

The second stage is development of specific recommendations for specific assumptions or algorithms (informed by the comparative analysis), along with rationales and references for the recommendations. These recommended assumptions identify cases where calculation of savings is required and where options exist (for example two coincidence factor values are provided for central AC measures, based on two definitions of peak coincidence factors) for calculation of impact. They also recommend deemed values where consistency can or should be achieved. The following criteria are used in the process of reviewing and adopting the proposed assumptions and establishing consensus on the final contents of the TRM:

* **Credibility.** The savings estimates and any related estimates of the cost-effectiveness of efficiency investments are credible.
* **Accuracy and completeness.** The individual assumptions or calculation protocols are accurate, and measure characterizations capture the full range of effects on savings.
* **Transparency.** The assumptions are considered by a variety of stakeholders to be transparent – that is, widely-known, widely accepted, and developed and refined through an open process that encourages and addresses challenges from a variety of stakeholders.
* **Cost efficiency.** The contents of the TRM addressed all inputs that were within the established project scope and constraints. Sponsors recognize that there are improvements and additions that can be made in future generations of this document.

Additional notes regarding the high level rationale for extrapolation for Mid-Atlantic estimates from the Northeast and other places are provided below under Intended Uses of the TRM.

**Task 3: Development of Recommendations for Update.**

The purpose of this task was to develop a recommended process for when and how information will be incorporated into the TRM in the future. This task assumes that the process of updating and maintaining the TRM is related to but distinct from processes for verification of annual savings claims by Program Administrators. It further assumes that verification remains the responsibility of individual organizations unlike the multi-sponsor, multi-jurisdictional TRM. The development of these recommendations was based on the following considerations:

* Review processes in other jurisdictions and newly available relevant research and data.
* Expected uses of the TRM. This assumes that the TRM will be used to conduct prospective cost-effectiveness screening of utility programs, to estimate progress towards goals and potentially to support bidding into capacity markets. Note that both the contents of the document and the process and timeline by which it is updated might need to be updated to conform to the PJM requirements, once sponsors have gained additional experience with the capacity market.
* Expected timelines required to implement updates to the TRM parameters and algorithms.
* Processes stakeholders envision for conducting annual reviews of utility program savings as well as program evaluations, and therefore what time frame TRM updates can accommodate these.
* Feasibility of merging or coordinating the Mid-Atlantic protocols with those of other States, such as Pennsylvania, New Jersey or entire the Northeast.

**Task 4: Delivery of Draft and Final Product.**

The final content of the TRM reflects the consensus approval of the results from Task 2 as modified following a peer review. By design, the final version of the TRM document is similar to other TRMs currently available, for ease of comparison and update and potential merging with others in the future.

**Use of the TRM**

As noted above, The TRM is intended to serve as an important tool to support rate-funded efficiency investments; for planning, implementation and assessment of success in meeting specific state goals. In addition, the TRM is intended to support the bidding of efficiency resources into capacity markets, such as PJM’s Reliability Pricing Model and in setting and tracking future environmental and climate change goals. It provides a common platform for the Mid-Atlantic stakeholders to characterize measures within their efficiency programs, analyze and meaningfully compare cost-effectiveness of measures and programs, communicate with policymakers about program details, and it can guide future evaluation and measurement activity and help identify priorities for investment in further study, needed either at a regional or individual organizational level.

The savings estimates are expected to serve as representative, recommended values, or ways to calculate savings based on program-specific information. All information is presented on a per measure basis. In using the measure-specific information in the TRM, it is helpful to keep the following notes in mind:

* The TRM clearly identifies whether the measure impacts pertain to “retrofit”, “time of sale”,[[2]](#footnote-2) or “early retirement” program designs.
* Additional information about the program design is sometimes included in the measure description because program design can affect savings and other parameters.
* Savings algorithms are typically provided for each measure. For a number of measures, prescriptive values for each of the variables in the algorithm are provided along with the output from the algorithm. That output is the deemed savings. For other measures, prescriptive values are provided for only some of the variables in the algorithm, with the term “actual” or “actual installed” provided for the others. In those cases – which one might call “deemed calculations” rather than “deemed savings” – users of the TRM are expected to use actual efficiency program data (e.g. capacities or rated efficiencies of central air conditioners) in the formula to compute savings. Note that the TRM typically provides *example calculations* for measures requiring “actual” values. These are for illustrative purposes only.
* All estimates of savings are annual savings and are assumed to be realized for each year of the measure life (unless otherwise noted).
* Unless otherwise noted, measure life is defined to be “The life of an energy consuming measure, including its equipment life and measure persistence (not savings persistence)” (EMV Forum Glossary). Conceptually it is similar to expected useful life, but the results are not necessarily derived from modeling studies, and many are from a report completed for New England program administrators’ and regulators’ State Program Working Group that is currently used to support the New England Forward Capacity Market M&V plans.
* Where deemed values for savings are provided, these represent average savings that could be expected from the average measures that might be installed in the region during the current program year.
* For measures that are not weather-sensitive, peak savings are estimated whenever possible as the average of savings between 2 pm and 6 pm across all summer weekdays (i.e. PJM’s EE Performance Hours for its Reliability Pricing Model). Where possible for cooling measures, we provide estimates of peak savings in two different ways. The primary way is to estimate peak savings during the most typical peak hour (assumed here to be 5 p.m.) on days during which system peak demand typically occurs (i.e., the hottest summer weekdays). This is most indicative of actual peak benefits. The secondary way – typically provided in a footnote – is to estimate peak savings as it is measured for non-cooling measures: the average between 2 pm and 6 pm across *all* summer weekdays (regardless of temperature). The second way is presented so that values can be bid into the PJM RPM.
* Wherever possible, savings estimates and other assumptions are based on mid-Atlantic data. For example, data from a BG&E metering study of residential central air conditioners was used to estimate both full load hours and system peak coincidence factors. However, a number of assumptions – including assumptions regarding peak coincidence factors – are based on New York and/or New England sources. While this information is not perfectly transferable, due to differences in definitions of peak periods as well as geography,climate and customer mix, it was used because it was the most transferable and usable source available at the time.[[3]](#footnote-3)
* Users will note that the TRM presents engineering equations for most measures. These were judged to be desirable because they convey information clearly and transparently, and they are widely accepted in the industry. Unlike simulation model results, they also provide flexibility and opportunity for users to substitute locally specific information and to update some or all parameters as they become available on an ad hoc basis. One limitation is that certain interactive effects between end uses, such as how reductions in waste heat from many efficiency measures impacts space conditioning, are not universally captured in this version of the TRM.[[4]](#footnote-4)
* For some of the whole-building program designs that are being planned or implemented in the Mid-Atlantic, simulation modeling may be needed to estimate savings. While they were beyond the scope of this TRM, it is recommended that a future version of the TRM may include the baseline specifications for any whole-building efficiency measures.
* In general, the baselines included in the TRM are intended to represent average conditions in the Mid-Atlantic. Some are based on data from the Mid-Atlantic, such as household consumption characteristics provided by the Energy Information Administration. Some are extrapolated from other areas, when Mid-Atlantic data are not available. Some are based on code.
* The TRM anticipates the effects of changes in efficiency standards for some measures, specifically incandescent lamps, CFLs and motors.

The following table outlines the terms used to describe programs with respect to when and how a measure is implemented. The third portion of each measure code for each measure described in this TRM includes the abbreviation of the program type for which the characterization is intended:

| **Program** | **Attributes** |
| --- | --- |
| Time of Sale (TOS) | Definition: A program in which the customer is incented to purchase or install higher efficiency equipment than if the program had not existed. This may include retail rebate (coupon) programs, upstream buydown programs, online store programs, contractor based programs, or CFL giveaways as examples. Also applies to End of Life  Baseline = New standard efficiency or code compliant equipment.  Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice.  Example: CFL rebate |
| New Construction (NC) | Definition: A program that intervenes during building design to support the use of more-efficient equipment and construction practices.  Baseline = Building code or federal standards.  Efficient Case = The program’s level of building specification  Example: Building shell and mechanical measures |
| Retrofit (RF) | Definition: A program that *upgrades* existing equipment before the end of its useful life.  Baseline = Existing equipment or the existing condition of the building or equipment.  A single baseline applies over the measure’s life.  Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice.  Example: Air sealing and insulation |
| Early Replacement (EREP) | Definition: A program that *replaces* existing equipment before the end of its expected life.  Baseline = Dual; it begins as the existing equipment and shifts to new baseline equipment after the expected life of the existing equipment is over.  Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice.  Example: Refrigerators, freezers |
| Early Retirement (ERET) | Definition: A program that *retires* duplicative equipment before its expected life is over.  Baseline = The existing equipment, which is retired and not replaced.  Efficient Case = Zero because the unit is retired.  Example: Appliance recycling |
| Direct Install (DI) | Definition: A program where measures are installed during a site visit.  Baseline = Existing equipment.  Efficient Case = New, premium efficiency equipment above federal and state codes and standard industry practice.  Example: Lighting and low-flow hot water measures |

Going forward, the project sponsors can use this TRM, along with other Forum products on common EM&V terminology, guidelines on common evaluation methods, and common reporting formats, along with the experience gained from implementation of the efficiency programs to inform decisions about what savings assumptions should be updated and how.

**TRM Update History**

Version                                Issued

1.1         October 2010

1.2         March 2011

2.0 July 2011

3.0 January 2013

4.0 June 2014

5.0 June 2015

6.0 May 2016

**RESIDENTIAL MARKET SECTOR**

*Lighting End Use*

**General Purpose CFL Screw base, Residential\***

**Unique Measure Code(s): RS\_LT\_TOS\_CFLSCR\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure characterizes the installation of a general purpose compact fluorescent light bulb (CFL) in place of an incandescent bulb. The measure provides assumptions for two implementation strategies (Time of Sale/Retail[[5]](#footnote-5) and Direct Install), and for two markets (Residential and Multi-Family).

This characterization is for a general purpose medium screw based CFL bulb (A-lamps/twists/spirals), and not a specialty bulb (e.g. reflector (PAR) lamps, globes <= 40 watts, candelabras <= 40 watts, 3-ways etc).

**Definition of Baseline Condition**

The baseline is the installation of an incandescent/halogen light bulb meeting the standards described in the Energy and Independence and Security Act of 2007[[6]](#footnote-6).

**Definition of Efficient Condition**

The efficient condition is the installation of a compact fluorescent light bulb.

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBase - WattsEE) /1000) \* ISR \* HOURS \* (WHFeHeat + (WHFeCool – 1))

*Where:*

WattsBase = Based on lumens of CFL bulb[[7]](#footnote-7):

For Non-decorative bulbs:

| **Minimum Lumens** | **Maximum Lumens** | **WattsBase** |
| --- | --- | --- |
| 4000 | 6000 | 300 |
| 3001 | 3999 | 200 |
| 2601 | 3000 | 150 |
| 1490 | 2600 | 72 |
| 1050 | 1489 | 53 |
| 750 | 1049 | 43 |
| 310 | 749 | 29 |
| 250 | 309 | 25 |

For decorative bulbs and non-G40 globes greater than 40 watts:

|  |  |  |  |
| --- | --- | --- | --- |
|  | **Minimum Lumens** | **Maximum Lumens** | **WattsBase** |
| **Decorative** | 500 | 699 | 43 |
| **Non-G40 globe** | 500 | 574 | 43 |
| 575 | 649 | 53 |
| 650 | 1099 | 72 |
| 1100 | 1300 | 150 |

WattsEE = Actual wattage of CFL purchased / installed

*ISR = In Service Rate or percentage of units rebated that are installed and operational.*

|  |  |
| --- | --- |
| **Program** | **In Service Rate (ISR)** |
| Time of Sale (Retail) | 0.96[[8]](#footnote-8) |
| Direct Install | 0.82[[9]](#footnote-9) |

*HOURS = Average hours of use per year*

| **Installation Location** | **Daily Hours** | **Annual Hours** |
| --- | --- | --- |
| Residential interior and  in-unit Multi Family | 2.46 | 898[[10]](#footnote-10) |
| Multi Family Common Areas | 16.3 | 5,950[[11]](#footnote-11) |
| Exterior | 4.5 | 1,643*[[12]](#footnote-12)* |
| Unknown[[13]](#footnote-13) | 2.46 | 898 |

*WHFeCool = Waste Heat Factor for Energy to account for cooling savings from reducing waste heat from efficient lighting.*

|  |  |
| --- | --- |
|  | **WHFeCool** |
| Building with cooling | 1.12[[14]](#footnote-14) |
| Building without cooling or exterior | 1.0 |
| Unknown | 1.10[[15]](#footnote-15) |

*WHFeHeat = Waste Heat Factor for Energy to account for electric heating savings from reducing waste heat from efficient lighting (if fossil fuel heating – see calculation of heating penalty in that section).*

= 1 - ((HF / ηHeat) \* %ElecHeat)

*If unknown assume 0.894[[16]](#footnote-16)*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[17]](#footnote-17) for interior or unknown location*

*= 0% for exterior or unheated location*

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[18]](#footnote-18):*

|  |  |  |  |
| --- | --- | --- | --- |
| **System Type** | **Age of Equipment** | **HSPF Estimate** | **ηHeat**  **(COP Estimate)** |
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| 2006-2014 | 7.7 | 2.26 |
| 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |
| Unknown | N/A | N/A | 1.67*[[19]](#footnote-19)* |

*%ElecHeat = Percentage of home with electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%ElecHeat** |
| Electric | 100% |
| Fossil Fuel | 0% |
| Unknown | 37.5%[[20]](#footnote-20) |

Illustrative examples – do not use as default assumption

A 13W, 850 lumen standard CFL bulb is purchased and installed in an unknown location:

ΔkWh = ((43-13)/1000) \* 0.96 \* 898 \* (0.894 + (1.10-1))

= 26.0 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBase - WattsEE) /1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting*

|  |  |
| --- | --- |
|  | **WHFd** |
| Building with cooling | *1.24*[[21]](#footnote-21) |
| Building without cooling or exterior | *1.0* |
| Unknown | *1.21*[[22]](#footnote-22) |

*CF = Summer Peak Coincidence Factor for measure*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Type** | **Coincidence Factor CF** |
| Residential interior and  in-unit Multi Family | Utility Peak CF | 0.082[[23]](#footnote-23) |
| PJM CF | 0.084[[24]](#footnote-24) |
| Multi Family Common Areas | PJM CF | 0.43[[25]](#footnote-25) |
| Exterior | PJM CF | 0.018[[26]](#footnote-26) |
| Unknown | Utility Peak CF | 0.082 |
| PJM CF | 0.084 |

Illustrative examples – do not use as default assumption

A 13W, 850 lumen CFL bulb is purchased and installed in an unknown location:

ΔkWPJM  = ((43-13) / 1000) \* 0.96 \* 1.21 \* 0.084

= 0.0029 kW

**Annual Fossil Fuel Savings Algorithm**

Heating Penalty if Fossil Fuel heated home (if heating fuel is unknown assume 62.5% of homes heated with fossil fuel):

ΔMMBtuPenalty = - ((((WattsBase - WattsEE) / 1000) \* ISR \* Hours \* HF \* 0.003412) / ηHeat) \* %FossilHeat

*Where:*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[27]](#footnote-27) for interior or unknown location*

*= 0% for exterior or unheated location*

*0.003412 =Converts kWh to MMBtu*

*ηHeat = Efficiency of heating system*

*=72%[[28]](#footnote-28)*

*%FossilHeat = Percentage of home with non-electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%FossilHeat** |
| Electric | 0% |
| Fossil Fuel | 100% |
| Unknown | 62.5%[[29]](#footnote-29) |

Illustrative examples – do not use as default assumption

A 13W, 780 CFL lumen bulb is purchased and installed in an unknown location:

∆MMBtuPenalty = - (((43-13)/1000) \* 0.96 \* 898 \* 0.47 \* 0.003412/0.72) \* 0.625

= - 0.036 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

For the Retail (Time of Sale) measure, the incremental capital cost is $1.80 from June 2014[[30]](#footnote-30).

For the Direct Install measure, the full cost of $3.20[[31]](#footnote-31) per bulb should be used plus $5 labor[[32]](#footnote-32) for a total measure cost of $8.20 per lamp.

**Measure Life**

The measure life is given in the table below. Note that a provision in the Energy Independence and Security Act of 2007 requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the CFL baseline. Therefore after 2014 the measure life will have to be reduced each year to account for the number of years remaining to 2020.

| **Installation Location** | **Measure Life** |
| --- | --- |
| Residential interior and  in-unit Multi Family | 4[[33]](#footnote-33) |
| Multi Family Common Areas | 1.7[[34]](#footnote-34) |
| Exterior | 4.0 |
| Unknown | 4.0 |

**Operation and Maintenance Impacts**

In order to account for the shift in baseline due to the Federal Legislation discussed above, the levelized baseline replacement cost over the lifetime of the CFL is calculated (see MidAtlantic Lighting adjustments and O&M\_042015.xls). The key assumptions used in this calculation are documented below:

|  |  |  |
| --- | --- | --- |
|  | **Standard Incandescent** | **Efficient Incandescent** |
| Replacement Cost | $0.50 | $1.40[[35]](#footnote-35) |
| Component Life[[36]](#footnote-36) (years)  Residential interior, in-unit Multi Family or unknown | 1.1[[37]](#footnote-37) | 1.1[[38]](#footnote-38) |
| Multi Family Common Areas | 0.17 | 0.17 |
| Exterior | 0.60 | 0.60 |

The calculated net present value of the baseline replacement costs for CFL type and installation year are presented below**[[39]](#footnote-39)**:

Residential interior and in-unit Multi Family

|  |  |
| --- | --- |
| **Year** | **NPV of baseline Replacement Costs** |
| 2015 | $3.83 |
| 2016 | $2.94 |
| 2017 | $2.01 |

Multi Family Common Areas

|  |  |
| --- | --- |
| **Year** | **NPV of baseline Replacement Costs** |
| 2015 | $5.60 |
| 2016 | $5.60 |
| 2017 | $5.60 |

Exterior

|  |  |
| --- | --- |
| **Year** | **NPV of baseline Replacement Costs** |
| 2015 | $5.65 |
| 2016 | $5.65 |
| 2017 | $4.32 |

**Specialty CFLs, Residential\***

**Unique Measure Code(s): RS\_LT\_TOS\_SPECCFL\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

An ENERGY STAR qualified specialty compact fluorescent bulb is installed in place of an incandescent specialty bulb. Specialty bulbs defined in this characterization are exempt of the EISA 2007 standard and include the following bulb types: three-way, plant light, daylight bulb, bug light, post light, globes G40, candelabra base, vibration service bulb, decorative candle with medium or intermediate base, shatter resistant, reflector.

The measure provides assumptions for two implementation strategies (Time of Sale/Retail[[40]](#footnote-40) and Direct Install), and for two markets (Residential and Multi-Family).

**Definition of Baseline Condition**

The baseline condition is a specialty incandescent light bulb.

**Definition of Efficient Condition**

The efficient condition is an ENERGY STAR qualified specialty CFL bulb as defined above that is exempt from EISA 2007.

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBase - WattsEE) /1000) \* ISR \* HOURS \* (WHFeHeat + (WHFeCool – 1))

*Where:*

*WattsBase = If actual CFL lumens is known – find the equivalent baseline wattage from the table below[[41]](#footnote-41); use 61.7W if unknown[[42]](#footnote-42)*

| **Bulb Type** | **Lower Lumen Range** | **Upper Lumen Range** | **WattsBase** |
| --- | --- | --- | --- |
| **3-Way, bug, marine, rough service, infrared** | 250 | 449 | 25 |
| 450 | 799 | 40 |
| 800 | 1099 | 60 |
| 1100 | 1599 | 75 |
| 1600 | 1999 | 100 |
| 2000 | 2549 | 125 |
| 2550 | 2999 | 150 |
| **Globe**  **(medium and intermediate bases less than 749 lumens)[[43]](#footnote-43)** | 90 | 179 | 10 |
| 180 | 249 | 15 |
| 250 | 349 | 25 |
| 350 | 749 | 40 |
| **Decorative**  **(Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)[[44]](#footnote-44)** | 70 | 89 | 10 |
| 90 | 149 | 15 |
| 150 | 299 | 25 |
| 300 | 500 | 40 |
| **Globe**  **(candelabra bases less than 1050 lumens)** | 90 | 179 | 10 |
| 180 | 249 | 15 |
| 250 | 349 | 25 |
| 350 | 499 | 40 |
| 500 | 1049 | 60 |
| **Decorative**  **(Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)** | 70 | 89 | 10 |
| 90 | 149 | 15 |
| 150 | 299 | 25 |
| 300 | 499 | 40 |
| 500 | 1049 | 60 |
| **Reflector with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649 | 50 |
| 650 | 1199 | 65 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter >2.5" (\*see exceptions below)** | 640 | 739 | 40 |
| 740 | 849 | 45 |
| 850 | 1179 | 50 |
| 1180 | 1419 | 65 |
| 1420 | 1789 | 75 |
| 1790 | 2049 | 90 |
| 2050 | 2579 | 100 |
| 2580 | 3429 | 120 |
| 3430 | 4270 | 150 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter > 2.26'' and ≤ 2.5" (\*see exceptions below)** | 540 | 629 | 40 |
| 630 | 719 | 45 |
| 720 | 999 | 50 |
| 1000 | 1199 | 65 |
| 1200 | 1519 | 75 |
| 1520 | 1729 | 90 |
| 1730 | 2189 | 100 |
| 2190 | 2899 | 120 |
| 2900 | 3850 | 150 |
| **\*ER30, BR30, BR40, or ER40** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649-1179[[45]](#footnote-45) | 50 |
| **\*BR30, BR40, or ER40** | 650 | 1419 | 65 |
| **\*R20** | 400 | 449 | 40 |
| 450 | 719 | 45 |
| **\*All reflector lamps**  **below lumen ranges specified above** | 200 | 299 | 20 |
| 300 | 399-639[[46]](#footnote-46) | 30 |

WattsEE = Actual wattage of energy efficient specialty bulb purchased, use 15W if unknown[[47]](#footnote-47)

*ISR = In Service Rate or percentage of units rebated that get installed.*

|  |  |
| --- | --- |
| **Program** | **In Service Rate (ISR)** |
| Time of Sale (Retail) | 0.96[[48]](#footnote-48) |
| Direct Install | 0.82[[49]](#footnote-49) |

*HOURS = Average hours of use per year*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Daily Hours** | **Annual Hours** |
| Residential and  in-unit Multi Family | 2.46 | 898[[50]](#footnote-50) |
| Multi Family Common Areas | 16.3 | 5,950[[51]](#footnote-51) |
| Exterior | 4.5 | 1,643*[[52]](#footnote-52)* |
| Unknown[[53]](#footnote-53) | 2.46 | 898 |

*WHFeCool = Waste Heat Factor for Energy to account for cooling savings from reducing waste heat from efficient lighting.*

|  |  |
| --- | --- |
|  | **WHFeCool** |
| Building with cooling | 1.12[[54]](#footnote-54) |
| Building without cooling or exterior | 1.0 |
| Unknown | 1.10[[55]](#footnote-55) |

*WHFeHeat = Waste Heat Factor for Energy to account for electric heating savings from reducing waste heat from efficient lighting (if fossil fuel heating – see calculation of heating penalty in that section).*

= 1 - ((HF / ηHeat) \* %ElecHeat)

*If unknown assume 0.894[[56]](#footnote-56)*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[57]](#footnote-57) for interior or unknown location*

*= 0% for exterior or unheated location*

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[58]](#footnote-58):*

|  |  |  |  |
| --- | --- | --- | --- |
| **System Type** | **Age of Equipment** | **HSPF Estimate** | **ηHeat**  **(COP Estimate)** |
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| 2006 - 2014 | 7.7 | 2.26 |
| 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |
| Unknown | N/A | N/A | 1.67*[[59]](#footnote-59)* |

*%ElecHeat = Percentage of home with electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%ElecHeat** |
| Electric | 100% |
| Fossil Fuel | 0% |
| Unknown | 37.5%[[60]](#footnote-60) |

Illustrative example – do not use as default assumption

An 800 lumen 15W Globe CFL is purchased and installed in an unknown location:

ΔkWh = ((60 - 15) / 1000) \* 0.96 \* 898 \* (0.894 + (1.10 – 1))

= 38.6 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBase - WattsEE) /1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting*

|  |  |
| --- | --- |
|  | **WHFd** |
| Building with cooling | *1.24*[[61]](#footnote-61) |
| Building without cooling or exterior | 1.0 |
| Unknown | *1.21*[[62]](#footnote-62) |

*CF = Summer Peak Coincidence Factor for measure*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Type** | **Coincidence Factor CF** |
| Residential interior and  in-unit Multi Family | Utility Peak CF | 0.082[[63]](#footnote-63) |
| PJM CF | 0.084[[64]](#footnote-64) |
| Multi Family Common Areas | PJM CF | 0.43[[65]](#footnote-65) |
| Exterior | PJM CF | 0.018[[66]](#footnote-66) |
| Unknown | Utility Peak CF | 0.082 |
| PJM CF | 0.084 |

Illustrative example – do not use as default assumption:

An 800 lumen 15W Globe CFL is purchased and installed in an unknown location:

ΔkWPJM  = ((60 – 15) / 1000) \* 0.96 \* 1.21 \* 0.084

= 0.0044 kW

**Annual Fossil Fuel Savings Algorithm**

Heating Penalty if Fossil Fuel heated home (if heating fuel is unknown assume 62.5% of homes heated with fossil fuel[[67]](#footnote-67)):

ΔMMBtuPenalty[[68]](#footnote-68) = - ((((WattsBase - WattsEE) / 1000) \* ISR \* Hours \* HF \* 0.003412) / ηHeat) \* %FossilHeat

*Where:*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[69]](#footnote-69) for interior or unknown location*

*= 0% for exterior or unheated location*

*0.003412 =Converts kWh to MMBtu*

*ηHeat = Efficiency of heating system*

*=72%[[70]](#footnote-70)*

*%FossilHeat = Percentage of home with non-electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%FossilHeat** |
| Electric | 0% |
| Fossil Fuel | 100% |
| Unknown | 62.5%[[71]](#footnote-71) |

Illustrative example – do not use as default assumption

An 800 lumen 15W Globe CFL is purchased and installed in a home with 75% AFUE gas furnace:

∆MMBtuPenalty = - (((60 – 15)/1000) \* 0.96 \* 898 \* 0.47 \* 0.003412/0.75) \* 1.0

= - 0.083 MMBtu

If home heating fuel is unknown:

∆MMBtuPenalty = - (((60 - 15)/1000) \* 0.96 \* 1100 \* 0.47 \* 0.003412/0.72) \* 0.625

= - 0.066 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

For the Retail (Time of Sale) measure, the incremental capital cost for this measure is $3.80[[72]](#footnote-72).

For the Direct Install measure, the full cost of $8.20 should be used plus $5 labor[[73]](#footnote-73) for a total measure cost of $13.20 per lamp.

**Measure Life**

The expected measure life is assumed to be:

|  |  |
| --- | --- |
| **Installation Location** | **Measure Life** |
| Residential interior and  in-unit Multi Family | 6.8[[74]](#footnote-74) |
| Multi Family Common Areas | 1.7[[75]](#footnote-75) |
| Exterior | 4.9*[[76]](#footnote-76)* |
| Unknown | 6.8 |

**Operation and Maintenance Impacts**

Life of the baseline bulb is assumed to be 1.1 years for Residential interior and in-unit Multi Family, 0.17 year for Multi Family common areas and 0.6 year for exterior[[77]](#footnote-77); baseline replacement cost is assumed to be $4.40[[78]](#footnote-78).

**Hardwired CFL Fixtures (Interior)\***

**Unique Measure Code(s): RS\_LT\_RTR\_CFLFIN\_0415 and RS\_LT\_INS\_CFLIN\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

An ENERGY STAR lighting fixture wired for exclusive use with pin-based (including the GU-24 base) compact fluorescent lamps is installed in an interior residential setting. This measure could relate to either retrofit or new installation, and for two markets (Residential and Multi-Family).

**Definition of Baseline Condition**

The baseline condition is a standard incandescent/halogen interior light fixture meeting the standards described in the Energy and Independence and Security Act of 2007[[79]](#footnote-79).

**Definition of Efficient Condition**

The efficient condition is an ENERGY STAR lighting interior fixture for pin-based compact fluorescent lamps.

**Annual Energy Savings Algorithm**

ΔkWh = #lamps \* ((WattsBase - WattsEE) /1000) \* ISR \* HOURS \* (WHFeHeat + (WHFeCool – 1))

*Where:*

WattsBase = Based on lumens of CFL bulb[[80]](#footnote-80):

| **Minimum Lumens** | **Maximum Lumens** | **WattsBase** |
| --- | --- | --- |
| 4000 | 6000 | 300 |
| 3001 | 3999 | 200 |
| 2600 | 3000 | 150 |
| 2000 | 2599 | 72 |
| 1600 | 1999 | 72 |
| 1100 | 1599 | 53 |
| 800 | 1099 | 43 |
| 450 | 799 | 29 |
| 250 | 449 | 25 |

#*lamps = Number of lamps in fixture. If unknown, assume 1.*

*ISR = In Service Rate or percentage of units rebated that get installed.*

*=0.95 [[81]](#footnote-81)*

*HOURS = Average hours of use per year*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Daily Hours** | **Annual Hours** |
| Residential interior and  in-unit Multi Family | 2.46 | 898[[82]](#footnote-82) |
| Multi Family Common Areas | 16.3 | 5,950[[83]](#footnote-83) |
| Unknown | 2.46 | 1,100 |

*WHFeCool = Waste Heat Factor for Energy to account for cooling savings from reducing waste heat from efficient lighting.*

|  |  |
| --- | --- |
|  | **WHFeCool** |
| Building with cooling | 1.12[[84]](#footnote-84) |
| Building without cooling or exterior | 1.0 |
| Unknown | 1.10[[85]](#footnote-85) |

*WHFeHeat = Waste Heat Factor for Energy to account for electric heating savings from reducing waste heat from efficient lighting (if fossil fuel heating – see calculation of heating penalty in that section).*

= 1 - ((HF / ηHeat) \* %ElecHeat)

*If unknown assume 0.894[[86]](#footnote-86)*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[87]](#footnote-87) for interior or unknown location*

*= 0% for exterior or unheated location*

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[88]](#footnote-88):*

|  |  |  |  |
| --- | --- | --- | --- |
| **System Type** | **Age of Equipment** | **HSPF Estimate** | **ηHeat**  **(COP Estimate)** |
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| 2006 - 2014 | 7.7 | 2.26 |
| 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |
| Unknown | N/A | N/A | 1.67*[[89]](#footnote-89)* |

*%ElecHeat = Percentage of home with electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%ElecHeat** |
| Electric | 100% |
| Fossil Fuel | 0% |
| Unknown | 37.5%[[90]](#footnote-90) |

Illustrative example – do not use as default assumption

A 3 x 11W, 600 lumen fixture is purchased and installed in an unknown location:

ΔkWh = (3 \* ((29-11)/1000)) \* 0.95 \* 898 \* (0.894 + (1.10 – 1))

= 46 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (#lamps \* ((WattsBase - WattsEE) /1000)) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting*

|  |  |
| --- | --- |
|  | **WHFd** |
| Building with cooling | *1.24*[[91]](#footnote-91) |
| Building without cooling or exterior | 1.0 |
| Unknown | *1.21*[[92]](#footnote-92) |

*CF = Summer Peak Coincidence Factor for measure*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Type** | **Coincidence Factor CF** |
| Residential interior and  in-unit Multi Family | Utility Peak CF | 0.082[[93]](#footnote-93) |
| PJM CF | 0.084[[94]](#footnote-94) |
| Multi Family Common Areas | PJM CF | 0.43[[95]](#footnote-95) |
| Unknown | Utility Peak CF | 0.082 |
| PJM CF | 0.084 |

Illustrative example – do not use as default assumption

A 3 x 11W, 600 lumen lamp fixture is purchased and installed in an unknown location:

ΔkWPJM  = (3 \* ((29-11) / 1000)) \* 0.95 \* 1.21 \* 0.084

= 0.0052 kW

**Annual Fossil Fuel Savings Algorithm**

Heating Penalty if Fossil Fuel heated home (if heating fuel is unknown assume 62.5% of homes heated with fossil fuel[[96]](#footnote-96)):

ΔMMBtuPenalty = - ((((#lamps \* (WattsBase - WattsEE) / 1000)) \* ISR \* Hours \* HF \* 0.003412) / ηHeat) \* %FossilHeat

*Where:*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[97]](#footnote-97) for interior or unknown location*

*= 0% for exterior or unheated location*

*0.003412 =Converts kWh to MMBtu*

*ηHeat = Efficiency of heating system*

*=72%[[98]](#footnote-98)*

*%FossilHeat = Percentage of home with non-electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%FossilHeat** |
| Electric | 0% |
| Fossil Fuel | 100% |
| Unknown | 62.5%[[99]](#footnote-99) |

Illustrative example – do not use as default assumption

A 3 x 11W, 600 lumen lamp fixture is purchased and installed in an unknown location:

∆MMBtuPenalty = - (((3 \* (29-11)/1000)) \* 0.95 \* 898 \* 0.47 \* 0.003412/0.72) \* 0.625

= - 0.064 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for an interior fixture is assumed to be $32[[100]](#footnote-100).

**Measure Life**

An additional provision in the Energy Independence and Security Act of 2007 requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the CFL baseline.

The measure life of an interior fixture[[101]](#footnote-101) will therefore need to be reduced each year and be equal to the remaining number of years before 2020, i.e. for installations in 2015 the measure life should be 5 years, for installations in 2016 the measure life should be 4 years etc.

**Operation and Maintenance Impacts**

In order to account for the shift in baseline due to the Federal Legislation discussed above, the levelized baseline replacement cost over the lifetime of the CFL is calculated (see MidAtlantic Lighting adjustments and O&M\_042015.xls). The key assumptions used in this calculation are documented below:

|  |  |  |  |
| --- | --- | --- | --- |
|  | **Baseline** | | **Efficient** |
|  | **Standard Incandescent** | **Efficient Incandescent** | **CFL** |
| Replacement Cost | $0.50 | $1.40[[102]](#footnote-102) | $3.20[[103]](#footnote-103) |
| Component Life[[104]](#footnote-104) (years)  Residential interior, in-unit Multi Family or unknown | 1.1[[105]](#footnote-105) | 1.1[[106]](#footnote-106) | 8.9[[107]](#footnote-107) |
| Multi Family Common Areas | 0.17 | 0.17 | 1.34 |

The calculated net present value of the baseline replacement costs for CFL type and installation year are presented below**[[108]](#footnote-108)**:

Residential interior, in-unit Multi Family or unknown

| **Year** | **NPV of baseline Replacement Costs** |
| --- | --- |
| 2015 | $4.24 |
| 2016 | $3.25 |
| 2017 | $2.22 |

Multi Family Common Areas

| **Year** | **NPV of baseline Replacement Costs** |
| --- | --- |
| 2015 | $26.63 |
| 2016 | $21.98 |
| 2017 | $17.09 |

**Hardwired CFL Fixtures (Exterior)\***

**Unique Measure Code(s): RS\_LT\_RTR\_CFLFEX\_0415 and RS\_LT\_INS\_CFLFEX\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

An ENERGY STAR lighting fixture wired for exclusive use with pin-based compact fluorescent lamps is installed in an exterior residential setting. This measure could relate to either retrofit or new installation, and for two markets (Residential and Multi-Family).

**Definition of Baseline Condition**

The baseline condition is a standard incandescent/halogen exterior light fixture meeting the standards described in the Energy and Independence and Security Act of 2007[[109]](#footnote-109).

**Definition of Efficient Condition**

The efficient condition is an ENERGY STAR lighting exterior fixture for pin-based compact fluorescent lamps.

**Annual Energy Savings Algorithm**

ΔkWh = #lamps \* ((WattsBase - WattsEE) /1000) \* ISR \* HOURS \* WHFeCool \* WHFeHeat

*Where:*

WattsBase = Based on lumens of CFL bulb[[110]](#footnote-110):

| **Minimum Lumens** | **Maximum Lumens** | **WattsBase** |
| --- | --- | --- |
| 4000 | 6000 | 300 |
| 3001 | 3999 | 200 |
| 2600 | 3000 | 150 |
| 2000 | 2599 | 72 |
| 1600 | 1999 | 72 |
| 1100 | 1599 | 53 |
| 800 | 1099 | 43 |
| 450 | 799 | 29 |
| 250 | 449 | 25 |

*#lamps = Number of lamps in fixture. If unknown, assume 1.*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 0.87 [[111]](#footnote-111)*

*HOURS = Average hours of use per year*

*= 1643 (4.5 hrs per day)[[112]](#footnote-112)*

Illustrative example – do not use as default assumption

A 2 x 23W, 1600 lumen fixture is purchased and installed in an unknown location:

ΔkWh = (2 \* ((72-23)/1000)) \* 0.87 \* 1643

= 138 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (#lamps \* ((WattsBase - WattsEE) /1000)) \* ISR \* CF

*Where:*

*CF =* *Summer Peak Coincidence Factor for measure*

*= 0.018 [[113]](#footnote-113)*

Illustrative example – do not use as default assumption

A 2 x 23W, 1600 lumen lamp fixture is purchased and installed in an unknown location:

ΔkW = (2\* (72-23) / 1000) \* 0.87 \* 0.018

= 0.0015 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for an exterior fixture is assumed to be $17[[114]](#footnote-114).

**Measure Life**

An additional provision in the Energy Independence and Security Act of 2007 requires that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in essence making the CFL baseline.

The measure life of an exterior fixture[[115]](#footnote-115) will therefore need to be reduced each year and be equal to the remaining number of years before 2020, i.e. for installations in 2016 the measure life should be 4 years etc.

**Operation and Maintenance Impacts**

In order to account for the shift in baseline due to the Federal Legislation discussed above, the levelized baseline replacement cost over the lifetime of the CFL is calculated (see MidAtlantic Lighting adjustments and O&M\_042015.xls). The key assumptions used in this calculation are documented below:

|  |  |  |  |
| --- | --- | --- | --- |
|  | **Baseline** | | **Efficient** |
|  | **Standard Incandescent** | **Efficient Incandescent** | **CFL** |
| Replacement Cost | $0.50 | $1.40[[116]](#footnote-116) | $3.20[[117]](#footnote-117) |
| Component Life (years)  (based on lamp life / assumed annual run hours) | 0.6[[118]](#footnote-118) | 0.6[[119]](#footnote-119) | 4.9[[120]](#footnote-120) |

The calculated net present value of the baseline replacement costs for CFL type and installation year are presented below:

|  |  |
| --- | --- |
| **Year** | **NPV of baseline Replacement Costs**[[121]](#footnote-121) |
| 2015 | $8.01 |
| 2016 | $6.34 |
| 2017 | $4.59 |

**Solid State Lighting (LED) Recessed Downlight Luminaire\***

**Unique Measure Code: RS\_LT\_TOS\_SSLDWN\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure describes savings from the purchase and installation of a Solid State Lighting (LED) Recessed Downlight luminaire in place of an incandescent downlight lamp (i.e. time of sale). The SSL downlight should meet the ENERGY STAR Luminaires Version 2.0 specification[[122]](#footnote-122). The characterization of this measure should not be applied to other types of LEDs.

Note, this measure assumes the baseline is a Bulged Reflector (BR) lamp. This lamp type is generally the cheapest and holds by far the largest market share for this fixture type. They currently are *not* subject to EISA regulations and so this characterization does not include the baseline shift provided in other lighting measures.

The measure provides assumptions for two markets (Residential and Multi-Family).

**Definition of Baseline Condition**

The baseline is the purchase and installation of a standard BR30-type incandescent downlight light bulb.

**Definition of Efficient Condition**

The efficient condition is the purchase and installation of an ENERGY STAR Solid State Lighting (LED) Recessed Downlight luminaire.

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBase - WattsEE) /1,000) \* ISR \* HOURS \* (WHFeHeat + (WHFeCool – 1))

*Where:*

*WattsBase = Connected load of baseline lamp*

*= Actual if retrofit, if LED lumens is known – find the equivalent baseline wattage from the table below[[123]](#footnote-123), if unknown assume 65W [[124]](#footnote-124)*

| **Bulb Type** | **Lower Lumen Range** | **Upper Lumen Range** | **WattsBase** |
| --- | --- | --- | --- |
| **Reflector with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649 | 50 |
| 650 | 1199 | 65 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter >2.5" (\*see exceptions below)** | 640 | 739 | 40 |
| 740 | 849 | 45 |
| 850 | 1179 | 50 |
| 1180 | 1419 | 65 |
| 1420 | 1789 | 75 |
| 1790 | 2049 | 90 |
| 2050 | 2579 | 100 |
| 2580 | 3429 | 120 |
| 3430 | 4270 | 150 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter > 2.26'' and ≤ 2.5" (\*see exceptions below)** | 540 | 629 | 40 |
| 630 | 719 | 45 |
| 720 | 999 | 50 |
| 1000 | 1199 | 65 |
| 1200 | 1519 | 75 |
| 1520 | 1729 | 90 |
| 1730 | 2189 | 100 |
| 2190 | 2899 | 120 |
| 2900 | 3850 | 150 |
| **\*ER30, BR30, BR40, or ER40** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649-1179[[125]](#footnote-125) | 50 |
| **\*BR30, BR40, or ER40** | 650 | 1419 | 65 |
| **\*R20** | 400 | 449 | 40 |
| 450 | 719 | 45 |
| **\*All reflector lamps**  **below lumen ranges specified above** | 200 | 299 | 20 |
| 300 | 399-639[[126]](#footnote-126) | 30 |

*WattsEE = Connected load of efficient lamp*

*= Actual. If unknown assume 9.2W [[127]](#footnote-127)*

*ISR = In Service Rate or percentage of units rebated that get installed.*

*= 1.0[[128]](#footnote-128)*

*HOURS = Average hours of use per year*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Daily Hours** | **Annual Hours** |
| Residential interior and  in-unit Multi Family | 2.46 | 898[[129]](#footnote-129) |
| Multi Family Common Areas | 16.3 | 5,950[[130]](#footnote-130) |
| Unknown | 2.46 | 898 |

*WHFeCool = Waste Heat Factor for Energy to account for cooling savings from reducing waste heat from efficient lighting.*

|  |  |
| --- | --- |
|  | **WHFeCool** |
| Building with cooling | 1.12[[131]](#footnote-131) |
| Building without cooling or exterior | 1.0 |
| Unknown | 1.10[[132]](#footnote-132) |

*WHFeHeat = Waste Heat Factor for Energy to account for electric heating savings from reducing waste heat from efficient lighting (if fossil fuel heating – see calculation of heating penalty in that section).*

= 1 - ((HF / ηHeat) \* %ElecHeat)

*If unknown assume 0.894[[133]](#footnote-133)*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[134]](#footnote-134) for interior or unknown location*

*= 0% for exterior or unheated location*

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[135]](#footnote-135):*

|  |  |  |  |
| --- | --- | --- | --- |
| **System Type** | **Age of Equipment** | **HSPF Estimate** | **ηHeat**  **(COP Estimate)** |
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| 2006 - 2014 | 7.7 | 2.26 |
| 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |
| Unknown | N/A | N/A | 1.67*[[136]](#footnote-136)* |

*%ElecHeat = Percentage of home with electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%ElecHeat** |
| Electric | 100% |
| Fossil Fuel | 0% |
| Unknown | 37.5%[[137]](#footnote-137) |

Illustrative example – do not use as default assumption

Residential interior and in-unit Multi Family

ΔkWh = ((65 – 9.2) / 1,000) \* 1.0 \* 898 \* (0.894 + (1.10 – 1))

= 49.8 kWh

Multi Family Common Areas

ΔkWh = ((65 – 9.2) / 1,000) \* 1.0 \* 5950 \* (0.894 + (1.10 – 1))

= 330 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBase - WattsEE) /1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting*

|  |  |
| --- | --- |
|  | **WHFd** |
| Building with cooling | *1.24*[[138]](#footnote-138) |
| Building without cooling | 1.0 |
| Unknown | *1.21*[[139]](#footnote-139) |

*CF = Summer Peak Coincidence Factor for measure*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Type** | **Coincidence Factor CF** |
| Residential interior and  in-unit Multi Family | Utility Peak CF | 0.082[[140]](#footnote-140) |
| PJM CF | 0.084[[141]](#footnote-141) |
| Multi Family Common Areas | PJM CF | 0.43[[142]](#footnote-142) |
| Unknown | Utility Peak CF | 0.082 |
| PJM CF | 0.084 |

Illustrative example – do not use as default assumption

ΔkWPJM = ((65 – 9.2) / 1,000) \* 1.0 \* 1.21 \* 0.084

= 0.0057 kW

**Annual Fossil Fuel Savings Algorithm**

Heating Penalty if Fossil Fuel heated home (if heating fuel is unknown assume 62.5% of homes heated with fossil fuel):

ΔMMBtuPenalty[[143]](#footnote-143) = - ((((WattsBase - WattsEE) / 1000) \* ISR \* Hours \* HF \* 0.003412) / ηHeat) \* %FossilHeat

*Where:*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[144]](#footnote-144) for interior or unknown location*

*= 0% for exterior or unheated location*

*0.003412 =Converts kWh to MMBtu*

*ηHeat = Efficiency of heating system*

*=72%[[145]](#footnote-145)*

*%FossilHeat = Percentage of home with non-electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%FossilHeat** |
| Electric | 0% |
| Fossil Fuel | 100% |
| Unknown | 62.5%[[146]](#footnote-146) |

Illustrative example – do not use as default assumption

A luminaire in a home with 75% AFUE gas furnace:

∆MMBtuPenalty = - (((65 – 9.2)/1000) \* 1.0 \* 898 \* 0.47 \* 0.003412/0.75) \* 1.0

= - 0.11 MMBtu

If home heating fuel is unknown:

∆MMBtuPenalty = - (((65 - 9.2)/1000) \* 1.0 \* 898 \* 0.47 \* 0.003412/0.72) \* 0.625

= - 0.070 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

If the implementation strategy allows the collection of actual costs, or an appropriate average, then that should be used. If not, the incremental cost for this measure is assumed to be $36[[147]](#footnote-147).

**Measure Life**

The measure life is assumed to be 20 yrs for Residential and Multi Family in-unit, and 8.4 years for Multi Family common areas for downlights featuring inseparable components, and 4.2 years for downlights with replaceable parts.[[148]](#footnote-148)

**Operation and Maintenance Impacts**

The levelized baseline replacement cost over the lifetime of the SSL is calculated (see MidAtlantic Lighting adjustments and O&M\_042015.xls). The key assumptions used in this calculation are documented below:

|  |  |
| --- | --- |
|  | **BR-type Incandescent** |
| Replacement Cost | $4.00 |
| Component Life[[149]](#footnote-149) (years)  Residential interior and in-unit Multi Family or unknown. | 2.2[[150]](#footnote-150) |
| Multi Family Common Areas | 0.34[[151]](#footnote-151) |

The calculated net present value of the baseline replacement costs is $19.99 for Residential interior and in-unit Multi Family and $77 for downlights featuring inseparable components installed in Multifamily common areas and $40 for downlights with replaceable parts installed in Multifamily common areas.

**ENERGY STAR Integrated Screw Based SSL (LED) Lamp\***

**Unique Measure Code: RS\_LT\_TOS\_SSLDWN\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

This measure describes savings from the purchase and installation of an ENERGY STAR Integrated Screw Based SSL (LED) Lamp in place of an incandescent lamp. Note: In December 2015, ENERGY STAR published V2.0 of the Product Specification for Lamps (Light Bulbs). Products that certify to both specifications are available until January 2, 2017, when only Lamps 2.0 products can carry the ENERGY STAR mark. Product brand owners may have products certified to V2.0 as early as December 31, 2015. Therefore, where applicable, this measure includes parameters for both the ENERGY STAR Product Specification for Lamps (Light Bulbs) V1.1 (i.e., the current version of the specification) and V2.0. Beginning January 2, 2017, the savings assumptions for the V1.1 specification will no longer be effective.

The measure provides assumptions for two markets (Residential and Multi-Family).

**Definition of Baseline Condition**

For time of sale replacement, the baseline wattage is assumed to be an incandescent or EISA compliant (where applicable) bulb installed in a screw-base socket[[152]](#footnote-152). Note that the baseline will be EISA compliant bulbs for all categories to which EISA applies. If the in situ lamp wattage is known and lower than the EISA mandated maximum wattage (where applicable), the baseline wattage should be assumed equal to the in situ lamp wattage.

**Definition of Efficient Condition**

The high efficiency wattage is assumed to be an ENERGY STAR qualified Integrated Screw Based SSL (LED) Lamp. As noted in the measure description, eligible products may be certified to either V1.1 or V2.0. The ENERGY STAR specifications can be viewed here: http://1.usa.gov/1QJFLgT

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBase - WattsEE) /1000) \* ISR \* HOURS \* (WHFeHeat + (WHFeCool – 1))

*Where:*

*WattsBase = Based on lumens of the LED – find the ewquivalent baseline wattage from the table below.*

*WattsEE = Actual LED wattage*

| **Bulb Type** | **Lower Lumen Range** | **Upper Lumen Range** | **WattsBase** |
| --- | --- | --- | --- |
| **Standard A-Type** | 250 | 449 | 25 |
| 450 | 799 | 29 |
| 800 | 1099 | 43 |
| 1100 | 1599 | 53 |
| 1600 | 1999 | 72 |
| 2000 | 2599 | 72 |
| 2600 | 3000 | 150 |
| 3001 | 3999 | 200 |
| 4000 | 6000 | 300 |
| **3-Way (Highest Setting), bug, marine, rough service, infrared** | 250 | 449 | 25 |
| 450 | 799 | 40 |
| 800 | 1099 | 60 |
| 1100 | 1599 | 75 |
| 1600 | 1999 | 100 |
| 2000 | 2549 | 125 |
| 2550 | 2999 | 150 |
| **Globe**  **(medium and intermediate bases less than 750 lumens)** | 90 | 179 | 10 |
| 180 | 249 | 15 |
| 250 | 349 | 25 |
| 350 | 749 | 40 |
| **Decorative**  **(Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)** | 70 | 89 | 10 |
| 90 | 149 | 15 |
| 150 | 299 | 25 |
| 300 | 749 | 40 |
| **Globe**  **(candelabra bases less than 1050 lumens)** | 90 | 179 | 10 |
| 180 | 249 | 15 |
| 250 | 349 | 25 |
| 350 | 499 | 40 |
| 500 | 1049 | 60 |
| **Decorative**  **(Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)** | 70 | 89 | 10 |
| 90 | 149 | 15 |
| 150 | 299 | 25 |
| 300 | 499 | 40 |
| 500 | 1049 | 60 |
| **Reflector with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649 | 50 |
| 650 | 1199 | 65 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter >2.5" (\*see exceptions below)** | 640 | 739 | 40 |
| 740 | 849 | 45 |
| 850 | 1179 | 50 |
| 1180 | 1419 | 65 |
| 1420 | 1789 | 75 |
| 1790 | 2049 | 90 |
| 2050 | 2579 | 100 |
| 2580 | 3429 | 120 |
| 3430 | 4270 | 150 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter > 2.26'' and ≤ 2.5" (\*see exceptions below)** | 540 | 629 | 40 |
| 630 | 719 | 45 |
| 720 | 999 | 50 |
| 1000 | 1199 | 65 |
| 1200 | 1519 | 75 |
| 1520 | 1729 | 90 |
| 1730 | 2189 | 100 |
| 2190 | 2899 | 120 |
| 2900 | 3850 | 150 |
| **\*ER30, BR30, BR40, or ER40** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649-1179[[153]](#footnote-153) | 50 |
| **\*BR30, BR40, or ER40** | 650 | 1419 | 65 |
| **\*R20** | 400 | 449 | 40 |
| 450 | 719 | 45 |
| **\*All reflector lamps**  **below lumen ranges specified above** | 200 | 299 | 20 |
| 300 | 399-639[[154]](#footnote-154) | 30 |

*ISR = In Service Rate or percentage of units rebated that get installed.*

*= 0.98[[155]](#footnote-155)*

*HOURS = Average hours of use per year*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Daily Hours** | **Annual Hours** |
| Residential interior and  in-unit Multi Family | 2.46 | 898[[156]](#footnote-156) |
| Multi Family Common Areas | 16.3 | 5,950[[157]](#footnote-157) |
| Exterior | 4.5 | 1,643*[[158]](#footnote-158)* |
| Unknown | 2.46 | 898[[159]](#footnote-159) |

*WHFeCool = Waste Heat Factor for Energy to account for cooling savings from reducing waste heat from efficient lighting.*

|  |  |
| --- | --- |
|  | **WHFeCool** |
| Building with cooling | 1.12[[160]](#footnote-160) |
| Building without cooling or exterior | 1.0 |
| Unknown | 1.10[[161]](#footnote-161) |

*WHFeHeat = Waste Heat Factor for Energy to account for electric heating savings from reducing waste heat from efficient lighting (if fossil fuel heating – see calculation of heating penalty in that section).*

= 1 - ((HF / ηHeat) \* %ElecHeat)

*If unknown assume 0.894[[162]](#footnote-162)*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[163]](#footnote-163) for interior or unknown location*

*= 0% for exterior or unheated location*

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[164]](#footnote-164):*

|  |  |  |  |
| --- | --- | --- | --- |
| **System Type** | **Age of Equipment** | **HSPF Estimate** | **ηHeat**  **(COP Estimate)** |
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| 2006 - 2014 | 7.7 | 2.26 |
| 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |
| Unknown | N/A | N/A | 1.67*[[165]](#footnote-165)* |

*%ElecHeat = Percentage of home with electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%ElecHeat** |
| Electric | 100% |
| Fossil Fuel | 0% |
| Unknown | 37.5%[[166]](#footnote-166) |

Illustrative example – do not use as default assumption

A 10W 550 lumen LED directional lamp with medium screw bases diameter <=2.25" is installed in a residential interior location.

ΔkWh = ((50 - 10)/ 1,000) \* 0.98 \* 898 \* (0.894 + (1.10 – 1))

= 35 kWh

**Baseline Adjustment**

Currently the EISA legislation only applies to omnidirectional bulbs, with Decorative <40 watts and Directional being exceptions. If additional legislation is passed, this TRM will be adjusted accordingly.

To account for these new standards, the savings for this measure should be reduced to account for the higher baselines in 2020. The following table shows the calculated adjustments. The calculated energy savings for omnidirectional lamps should be multiplied by the appropriate factor from the table below for years 2020 and beyond[[167]](#footnote-167):

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Lower Lumen Range** | **Upper Lumen Range** | **Mid-life Adjustment in 2020** | | |
| **ENERGY STAR V1.1** | **ENERGY STAR V2.0** | |
| **CRI>=90** | **CRI<90** |
| 200 | 449 | 100% | 100% | 100% |
| 450 | 799 | 100% | 100% | 100% |
| 800 | 1099 | 9% | 16% | 19% |
| 1,100 | 1599 | 11% | 20% | 24% |
| 1,600 | 1999 | 21% | 23% | 27% |
| 2,000 | 2599 | 23% | 26% | 30% |
| 2,600 | 3000 | 100% | 100% | 100% |
| 3001 | 3999 | 100% | 100% | 100% |
| 4000 | 6000 | 100% | 100% | 100% |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBase - WattsEE) /1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling savings from efficient lighting*

|  |  |
| --- | --- |
|  | **WHFd** |
| Building with cooling | *1.24*[[168]](#footnote-168) |
| Building without cooling or exterior | 1.0 |
| Unknown | *1.21*[[169]](#footnote-169) |

*CF = Summer Peak Coincidence Factor for measure*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Type** | **Coincidence Factor CF** |
| Residential interior and  in-unit Multi Family | Utility Peak CF | 0.082[[170]](#footnote-170) |
| PJM CF | 0.084[[171]](#footnote-171) |
| Multi Family Common Areas | PJM CF | 0.43[[172]](#footnote-172) |
| Exterior | PJM CF | 0.018[[173]](#footnote-173) |
| Unknown | Utility Peak CF | 0.082 |
| PJM CF | 0.084 |

Illustrative example – do not use as default assumption

A 10W 550 lumen LED directional lamp with medium screw bases diameter <=2.25" is installed in a residential interior location.

ΔkWPJM  = ((50 – 10)/ 1,000) \* 0.98 \* 1.21 \* 0.084

= 0.0040 kW

**Annual Fossil Fuel Savings Algorithm**

Heating Penalty if Fossil Fuel heated home (if heating fuel is unknown assume 62.5% of homes heated with fossil fuel):

ΔMMBtuPenalty = - ((((WattsBase - WattsEE) / 1000) \* ISR \* Hours \* HF \* 0.003412) / ηHeat) \* %FossilHeat

*Where:*

*HF = Heating Factor or percentage of light savings that must be heated*

*= 47%[[174]](#footnote-174) for interior or unknown location*

*= 0% for exterior or unheated location*

*0.003412 =Converts kWh to MMBtu*

*ηHeat = Efficiency of heating system*

*=72%[[175]](#footnote-175)*

*%FossilHeat = Percentage of home with non-electric heat*

|  |  |
| --- | --- |
| **Heating fuel** | **%FossilHeat** |
| Electric | 0% |
| Fossil Fuel | 100% |
| Unknown | 62.5%[[176]](#footnote-176) |

Illustrative example – do not use as default assumption

A 10W 550 lumen LED directional lamp with medium screw bases diameter <=2.25" is installed in a residential interior location with unknown heating fuel.

∆MMBtuPenalty = - ((50 - 10)/ 1,000) \* 0.98 \* 898 \* 0.47 \* 0.003412/0.72) \* 0.625

= - 0.049 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

If the implementation strategy allows the collection of actual costs, or an appropriate average, then that should be used. If not, the incremental cost for this measure is presented below[[177]](#footnote-177):

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Lamp Type** | **LED Wattage** | **Lamp Costs** | | |
| **Efficient** | **Baseline** | **Incremental Cost** |
| **LED** | **Incandescent or EISA compliant** | **Incandescent or EISA complaint** |
| **Omnidirectional** | <15W | $6.11 | $1.50 | $4.61 |
| >=15W | $6.81 | $1.50 | $5.31 |
| **Decorative** | <15W | $8.00 | $1.00 | $7.00 |
| 15<= to <25W | $25.00 | $1.00 | $24.00 |
| >=25W | $25.00 | $1.00 | $24.00 |
| **Directional** | < 20W | $17.63 | $5.00 | $12.63 |
| >=20W | $70.78 | $5.00 | $65.78 |

**Measure Life**

The tables below show the assumed measure life for ENERGY STAR Versions 1.1 and 2.0.

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
|  | **Measure Life, Energy Star V1.1** | | | | |
| **Lamp Type** | **Rated Life**[[178]](#footnote-178) | **Residential interior,**  **in-unit Multi Family or unknown** | **Multi Family Common Areas** | **Exterior** | **Unknown** |
| **Omnidirectional** | 25,000 | 20 | 4.2 | 15.2 | 20 |
| **Decorative** | 15,000 | 16.7 | 2.5 | 9.1 | 13.6 |
| **Directional** | 25,000 | 20 | 4.2 | 15.2 | 20 |

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
|  | **Measure Life, Energy Star V2.0** | | | | |
| **Lamp Type** | **Rated Life**[[179]](#footnote-179) | **Residential interior,**  **in-unit Multi Family or unknown** | **Multi Family Common Areas** | **Exterior** | **Unknown** |
| **Omnidirectional** | 15,000 | 16.7 | 2.5 | 8.1 | 13.6 |
| **Decorative** | 15,000 | 16.7 | 2.5 | 9.1 | 13.6 |
| **Directional** | 25,000 | 20 | 4.2 | 15.2 | 20 |

**Operation and Maintenance Impacts**

For Decorative and Directional bulbs, without a baseline shift, the following component costs and lifetimes will be used to calculate O&M savings:

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
| **Lamp Type** | **Baseline Lamp Cost** | **Lamp Lifetime**[[180]](#footnote-180) | | | |
| **Residential interior,**  **in-unit Multi Family** | **Multi Family Common Areas** | **Exterior** | **Unknown** |
| **Decorative** | $1.00 | 1.1 | 0.2 | 0.6 | 0.9 |
| **Directional <15W** | $5.00 | 1.1 | 0.2 | 0.6 | 0.9 |
| **Directional >=15W** | $5.00 | 1.1 | 0.2 | 0.6 | 0.9 |

For Omni-directional bulbs, to account for the shift in baseline due to the Federal Legislation, the levelized baseline replacement cost over the lifetime of the LED is calculated (see ‘ESTAR Integrated Screw SSL Lamp\_041816.xls’). The key assumptions used in this calculation are documented below:

|  |  |  |
| --- | --- | --- |
|  | **EISA**  **2012-2014 Compliant** | **EISA 2020 Compliant** |
| **Replacement Cost <10W** | $1.50 | $2.86 |
| **Replacement Cost >=10W** | $1.50 | $3.19 |
| **Component Life (hours)** | 1000 | 8,000 (for Residential Interior and Exterior)  10,000 (for MF Common Areas)[[181]](#footnote-181) |

The calculation results in the following assumptions of equivalent annual baseline replacement cost:

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
|  | **ENERGY STAR V1.1** |  | **NPV of baseline Replacement Costs** | |
| **Omnidirectional** | **Location** | **LED Wattage** | **2016** | **2017** |
| **Residential interior,** | <10W | $5.67 | $4.84 |
| **in-unit Multi Family and unknown** | >=10W | $5.98 | $5.15 |
| **Multi Family Common Areas** | <10W | $29.23 | $23.44 |
| >=10W | $29.23 | $23.44 |
| **Exterior** | <10W | $10.72 | $9.20 |
| >=10W | $10.72 | $9.20 |

|  | **ENERGY STAR V2.0** |  | **NPV of baseline**  **Replacement Costs** | |
| --- | --- | --- | --- | --- |
| **Omnidirectional** | **Location** | **LED Wattage** | **2016** | **2017** |
| **Residential interior,** | <10W | $5.32 | $4.49 |
| **in-unit Multi Family and unknown** | >=10W | $5.55 | $4.80 |
| **Multi Family Common Areas** | <10W | $18.43 | $18.43 |
| >=10W | $18.43 | $18.43 |
| **Exterior** | <10W | $9.12 | $7.60 |
| >=10W | $9.12 | $7.60 |

*Refrigeration End Use*

**Freezer**

**Unique Measure Code(s): RS\_RF\_TOS\_FREEZER\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

A freezer meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard (NAECA). Energy usage specifications are defined in the table below (note, AV is the freezer Adjusted Volume and is calculated as 1.73\*Total Volume):[[182]](#footnote-182)

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
| **Product Category** | **Volume (cubic feet)** | **Assumptions up to September 2014** | | **Assumptions after September 2014** | |
| **Federal Baseline Maximum Energy Usage in kWh/year[[183]](#footnote-183)** | **ENERGY STAR Maximum Energy Usage in kWh/year[[184]](#footnote-184)** | **Federal Baseline  Maximum Energy Usage in kWh/year[[185]](#footnote-185)** | **ENERGY STAR Maximum Energy Usage in kWh/year[[186]](#footnote-186)** |
| Upright Freezers with Manual Defrost | 7.75 or greater | 7.55\*AV+258.3 | 6.795\*AV+232.47 | 5.57\*AV + 193.7 | 5.01\*AV + 174.3 |
| Upright Freezers with Automatic Defrost | 7.75 or greater | 12.43\*AV+326.1 | 11.187\*AV+293.49 | 8.62\*AV + 228.3 | 7.76\*AV + 205.5 |
| Chest Freezers and all other Freezers except Compact Freezers | 7.75 or greater | 9.88\*AV+143.7 | 8.892\*AV+129.33 | 7.29\*AV + 107.8 | 6.56\*AV + 97.0 |
| Compact Upright Freezers with Manual Defrost | < 7.75 and <=36 inches in height | 9.78\*AV+250.8 | 7.824\*AV+200.64 | 8.65\*AV + 225.7 | 7.79\*AV + 203.1 |
| Compact Upright Freezers with Automatic Defrost | < 7.75 and <=36 inches in height | 11.40\*AV+391 | 9.12\*AV+312.8 | 10.17\*AV + 351.9 | 9.15\*AV + 316.7 |
| Compact Chest Freezers | <7.75 and <=36 inches in height | 10.45\*AV+152 | 8.36\*AV+121.6 | 9.25\*AV + 136.8 | 8.33\*AV + 123.1 |

**Definition of Baseline Condition**

The baseline equipment is assumed to be a model that meets the federal minimum standard for energy efficiency. The standard varies depending on the size and configuration of the freezer (chest freezer or upright freezer, automatic or manual defrost) and is defined in the table above. Note that the Federal Standard will increase on September 1, 2014.

**Definition of Efficient Condition**

The efficient equipment is defined as a freezer meeting the efficiency specifications of ENERGY STAR, as defined below and calculated above:

|  |  |  |
| --- | --- | --- |
| **Equipment** | **Volume** | **Criteria** |
| Full Size Freezer | 7.75 cubic feet or greater | At least 10% more energy efficient than the minimum federal government standard (NAECA). |
| Compact Freezer | Less than 7.75 cubic feet and 36 inches or less in height | At least 20% more energy efficient than the minimum federal government standard (NAECA). |

Note that the ENERGY STAR level will increase in line with the Federal Standard increase on September 1, 2014.

**Annual Energy Savings Algorithm**

ΔkWh = kWhBase- kWhESTAR

*Where:*

*kWhBASE = Baseline kWh consumption per year as calculated in algorithm provided in table above.*

*kWhESTAR = ENERGY STAR kWh consumption per year as calculated in algorithm provided in table above.*

Illustrative example – do not use as default assumption

A 12 cubic foot Upright Freezers with Manual Defrost before September 2014:

ΔkWh*=*(7.55 \* (12 \* 1.73) + 258.3) – (6.795 \* (12 \* 1.73) + 232.47)

*=* 359.5 – 323.6

= 41.5 kWh

If volume is unknown, use the following default values:

|  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- |
| **Product Category** | **Volume Used[[187]](#footnote-187)** | **Assumptions up to September 2014** | | | **Assumptions after September 2014** | | | **Weighting for unknown configuration** |
| **kWhBASE** | **kWhESTAR** | **kWh Savings** | **kWhBASE** | **kWhESTAR** | **kWh Savings** |
| Upright Freezers with Manual Defrost | 27.9 | 469.1 | 422.2 | 46.9 | 349.2 | 314.2 | 35.0 | 0.0% |
| Upright Freezers with Automatic Defrost | 27.9 | 673.2 | 605.9 | 67.3 | 469.0 | 422.2 | 46.8 | 39.5% |
| Chest Freezers and all other Freezers except Compact Freezers | 27.9 | 419.6 | 377.6 | 42.0 | 311.4 | 280.2 | 31.2 | 40.5% |
| Compact Upright Freezers with Manual Defrost | 10.4 | 352.3 | 281.9 | 70.5 | 467.2 | 420.6 | 46.6 | 10.0% |
| Compact Upright Freezers with Automatic Defrost | 10.4 | 509.3 | 407.5 | 101.9 | 635.9 | 572.2 | 63.7 | 6.0% |
| Compact Chest Freezers | 10.4 | 260.5 | 208.4 | 52.1 | 395.1 | 355.7 | 39.4 | 4.0% |

If configuration is unknown assume 58.8 kWh[[188]](#footnote-188) for installations before September 1, 2014 and 41.2kWh for installations after September 1, 2014.

**Summer Coincident Peak kW Savings Algorithm**

∆kW*=* (ΔkWh/8760) \* TAF \* LSAF

*Where:*

TAF = Temperature Adjustment Factor

= 1.23 [[189]](#footnote-189)

LSAF = Load Shape Adjustment Factor

= 1.15 [[190]](#footnote-190)

Illustrative example – do not use as default assumption

A 12 cubic foot Upright Freezers with Manual Defrost installed before September 1, 2014:

ΔkW *=* 41.5 / 8760 \* 1.23 \* 1.15

= 0.0067 kW

If volume is unknown, use the following default values:

|  |  |  |
| --- | --- | --- |
| **Product Category** | **Assumptions up to September 2014** | **Assumptions after September 2014** |
| **kW Savings** | **kW Savings** |
| Upright Freezers with Manual Defrost | 0.0076 | 0.0057 |
| Upright Freezers with Automatic Defrost | 0.0109 | 0.0076 |
| Chest Freezers and all other Freezers except Compact Freezers | 0.0068 | 0.0050 |
| Compact Upright Freezers with Manual Defrost | 0.0114 | 0.0075 |
| Compact Upright Freezers with Automatic Defrost | 0.0164 | 0.0103 |
| Compact Chest Freezers | 0.0084 | 0.0064 |

If configuration is unknown assume 0.0095 kW for installations before September 1, 2014 and 0.0067kW for installations after September 1, 2014.

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is $35[[191]](#footnote-191).

**Measure Life**

The measure life is assumed to be 12 years[[192]](#footnote-192).

**Operation and Maintenance Impacts**

n/a

**Refrigerator**

**Unique Measure Code(s): RS\_RF\_TOS\_REFRIG\_V0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the purchase and installation of a new refrigerator meeting either ENERGY STAR or Consortium for Energy Efficiency (CEE) TIER 2 specifications (defined as requiring >= 20% or >= 25% less energy consumption than an equivalent unit meeting federal standard requirements respectively). The algorithms for calculating Federal Baseline and ENERGY STAR consumption are provided below (note, Adjusted Volume is calculated as the fresh volume + (1.63 \* Freezer Volume). This is a time of sale measure characterization.

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Product Category** | **Assumptions up to September 2014** | | **Assumptions after September 2014** | |
| **Federal Baseline  Maximum Energy Usage in kWh/year[[193]](#footnote-193)** | **ENERGY STAR Maximum Energy Usage in kWh/year**[[194]](#footnote-194) | **Federal Baseline  Maximum Energy Usage in kWh/year[[195]](#footnote-195)** | **ENERGY STAR Maximum Energy Usage in kWh/year[[196]](#footnote-196)** |
| 1. Refrigerators and Refrigerator-freezers with manual defrost | 8.82\*AV+248.4 | 7.056\*AV+198.72 | 6.79AV + 193.6 | 6.11 \* AV + 174.2 |
| 2. Refrigerator-Freezer--partial automatic defrost | 8.82\*AV+248.4 | 7.056\*AV+198.72 | 7.99AV + 225.0 | 7.19 \* AV + 202.5 |
| 3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost | 9.80\*AV+276 | 7.84\*AV+220.8 | 8.07AV + 233.7 | 7.26 \* AV + 210.3 |
| 4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service | 4.91\*AV+507.5 | 3.928\*AV+406 | 8.51AV + 297.8 | 7.66 \* AV + 268.0 |
| 5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service | 4.60\*AV+459 | 3.68\*AV+367.2 | 8.85AV + 317.0 | 7.97 \* AV + 285.3 |
| 6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service | 10.20\*AV+356 | 8.16\*AV+284.8 | 8.40AV + 385.4 | 7.56 \* AV + 355.3 |
| 7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service | 10.10\*AV+406 | 8.08\*AV+324.8 | 8.54AV + 432.8 | 7.69 \* AV + 397.9 |

Note CEE Tier 2 standard criteria is 15% less consumption than a new baseline unit

**Definition of Baseline Condition**

The baseline condition is a new refrigerator meeting the minimum federal efficiency standard for refrigerator efficiency as presented above. Note that the Federal Standard will increase on September 1, 2014.

**Definition of Efficient Condition**

The efficient condition is a new refrigerator meeting either the ENERGY STAR or CEE TIER 2 efficiency standards as presented above. Note that the Federal Standard will increase on September 1, 2014.

**Annual Energy Savings Algorithm**

ΔkWh = kWhBASE – kWhES

*Where:*

*kWhBASE = Annual energy consumption of baseline unit as calculated in algorithm provided in table above.*

*kWhEE = Annual energy consumption of energy efficient unit*

*as calculated in algorithm provided in table above.*

Illustrative example – do not use as default assumption

A 14 cubic foot Refrigerator and 6 cubic foot Freezer, with automatic defrost with side-mounted freezer without through-the-door ice service, installed before September 2014:

ΔkWh*=*((4.91 \* (14 + (6 \* 1.63))) + 507.5) – ((3.928 \* (14 + (6 \* 1.63))) + 406)

*=* 624.3 – 499.4

= 124.9 kWh

If volume is unknown, use the following defaults, based on an assumed Adjusted Volume of 25.8**[[197]](#footnote-197)**:

|  |  |  |  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- |
| **Product Category** | **Assumptions prior to September 1st, 2014** | | | | | **Assumptions after September 1st, 2014** | | | | | **Weighting (%)** |
| **New Baseline UECBASE** | **New Efficient**  **UECEE** | | **ΔkWh** | | **New Baseline UECBASE** | **New Efficient**  **UECEE** | | **ΔkWh** | |
| **ENERGY STAR** | **CEE T2** | **ENERGY STAR** | **CEE T2** | **ENERGY STAR** | **CEE T2** | **ENERGY STAR** | **CEE T2** |
| 1. Refrigerators and Refrigerator-freezers with manual defrost | 475.7 | 380.5 | 356.8 | 95.1 | 118.9 | 368.6 | 331.6 | 276.4 | 36.9 | 92.1 | 0.27 |
| 2. Refrigerator-Freezer--partial automatic defrost | 475.7 | 380.5 | 356.8 | 95.1 | 118.9 | 430.9 | 387.8 | 323.2 | 43.1 | 107.7 | 0.27 |
| 3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost | 528.5 | 422.8 | 396.4 | 105.7 | 132.1 | 441.7 | 397.4 | 331.2 | 44.3 | 110.4 | 57.45 |
| 4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service | 634.0 | 507.2 | 475.5 | 126.8 | 158.5 | 517.1 | 465.4 | 387.8 | 51.7 | 129.3 | 1.40 |
| 5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service | 577.5 | 462.0 | 433.2 | 115.5 | 144.4 | 545.1 | 490.7 | 408.8 | 54.4 | 136.3 | 16.45 |
| 6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service | 618.8 | 495.1 | 464.1 | 123.8 | 154.7 | 601.9 | 550.1 | 451.4 | 51.7 | 150.5 | 0.27 |
| 7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service | 666.3 | 533.0 | 499.7 | 133.3 | 166.6 | 652.9 | 596.1 | 489.6 | 56.8 | 163.2 | 24.10 |

If configuration is unknown assume 114.5 kWh[[198]](#footnote-198) for ENERGY STAR and 143.1 kWh for CEE T2 for installations before September 1, 2014 and 49.1 kWh for ENERGY STAR and 127.9 kWh for CEE T2 for installations after September 1, 2014.

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (ΔkWh/8760) \* TAF \* LSAF

*Where:*

TAF = Temperature Adjustment Factor

= 1.23 [[199]](#footnote-199)

LSAF = Load Shape Adjustment Factor

= 1.15 [[200]](#footnote-200)

If volume is unknown, use the following defaults:

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Product Category** | **Assumptions prior to September 2014 standard change ΔkW** | | **Assumptions after September 2014 standard change**  **ΔkW** | |
| **ENERGY STAR** | **CEE T2** | **ENERGY STAR** | **CEE T2** |
| 1. Refrigerators and Refrigerator-freezers with manual defrost | 0.014 | 0.018 | 0.006 | 0.014 |
| 2. Refrigerator-Freezer--partial automatic defrost | 0.014 | 0.018 | 0.007 | 0.016 |
| 3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost | 0.016 | 0.020 | 0.007 | 0.017 |
| 4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service | 0.019 | 0.024 | 0.008 | 0.019 |
| 5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service | 0.017 | 0.022 | 0.008 | 0.021 |
| 6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service | 0.019 | 0.023 | 0.008 | 0.023 |
| 7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service | 0.020 | 0.025 | 0.009 | 0.025 |

If configuration is unknown assume 0.017 kW for ENERGY STAR and 0.022 kW for CEE T2 for installations before September 1, 2014 and 0.007 kW for ENERGY STAR and 0.019 kW for CEE T2 for installations after September 1, 2014.

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $26 for an ENERGY STAR unit[[201]](#footnote-201) and $140 for a CEE Tier 2 unit.[[202]](#footnote-202)

**Measure Life**

The measure life is assumed to be 12 Years.[[203]](#footnote-203)

**Operation and Maintenance Impacts**

n/a

**Refrigerator Early Replacement**

**Unique Measure Code(s): RS\_RF\_RTR\_REFRIG\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the early removal of an existing inefficient Refrigerator unit from service, prior to its natural end of life, and replacement with a new ENERGY STAR or CEE Tier 2 qualifying unit. This measure is suitable for a Low Income or a Home Performance program.

Savings are calculated between the existing unit and the new efficient unit consumption during the assumed remaining life of the existing unit, and between a hypothetical new baseline unit and the efficient unit consumption for the remainder of the measure life.

This is a retrofit measure.

**Definition of Baseline Condition**

The baseline condition is the existing inefficient refrigerator unit for the remaining assumed useful life of the unit, and then for the remainder of the measure life the baseline becomes a new replacement unit meeting the minimum federal efficiency standard.

**Definition of Efficient Condition**

The efficient condition is a new refrigerator meeting either the ENERGY STAR, or CEE TIER 2 efficiency standards (defined as 20% or 25% above federal standards respectively).

**Annual Energy Savings Algorithm**

Remaining life of existing unit (first 4 years[[204]](#footnote-204))

ΔkWh = kWhEXIST – kWhEE

Remaining measure life (next 8 years)

ΔkWh = kWhBASE – kWhEE

*Where:*

*kWhEXIST = Annual energy consumption of existing unit*

*= 1146* [[205]](#footnote-205)

kWhBASE = *Annual energy consumption of new baseline unit*

*= 572.3 for units prior to September 2014*

*= 511.7 for units after September 2014[[206]](#footnote-206)*

*kWhEE = Annual energy consumption of ENERGY STAR unit*

*= 457.8 for units prior to September 2014*

*= 462.6 for units after September 2014[[207]](#footnote-207)*

*Or = Annual energy consumption of CEE Tier 2 unit*

*= 429.2 for units prior to September 2014*

*= 383.8 for units after September 2014[[208]](#footnote-208)*

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
| **Timing** | **Efficient unit specification** | **First 4 years ΔkWh** | **Remaining 8 years**  **ΔkWh** | **Equivalent Mid Life Savings Adjustment (after 4 years)** | **Equivalent Weighted Average Annual Savings**[[209]](#footnote-209) |
| Assumptions prior to September 2014 | ENERGY STAR | 688.2 | 114.5 | 16.6% | 344.0 |
| CEE T2 | 716.8 | 143.1 | 20.0% | 372.6 |
| Assumptions after September 2014 | ENERGY STAR | 683.4 | 49.1 | 7.2% | 302.9 |
| CEE T2 | 762.2 | 127.9 | 16.8% | 381.7 |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (ΔkWh/8760) \* TAF \* LSAF

*Where:*

TAF = Temperature Adjustment Factor

= 1.23 [[210]](#footnote-210)

LSAF = Load Shape Adjustment Factor

= 1.15 [[211]](#footnote-211)

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
| **Timing** | **Efficient unit specification** | **First 4 years**  **ΔkW** | **Remaining 8 years**  **ΔkW** | **Equivalent Mid Life Savings Adjustment (after 4 years)** | **Equivalent Weighted Average Annual Savings** |
| Assumptions prior to September 2014 | ENERGY STAR | 0.111 | 0.018 | 16.6% | 0.056 |
| CEE T2 | 0.116 | 0.023 | 20.0% | 0.060 |
| Assumptions after September 2014 | ENERGY STAR | 0.110 | 0.008 | 7.2% | 0.049 |
| CEE T2 | 0.123 | 0.021 | 16.8% | 0.062 |

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The initial full measure cost for an Energy Star refrigerator is assumed to be $748 and Tier 2 is $862. The avoided replacement cost (after 4 years) of a baseline replacement refrigerator is $722.[[212]](#footnote-212)

**Measure Life**

The measure life is assumed to be 12 Years. [[213]](#footnote-213)

**Operation and Maintenance Impacts**

n/a

**Refrigerator and Freezer Early Retirement**

**Unique Measure Code(s): RS\_RF\_ERT\_REFRIG\_0414, RS\_RF\_ERT\_FREEZE\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure involves the removal of an existing inefficient refrigerator[[214]](#footnote-214) from service, prior to its natural end of life (early retirement). The program should target refrigerators with an age greater than 10 years, though it is expected that the average age will be greater than 20 years based on other similar program performance. Savings are calculated for the estimated energy consumption during the remaining life of the existing unit[[215]](#footnote-215).

**Definition of Baseline Condition**

The existing refrigerator baseline efficiency is based upon evaluation of a number of existing programs and evaluations.

**Definition of Efficient Condition**

The existing inefficient refrigerator is removed from service and not replaced.

**Annual Energy Savings Algorithm**

**Refrigerators**:

Energy savings for retired refrigerators are based upon a linear regression model using the following coefficients[[216]](#footnote-216):

|  |  |
| --- | --- |
| **Independent Variable Description** | **Estimate Coefficient** |
| Intercept | 0.80460 |
| Age (years) | 0.02107 |
| Pre-1990 (=1 if manufactured pre-1990) | 1.03605 |
| Size (cubic feet) | 0.05930 |
| Dummy: Single Door (=1 if single door) | -1.75138 |
| Dummy: Side-by-Side (= 1 if side-by-side) | 1.11963 |
| Dummy: Primary Usage Type (in absence of the program)  (= 1 if primary unit) | 0.55990 |
| Interaction: Located in Unconditioned Space x HDD/365.25 | -0.04013 |
| Interaction: Located in Unconditioned Space x CDD/365.25 | 0.02622 |

ΔkWh = [0.80460 + (Age \* 0.02107) + (Pre-1990 \* 1.03605) + (Size \* 0.05930) + (Single-Door \* -1.75138) + (Side-by-side \* 1.11963) + (Primary \* 0.55990) + (HDD/365.25 \* Unconditioned \* -0.04013) + (CDD/365.25 \* Unconditioned \* 0.02622)] \* 365.25 \* Part Use

*Where:*

*HDD = Heating Degree Days*

*= dependent on location. Use actual for location or defaults below[[217]](#footnote-217)*

|  |  |  |
| --- | --- | --- |
| ***Location*** | ***Heating Degree Days***  ***(65°F set point)*** | ***HDD / 365.25*** |
| *Wilmington, DE* | *4,298* | *11.8* |
| *Baltimore, MD* | *4,529* | *12.4* |
| *Washington, DC* | *3,947* | *10.8* |

*CDD = Cooling Degree Days*

*= dependent on location. Use actual for location or defaults below[[218]](#footnote-218)*

|  |  |  |
| --- | --- | --- |
| ***Location*** | ***Cooling Degree Days***  ***(65°F set point)*** | ***CDD / 365.25*** |
| *Wilmington, DE* | *1,162* | *3.2* |
| *Baltimore, MD* | *1,266* | *3.5* |
| *Washington, DC* | *1,431* | *3.9* |

*Part Use Factor = To account for those units that are not running throughout the entire year as reported by the customer.  Default of 0.95 for refrigerations and 0.86 for freezers.[[219]](#footnote-219)*

Illustrative example – can be used as default assumption only if required data tracking is not available

Using participant population mean values from BGE EY4 and default part use factor:

ΔkWh = [0.80460 + (18.61 \* 0.02107) + (0.20 \* 1.03605) + (19.43 \* 0.05930) + (0.02 \* -1.75138) + (0.34 \* 1.11963) + (0.64 \* 0.55990) + (2.91 \* -0.04013) + (0.77 \* 0.02622)] \* 365.25 \* 0.95

= 1,098 kWh

**Freezers:**

Energy savings for freezers are based upon a linear regression model using the following coefficients[[220]](#footnote-220):

|  |  |
| --- | --- |
| **Independent Variable Description** | **Estimate Coefficient** |
| Intercept | -0.95470 |
| Age (years) | 0.0453 |
| Pre-1990 (=1 if manufactured pre-1990) | 0.54341 |
| Size (cubic feet) | 0.12023 |
| Chest Freezer Configuration (=1 if chest freezer) | 0.29816 |
| Interaction: Located in Unconditioned Space x HDD/365.25 | -0.03148 |
| Interaction: Located in Unconditioned Space x CDD/365.25 | 0.08217 |

ΔkWh = [-0.95470 + (Age \* 0.04536) + (Pre-1990 \* 0.54341) + (Size \* 0.12023) + (Chest Freezer \* 0.29816) + (HDDs/365.25 \* Unconditioned \* -0.03148) + (CDDs/365.25 \* Unconditioned \* 0.08217)] \* 365.25 \* Part Use Factor

Illustrative example – can be used as default assumption only if required data tracking is not available.

Using participant population mean values from BGE EY4 and default part use factor:

ΔkWh = [-0.95470 + (23.79 \* 0.04536) + (0.46 \* 0.54341) + (15.86 \* 0.12023) + (0.21 \* 0.29816) + (6.83 \* -0.03148) + (1.80 \* 0.08217)] \* 365.25 \* 0.86

= 715 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (ΔkWh/8760) \* TAF \* LSAF

*Where:*

TAF = Temperature Adjustment Factor

= 1.23 [[221]](#footnote-221)

LSAF = Load Shape Adjustment Factor

= 1.066 [[222]](#footnote-222)

Illustrative example – can be used as default assumption only if required data tracking is not available

Using participant population mean values from BGE EY4 and default part use factor:

Refrigerator:

ΔkW = 1098/8760 \* 1.23 \* 1.066

= 0.164 kW

Freezer:

ΔkW = 715/8760 \* 1.23 \* 1.066

= 0.107 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure will be the actual cost associated with the removal and recyling of the secondary refrigerator.

**Measure Life**

The measure life is assumed to be 8 Years.[[223]](#footnote-223)

**Operation and Maintenance Impacts**

n/a

*Heating Ventilation and Air Conditioning (HVAC) End Use*

**Central Furnace Efficient Fan Motor**

**Unique Measure Code(s): RS\_HV\_RTR\_FANMTR\_0510 and RS\_HV\_TOS\_FANMTR\_0510**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure involves the installation of a high efficiency brushless permanent magnet fan motor (BPM or ECM), hereafter referred to as “efficient fan motor”. This measure could apply to fan motors installed with a furnace or with a central air conditioning unit and could apply when retrofitting an existing unit or installing a new one.

If a new unit is installed, the program should require that it meet ENERGY STAR efficiency criteria in order to qualify for the incentive, although the savings estimations below relate only to the efficiency gains associated with an upgrade to the efficient fan motor.

For homes that install an efficient furnace fan and have central A/C, both the cooling and heating savings values should be included.

**Definition of Baseline Condition**

A standard low-efficiency permanent split capacitor (PSC) fan motor.

**Definition of Efficient Condition**

A high efficiency brushless permanent magnet fan motor (BPM or ECM).

**Annual Energy Savings Algorithm**

Heating Season kWh Savings from efficient fan motor = 241kWh [[224]](#footnote-224)

Cooling Season kWh Savings from efficient fan motor = 178kWh [[225]](#footnote-225)

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = 0 [[226]](#footnote-226)

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $200.[[227]](#footnote-227)

**Measure Life**

The measure life is assumed to be 18 years.245

**Operation and Maintenance Impacts**

n/a

**Room Air Conditioner\***

**Unique Measure Code(s): RS\_HV\_TOS\_RA/CES\_0414 and RS\_HV\_TOS\_RA/CT2\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the purchase (time of sale) and installation of a room air conditioning unit that meets the ENERGY STARminimum qualifying efficiency specifications presented below. Note that if the AC unit is connected to a network in a way to enable it to respond to energy related commands, it gets a 5% extra CEER allowance. In these instances, the efficient CEER, would be 0.95 multiplied by the appropriate CEER from the table below.

|  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- |
| **Product Type and Class (Btu/hour)** | | **Federal Standard with louvered sides**  **(CEER)** | **Federal Standard without louvered sides**  **(CEER)** | **ENERGY STAR with louvered sides (CEER)** | **ENERGY STAR without louvered sides (CEER)** |  |
| Without Reverse Cycle | < 8,000 | 11.0 | 10.0 | 12.1 | 11.0 |  |
| 8,000 to 10,999 | 10.9 | 9.6 | 12.0 | 10.6 |  |
| 11,000 to 13,999 | 10.9 | 9.5 | 12.0 | 10.5 |  |
| 14,000 to 19,999 | 10.7 | 9.3 | 11.8 | 10.2 |  |
| 20,000 to 24,999 | 9.4 | 9.4 | 10.3 | 10.3 |  |
| 25,000 to 27,999 | 9.0 | 9.4 | 10.3 | 10.3 |  |
| >=28,000 | 9.0 | 9.4 | 9.9 | 10.3 |  |
| With Reverse Cycle | <14,000 | NA | 9.3 |  | 10.2 |  |
| >= 14,000 | NA | 8.7 |  | 9.6 |  |
| <20,000 | 9.8 | NA | 10.8 | NA |  |
| >=20,000 | 9.3 | NA | 10.2 | NA |  |
| Casement only | | 9.5 | | 10.5 | |  |
| Casement-Slider | | 10.4 | | 11.4 | |  |

**Definition of Baseline Condition**

The baseline condition is a window AC unit that meets the minimum federal efficiency standards as of June 1, 2014[[228]](#footnote-228) presented above.

**Definition of Efficient Condition**

The baseline condition is a window AC unit that meets the ENERGY STAR v4.0.

**Annual Energy Savings Algorithm**

ΔkWH = (Hours \* Btu/hour \* (1/CEERbase - 1/CEERee))/1000

*Where:*

*Hours = Run hours of Window AC unit*

*= 325 [[229]](#footnote-229)*

*Btu/hour = Size of rebated unit*

*When available, the actual size of the rebated unit should be used in the calculation. In the absence of this data, the following default value can be used:*

*= 8500 [[230]](#footnote-230)*

*CEERbase = Efficiency of baseline unit in Btus per Watt-hour*

*= Actual (see table above)*

*If average deemed value required use 10.9 [[231]](#footnote-231)*

*CEERee = Efficiency of ENERGY STAR unit in Btus per Watt-hour*

*= Actual*

*If average deemed value required use 12.0[[232]](#footnote-232) for an ENERGY STAR[[233]](#footnote-233)*

Using deemed values above:

ΔkWHENERGY STAR

= (325 \* 8500 \* (1/10.9 – 1/11.3)) / 1000

= 23.2 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = Btu/hour \* (1/CEERbase - 1/CEERee))/1000 \* CF

*Where:*

*CF = Summer Peak Coincidence Factor for measure*

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.31 [[234]](#footnote-234)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.3[[235]](#footnote-235)*

Using deemed values above:

ΔkWENERGY STAR SSP

= (8500 \* (1/10.9 – 1/12)) / 1000 \* 0.31

= 0.022 kW

ΔkWENERGY STAR PJM

= (8500 \* (1/10.9 – 1/12)) / 1000 \* 0.30

= 0.021 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $54 for units less than 6,000 Btu/hr, and $39 for units greater or equal to 6,000 Btu/hr.[[236]](#footnote-236)

**Measure Life**

The measure life is assumed to be 12 years.[[237]](#footnote-237)

**Operation and Maintenance Impacts**

n/a

**ENERGY STAR Central A/C\***

**Unique Measure Code(s): RS\_HV\_TOS\_CENA/C\_0415, RS\_HV\_RTR\_CENA/C\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a new Central Air Conditioning ducted split system meeting ENERGY STAR efficiency standards presented below.

|  |  |  |
| --- | --- | --- |
| **Efficiency Level** | **SEER Rating** | **EER Rating** |
| Federal Standard | 14 | 11.5[[238]](#footnote-238) |
| ENERGY STAR | 15 | 12.5 |

This measure could relate to:

1. Time of Sale – the installation of a new Central AC system meeting ENERGY STAR specifications replacing an existing unit at the end of its useful life or the installation of a new system in a new home.
2. Early Replacement – the early removal of an existing functioning unit prior to its natural end of life and replacement with an ENERGY STAR unit. Savings are calculated between existing unit and efficient unit consumption during the assumed remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

The savings methodology provided is applicable only where the baseline and efficient capacities are equal.

Evaluators should be aware that there will be an interaction between this measure and others, e.g. duct sealing, air sealing and insulation measures. Attempt should be made to account for this interaction where the measures occur in the same home within the same program period.

**Definition of Baseline Condition**

The baseline condition for the Time of Sale is a central air conditioning ducted split system that meets the minimum Federal standards as presented above.

The baseline condition for the Early Replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit, and the new baseline as defined above for the remainder of the measure life.

**Definition of Efficient Condition**

The efficient condition is a central air conditioning ducted split system that meets the ENERGY STAR standards presented above.

**Annual Energy Savings Algorithm**

Time of Sale:

ΔkWH = (Hours \* Btu/hour \* (1/SEERbase - 1/SEERee))/1000

Early replacement[[239]](#footnote-239):

ΔkWH for remaining life of existing unit (1st 6 years):

= ((Hours \* Btu/hour \* (1/SEERexist - 1/SEERee))/1000)

ΔkWH for remaining measure life (next 12 years):

= ((Hours \* Btu/hour \* (1/SEERbase - 1/SEERee))/1000)

*Where:*

*Hours = Full load cooling hours*

*Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | ***Run Hours*** |
| *Wilmington, DE* | *524* *[[240]](#footnote-240)* |
| *Baltimore, MD* | *542 [[241]](#footnote-241)* |
| *Washington, DC* | *681* |

*Btu/Hour = Size of equipment in Btu/hour (note 1 ton = 12,000Btu/hour)*

*= Actual installed*

*SEERbase = Seasonal Energy Efficiency Ratio Efficiency of baseline unit*

*= 14 [[242]](#footnote-242)*

*SEERexist = Seasonal Energy Efficiency Ratio of existing unit (kBtu/kWh)*

*= Use actual SEER rating where it is possible to measure or reasonably estimate. If unknown assume 10.0*[[243]](#footnote-243)*.*

*SEERee = Seasonal Energy Efficiency Ratio Efficiency of ENERGY STAR unit*

*= Actual installed*

Illustrative example – do not use as default assumption

Time of Sale example: a 3 ton unit with SEER rating of 15, in Baltimore:

ΔkWH = (542 \* 36000 \* (1/14 – 1/15)) / 1000

= 93 kWh

Early Replacement example: a 3 ton unit with SEER rating of 15 replaces an existing unit in Baltimore:

ΔkWH (for first 6 years) = (542 \* 36000 \* (1/10 – 1/15)) / 1000

= 650 kWh

ΔkWH (for next 12 years) = (542 \* 36000 \* (1/14 – 1/15)) / 1000

= 93 kWh

**Summer Coincident Peak kW Savings Algorithm**

Time of Sale:

ΔkW = Btu/hour \* (1/EERbase - 1/EERee)/1000 \* CF

Early replacement:

ΔkW for remaining life of existing unit (1st 6 years):

= Btu/hour \* (1/EERexist - 1/EERee)/1000 \* CF

ΔkW for remaining measure life (next 12 years):

= Btu/hour \* (1/EERbase - 1/EERee)/1000 \* CF

*Where:*

*EERbase = Energy Efficiency Ratio Efficiency of baseline unit*

*= 11.5*

*EERexist = EER Efficiency of existing unit*

*= Actual EER of unit should be used, if EER is unknown, use 9.2[[244]](#footnote-244)*

*EERee = Energy Efficiency Ratio Efficiency of ENERGY STAR unit*

*= Actual installed*

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[245]](#footnote-245)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[246]](#footnote-246)*

Illustrative example – do not use as default assumption

Time of Sale example: a 3 ton unit with EER rating of 12.5:

ΔkW*SSP* = (36000 \* (1/11.5 – 1/12.5)) / 1000 \* 0.69

= 0.17 kW

ΔkW*PJM* = (36000 \* (1/11.5 – 1/12.5)) / 1000 \* 0.66

= 0.17 kW

Early Replacement example: a 3 ton unit with SEER rating of 15 replaces an existing unit in Baltimore:

ΔkW for remaining life of existing unit (1st 6 years):

ΔkW*SSP* = (36000 \* (1/9.2 – 1/12.5)) / 1000 \* 0.69

= 0.71 kW

ΔkW*PJM* = (36000 \* (1/9.2 – 1/12.5)) / 1000 \* 0.66

= 0.68 kW

ΔkW for remaining measure life (next 12 years):

ΔkW*SSP* = (36000 \* (1/11.5 – 1/12.5)) / 1000 \* 0.69

= 0.17 kW

ΔkW*PJM* = (36000 \* (1/11.5 – 1/12.5)) / 1000 \* 0.66

= 0.17 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

Time of Sale:

The incremental cost for this measure is provided below:[[247]](#footnote-247)

|  |  |
| --- | --- |
| **Efficiency Level** | **Cost per Ton** |
| SEER 15 | $92 |
| SEER 16 | $184 |
| SEER 17 | $276 |
| SEER 18 | $369 |
| SEER 19 | $461 |
| SEER 20 | $553 |
| SEER 21 | $645 |

Early replacement:

The incremental capital cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume (note these costs are per ton of unit capacity)[[248]](#footnote-248):

|  |  |
| --- | --- |
| **Efficiency (SEER)** | **Full Retrofit Cost (including labor) per Ton of Capacity ($/ton)** |
| 15 | $2,403 |
| 16 | $2,495 |
| 17 | $2,588 |
| 18 | $2,680 |
| 19 | $2,772 |
| 20 | $2,864 |
| 21 | $2,956 |

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be $2,286 per ton[[249]](#footnote-249). This cost should be discounted to present value using the utilities discount rate.

**Measure Life**

The measure life is assumed to be 18 years.[[250]](#footnote-250)

Remaining life of existing equipment is assumed to be 6 years[[251]](#footnote-251).

**Operation and Maintenance Impacts**

n/a

**Air Source Heat Pump\***

**Unique Measure Code: RS\_HV\_TOS\_ASHP\_0415, RS\_HV\_RTR\_ASHP\_0415,**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a new Air Source Heat Pump split system meeting ENERGY STAR efficiency standards presented below:

|  |  |  |  |
| --- | --- | --- | --- |
| **Efficiency Level** | **HSPF** | **SEER Rating** | **EER Rating**[[252]](#footnote-252) |
| Federal Standard as of 1/1/2015 | 8.2 | 14 | 11.8[[253]](#footnote-253) |
| ENERGY STAR | 8.5 | 15 | 12 |

This measure could relate to:

1. Time of Sale – the installation of a new Air Source Heat Pump system meeting ENERGY STAR specifications replacing an existing unit at the end of its useful life or the installation of a new system in a new home.
2. Early Replacement – the early removal of existing functioning electric heating and cooling prior to its natural end of life and replacement with an ENERGY STAR unit. Savings are calculated between existing unit and efficient unit consumption during the assumed remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life.

The savings methodology provided is applicable only where the baseline and efficient capacities are equal.

Evaluators should be aware that there will be an interaction between this measure and others, e.g. duct sealing, air sealing and insulation measures. Attempt should be made to account for this interaction where the measures occur in the same home within the same program period.

**Definition of Baseline Condition**

The baseline condition for the Time of Sale measure is an Air Source Heat Pump split system that meets the minimum Federal standards defined above.

The baseline condition for the Early Replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit, and the new baseline of the same equipment type for the remainder of the measure life as provided below:

|  |  |  |  |
| --- | --- | --- | --- |
| **Existing Equipment Type** | **HSPF** | **SEER Rating** | **EER Rating** |
| ASHP | 8.2 | 14 | 11.8 |
| Electric Resistance and Central AC | 3.41 | 13 | 11.0 |

**Definition of Efficient Condition**

The efficient condition is an Air Source Heat Pump split system that meets the ENERGY STAR standards defined above or other specifications as determined by the programs.

**Annual Energy Savings Algorithm**

Time of Sale:

ΔkWH = (FLHcool \* BtuH \* (1/SEERbase - 1/SEERee))/1,000 + (FLHheat \* BtuH \* (1/HSPFbase – 1/HSPFee))/1,000

Early replacement[[254]](#footnote-254):

ΔkWH for remaining life of existing unit (1st 6 years):

= (FLHcool \* BtuHCool \* (1/SEERexist - 1/SEERee))/1,000 + (FLHheat \* BtuHHeat \* (1/HSPFexist – 1/HSPFee))/1,000

ΔkWH for remaining measure life (next 12 years):

= (FLHcool \* BtuHCool \* (1/SEERbasereplace - 1/SEERee))/1,000 + ( FLHheat \* BtuHHeat\* (1/HSPFbasereplace – 1/HSPFee))/1,000

*Where:*

*FLHcool = Full Load Cooling Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | ***FLHcool*** |
| *Wilmington, DE* | *719 [[255]](#footnote-255)* |
| *Baltimore, MD* | *744 [[256]](#footnote-256)* |
| *Washington, DC* | *935* |

*BtuHCool  = Cooling capacity of Air Source Heat Pump (1 ton = 12,000Btuh)*

*= Actual*

*SEERbase = Seasonal Energy Efficiency Ratio of baseline Air Source Heat Pump*

*= 14[[257]](#footnote-257)*

*SEERexist = Seasonal Energy Efficiency Ratio of existing cooling system (kBtu/kWh)*

*= Use actual SEER rating where it is possible to measure or reasonably estimate. If not, assume the following dependent on type of existing cooling system:*

|  |  |
| --- | --- |
| ***Existing Cooling System*** | ***SEERexist****[[258]](#footnote-258)* |
| *Air Source Heat Pump or Central AC* | *10.0* |
| *No central cooling[[259]](#footnote-259)* | *Make ‘1/SEERexist’ = 0* |

*SEERee = Seasonal Energy Efficiency Ratio of efficient Air Source Heat Pump*

*= Actual*

SEERbasereplace *= Seasonal Energy Efficiency Ratio of new baseline replacement of same equipment type as existing:*

|  |  |
| --- | --- |
| **Existing Equipment Type** | **SEER Rating** |
| ASHP | 14 |
| Central AC or no replaced cooling | 13 |

*FLHheat = Full Load Heating Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | **FLHheat** |
| *Wilmington, DE* | *935[[260]](#footnote-260)* |
| *Baltimore, MD* | *866[[261]](#footnote-261)* |
| *Washington, DC* | *822* |

*BtuHHeat  = Heating capacity of Air Source Heat Pump (1 ton = 12,000Btuh)*

*= Actual*

*HSPFbase = Heating Seasonal Performance Factor of baseline Air Source Heat Pump*

*= 8.2 [[262]](#footnote-262)*

*HSPFexist = Heating System Performance Factor[[263]](#footnote-263) of existing heating system (kBtu/kWh)*

*= Use actual HSPF rating where it is possible to measure or reasonably estimate. If not available use:*

|  |  |
| --- | --- |
| ***Existing Heating System*** | ***HSPF\_exist*** |
| *Air Source Heat Pump* | *5.96 [[264]](#footnote-264)* |
| *Electric Resistance* | *3.41 [[265]](#footnote-265)* |

*HSPFee = Heating Seasonal Performance Factor of efficient Air Source Heat Pump*

*= Actual*

*HSPFbasereplace = Heating System Performance Factor of new baseline replacement of same equipment type as existing (kBtu/kWh)*

|  |  |
| --- | --- |
| **Existing Equipment Type** | **HSPF** |
| ASHP | 8.2 |
| Electric Resistance and Central AC | 3.41 |

Illustrative example – do not use as default assumption

Time of Sale example: a 3 ton unit with a SEER rating of 15 and HSPF of 8.5 in Baltimore, MD:

ΔkWH = (744 \* 36,000 \* (1/14 - 1/15))/1,000 + (866 \* 36,000 \* (1/8.2 – 1/8.5))/1,000

= 261.7 kWh

Early Replacement example: a 3 ton unit with a SEER rating of 14.5 and HSPF of 8.4 in Baltimore, MD is installed replacing an existing working Central AC system with a SEER rating of 10 and electric resistance heating:

ΔkWH (for first 6 years) = (744 \* 36,000 \* (1/10 - 1/15))/1,000 + (866 \* 36,000 \* (1/3.41 – 1/8.5))/1,000

= 6,368 kWh

ΔkWH (for remaining 12 years) = (744 \* 36,000 \* (1/13 - 1/15))/1,000 + (866 \* 36,000 \* (1/3.41 – 1/8.5))/1,000

= 5,749 kWh

**Summer Coincident Peak kW Savings Algorithm**

Time of Sale:

ΔkW = BtuHCool \* (1/EERbase - 1/EERee))/1,000 \* CF

Early replacement:

ΔkW for remaining life of existing unit (1st 6 years):

= BtuHCool \* (1/EERexist - 1/EERee)/1000 \* CF

ΔkW for remaining measure life (next 12 years):

= BtuHCool \* (1/EERbasereplace - 1/EERee)/1000 \* CF

*Where:*

*EERbase = Energy Efficiency Ratio (EER) of Baseline Air Source Heat Pump*

*= 11.8 [[266]](#footnote-266)*

EERexist = Energy Efficiency Ratio of existing cooling system (kBtu/hr / kW)

= Use actual EER rating where it is possible to measure or reasonably estimate. If EER unknown but SEER available convert using the equation:

EER = (-0.02 \* SEER2) + (1.12 \* SEER) [[267]](#footnote-267)

If SEER rating unavailable use:

|  |  |
| --- | --- |
| **Existing Cooling System** | **EERexist[[268]](#footnote-268)** |
| Air Source Heat Pump or Central AC | 9.2 |
| No central cooling[[269]](#footnote-269) | Make ‘1/EERexist’ = 0 |

*EERee = Energy Efficiency Ratio (EER) of Efficient Air Source Heat Pump*

*= Actual*

*If EER is unknown, calculate based on formula presented above.*

EERbasereplace *= Energy Efficiency Ratio of new baseline replacement of same equipment type as existing:*

|  |  |
| --- | --- |
| **Existing Equipment Type** | **EER Rating** |
| ASHP | 11.8 |
| Electric Resistance and Central AC | 11.0 |

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[270]](#footnote-270)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[271]](#footnote-271)*

Illustrative example – do not use as default assumption

Time of Sale example: a 3 ton unit with EER rating of 12.0 in Baltimore, MD:

ΔkWSSP  = 36,000 \* (1/11.8 - 1/12))/1,000 \* 0.69

= 0.035kW

Early Replacement example: a 3 ton unit with an EER rating of 12 in Baltimore, MD is installed replacing an existing working Central AC system with an EER rating of 9.2 and electric resistance heating:

ΔkW for remaining life of existing unit (1st 6 years):

ΔkW SSP = 36,000 \* (1/9.2 - 1/12))/1,000 \* 0.69

= 0.63 kW

ΔkW for remaining measure life (next 12 years):

ΔkW SSP = 36,000 \* (1/11 - 1/12))/1,000 \* 0.69

= 0.15 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is provided in the table below[[272]](#footnote-272). Note these incremental costs are per ton of capacity, so for example a 3 ton, 15 SEER unit would have an incremental cost of $510.

|  |  |
| --- | --- |
| **Efficiency (SEER)** | **Incremental Cost per Ton of Capacity** |
| 15 | $170 |
| 16 | $340 |
| 17 | $529 |
| 18 | $710 |

Early replacement: The capital cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume the following (note these costs are per ton of unit capacity)[[273]](#footnote-273):

|  |  |
| --- | --- |
| **Efficiency (SEER)** | **Full Retrofit Cost (including labor) per Ton of Capacity ($/ton)** |
| 15 | $2,544 |
| 16 | $3,120 |
| 17 | $3,309 |
| 18 | $3,614 |

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be $2,355 per ton of capacity[[274]](#footnote-274). This cost should be discounted to present value using the utilities discount rate.

**Measure Life**

The measure life is assumed to be 18 years[[275]](#footnote-275).

**Operation and Maintenance Impacts**

n/a

**Duct Sealing**

**Unique Measure Code: RS\_HV\_RTR\_DCTSLG\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure is the sealing of ducts using mastic sealant, aerosol or UL-181 compliant duct sealing tape.

Three methodologies for evaluating the savings associated with sealing the ducts are provided. The first method is provided only as a tool for prescreening potential measures involving a careful visual inspection of the duct work, followed by two further methods that requires the use of a blower door either of which can be used to evaluate savings.

1. **Feasibility Evaluation of Distribution Efficiency** – this methodology should **not be used for claiming savings** but can be a useful tool to help evaluate the potential from duct sealing. It requires evaluation of three duct characteristics below, and use of the Building Performance Institutes ‘Distribution Efficiency Look-Up Table’;

<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>

* 1. Percentage of duct work found within the conditioned space
  2. Duct leakage evaluation
  3. Duct insulation evaluation

1. **Modified Blower Door Subtraction** – this technique is described in detail on p44 of the Energy Conservatory Blower Door Manual; <http://www.energyconservatory.com/download/bdmanual.pdf>

It involves performing a whole house depressurization test and repeating the test with the ducts excluded.

1. **Duct Blaster Testing**  - as described in RESNET Test 803.7;

<http://www.resnet.us/standards/DRAFT_Chapter_8_July_22.pdf>

This involves using a blower door to pressurize the house to 25 Pascals, and pressurizing the duct system using a duct blaster to reach equilibrium with the inside. The air required to reach equilibrium provides a duct leakage estimate.

This is a retrofit measure. Evaluators should be aware that there will be an interaction between this measure and others, e.g. duct sealing, air sealing and insulation measures. Attempt should be made to account for this interaction where the measures occur in the same home within the same program period.

**Definition of Baseline Condition**

The existing baseline condition is leaky duct work within the unconditioned space in the home.

**Definition of Efficient Condition**

The efficient condition is sealed duct work throughout the unconditioned space in the home.

**Annual Energy Savings Algorithm**

***Methodology 1: Feasibility Evaluation of Distribution Efficiency (not for claiming savings)***

Estimate of Cooling savings from reduction in Air Conditioning Load:

Determine Distribution Efficiency by evaluating duct system before and after duct sealing using Building Performance Institute “Distribution Efficiency Look-Up Table”

ΔkWh cooling = ((((DEafter – DEbefore)/ DEafter)) \* FLHcool \* BtuH) / 1,000 / ηCool

Where:

*DEafter = Distribution Efficiency after duct sealing*

*DEbefore = Distribution Efficiency before duct sealing*

*FLHcool = Full Load Cooling Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | ***FLHcool*** |
| *Wilmington, DE* | *524 [[276]](#footnote-276)* |
| *Baltimore, MD* | *542 [[277]](#footnote-277)* |
| *Washington, DC* | *681* |

*BtuH = Size of equipment in Btuh (note 1 ton = 12,000Btuh)*

*= Actual*

*ηCool = Efficiency in SEER of Air Conditioning equipment*

*= actual. If not available use[[278]](#footnote-278):*

|  |  |  |
| --- | --- | --- |
| ***Equipment Type*** | ***Age of Equipment*** | ***SEER Estimate*** |
| *Central AC* | *Before 2006* | *10* |
| *After 2006* | *13* |
| *Heat Pump* | *Before 2006* | *10* |
| *2006-2014* | *13* |
| *2015 on* | *14* |

Illustrative example – do not use as default assumption

Duct sealing in a house in Wilmington DE, with 3-ton SEER 11 central air conditioning and the following duct evaluation results:

DEbefore = 0.80

DEafter = 0.90

Energy Savings:

ΔkWh = ((0.90 – 0.80)/0.90) \* 524 \* 36,000) / 1,000 / 11

= 191 kWh

Estimate of Heating savings for homes with electric heat (Heat Pump of resistance):

kWh = (((((DEafter – DEbefore)/ DEafter)) \* FLHheat \* BtuH ) / 1,000,000 / ηHeat ) \* 293.1

*Where:*

*FLHheat = Full Load Heating Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | **FLHheat** |
| *Wilmington, DE* | *935[[279]](#footnote-279)* |
| *Baltimore, MD* | *866[[280]](#footnote-280)* |
| *Washington, DC* | *822* |

BtuH = Size of equipment in Btuh (note 1 ton = 12,000Btuh)

= Actual

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[281]](#footnote-281):*

|  |  |  |  |
| --- | --- | --- | --- |
| ***System Type*** | ***Age of Equipment*** | ***HSPF Estimate*** | ***COP Estimate*** |
| *Heat Pump* | *Before 2006* | *6.8* | *2.00* |
| *2006-2014* | *7.7* | *2.26* |
| 2015 on | 8.2 | 2.40 |
| *Resistance* | *n/a* | *n/a* | *1.00* |

Illustrative example – do not use as default assumption

Duct sealing in a 2.5 COP heat pump heated house in Baltimore, MD with the following duct evaluation results:

DEbefore = 0.80

DEafter = 0.90

Energy Savings:

ΔkWh = ((((0.90 – 0.80)/0.90) \* 866 \* 36,000) / 1,000,000 / 2.5 ) \* 293.1

= 406 kWh

***Methodology 2: Modified Blower Door Subtraction***

Claiming Cooling savings from reduction in Air Conditioning Load:

1. Determine Duct Leakage rate before and after performing duct sealing:

Duct Leakage (CFM50DL) = (CFM50Whole House – CFM50Envelope Only) \* SCF

*Where:*

*CFM50Whole House = Standard Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differential*

*CFM50Envelope Only = Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differential with all supply and return registers sealed.*

*SCF = Subtraction Correction Factor to account for underestimation of duct leakage due to connections between the duct system and the home. Determined by measuring pressure in duct system with registers sealed and using look up table provided by Energy Conservatory.*

1. Calculate duct leakage reduction, convert to CFM25DL[[282]](#footnote-282)and factor in Supply and Return Loss Factors

Duct Leakage Reduction (∆CFM25DL) = (Pre CFM50DL – Post CFM50DL) \* 0.64 \* (SLF + RLF)

*Where :*

*SLF = Supply Loss Factor*

*= % leaks sealed located in Supply ducts \* 1 [[283]](#footnote-283)*

*Default = 0.5[[284]](#footnote-284)*

*RLF = Return Loss Factor*

*= % leaks sealed located in Return ducts \* 0.5[[285]](#footnote-285)*

*Default = 0.25[[286]](#footnote-286)*

1. Calculate Energy Savings:

ΔkWhcooling = ((∆CFM25DL)/ (Capacity \* 400)) \* FLHcool \* BtuH) / 1000 / ηCool

*Where:*

*∆CFM25DL = Duct leakage reduction in CFM25*

*Capacity = Capacity of Air Cooling system (tons)*

*400 = Conversion of Capacity to CFM (400CFM / ton)*

*FLHcool = Full Load Cooling Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | ***FLHcool*** |
| *Wilmington, DE* | *524 [[287]](#footnote-287)* |
| *Baltimore, MD* | *542 [[288]](#footnote-288)* |
| *Washington, DC* | *681* |

*BtuH = Size of equipment in Btuh (note 1 ton = 12,000Btuh)*

*= Actual*

*ηCool = Efficiency in SEER of Air Conditioning equipment*

*= actual. If not available use[[289]](#footnote-289):*

|  |  |  |
| --- | --- | --- |
| ***Equipment Type*** | ***Age of Equipment*** | ***SEER Estimate*** |
| *Central AC* | *Before 2006* | *10* |
| *After 2006* | *13* |
| *Heat Pump* | *Before 2006* | *10* |
| *2006-2014* | *13* |
| *2015 on* | *14* |

Illustrative example – do not use as default assumption

Duct sealing in a house in Wilmington, DE with 3 ton, SEER 11 central air conditioning and the following blower door test results:

Before:

CFM50Whole House = 4,800 CFM50

CFM50Envelope Only = 4,500 CFM50

House to duct pressure = 45 Pascals

= 1.29 SCF (Energy Conservatory look up table)

After:

CFM50Whole House = 4,600 CFM50

CFM50Envelope Only = 4,500 CFM50

House to duct pressure = 43 Pascals

= 1.39 SCF (Energy Conservatory look up table)

Duct Leakage at CFM50:

CFM50DL before = (4,800 – 4,500) \* 1.29

= 387 CFM50

CFM50DL after = (4,600 – 4,500) \* 1.39

= 139 CFM50

Duct Leakage reduction at CFM25:

∆CFM25DL = (387 – 139) \* 0.64 \* (0.5 + 0.25)

= 119 CFM25

Energy Savings:

ΔkWh = ((119 / (3 \* 400)) \* 524 \* 36,000) / 1,000 / 11

= 170 kWh

Claiming Heating savings for homes with electric heat (Heat Pump):

ΔkWh = (((∆CFM25DL / (Capacity \* 400)) \* FLHheat \* BtuH) / 1,000,000 / ηHeat) \* 293.1

*Where:*

*∆CFM25DL = Duct leakage reduction in CFM25*

*Capacity = Capacity of Air Cooling system (tons)*

*400 = Conversion of Capacity to CFM (400CFM / ton)*

*FLHheat = Full Load Heating Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | **FLHheat** |
| *Wilmington, DE* | *935[[290]](#footnote-290)* |
| *Baltimore, MD* | *866[[291]](#footnote-291)* |
| *Washington, DC* | *822* |

BtuH = Size of equipment in Btuh (note 1 ton = 12,000Btuh)

= Actual

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[292]](#footnote-292):*

|  |  |  |  |
| --- | --- | --- | --- |
| ***System Type*** | ***Age of Equipment*** | ***HSPF Estimate*** | ***COP Estimate*** |
| *Heat Pump* | *Before 2006* | *6.8* | *2.00* |
| *2006-2014* | *7.7* | *2.26* |
| 2015 on | 8.2 | 2.40 |
| *Resistance* | *n/a* | *n/a* | *1.00* |

Illustrative example – do not use as default assumption

Duct sealing in a 3-ton 2.5 COP heat pump heated house in Baltimore, MD with the blower door results described above:

ΔkWh = (((119 / (3 \* 400)) \* 866 \* 36,000) / 1,000,000 / 2.5) \* 293.1

= 362 kWh

***Methodology 3: Duct Blaster Testing***

Claiming Cooling savings from reduction in Air Conditioning Load:

ΔkWhcooling = (((Pre\_CFM25 –Post\_CFM25)/ (Capacity \* 400)) \* FLHcool \* BtuH) / 1000 / ηCool

*Where:*

*Pre\_CFM25 = Duct leakage in CFM25 as measured by duct blaster test before sealing*

*Post\_CFM25 = Duct leakage in CFM25 as measured by duct blaster test after sealing*

*All other variables as provided above.*

Illustrative example – do not use as default assumption

Duct sealing in a house in Wilmington, DE with 3 ton, SEER 11 central air conditioning and the following duct blaster test results:

*Pre\_CFM25* = 220 CFM25

*Post\_CFM25 = 80 CFM25*

ΔkWh = (((220 - 80) / (3 \* 400)) \* 524 \* 36,000) / 1,000 / 11

= 200 kWh

Claiming Heating savings for homes with electric heat (Heat Pump):

ΔkWh = (((Pre\_CFM25 –Post\_CFM25/ (Capacity \* 400)) \* FLHheat \* BtuH) / 1,000,000 / ηHeat) \* 293.1

*Where:*

*All other variables as provided above.*

Illustrative example – do not use as default assumption

Duct sealing in a 3-ton 2.5 COP heat pump heated house in Baltimore, MD with the duct blaster results described above:

ΔkWh = ((((220 - 80) / (3 \* 400)) \* 866 \* 36,000) / 1,000,000 / 2.5) \* 293.1

= 426 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh / FLHcool \* CF

*Where:*

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[293]](#footnote-293)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[294]](#footnote-294)*

**Annual Fossil Fuel Savings Algorithm**

For homes with Fossil Fuel Heating:

***Methodology 1: Feasibility Evaluation of Distribution Efficiency (not for claiming savings)***

ΔMMBTUfossil fuel = ((((DEafter – DEbefore)/ DEafter)) \* FLHheat \* BtuH ) / 1,000,000 / ηHeat

*Where:*

*DEafter = Distribution Efficiency after duct sealing*

*DEbefore = Distribution Efficiency before duct sealing*

*FLHheat = Full Load Heating Hours*

*= 620[[295]](#footnote-295)*

*BtuH = Capacity of Heating System*

*= Actual*

*ηHeat = Efficiency of Heating equipment*

*= Actual[[296]](#footnote-296). If not available use 84%[[297]](#footnote-297).*

Illustrative example – do not use as default assumption

Duct sealing in a fossil fuel heated house with a 100,000Btuh, 80% AFUE natural gas furnace, with the following duct evaluation results:

DEbefore = 0.80

DEafter = 0.90

Energy Savings:

ΔMMBTU = ((0.90 – 0.80)/0.90) \* 620 \* 100,000 ) / 1,000,000 / 0.80

= 8.6 MMBtu

***Methodology 2: Modified Blower Door Subtraction***

ΔMMBTU = (((∆CFM25DL / (BtuH \* 0.0126)) \* FLHheat \* BtuH ) / 1,000,000 / ηHeat

*Where:*

*∆CFM25DL = Duct leakage reduction in CFM25*

*BtuH = Capacity of Heating System (Btuh)*

*= Actual*

*0.0126 = Conversion of Capacity to CFM (0.0126CFM / Btuh)[[298]](#footnote-298)*

*FLHheat = Full Load Heating Hours*

*= 620[[299]](#footnote-299)*

*ηHeat = Efficiency of Heating equipment*

*= Actual[[300]](#footnote-300). If not available use 84%[[301]](#footnote-301).*

Illustrative example – do not use as default assumption

Duct sealing in a house with a 100,000Btuh, 80% AFUE natural gas furnace and with the blower door results described above:

Energy Savings:

ΔMMBTU = (((119 / (100,000 \* 0.0126)) \* 620 \* 100,000 ) / 1,000,000 / 0.80

= 7.3 MMBtu

***Methodology 3: Duct Blaster Testing***

ΔMMBTU = (((Pre\_CFM25 –Post\_CFM25/ (BtuH \* 0.0126)) \* FLHheat \* BtuH ) / 1,000,000 / ηHeat

*Where:*

*All variables as provided above*

Illustrative example – do not use as default assumption

Duct sealing in a house with a 100,000Btuh, 80% AFUE natural gas furnace and with the duct blaster results described above:

Energy Savings:

ΔMMBTU = (((220 - 80 / (100,000 \* 0.0126)) \* 620 \* 100,000 ) / 1,000,000 / 0.80

= 8.6 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure should be the actual labor and material cost to seal the ducts.

**Measure Life**

The measure life is assumed to be 20 years[[302]](#footnote-302).

**Operation and Maintenance Impacts**

n/a

**Ductless Mini-Split Heat Pump\***

**Unique Measure Code: RS\_HV\_TOS\_MSHP\_0415, RS\_HV\_RTR\_ASHP\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of new ENERGY STAR rated ductless “mini-split” heat pump(s) (DMSHP). A ductless mini-split heat pump (DMSHP) is a type of heat pump with an outdoor condensing unit connected via refrigerant line to one or more indoor evaporator coils. Ductless mini-split heat pumps deliver cooling at the same or higher efficiency as standard central AC units, but can also deliver heat. Further, since the units do not require ductwork, they avoid duct losses.

This measure could be installed in either an existing or in a new home and the characterization is designed to allow the calculation of the impact on electric and/or gas consumption following the installation of a DHP system. The characterization requires that the program implementer perform a custom calculation to determine how much existing and supplemental heating and/or cooling load the DHP will replace based on a combination of billing data, the percentage of conditioned space covered by the DMSHP, the existing equipment and its hours of operation, proposed hours of operation, and the size of the conditioned space. Where possible, this should be treated as a custom measure, due to the number of variables needed, including usage patterns and types of baseline systems.

**Definition of Baseline Condition**

The baseline condition for early replacement is the existing heating and cooling (if applicable) systems within the home. If cooling equipment is not previously present, it is presumed that some type of cooling equipment would have been installed and the time of sale baseline described next should be used for the cooling baseline assumption.

The baseline condition in time of sale / new construction is a standard-efficiency ductless unit meeting the following efficiency standards:

|  |  |  |  |
| --- | --- | --- | --- |
| **Year** | **SEER** | **EER** | **HSPF** |
| 2015 | 14 | 11.8[[303]](#footnote-303) | 8.2 |

**Definition of Efficient Condition**

The efficient condition is an ENERGY STAR ductless heat pump exceeding all of the following efficiency standards; 14.5 SEER, 12 EER, 8.2 HSPF.

**Annual Energy Savings Algorithm**

If displacing/replacing electric heat:

ΔkWhtotal = ΔkWhcool + ΔkWhheat

ΔkWhcool = CoolingLoadDHP \* (1/SEERbase\* (1 + ΔDLimpr x DLcool)

– 1/SEERee)

ΔkWhheat =HeatLoadElectricDHP \* (3.412/HSPFbase \* (1 + ΔDLimpr \* DLheat) – 3.412/HSPFee)

If displacing/replacing gas heat:

ΔkWhtotal = ΔkWhcool - Total\_kWhheat

ΔkWhcool = CoolingLoadDHP \* (1/SEERbase\* (1 + ΔDLimpr x DLcool)

– 1/SEERee)

Total\_kWhheat = (HeatLoadGasDHP \* 293.1 \* 3.412 / HSPFee)

*Where:*

*CoolingLoadDHP*

*= Cooling load (kWh) that the DHP will now provide*

*= Actual*

*SEERbase = Efficiency in SEER of existing Air Conditioner or baseline ductless heat pump (kBtu cooling/ kWh consumed)*

*Early Replacement = Use actual SEER rating where it is possible to measure or reasonably estimate. If unknown assume 10.0[[304]](#footnote-304) for Central AC or 8.5 for Room AC[[305]](#footnote-305). If no cooling exists but the customer is looking for a cooling solution, assume 14.0. If no cooling exists or was planned at the home, make 1/SEER = 0 (resulting in a negative value i.e. increase in cooling load).*

*Time of Sale / New Construction = 14.0[[306]](#footnote-306)*

*SEERee = Efficiency i n SEER of efficient ductless heat pump*

*= Actual (kBtu cooling/ kWh consumed)*

*HeatLoadElectricDHP*

*= Heating load (kWh) that the DHP will now provide*

*= Actual[[307]](#footnote-307)*

*DLcool  = 1 if duct leakage applies based on baseline cooling equipment (0 otherwise)*

*DLheat  = 1 if duct leakage applies based on baseline heating equipment (0 otherwise)*

ΔDLimpr = Duct loss improvement factor, 0.15

*3.412 = Converts 1/HSPF to 1/COP*

*HSPFbase = Heating Seasonal Performance Factor of existing system or baseline ductless heat pump for new construction*

*Early Replacement = Use actual HSPF rating where it is possible to measure or reasonably estimate.*

*If unknown assume 3.412[[308]](#footnote-308) for resistance heat,* 5.96**[[309]](#footnote-309)** for ASHP.

*Time of Sale / New Construction = 8.2 [[310]](#footnote-310)*

*HSPFee = Heating Seasonal Performance Factor of ENERGY STAR ductless heat pump[[311]](#footnote-311)*

*= Actual*

*HeatLoadGasDHP = Heating load (MMBtu) that the DHP will now provide*

*= Actual[[312]](#footnote-312)*

*293.1 = Converts MMBtu to kWh*

*AFUEexist = Efficiency of existing furnace or boiler*

*= Use actual AFUE rating where it is possible to measure or reasonably estimate. If unknown assume 78%[[313]](#footnote-313).*

*3.412 = Converts heat pump HSPF in to COP*

See example calculations at end of characterization.

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = BtuHCool \* (1/EERbase \* (1 + ΔDLimpr \* DLcool)

- 1/EERee))/1,000 \* CF

*Where:*

*BtuHCool = Cooling capacity of ductless heat pump (1 ton = 12,000Btuh)*

*= Actual*

*EERbase = Energy Efficiency Ratio (EER) of Baseline Air Source Heat Pump*

*Early Replacement = Use actual EER rating where it is possible to measure or reasonably estimate.*

*If unknown assume 9.1[[314]](#footnote-314) for Central AC or 7.7 for Room AC[[315]](#footnote-315).*

*If no cooling is at the home, make 1/EER = 0 (resulting in a negative value i.e. increase in load).*

*Time of Sale / New Construction = 8.5[[316]](#footnote-316)*

*EERee = Energy Efficiency Ratio (EER) of Efficient ductless heat pump*

*= Actual.*

*DLcool  = 1 if duct leakage applies based on baseline cooling equipment (0 otherwise)*

ΔDLimpr = Duct loss improvement factor, 0.15

*CF = Coincidence Factor for measure. Assumptions for both Central AC and Room AC are provided below. The appropriate selection depends on whether the DHP is being used similarly to a central AC (thermostatically controlled) or a room AC (controlled with need). If unknown assume Room AC.*

*CFSSP Room AC = Summer System Peak Coincidence Factor for Room A/C (hour ending 5pm on hottest summer weekday)*

*= 0.31 [[317]](#footnote-317)*

*CFPJM Room AC = PJM Summer Peak Coincidence Factor for Room A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.3[[318]](#footnote-318)*

*CFSSP Central AC = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[319]](#footnote-319)*

*CFPJM Central AC = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[320]](#footnote-320)*

See example calculations at end of characterization.

**Annual Fossil Fuel Savings Algorithm**

If the existing heating system is gas fired, the savings from the measure represent the displaced gas heating consumption, and the DHP represents added electric load.

ΔMMBtu = HeatLoadGasReplaced / AFUEexist \* (1 + ΔDLimpr \* DLheat)

*Where:*

*HeatLoadGasReplaced*

*= Heating load (MMBtu) that the DHP will now provide in place of gas unit*

*= Actual[[321]](#footnote-321)*

*AFUEexist = Efficiency of existing heating system*

*= Use actual AFUE rating where it is possible to measure or reasonably estimate. If unknown assume 78% for early retiremens, or 82% for replace on burnouts[[322]](#footnote-322).*

*DLheat  = 1 if duct leakage applies based on baseline heating equipment (0 otherwise)*

ΔDLimpr = Duct loss improvement factor, 0.15

See example calculations at end of characterization.

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

**Early Replacement:** the actual full cost of the DHP installation should be used if available, if not defaults are provided in the table below:

|  |  |
| --- | --- |
| **Unit Size** | **Early Replacement: Full Install Cost[[323]](#footnote-323)** |
| 1-Ton | $3,000 |
| 1.5-Ton | $3750 |
| 2-Ton | $4,500 |
| 2.5-Ton | $5,250 |
| 3-Ton | $6,000 |

If the DHP installation results in the early removal of existing operating heating or cooling equipment (that otherwise would have needed to be replaced in the future) then the deferred replacement of that equipment should be accounted for. This deferred replacement cost should be estimated based on the existing equipment or the following defaults can be used:

Central AC - $2,185 per ton[[324]](#footnote-324).

Central Ducted Air Source Heat Pump - $2,166 per ton[[325]](#footnote-325)

Furnace - $2,311 [[326]](#footnote-326)

Boiler - $3,834 [[327]](#footnote-327)

The deferred replacement cost should be discounted to today’s dollar assuming it would have occurred in 6 years (3rd of measure life) and subtracted from the full DHP install cost presented above.

**Time of Sale / New construction:** an estimated incremental cost from a SEER 14 baseline is provided below:

|  |  |
| --- | --- |
| **Unit Size** | **Time of Sale / New Construction: Incremental Cost**[[328]](#footnote-328) |
| 1-Ton | $603 |
| 1.5-Ton | $624 |
| 2-Ton | $601 |
| 2.5-Ton | $600 |
| 3-Ton | $600 |

**Measure Life**

The measure life is assumed to be 18 years[[329]](#footnote-329). If an early replacement measure results in the removal of existing operating heating or cooling equipment, it is assumed that it would have needed replacing in 6 years.

**Operation and Maintenance Impacts**

n/a

Illustrative examples – do not use as default assumption

Early Replacement:

A 1.5 ton, 20 SEER, 14 EER, 12 HSPF, DHP replaces 5000 kWh of existing electric resistance heat load in a home without existing cooling in Baltimore, MD. DHP is estimated to provide 2,000kWh of cooling load.

ΔkWH = (CoolingLoadDHP \* (1/SEERbase - 1/SEERee)) + (HeatLoadElectricDRP \* (3.412/HSPFbase – 3.412/HSPFee)

= (2000 \* (0 – 1/20)) + (5000 \* (3.412/3.4 – 3.412/12))

= 3,496 kWh

ΔkW*SSP* = BtuHCool \* (1/EERbase - 1/EERee))/1,000 \* CF

= (18,000 \* (0 – 1/14)) / 1000) \* 0.31

= - 0.40kW

A 2.5 ton, 18 SEER, 13.5 EER, 11 HSPF, DHP displaces all of existing gas heat (78% AFUE) in a home with central cooling in Baltimore, MD. The heating load is estimated as 40 MMBtu and cooling load of 4000 kWh.

ΔkWH = (CoolingLoadDHP \* (1/SEERbase - 1/SEERee)) - (HeatLoadGasDHP \* 293.1 \* 0.85 \* (3.412/HSPFee))

= (4000 \* (1/10 – 1/18)) - (40 \* 293.3 \* 0.85 \* (3.412/11))

= -2,915 kWh (i.e. this results in an increase in electric consumption)

ΔkW*SSP* = BtuHCool \* (1/EERbase - 1/EERee))/1,000 \* CF

= (30,000 \* (1/9.1 – 1/13.5)) / 1000) \* 0.31

= 0.33 kW (in the summer you see demand savings)

ΔMMBtu = HeatLoadGasReplaced / AFUEexist

= 40 / 0.78

= 51.3 MMBtu

Time of Sale / New Construction

Two 1.5 ton, 18 SEER, 13.5 EER, 11 HSPF, DHP is installed in a new home in Baltimore, MD. The estimated heat load is 12,000kWh and the cooling load is 6,000kWh

ΔkWH = (CoolingLoadDHP \* (1/SEERbase - 1/SEERee)) + (HeatLoadElectricDHP \* (3.412/HSPFbase – 3.412/HSPFee)

= (6000 \* (1/14 – 1/18)) + (12,000 \* (3.412/8.2– 3.412/11))

= 1,366kWh

ΔkW*SSP* = BtuHCool \* (1/EERbase - 1/EERee))/1,000 \* CF

= (36,000 \* (1/11.8 – 1/13.5)) / 1000) \* 0.31

= 0.12 kW

**HE Gas Boiler**

**Unique Measure Code: RS\_HV\_TOS\_GASBLR\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure characterization provides savings for the purchase and installation of a new residential sized ENERGY STAR-qualified high efficiency gas-fired boiler for residential space heating, instead of a new baseline gas boiler. The measure could be installed in either an existing or new home. The installation is assumed to occur during a natural time of sale.

Evaluators should be aware that there will be an interaction between this measure and others, e.g. duct sealing, air sealing and insulation measures. Attempt should be made to account for this interaction where the measures occur in the same home within the same program period.

**Definition of Baseline Condition**

The baseline condition is a boiler that meets the minimum Federal baseline AFUE for boilers. For boilers manufactured after September 2012, the Federal baseline is 82% AFUE.

**Definition of Efficient Condition**

The efficient condition is an ENERGY STAR qualified boiler with an AFUE rating ≥ 85%.

**Annual Energy Savings Algorithm**

n/a

**Summer Coincident Peak kW Savings Algorithm**

n/a

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = (EFLHheat \* Btuh \* ((AFUEee/AFUEbase) - 1)) /1,000,000

*Where:*

*EFLHheat = Equivalent Full Load Heating Hours*

|  |  |
| --- | --- |
| ***Location*** | ***EFLH*** |
| *Wilmington, DE* | *848[[330]](#footnote-330)* |
| *Baltimore, MD* | *620[[331]](#footnote-331)* |
| *Washington, DC* | *528*[[332]](#footnote-332) |

*BtuH = Input Capacity of Boiler*

*= Actual*

*AFUEbase = Efficiency in AFUE of baseline boiler*

*= 82%*

*AFUEee = Efficiency in AFUE of efficient boiler*

*= Actual*

Illustrative example – do not use as default assumption

The purchase and installation of a 100,000 Btuh input capacity, 90% AFUE boiler in Maryland:

ΔMMBtu = (620 \* 100,000 \* ((0.9/0.82) – 1)) /1,000,000

= 6.0 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental install cost for this measure is provided below[[333]](#footnote-333):

|  |  |
| --- | --- |
| **Efficiency of Boiler (AFUE)** | **Incremental Cost** |
| 85% - 90% | $725 |
| 91% + | $1272 |

**Measure Life**

The measure life is assumed to be 18 years[[334]](#footnote-334) .

**Operation and Maintenance Impacts**

n/a

**Condensing Furnace (gas)\***

**Unique Measure Code: RS\_HV\_TOS\_GASFUR\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure characterization provides savings for the purchase and installation of a new residential sized ENERGY STAR-qualified high efficiency gas-fired condensing furnace for residential space heating, instead of a new baseline gas furnace. The measure could be installed in either an existing or new home. The installation is assumed to occur during a natural time of sale.

Evaluators should be aware that there will be an interaction between this measure and others, e.g. duct sealing, air sealing and insulation measures. Attempt should be made to account for this interaction where the measures occur in the same home within the same program period.

**Definition of Baseline Condition**

The baseline condition is a non-condensing gas furnace with an AFUE of 80%*[[335]](#footnote-335)*.

**Definition of Efficient Condition**

The efficient condition is an ENERGY STAR qualified gas-fired condensing furnace with an AFUE rating ≥ 90%.

**Annual Energy Savings Algorithm**

n/a. Note, if the furnace has an ECM fan, electric savings should be claimed as characterized inthe “Central Furnace Efficient Fan Motor” section of the TRM.

**Summer Coincident Peak kW Savings Algorithm**

n/a

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = (EFLHheat \* Btuh \* ((AFUEee/AFUEbase) - 1) /1,000,000

*Where:*

*EFLHheat = Equivalent Full Load Heating Hours*

|  |  |
| --- | --- |
| ***Location*** | ***EFLH*** |
| *Wilmington, DE* | *848[[336]](#footnote-336)* |
| *Baltimore, MD* | *620[[337]](#footnote-337)* |
| *Washington, DC* | *528*[[338]](#footnote-338) |

*BtuH = Input Capacity of Furnace*

*= Actual*

*AFUEbase = Efficiency in AFUE of baseline Furnace*

*= 0.80*

*AFUEee = Efficiency in AFUE of efficient Furnace*

*= Actual*

Illustrative example – do not use as default assumption

The purchase and installation of a 100,000 Btuh, 92% AFUE furnace in Maryland:

ΔMMBtu = (620 \* 100,000 \* ((0.92/0.8) – 1) /1,000,000

= 9.3 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is provided below[[339]](#footnote-339):

|  |  |
| --- | --- |
| **Efficiency of Furnace (AFUE)** | **Incremental Cost** |
| 90% | $630 |
| 92% | $802 |
| 96% | $1,747 |

**Measure Life**

The measure life is assumed to be 18 years[[340]](#footnote-340).

**Operation and Maintenance Impacts**

n/a

**Programmable Thermostat**

**Unique Measure Code: RS\_HV\_RTR\_PRGTHE\_0711**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

Programmable Thermostats can save energy through the advanced scheduling of setbacks to heating setpoints. Typical usage reduces the heating setpoint during times of the day when occupants are usually not at home (e.g. work hours) or during the night.

Note, savings are only provided for the reduction in heating load for fossil fuel fired heating systems. A literature review could not find any appropriate defensible source of cooling savings from programmable thermostats. It is inappropriate to assume a similar pattern of savings from setting your thermostat down during the heating season and up during the cooling season.

This is a retrofit measure.

**Definition of Baseline Condition**

A standard, non-programmable thermostat for central heating system (baseboard electric is excluded from this characterization).

**Definition of Efficient Condition**

A programmable thermostat is installed and programmed by a professional.

**Annual Energy Savings Algorithm**

n/a

**Summer Coincident Peak kW Savings Algorithm**

n/a

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = (Savings %) x (Heat Consumption)

*Where:*

*Savings % = Estimated percent reduction in heating load due to programmable thermostat*

*= 6.8% [[341]](#footnote-341)*

*Heat* Consumption *= Annual Home Heating* Consumption *(MMBtu)*

*= 50.1 [[342]](#footnote-342)*

ΔMMBtu = 0.068 \* 50.1

= 3.41 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure should be the actual unit cost and if installed via program administrators should also include labor cost[[343]](#footnote-343).

**Measure Life**

The measure life is assumed to be 10 years[[344]](#footnote-344).

**Operation and Maintenance Impacts**

n/a

**Room Air Conditioner Early Replacement**

**Unique Measure Code: RS\_HV\_RTR\_RA/CES\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes the early removal of an existing inefficient Room Air Conditioner unit from service, prior to its natural end of life, and replacement with a new ENERGY STAR qualifying unit. This measure is suitable for a Low Income or a Home Performance program.

Savings are calculated between the existing unit and the new efficient unit consumption during the assumed remaining life of the existing unit, and between a hypothetical new baseline unit and the efficient unit consumption for the remainder of the measure life.

This is a retrofit measure.

**Definition of Baseline Condition**

The baseline condition is the existing inefficient room air conditioning unit for the remaining assumed useful life of the unit, and then for the remainder of the measure life the baseline becomes a new replacement unit meeting the minimum federal efficiency standard (i.e. with an efficiency rating of 10.9 CEER*[[345]](#footnote-345)*).

**Definition of Efficient Condition**

The efficient condition is a new replacement room air conditioning unit meeting the ENERGY STAR efficiency standard (i.e. with an efficiency rating greater than or equal to 11.3 [[346]](#footnote-346)).

**Annual Energy Savings Algorithm**

Savings for remaining life of existing unit (1st 3 years)

ΔkWh = (Hours \* BtuH \* (1/EERexist - 1/EERee))/1,000

Savings for remaining measure life (next 9 years)

ΔkWh = (Hours \* BtuH \* (1/EERbase - 1/EERee))/1,000

*Where:*

*Hours = Run hours of Window AC unit*

*= 325 [[347]](#footnote-347)*

*Btuh = Capacity of replaced unit*

*= Actual or 8,500 if unknown [[348]](#footnote-348)*

*EERexist = Efficiency of existing unit in Btus per Watt-hour*

*= 7.7 [[349]](#footnote-349)*

*EERbase = Efficiency of baseline unit in Btus per Watt-hour*

*= 10.9 [[350]](#footnote-350)*

*EERee = Efficiency of ENERGY STAR unit in Btus per Watt-hour*

*= Actual*

Illustrative example – do not use as default assumption

Replacing existing 8,500 Btuh Room AC unit with a new ENERGY STAR unit with EER rating of 11.3:

Savings for remaining life of existing unit (1st 3 years)

ΔkWh = (325 \* 8,500 \* (1/7.7– 1/11.3)) / 1,000

= 114 kWh

Savings for remaining measure life (next 9 years)

ΔkWh = (325 \* 8,500 \* (1/10.9 – 1/11.3)) / 1,000

= 9 kWh

**Summer Coincident Peak kW Savings Algorithm**

Savings for remaining life of existing unit (1st 3 years)

ΔkW = ((BtuH \* (1/EERexist - 1/EERee))/1000) \* CF

Savings for remaining measure life (next 9 years)

ΔkW = ((BtuH \* (1/EERbase - 1/EERee))/1000) \* CF

*Where:*

*CFSSP  = Summer System Peak Coincidence Factor for Room A/C (hour ending 5pm on hottest summer weekday)*

*= 0.31 [[351]](#footnote-351)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Room A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.3[[352]](#footnote-352)*

Illustrative example – do not use as default assumption

Replacing existing 8,500 Btuh Room AC unit with a new ENERGY STAR unit with EER rating of 11.3.

Savings for remaining life of existing unit (1st 3 years)

ΔkW*SSP*  = ((8,500 \* (1/7.7 – 1/11.3)) / 1,000) \* 0.31

= 0.11 kW

Savings for remaining measure life (next 9 years)

ΔkW*SSP* = ((8,500 \* (1/10.9 – 1/11.3)) / 1,000) \* 0.31

= 0.0086 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure should be the actual cost of the replacement unit and any cost of installation labor.

Note, the deferred baseline replacement cost is presented under Operation and Maintenance Impacts.

**Measure Life**

The measure life is assumed to be 12 years[[353]](#footnote-353). Note this characterization also assumes there is 3 years of remaining useful life of the unit being replaced[[354]](#footnote-354).

**Operation and Maintenance Impacts**

The net present value of the deferred replacement cost (the cost associated with the replacement of the existing unit with a standard unit that would have occurred in 3 years, had the existing unit not been replaced) should be calculated as:

NPVdeferred replacement cost = (Actual Cost of ENERGY STAR unit - $40[[355]](#footnote-355)) \* 69%[[356]](#footnote-356).

Note that this is a lifecycle cost savings (i.e. a negative cost).

**Room Air Conditioner Early Retirement / Recycling**

**Unique Measure Code: RS\_HV\_ERT\_RA/C\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes the savings resulting from implementing a drop off service taking existing working inefficient Room Air Conditioner units from service, prior to their natural end of life. This measure assumes that a percentage of these units will ultimately be replaced with a baseline standard efficiency unit (note that if it is actually replaced by a new ENERGY STAR qualifying unit, the savings increment between baseline and ENERGY STAR should be captured under the ENERGY STAR Room AC Time of Sale measure).

**Definition of Baseline Condition**

The baseline condition is the existing inefficient room air conditioning unit.

**Definition of Efficient Condition**

Not applicable. This measure relates to the retiring of an existing inefficient unit. A percentage of units however are assumed to be replaced with a baseline new unit and the savings are therefore reduced to account for these replacement units.

**Annual Energy Savings Algorithm**

ΔkWh = ((Hours \* BtuH \* (1/EERexist))/1,000) -

(%replaced \* ((Hours \* BtuH \* (1/EERnewbase))/ 1,000)

*Where:*

*Hours = Run hours of Window AC unit*

*= 325 [[357]](#footnote-357)*

*Btu/hour = Capacity of replaced unit*

*= Actual or 8,500 if unknown [[358]](#footnote-358)*

*EERexist = Efficiency of existing unit in Btus per Watt-hour*

*= Actual or 7.7 if unknown [[359]](#footnote-359)*

%replaced = Percentage of units dropped off that are replaced in the home

= 76% [[360]](#footnote-360)

EERnewbase *= Efficiency of new baseline unit in Btus per Watt-hour*

*=* 10.9[[361]](#footnote-361)

Illustrative example – do not use as default assumption

The turn in of an 8,500 Btuh, 7.7 EER unit:

ΔkWh = ((325 \* 8,500 \* (1/7.7))/1,000) -

(0.76 \* ((325 \* 8,500 \* (1/10.9))/1,000)

= 166 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((BtuH \* (1/EERexist))/1,000) -

(%replaced \* ((BtuH \* (1/EERnewbase))/1,000) \* CF

*Where:*

*CFSSP  = Summer System Peak Coincidence Factor for Room A/C (hour ending 5pm on hottest summer weekday)*

*= 0.31 [[362]](#footnote-362)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Room A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.3[[363]](#footnote-363)*

Illustrative example – do not use as default assumption

The turn in of an 8500 Btuh, 7.7 EER unit:

ΔkWSSM = ((8,500 \* (1/7.7))/1,000) -

(0.76 \* ((8,500 \* (1/10.9))/1,000) \* 0.31

= 0.16 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure should be the actual implementation cost for recycling the existing unit, plus $129 to account for the replacement of 76% of the units[[364]](#footnote-364).

**Measure Life**

The measure life is assumed to be 3 years[[365]](#footnote-365).

**Operation and Maintenance Impacts**

The net present value of the deferred replacement cost (the cost associated with the replacement of those units that would be replaced, with a standard unit that would have had to have occurred in 3 years, had the existing unit not been replaced) is calculated as $89.36[[366]](#footnote-366).

**Boiler Pipe Insulation**

**Unique Measure Code: RS\_HV\_RTR\_PIPEIN\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure describes adding insulation to un-insulated boiler pipes in un-conditioned basements or crawlspaces.

Note, the algorithm provided to calculate savings may be used to determine an appropriate deemed savings value if the programs can provide appropriate average values for each of the variables.

This is a retrofit measure.

**Definition of Baseline Condition**

The baseline condition is an un-insulated boiler pipe.

**Definition of Efficient Condition**

The efficient condition is installing pipe wrap insulation to a length of boiler pipe.

**Annual Energy Savings Algorithm**

N/A

**Summer Coincident Peak kW Savings Algorithm**

N/A

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = (((1/Rexist \* Cexist) – (1/Rnew \* Cnew)) \* FLH\_heat \* L \* ΔT) / ηBoiler /1,000,000

*Where:*

*Rexist = Pipe heat loss coefficient of uninsulated pipe [(hr-°F-ft2)/Btu]*

*= 0.5[[367]](#footnote-367)*

*Rnew = Pipe heat loss coefficient of insulated pipe [(hr-°F-ft2)/Btu]*

*= Actual (0.5 + R value of insulation)*

*EFLH\_heat = Equivalent Full load hours of heating*

|  |  |
| --- | --- |
| ***Location*** | ***EFLH*** |
| *Wilmington, DE* | *848 [[368]](#footnote-368)* |
| *Baltimore, MD* | *620 [[369]](#footnote-369)* |
| *Washington, DC* | *528 [[370]](#footnote-370)* |

*L = Length of boiler pipe in unconditioned space covered by pipe wrap (ft)*

*= Actual*

*Cexist = Circumference of bare pipe (ft) (Diameter (in) \* π/12)*

*= Actual (0.5” pipe = 0.131ft, 0.75” pipe = 0.196ft)*

*Cnew  = Circumference of pipe with insulation(ft) (Diameter (in) \* π/12)*

*= Actual*

*ΔT = Average temperature difference between circulated heated water and unconditioned space air temperature (°F) [[371]](#footnote-371)*

|  |  |  |
| --- | --- | --- |
| ***Pipes location*** | ***Outdoor Reset Controls*** | ***ΔT (°F)*** |
| *Unconditioned basement* | *Boiler without reset control* | *110* |
| *Boiler with reset control* | *70* |
| *Crawlspace* | *Boiler without reset control* | *120* |
| *Boiler with reset control* | *80* |

*ηBoiler = Efficiency of boiler*

*= 0.8 [[372]](#footnote-372)*

Illustrative example – do not use as default assumption

Insulating 15 feet of 0.75” pipe with R-3 wrap (0.75” thickness) in a crawl space in Wilmington, DE with a boiler without reset controls:

ΔMMBtu = (((1/Rexist \* Cexist) – (1/Rnew \* Cnew)) \* FLH\_heat \* L \* ΔT) / ηBoiler /1,000,000

= (((1/0.5 \* 0.196) – (1/3.5 \* ((0.75+0.75+0.75) \* π/12))) \* 848 \* 15 \* 120) / 0.8 / 1,000,000

= 0.43 MMBtu

**Annual Water Savings Algorithm**

N/A

**Incremental Cost**

The measure cost including material and installation is assumed to be $3 per linear foot[[373]](#footnote-373).

**Deemed Lifetime of Efficient Equipment**

The assumed lifetime of the measure is 15 years[[374]](#footnote-374).

**Operation and Maintenance Impacts**

N/A

**Boiler Reset Controls\***

**Unique Measure Code: RS\_HV\_TOS\_BLRRES\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to improving system efficiency by adding controls to residential heating boilers to vary the boiler entering water temperature relative to heating load as a function of the outdoor air temperature to save energy. The water can be run a little cooler during fall and spring, and a little hotter during the coldest parts of the winter. A boiler reset control has two temperature sensors - one outside the house and one in the boiler water. As the outdoor temperature goes up and down, the control adjusts the water temperature setting to the lowest setting that is meeting the house heating demand. There are also limits in the controls to keep a boiler from operating outside of its safe performance range.

**Definition of Baseline Condition**

Existing condensing boiler in a single family residential setting without boiler reset controls.

**Definition of Efficient Condition**

Natural gas single family residential customer adding boiler reset controls capable of resetting the boiler supply water temperature in an inverse fashion with outdoor air temperature. The system must be set so that the minimum temperature is not more than 10 degrees above manufacturer’s recommended minimum return temperature. This boiler reset measure is limited to existing condensing boilers serving a single family residence. Boiler reset controls for non-condensing boilers in single family residences should be implemented as a custom measure, and the cost-effectiveness should be confirmed.

**Annual Energy Savings Algorithm**

n/a

**Summer Coincident Peak kW Savings Algorithm**

n/a

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = (Savings %) \* (EFLHheat \* Btuh)/ 1,000,000

*Where:*

*Savings % = Estimated percent reduction in heating load due to boiler reset controls being installed*

*= 5%[[375]](#footnote-375)*

*EFLHheat = Equivalent Full Load Heating Hours*

|  |  |
| --- | --- |
| ***Location*** | ***EFLH*** |
| *Wilmington, DE* | *848[[376]](#footnote-376)* |
| *Baltimore, MD* | *620[[377]](#footnote-377)* |
| *Washington, DC* | *528*[[378]](#footnote-378) |

*BtuH = Input Capacity of Boiler*

*= Actual*

Illustrative example – do not use as default

A boiler reset control is applied to a 80,000 BtuH boiler in Baltimore, MD.

ΔMMBtu = 0.05 \* (620 \* 80,000)/1,000,000

= 2.48 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The cost of this measure is $612[[379]](#footnote-379)

**Measure Life**

The life of this measure is 18 years[[380]](#footnote-380)

**Operation and Maintenance Impacts**

n/a

**Ground Source Heat Pumps**

**Unique Measure Code: RS\_HV\_TOS\_GSHPS\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure characterizes the installation of an ENERGY STAR qualified Ground Source Heat Pump (GSHP) either during new construction or at Time of Sale/Replacement of an existing system(s). The baseline is always assumed to be a new baseline Air Source Heat Pump. Savings are calculated due to the GSHP providing heating and cooling more efficiently than a baseline ASHP, and where a desuperheater is installed, additional Domestic Hot Water (DHW) savings due to displacing existing water heating.

The ENERGY STAR efficiency standards are presented below.

|  |  |  |
| --- | --- | --- |
| ENERGY STAR Requirements (Effective January 1, 2012) | | |
| **Product Type** | **Cooling EER** | **Heating COP** |
| **Water-to-air** | | |
| Closed Loop | 17.1 | 3.6 |
| Open Loop | 21.1 | 4.1 |
| **Water-to-Water** | | |
| Closed Loop | 16.1 | 3.1 |
| Open Loop | 20.1 | 3.5 |
| DGX | 16 | 3.6 |

Evaluators should be aware that there will be an interaction between this measure and others, e.g. duct sealing, air sealing and insulation measures. Attempt should be made to account for this interaction where the measures occur in the same home within the same program period.

**Definition of Baseline Condition**

New Construction:

The baseline equipment is assumed to be an Air Source Heat Pump meeting the Federal Standard efficiency level; 14 SEER, 8.2 HSPF and 11.8[[381]](#footnote-381) EER. If a desuperheater is installed, the baseline for DHW savings is assumed to be a Federal Standard electric hot water heater, with Energy Factor calculated as follows[[382]](#footnote-382):

For <=55 gallons: EF = 0.96 – (0.0003 \* rated volume in gallons)

For >55 gallons: EF = 2.057 – (0.00113 \* rated volume in gallons)

If size is unknown, assume 50 gallon; 0.945 EF.

Time of Sale:

The baseline equipment is assumed to be an Air Source Heat Pump meeting the Federal Standard efficiency level; 14 SEER, 8.2 HSPF and 11.8 EER. If a desuperheater is installed, the baseline for DHW savings is assumed to be the existing home’s hot water heater fuel and efficiency.

If electric DHW, and unknown efficiency – assume efficiency is equal to pre 4/2015 Federal Standard:

EF = 0.93 – (0.00132 \* rated volume in gallons)[[383]](#footnote-383)

If size is unknown, assume 50 gallon; 0.864 EF

If gas water heater, and unknown efficiency – assume efficiency is equal to pre 4/2015 Federal Standard:

EF = (0.67 – 0.0019 \* rated volume in gallons)[[384]](#footnote-384).

If size is unknown, assume 40 gallon; 0.594 EF

If DHW fuel is unknown, assume electric DHW provided above.

**Definition of Efficient Condition**

In order for this characterization to apply, the efficient equipment must be a Ground Source Heat Pump unit meeting the minimum ENERGY STAR efficiency level standards effective at the time of installation as detailed above.

**Annual Energy Savings Algorithm**

ΔkWh = [Cooling savings] + [Heating savings] + [DHW savings]

= [(FLHcool \* Capacity\_cooling \* (1/SEERbase– (1/EERPL)/1000] +

[FLHheat \* Capacity\_heating \* (1/HSPFbase – (1/COPPL \* 3.412)))/1000] + [ElecDHW \* %DHWDisplaced \* (((1/EFELEC) \* GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 3412)]

*Where:*

*FLHcool = Full load cooling hours*

*Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | ***Run Hours*** |
| *Wilmington, DE* | *524 [[385]](#footnote-385)* |
| *Baltimore, MD* | *542 [[386]](#footnote-386)* |
| *Washington, DC* | *681* |

*Capacity\_cooling = Cooling Capacity of Ground Source Heat Pump (Btu/hr)*

*= Actual (1 ton = 12,000Btu/hr)*

*SEERbase = SEER Efficiency of new replacement baseline unit*

*= 14[[387]](#footnote-387)*

*EERPL = Part Load EER Efficiency of efficient GSHP unit[[388]](#footnote-388)*

*= Actual installed*

*FLHheat = Full load heating hours*

|  |  |
| --- | --- |
| ***Location*** | ***EFLH*** |
| *Wilmington, DE* | *848[[389]](#footnote-389)* |
| *Baltimore, MD* | *620[[390]](#footnote-390)* |
| *Washington, DC* | *528*[[391]](#footnote-391) |

*Capacity\_heating = Heating Capacity of Ground Source Heat Pump (Btu/hr)*

*= Actual (1 ton = 12,000Btu/hr)*

*HSPFbase =Heating System Performance Factor of new replacement baseline heating system (kBtu/kWh)*

*=8.2 [[392]](#footnote-392)*

*COPPL = Part Load Coefficient of Performance of efficient unit[[393]](#footnote-393)*

*= Actual Installed*

*3.412 = Constant to convert the COP of the unit to the Heating Season Performance Factor (HSPF).*

*ElecDHW = 1 if existing DHW is electrically heated*

*= 0 if existing DHW is not electrically heated*

*%DHWDisplaced = Percentage of total DHW load that the GSHP will provide*

*= Actual if known*

*= If unknown and if desuperheater installed assume 44%[[394]](#footnote-394)*

*= 0% if no desuperheater installed*

*EFELEC = Energy Factor (efficiency) of electric water heater*

*For new construction assume federal standard[[395]](#footnote-395):*

*For <=55 gallons: 0.96 – (0.0003 \* rated volume in gallons)*

*For >55 gallons: 2.057 – (0.00113 \* rated volume in gallons)*

*If size is unknown, assume 50 gallon; 0.945 EF.*

*For Time of Sale, if electric DHW use Actual efficiency. If unknown – assume efficiency is equal to pre 4/2015 Federal Standard:*

*EF = 0.93 – (0.00132 \* rated volume in gallons)[[396]](#footnote-396)*

*If size is unknown, assume 50 gallon; 0.864 EF*

*GPD = Gallons Per Day of hot water use per person*

*= 45.5 gallons hot water per day per household/2.59 people per household[[397]](#footnote-397)*

*= 17.6*

*Household = Average number of people per household*

*= 2.53 [[398]](#footnote-398)*

*365.25 = Days per year*

*γWater = Specific weight of water*

*= 8.33 pounds per gallon*

*Tout = Tank temperature*

*= 125°F*

*Tin = Incoming water temperature from well or municipal system*

*= 60.9 [[399]](#footnote-399)*

1. *= Heat Capacity of water (1 Btu/lb\*°F)*

*3412 = Conversion from Btu to kWh*

Illustrative Example – do not use as default assumption

New Construction:

For example, a 3 ton unit with Part Load EER rating of 19 and Part Load COP of 4.4 with desuperheater is installed with a 50 gallon electric water heater in single family house in Baltimore:

ΔkWh = [(FLHcool \* Capacity\_cooling \* (1/SEERbase – (1/EERPL)/1000] + [(FLHheat \* Capacity\_heating \* (1/HSPFbase – (1/COPPL \* 3.412)))/1000] + [ElecDHW \* %DHWDisplaced \* (((1/ EFELEC exist) \* GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 3412)]

ΔkWh = [(542 \* 36,000 \* (1/14 – 1/19)) / 1000] + [(620 \* 36,000 \* (1/8.2 – 1/ (4.4\*3.412))) / 1000] + [1 \* 0.44 \* (((1/0.945) \* 17.6 \* 2.53 \*365.25 \* 8.33 \* (125-60.9) \* 1)/3412)]

= 367 + 1235 + 1185

= 2787 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (Capacity\_cooling \* (1/EERbase - 1/EERFL))/1000) \* CF

*Where:*

*EERbase = EER Efficiency of new replacement unit*

*= 11.8[[400]](#footnote-400)*

*EERFL = Full Load EER Efficiency of ENERGY STAR GSHP unit [[401]](#footnote-401)*

*= Actual*

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[402]](#footnote-402)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[403]](#footnote-403)*

Illustrative Example– do not use as default assumption

New Construction or Time of Sale:

For example, a 3 ton unit with Full Load EER rating of 19:

ΔkWSSP = ((36,000 \* (1/11.8 – 1/19))/1000) \* 0.69

= 0.80 kW

ΔkWPJM = ((36,000 \* (1/11 – 1/19))/1000) \* 0.66

= 0.76 kW

**Annual Fossil Fuel Savings Algorithm**

Savings for Time of Sale where existing hot water heater is gas fired:

ΔMMBtu = [DHW Savings]

= [(1 – ElecDHW) \* %DHWDisplaced \* (1/ EFGas BASE \* GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 1,000,000)]

*Where:*

*EFGas exist = Energy Factor (efficiency) of existing gas water heater*

*= Actual. If unknown assume efficiency is equal to pre 4/2015 Federal Standard:*

*= (0.67 – 0.0019 \* rated volume in gallons)[[404]](#footnote-404).*

*If size is unknown, assume 40 gallon; 0.594 EF*

*All other variables provided above*

Illustrative Example – do not use as default assumption

Time of Sale:

For example, a GSHP with desuperheater is installed with a 40 gallon gas water heater in single family house in Baltimore:

ΔMMBtu = [(1 – ElecDHW) \* %DHWDisplaced \* (1/ EFGas BASE \* GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 1,000,000)]

= [(1 – 0) \* 0.44 \* (((1/0.594) \* 17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 - 60.9) \* 1)/1,000,000)]

= 6.4 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

New Construction and Time of Sale: The actual installed cost of the Ground Source Heat Pump should be used (default of $3957 per ton[[405]](#footnote-405)), minus the assumed installation cost of the baseline equipment ($2355 per ton for ASHP[[406]](#footnote-406)).

**Measure Life**

The expected measure life is assumed to be 25 years[[407]](#footnote-407).

**Operation and Maintenance Impacts**

N/A

**High Efficiency Bathroom Exhaust Fan\***

**Unique Measure Code(s): RS\_HV\_TOS\_BTHFAN\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This market opportunity is defined by the need for continuous mechanical ventilation due to reduced air-infiltration from a tighter building shell. In retrofit projects, existing fans may be too loud, or insufficient in other ways, to be operated as required for proper ventilation. This measure assumes a fan capacity of 50 CFM rated at a sound level of less than 2.0 sones at 0.1 inches of water column static pressure. This measure may be applied to larger capacity, up to 130 CFM, efficient fans with bi-level controls because the savings and incremental costs are very similar. All eligible installations shall be sized to provide the mechanical ventilation rate indicated by ASHRAE 62.2.

**Definition of Baseline Condition**

New standard efficiency (average CFM/Watt of 3.1[[408]](#footnote-408)) exhaust-only ventilation fan, quiet (< 2.0 sones) operating in accordance with recommended ventilation rate indicated by ASHRAE 62.2[[409]](#footnote-409).

**Definition of Efficient Condition**

New efficient (average CFM/watt of 8.3[[410]](#footnote-410)) exhaust-only ventilation fan, quiet (< 2.0 sones) Continuous operation in accordance with recommended ventilation rate indicated by ASHRAE 62.2[[411]](#footnote-411)

**Annual Energy Savings Algorithm**

ΔkWh = (CFM \* (1/ηBaseline - 1/ηEfficient)/1000) \* Hours

*Where:*

*CFM = Nominal Capacity of the exhaust fan*

*= 50 CFM*[[412]](#footnote-412)

*ηBaseline = Average efficacy for baseline fan*

*= 3.1 CFM/Watt*[[413]](#footnote-413)

*ηEffcient = Average efficacy for efficient fan*

*= 8.3 CFM/Watt*[[414]](#footnote-414)

*Hours = assumed annual run hours,*

*= 8760 for continuous ventilation.*

ΔkWh = (50 \* (1/3.1 – 1/8.3)/1000) \* 8760

= 88.5 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (CFM \* (1/ηBaseline - 1/ηEfficient)/1000) \* CF

*Where:*

*CF = Summer Peak Coincidence Factor*

*= 1.0 (continuous operation)*

*Other variables as defined above*

ΔkW = (50 \* (1/3.1 – 1/8.3)/1000) \* 1.0

= 0.0101 kW

**Deemed Lifetime of Efficient Equipment**

The expected measure life is assumed to be 19 years[[415]](#footnote-415).

**Deemed Measure Cost**

Incremental cost per installed fan is $43.50 for quiet, efficient fans[[416]](#footnote-416).

**ENERGY STAR Ceiling Fan**

**Unique Measure Code: RS\_HV\_TOS\_ESCFN\_0415, RS\_HV\_NC\_ESCFN\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

A ceiling fan/light unit meeting the ENERGY STAR efficiency specifications is installed in place of a model meeting the federal standard. ENERGY STAR qualified ceiling fan/light combination units are over 60% more efficient than conventional fan/light units, and use improved motors and blade designs[[417]](#footnote-417).

Due to the savings from this measure being derived from more efficient ventilation and more efficient lighting, and the loadshape and measure life for each component being very different, the savings are split in to the component parts and should be claimed together. Lighting savings should be estimated utilizing the General Purpose CFL Screw Based, Residential measure.

**Definition of Baseline Equipment**

The baseline equipment is assumed to be a standard fan with EISA qualified incandescent or halogen light bulbs.

**Definition of Efficient Equipment**

The efficient equipment is defined as an ENERGY STAR certified ceiling fan with integral CFL bulbs.

**Annual Energy Savings Algorithm**

∆kWh = ΔkWhfan + ΔkWhLight

∆kWhfan = [Days\* FanHours \* ((%Lowbase \* WattsLowbase) + (%Medbase \* WattsMedbase) + (%Highbase \* WattsHighbase))/1000 ] - [Days\* FanHours \* ((%LowES \* WattsLowES) + (%MedES \* WattsMedES) + (%HighES \* WattsHighES))/1000]

∆kWhlight = ((WattsBase - WattsEE)/1000) \* ISR \* HOURS \* (WHFeHeat + (WHFeCool – 1))

See General Purpose CFL Screw Based, Residential measure (assume ISR = 1.0)

*Where[[418]](#footnote-418):*

*Days = Days used per year*

*= Actual. If unknown use 365.25 days/year*

*FanHours = Daily Fan “On Hours”*

*= Actual. If unknown use 3 hours*

*%Lowbase = Percent of time spent at Low speed of baseline*

*= 40%*

*WattsLowbase = Fan wattage at Low speed of baseline*

*= Actual. If unknown use 15 watts*

*%Medbase = Percent of time spent at Medium speed of baseline*

*= 40%*

*WattsMedbase = Fan wattage at Medium speed of baseline*

*= Actual. If unknown use 34 watts*

*%Highbase = Percent of time spent at High speed of baseline*

*= 20%*

*WattsHighbase = Fan wattage at High speed of baseline*

*= Actual. If unknown use 67 watts*

*%LowES = Percent of time spent at Low speed of ENERGY STAR*

*= 40%*

*WattsLowES = Fan wattage at Low speed of ENERGY STAR*

*= Actual. If unknown use 6 watts*

*%MedES = Percent of time spent at Medium speed of ENERGY STAR*

*= 40%*

*WattsMedES = Fan wattage at Medium speed of ENERGY STAR*

*= Actual. If unknown use 23 watts*

*%HighES = Percent of time spent at High speed of ENERGY STAR*

*= 20%*

*WattsHighES = Fan wattage at High speed of ENERGY STAR*

*= Actual. If unknown use 56 watts*

*For ease of reference, the fan assumptions are provided below in table form:*

|  |  |  |  |
| --- | --- | --- | --- |
|  | ***Low Speed*** | ***Medium Speed*** | ***High Speed*** |
| *Percent of Time at Given Speed* | *40%* | *40%* | *20%* |
| *Conventional Unit Wattage* | *15* | *34* | *67* |
| *ENERGY STAR Unit Wattage* | *6* | *23* | *56* |
| *∆W* | *9* | *11* | *11* |

*If the lighting WattsBase and WattsEE is unknown, assume the following*

*WattsBase = 3 x 43 = 129 W*

*WattsEE = 1 x 42 = 42 W*

Deemed savings if using defaults provided above:

ΔkWhfan = [365.25 \* 3 \* ((0.4 \* 15) + (0.4 \* 34)+(0.2 \* 67))/1000] – [365.25 \* 3 \*((0.4 \* 6)+(0.4 \* 23)+(0.2 \* 56))/1000]

= 36.2 – 25.0

= 11.2 kWh

ΔkWhlight =((129 – 42)/1000) \* 1.0 \* 898 \* (0.894 + (1.09-1))

= 76.9 kWh

ΔkWh = 11.2 + 76.9

= 88.1 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWFan + ΔkWlight

ΔkWFan = ((WattsHighbase - WattsHighES)/1000) \* CFfan

ΔkWLight = ((WattsBase - WattsEE) /1000) \* ISR \* WHFd \* CFlight

See General Purpose CFL Screw Based, Residential measure (assume ISR = 1.0)

*Where:*

*CFfanSSP  = Summer System Peak Coincidence Factor (hour ending 5pm on hottest summer weekday)*

*= 0.31 [[419]](#footnote-419)*

*CFfanPJM  = PJM Summer Peak Coincidence Factor (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.3[[420]](#footnote-420)*

*CFlight= Summer Peak coincidence factor for lighting savings*

|  |  |  |
| --- | --- | --- |
| **Installation Location** | **Type** | **Coincidence Factor CF** |
| Residential interior and  in-unit Multi Family | Utility Peak CF | 0.082[[421]](#footnote-421) |
| PJM CF | 0.084[[422]](#footnote-422) |

Deemed savings if using defaults provided above:

ΔkWfan ssp = ((67-56)/1000) \* 0.31

=0.0034 kW

ΔkWlight ssp =((129 – 42)/1000) \* 1.0 \* 1.18 \* 0.073

= 0.0075 kW

ΔkWssp  = 0.0034 + 0.0075

= 0.011 kW

ΔkWfan pjm = ((67-56)/1000) \* 0.3

=0.0033 kW

ΔkWlight pjm =((129 – 42)/1000) \* 1.0 \* 1.18 \* 0.084

= 0.0086 kW

ΔkWpjm  = 0.0033 + 0.0086

= 0.012 kW

**Annual Fossil Fuel Savings Algorithm**

Heating penalty from improved lighting:

ΔMMBtuPenalty = - ((((WattsBase - WattsEE) / 1000) \* ISR \* Hours \* HF \* 0.003412) / ηHeat) \* %FossilHeat

See General Purpose CFL Screw Based, Residential measure (assume ISR = 1.0)

Deemed savings if using defaults provided above:

ΔMMBtuPenalty = - ((((129 – 42) / 1000) \* 1.0 \* 898 \* 0.47 \* 0.003412) / 0.72) \* 0.625

= -0.11

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

Incremental cost of unit is assumed to be $46.[[423]](#footnote-423)

**Measure Life**

The fan savings measure life is assumed to be 10 years.2

The lighting savings measure life is assumed to be 5 years as per General Purpose CFL Screw Based, Residential measure.

**Operation and Maintenance Impacts**

See General Purpose CFL Screw Based, Residential measure.

Deemed baseline O&M cost if using defaults provided above:

|  |  |  |
| --- | --- | --- |
| **Year** | **NPV of baseline Replacement Costs**  **Per bulb** | **Total NPV of baseline Replacement Costs (assuming 3 bulbs)** |
| 2015 | $3.83 | $11.49 |
| 2016 | $2.94 | $8.82 |
| 2017 | $2.01 | $6.03 |

*Domestic Hot Water (DHW) End Use*

**Low Flow Shower Head**

**Unique Measure Code(s): RS\_WT\_INS\_SHWRHD\_0414 and RS\_WT\_TOS\_SHWRHD\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a low flow (≤2.0 GPM) showerhead in a home. This is a retrofit direct install measure or a new installation.

**Definition of Baseline Condition**

The baseline is a standard showerhead using 2.5 GPM. For direct install programs, utilities may choose to measure the actual flow rate of the existing showerhead and use that in the algorithm below

**Definition of Efficient Condition**

The efficient condition is an energy efficient showerhead using the rated GPM of the installed showerhead. If actual flow rates of the baseline fixtures are used in a direct install program, then the actual flow rate of the installed aerators should be used as well.

**Annual Energy Savings Algorithm**

If electric domestic water heater:

ΔkWH[[424]](#footnote-424) = ((((GPMbase - GPMlow) / GPMbase) \* # people \* gals/day/person \* days/year) / SH/home \* 8.3 \* (TEMPsh - TEMPin) / 1,000,000) / DHW Recovery Efficiency / 0.003412

*Where:*

*GPMbase = Gallons Per Minute of baseline showerhead*

*= 2.5 [[425]](#footnote-425) or actual flow rate if recorded*

*GPMlow = Gallons Per Minute of low flow showerhead*

*= Rated flow rate of unit installed or actual flow rate if baseline flow rate used.*

*# people = Average number of people per household*

*= 2.56 [[426]](#footnote-426)*

*gals/day/person = Average gallons per day used for showering*

*= TimeShower \* GPMBase \* ShowersPerson*

*= if unknown, use 11.6 [[427]](#footnote-427)*

*days/y = Days shower used per year*

*= 365*

*Showers/home = Average number of showers in the home*

*= 1.6 [[428]](#footnote-428)*

*8.3 = Constant to convert gallons to lbs*

*TEMPsh = Assumed temperature of water used for shower*

*= 105*

*TEMPin = Assumed temperature of water entering house*

*= 60.9 [[429]](#footnote-429)*

*DHW Recovery Efficiency = Recovery efficiency of electric water heater*

*= 0.98 [[430]](#footnote-430)*

*0.003412 = Constant to convert MMBtu to kWh*

Illustrative example – do not use as default assumption

For a 2.0GPM rated showerhead:

ΔkWH = ((((2.5 – 2.0) / 2.5) \*2.56 \* 11.6 \* 365) / 1.6 \* 8.3 \* (105-60.9) / 1,000,000) / 0.98 / 0.003412

= 148 kWh

Note, utilities may consider whether it is appropriate to claim kWh savings from the reduction in water consumption arising from this measure. The kWh savings would be in relation to the pumping and wastewater treatment. See water savings for characterization.

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/hours \* CF

*Where:*

*Hours = Average number of hours per year spent using shower head*

*= (Gal/person \* # people \* 365) / SH/home / GPM / 60*

*= (11.6 \* 2.56 \* 365) / 1.6 / 2.5 / 60*

*= 45 hours*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.00371 [[431]](#footnote-431)*

Illustrative example – do not use as default assumption

For a 2.0GPM rated showerhead:

ΔkW = 148 / 45 \* 0.00371

= 0.0122 kW

**Annual Fossil Fuel Savings Algorithm**

If fossil fuel domestic water heater:

ΔMMBtu = ((((GPMbase - GPMlow) / GPMbase) \* # people \* gals/day \* days/year)) / SH/home \* 8.3 \* (TEMPsh - TEMPin) / 1,000,000) / Gas DHW Recovery Efficiency

*Where:*

*Gas DHW Recovery Efficiency = Recovery efficiency of electric water heater*

*= 0.75 [[432]](#footnote-432)*

*All other variables As above*

Illustrative example – do not use as default assumption

For a 2.0GPM rated showerhead:

ΔMMBtu = ((((2.5 – 2.0) / 2.5) \* 2.56 \* 11.6 \* 365) / 1.6 \* 8.3 \* (105-60.9) / 1,000,000) / 0.75

= 0.661 MMBtu

**Annual Water Savings Algorithm**

Water Savings = (((GPMbase - GPMlow) / GPMbase) \* # people \* gals/day \* days/year) / SH/home /748

*Where:*

*748 = Constant to convert from gallons to CCF*

*All other variables as above*

Illustrative example – do not use as default assumption

For a 2.0GPM rated showerhead:

Water Savings = ((((2.5 – 2.0) / 2.5) \* 2.56 \* 11.6 \* 365)) / 1.6 / 748

= 1.81 CCF

kWh Savings from Water Reduction

The kWh savings from the waste reduction characterized above is now estimated. Please note that utilities’ must be careful not to double count the monetary benefit of these savings within cost effectiveness testing if the avoided costs of water already include the associated electric benefit.

ΔkWhwater[[433]](#footnote-433)  = 2.07 kWh/CCF \* ∆Water (CCF)

Illustrative example – do not use as default assumption

For a 2.0GPM rated showerhead:

ΔkWhwater = 2.07 \* 1.81

= 3.7kWh

**Incremental Cost**

As a retrofit measure, the incremental cost will be the actual cost of installing the new showerhead. As a time of sale measure, the incremental cost is assumed to be $6.[[434]](#footnote-434)

**Measure Life**

The measure life is assumed to be 10 years.[[435]](#footnote-435)

**Operation and Maintenance Impacts**

When a retrofit measure, there would be a very small O&M benefit associated with the deferral of the next replacement, but this has conservatively not been characterized.

**Faucet Aerators**

**Unique Measure Code(s): RS\_WT\_INS\_FAUCET\_0414 and RS\_WT\_TOS\_FAUCET\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a low flow (≤1.5 GPM) faucet aerator in a home. This could be a retrofit direct install measure or a new installation.

**Definition of Baseline Condition**

The baseline is a standard faucet aerator using 2.2 GPM. For direct install programs, utilities may choose to measure the actual flow rate of the existing aerator and use that in the algorithm below

**Definition of Efficient Condition**

The efficient condition is an energy efficient faucet aerator using rated GPM of the installed aerator. If actual flow rates of the baseline fixtures are used in a direct install program, then the actual flow rate of the installed aerators should be used as well.

**Annual Energy Savings Algorithm**

If electric domestic water heater:

ΔkWH[[436]](#footnote-436) = (((((GPMbase - GPMlow) / GPMbase) \* # people \* gals/day/person \* days/year \* DR) / (F/home)) \* 8.3 \* (TEMPft - TEMPin) / 1,000,000) / DHW Recovery Efficiency / 0.003412

*Where:*

*GPMbase = Gallons Per Minute of baseline faucet*

*= 2.2 [[437]](#footnote-437) or actual flow rate if recorded*

*GPMlow = Gallons Per Minute of low flow faucet*

*= Rated flow rate of unit installed or actual flow rate if baseline flow rate used.*

*# people = Average number of people per household*

*= 2.56 [[438]](#footnote-438)*

*gals/day/person = Average gallons per day used by faucet per person*

*=* Time*faucet* \*GPM*base* \* Flow***base***

*= if unknown, use**10.9 [[439]](#footnote-439)*

*days/y = Days faucet used per year*

*= 365*

*DR = Percentage of water flowing down drain (if water is collected in a sink, a faucet aerator will not result in any saved water)*

*= 50% [[440]](#footnote-440)*

*F/home = Average number of faucets in the home*

*= 3.5 [[441]](#footnote-441)*

*8.3 = Constant to convert gallons to lbs*

*TEMPft = Assumed temperature of water used by faucet*

*= 80*

*TEMPin = Assumed temperature of water entering house*

*= 60.9 [[442]](#footnote-442)*

*DHW Recovery Efficiency = Recovery efficiency of electric water heater*

*= 0.98 [[443]](#footnote-443)*

*0.003412 = Constant to converts MMBtu to kWh*

Illustrative example – do not use as default assumption

For a 1.5 GPM rated aerator:

ΔkWH = ((((2.2 – 1.5) / 2.2) \* 2.56 \* 10.9 \* 365 \* 0.5) / 3.5 \* 8.3 \* (80-60.9) / 1,000,000) / 0.98 / 0.003412

= 22 kWh

Note, utilities may consider whether it is appropriate to claim kWh savings from the reduction in water consumption arising from this measure. The kWh savings would be in relation to the pumping and wastewater treatment. See water savings for characterization.

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/hours \* CF

*Where:*

*Hours = Average number of hours per year spent using faucet*

*= (Gal/person \* # people \* 365) /(F/home) / GPM / 60*

*= (10.9 \* 2.56 \* 365) / 3.5 / 2.2 / 60*

*= 22 hours*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.00262 [[444]](#footnote-444)*

Illustrative example – do not use as default assumption

For a 1.5 GPM rated aerator:

ΔkW = 22 / 22 \* 0.00262

= 0.0026 kW

**Annual Fossil Fuel Savings Algorithm**

If fossil fuel domestic water heater, MMBtu savings provided below:

ΔMMBtu = ((((GPMbase - GPMlow) / GPMbase) \* # people \* gals/day \* days/year \* DR) / (F/home) \* 8.3 \* (TEMPft - TEMPin) / 1,000,000) / Gas DHW Recovery Efficiency

*Where:*

*Gas DHW Recovery Efficiency = Recovery efficiency of electric water heater*

*= 0.75 [[445]](#footnote-445)*

*All other variables As above*

Illustrative example – do not use as default assumption

For a 1.5 GPM rated aerator:

ΔMMBtu = ((((2.2 – 1.5) / 2.2) \* 2.56 \* 10.9 \* 365 \* 0.5) / 3.5 \* 8.3 \* (80-60.9) / 1,000,000) / 0.75

= 0.098 MMBtu

**Annual Water Savings Algorithm**

Water Savings = (((GPMbase - GPMlow) / GPMbase) \* # people \* gals/day \* days/year \* DR) / (F/home) /748

*Where:*

*748 = Constant to convert from gallons to CCF*

*All other variables As above*

Illustrative example – do not use as default assumption

For a 1.5 GPM rated aerator:

Water Savings = (((2.2 – 1.5) / 2.2) \* 2.56 \* 10.9 \* 365 \* 0.5) / 3.5 / 748

= 0.619 CCF

kWh Savings from Water Reduction

The kWh savings from the waste reduction characterized above is now estimated. Please note that utilities’ must be careful not to double count the monetary benefit of these savings within cost effectiveness testing if the avoided costs of water already include the associated electric benefit.

ΔkWhwater[[446]](#footnote-446)  = 2.07 kWh/CCF \* ∆Water (CCF)

Illustrative example – do not use as default assumption

For a 1.5 GPM rated aerator:

ΔkWhwater = 2.07 kWh/CCF \* 0.619 CCF

= 1.3 kWh

**Incremental Cost**

As a retrofit measure, the incremental cost will be the actual cost of installing the new aerator. As a time of sale measure, the incremental cost is assumed to be $2.[[447]](#footnote-447)

**Measure Life**

The measure life is assumed to be 5 years.[[448]](#footnote-448)

**Operation and Maintenance Impacts**

When a retrofit measure, there would be a very small O&M benefit associated with the deferral of the next replacement, but this has conservatively not been characterized.

**Domestic Hot Water Tank Wrap**

**Unique Measure Code(s): RS\_WT\_INS\_HWWRAP\_0113**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to a Tank Wrap or insulation “blanket” that is wrapped around the outside of a hot water tank to reduce stand-by losses. This measure applies only for homes that have an electric water heater that is not already well insulated.

**Definition of Baseline Condition**

The baseline condition is a standard electric domestic hot water tank without an additional tank wrap.

**Definition of Efficient Condition**

The efficient condition is the same standard electric domestic hot water tank with an additional tank wrap.

**Annual Energy Savings Algorithm**

ΔkWh = ((UbaseAbase – UinsulAinsul) \* ΔT \* Hours) / (3412 \* ηDHW)

*Where:*

*ΔkWh= gross customer annual kWh savings for the measure*

*Ubase = Overall heat transfer coefficient prior to adding tank wrap (Btu/Hr-F-ft­­2)*

*= See table below. If unknown assume 1/8 [[449]](#footnote-449)*

*Uinsul = Overall heat transfer coefficient after addition of tank wrap (Btu/Hr-F-ft­­2)*

*= See table below. If unknown assume 1/18 [[450]](#footnote-450)*

*Abase  = Surface area of storage tank prior to adding tank wrap (square feet)*

*= See table below. If unknown assume 23.18 [[451]](#footnote-451)*

*Ainsul = Surface area of storage tank after addition of tank wrap (square feet)*

*= See table below. If unknown assume 25.31 [[452]](#footnote-452)*

*ΔT = Average temperature difference between tank water and outside air temperature (°F)*

*= 60°F [[453]](#footnote-453)*

*Hours = Number of hours in a year (since savings are assumed to be constant over year).*

*= 8760*

*3412 = Conversion from BTU to kWh*

*ηDHW = Recovery efficiency of electric hot water heater*

*= 0.98 [[454]](#footnote-454)*

The following table has default savings for various tank capacity and pre and post R-values.

|  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- |
| **Capacity (gal)** | **Rbase** | **Rinsul** | **Abase (ft2)** | **Ainsul (ft2)** | **ΔkWh** | **ΔkW** |
| 30 | 8 | 16 | 19.16 | 20.94 | 171 | 0.019 |
| 30 | 10 | 18 | 19.16 | 20.94 | 118 | 0.014 |
| 30 | 12 | 20 | 19.16 | 20.94 | 86 | 0.010 |
| 30 | 8 | 18 | 19.16 | 20.94 | 194 | 0.022 |
| 30 | 10 | 20 | 19.16 | 20.94 | 137 | 0.016 |
| 30 | 12 | 22 | 19.16 | 20.94 | 101 | 0.012 |
| 40 | 8 | 16 | 23.18 | 25.31 | 207 | 0.024 |
| 40 | 10 | 18 | 23.18 | 25.31 | 143 | 0.016 |
| 40 | 12 | 20 | 23.18 | 25.31 | 105 | 0.012 |
| 40 | 8 | 18 | 23.18 | 25.31 | 234 | 0.027 |
| 40 | 10 | 20 | 23.18 | 25.31 | 165 | 0.019 |
| 40 | 12 | 22 | 23.18 | 25.31 | 123 | 0.014 |
| 50 | 8 | 16 | 24.99 | 27.06 | 225 | 0.026 |
| 50 | 10 | 18 | 24.99 | 27.06 | 157 | 0.018 |
| 50 | 12 | 20 | 24.99 | 27.06 | 115 | 0.013 |
| 50 | 8 | 18 | 24.99 | 27.06 | 255 | 0.029 |
| 50 | 10 | 20 | 24.99 | 27.06 | 180 | 0.021 |
| 50 | 12 | 22 | 24.99 | 27.06 | 134 | 0.015 |
| 80 | 8 | 16 | 31.84 | 34.14 | 290 | 0.033 |
| 80 | 10 | 18 | 31.84 | 34.14 | 202 | 0.023 |
| 80 | 12 | 20 | 31.84 | 34.14 | 149 | 0.017 |
| 80 | 8 | 18 | 31.84 | 34.14 | 327 | 0.037 |
| 80 | 10 | 20 | 31.84 | 34.14 | 232 | 0.027 |
| 80 | 12 | 22 | 31.84 | 34.14 | 173 | 0.020 |

If tank specifics are unknown assume 40 gallons as an average tank size[[455]](#footnote-455), and savings from adding R-10 to a poorly insulated R-8 tank:

ΔkWh = ((23.18/8 – 25.31/18) \* 60 \* 8760) / (3412 \* 0.98)

= 234 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/8760

*Where:*

*ΔkWh = kWh savings from tank wrap installation*

*8760 = Number of hours in a year (since savings are assumed to be constant over year).*

The table above has default savings for various tank capacity and pre and post R-values.

If tank specifics are unknown assume 40 gallons as an average tank size[[456]](#footnote-456), and savings are from adding R-10 to a poorly insulated R-8 tank:

ΔkW = 234 / 8760

= 0.027 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure will be the actual cost of installing the tank wrap. If unknown assume $35 average cost[[457]](#footnote-457).

**Measure Life**

The measure life is assumed to be 5 years.[[458]](#footnote-458)

**Operation and Maintenance Impacts**

n/a

**DHW pipe insulation**

**Unique Measure Code: RS\_WT\_RTR\_PIPEIN\_0711**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes adding insulation to un-insulated domestic hot water pipes. The measure assumes the pipe wrap is installed to the first elbow of the hot water carrying pipe.

Note, the algorithm provided to calculate savings may be used to determine an appropriate deemed savings value if the programs can provide appropriate average values for each of the variables.

This is a retrofit measure.

**Definition of Baseline Condition**

The baseline condition is un-insulated hot water carrying copper pipes.

**Definition of Efficient Condition**

To efficiency case is installing pipe wrap insulation to the first elbow of the hot water carrying copper pipe.

**Annual Energy Savings Algorithm**

If electric domestic hot water tank:

ΔkWh = ((1/Rexist – 1/Rnew) \* (L \* C) \* ΔT \* 8,760)/ ηDHW / 3413

*Where:*

*Rexist = Assumed R-value of existing uninsulated piping*

*= 1.0 [[459]](#footnote-459)*

*Rnew = R-value of existing pipe plus installed insulation*

*= Actual*

*Length = Length of piping insulated*

*= Actual*

*Circumference = Circumference of piping*

*= Actual (0.5” pipe = 0.13ft, 0.75” pipe = 0.196ft)*

*ΔT = Temperature difference between water in pipe and ambient air*

*= 65°F [[460]](#footnote-460)*

*8,760 = Hours per year*

*ηDHW = DHW Recovery efficiency (ηDHW)*

*= 0.98 [[461]](#footnote-461)*

*3413 = Conversion from Btu to kWh*

Illustrative example – do not use as default assumption

Insulating 4 feet of 0.75” pipe with R-3.5 wrap:

ΔkWh = ((1/1.0 – 1/4.5) \* (4 \* 0.196) \* 65 \* 8,760)/ 0.98 / 3,413

= 104 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/8,760

Illustrative example – do not use as default assumption

Insulating 4 feet of 0.75” pipe with R-3.5 wrap:

ΔkW = 104 /8,760

= 0.012 kW

**Annual Fossil Fuel Savings Algorithm**

If fossil fuel DHW unit:

ΔMMBtu = ((1/Rexist – 1/Rnew) \* (L \* C) \* ΔT \* 8,760) / ηDHW /1,000,000

*Where:*

*ηDHW = Recovery efficiency of gas hot water heater*

*= 0.75 [[462]](#footnote-462)*

Illustrative example – do not use as default assumption

Insulating 4 feet of 0.75” pipe with R-3.5 wrap:

ΔMMBtu = ((1/1.0 – 1/4.5) \* (4 \* 0.196) \* 65 \* 8,760)/ 0.75 / 1,000,000

= 0.46 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure should be the actual cost of material and labor. If this is not available, assume $3 per foot of insulation[[463]](#footnote-463).

**Measure Life**

The measure life is assumed to be 15 years[[464]](#footnote-464).

**Operation and Maintenance Impacts**

n/a

**High Efficiency Gas Water Heater**

**Unique Measure Code: RS\_WT\_TOS\_GASDHW\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure describes the purchase of a high efficiency gas water heater meeting or exceeding ENERGY STAR criteria for the water heater category provided below, in place of a new unit rated at the minimum Federal Standard. The measure could be installed in either an existing or new home. The installation is assumed to occur during a natural time of sale.

**Definition of Baseline Condition**

The baseline condition is a new conventional gas storage water heater rated at the federal minimum[[465]](#footnote-465).

For 20 - 55 gallons: EF = 0.675 – (0.0015 \* rated volume in gallons)

For 55 - 100 gallons: EF = 0.8012 – (0.00078 \* rated volume in gallons)

If size is unknown, assume 40 gallon; 0.615 EF.

**Definition of Efficient Condition**

The efficient condition is a new high efficiency gas water heater meeting or exceeding the minimum efficiency Energy Star qualification criteria provided below[[466]](#footnote-466):

| **Water Heater Type** | **Energy Factor** |
| --- | --- |
| High Efficiency Gas Storage | 0.67 |
| Gas Condensing | 0.80 |
| Whole Home Gas Tankless | 0.82 |

**Annual Energy Savings Algorithm**

n/a

**Summer Coincident Peak kW Savings Algorithm**

n/a

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = (1/ EFbase - 1/EFefficient) \* (GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0 )/1,000,000

Where:

EF\_Baseline = Energy Factor rating for baseline equipment

For <=55 gallons: 0.675 – (0.0015 \* tank\_size)

For > 55 gallons: 0.8012 – (0.00078 \* tank size)

= If tank size unknown assume 40 gallons and EF\_Baseline of 0.615

EF\_Efficient = Energy Factor Rating for efficient equipment

= Actual. If Tankless whole-house multiply rated efficiency by 0.91[[467]](#footnote-467). If unknown assume values in look up in table below

|  |  |
| --- | --- |
| **Water Heater Type** | **EF\_Efficient** |
| Condensing Gas Storage | 0.80 |
| Gas Storage | 0.67 |
| Tankless whole-house | 0.82 \* 0.91 = 0.75 |

GPD = Gallons Per Day of hot water use per person

= 45.5 gallons hot water per day per household/2.59 people per household[[468]](#footnote-468)

= 17.6

Household = Average number of people per household

*= 2.53 [[469]](#footnote-469)*

365.25 = Days per year, on average

γWater  = Specific Weight of water

= 8.33 pounds per gallon

Tout = Tank temperature

= 125°F

Tin = Incoming water temperature from well or municipal system

= *60.9 [[470]](#footnote-470)*

1. = Heat Capacity of water (1 Btu/lb\*°F)

Illustrative example – do not use as default assumption

For example, installing a 40 gallon condensing gas storage water heater, with an energy factor of 0.82 in a single family house:

ΔMMBtu = (1/0.615 - 1/0.82) \* (17.6 \* 2.53 \* 365.25\* 8.33 \* (125 – 60.9) \* 1) / 1,000,000

= 3.53 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental capital cost for this measure is dependent on the type of water heater as listed below[[471]](#footnote-471).

|  |  |
| --- | --- |
| **Water heater Type** | **Incremental Cost** |
| Gas Storage | $400 |
| Condensing gas storage | $685 |
| Tankless whole-house unit | $605 |

**Measure Life**

The measure life is assumed to be 13 years[[472]](#footnote-472).

**Operation and Maintenance Impacts**

n/a

**Heat Pump Domestic Water Heater**

**Unique Measure Code(s): RS\_WT\_TOS\_HPRSHW\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a Heat Pump domestic water heater in place of a standard electric water heater in conditioned space. This is a time of sale measure.

**Definition of Baseline Condition**

The baseline condition is assumed to be a new electric water heater meeting federal minimum efficiency standards[[473]](#footnote-473):

For <=55 gallons: 0.96 – (0.0003 \* rated volume in gallons)

For >55 gallons: 2.057 – (0.00113 \* rated volume in gallons)

**Definition of Efficient Condition**

The efficient condition is a heat pump water heater.

**Annual Energy Savings Algorithm**

ΔkWh = (((1/EFBASE – 1/EFefficient) \* GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 3412) + kWh\_cooling - kWh\_heating

*Where:*

*EFbase = Energy Factor (efficiency) of standard electric water heater according to federal standards[[474]](#footnote-474):*

*For <=55 gallons: 0.96 – (0.0003 \* rated volume in gallons)*

*For >55 gallons: 2.057 – (0.00113 \* rated volume in gallons)*

*= 0.945 for a 50 gallon tank, the most common size for HPWH*

*EFefficient = Energy Factor (efficiency) of Heat Pump water heater*

*= Actual. If unknown assume 2.0 [[475]](#footnote-475)*

*GPD = Gallons Per Day of hot water use per person*

*= 45.5 gallons hot water per day per household/2.59 people per household[[476]](#footnote-476)*

*= 17.6*

*Household = Average number of people per household*

*= 2.53 [[477]](#footnote-477)*

*365.25 = Days per year*

*γWater = Specific weight of water*

*= 8.33 pounds per gallon*

*Tout = Tank temperature*

*= 125°F*

*Tin = Incoming water temperature from well or municiple system*

*= 60.9 [[478]](#footnote-478)*

*1.0 = Heat Capacity of water (1 Btu/lb\*°F)*

*3412 = Conversion from Btu to kWh*

*kWh\_cooling[[479]](#footnote-479) = Cooling savings from conversion of heat in home to water heat*

*=(((((GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 3412) – ((1/ EFNEW \* GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 3412)) \* LF \* 33%) / COPCOOL) \* LM*

Where:

LF = Location Factor

= 1.0 for HPWH installation in a conditioned space

= 0.5 for HPWH installation in an unknown location

= 0.0 for installation in an unconditioned space

33% = Portion of removed heat that results in cooling savings[[480]](#footnote-480)

COPCOOL = COP of central air conditioning

= Actual, if unknown, assume 3.08 (10.5 SEER / 3.412)

LM = Latent multiplier to account for latent cooling demand

= 1.33 [[481]](#footnote-481)

*kWh\_heating = Heating cost from conversion of heat in home to water heat (dependent on heating fuel)*

*For Natural Gas heating, kWh\_heating = 0*

*For electric heating:*

*= ((((GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 3412) – ((1/ EFNEW \* GPD \* Household \* 365.25 \* γWater \* (TOUT – Tin) \* 1.0) / 3412)) \* LF \* 49%) / COPHEAT*

Where:

47% = Portion of removed heat that results in increased heating load[[482]](#footnote-482)

COPHEAT = COP of electric heating system

= actual. If not available use[[483]](#footnote-483):

| **System Type** | **Age of Equipment** | **HSPF Estimate** | **COPHEAT**  **(COP Estimate)** |
| --- | --- | --- | --- |
| Heat Pump | Before 2006 | 6.8 | 2.00 |
| After 2006 – 2014 (default) | 7.7 | 2.26 |
| 2015 on | 8.2 | 2.40 |
| Resistance | N/A | N/A | 1.00 |

Prescriptive savings based on defaults provided above:

ΔkWH electric resistance heat = (((1/0.945– 1/2.0) \* 17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 – 60.9) \* 1.0) / 3412) + kWh\_cooling - kWh\_heating

kWh\_cooling = (((((17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 – 60.9) \* 1.0) / 3412) – ((1/ 2.0 \* 17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 – 60.9) \* 1.0) / 3412)) \* 0.5 \* 0.33) / 3.08) \* 1.33

= 90.7 kWh

kWh\_heating= ((((17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 – 60.9) \* 1.0) / 3412) – ((1/ 2.0 \* 17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 – 60.9) \* 1.0) / 3412)) \* 0.5 \* 0.47) / 1.0

= 299.1 kWh

ΔkWH electric resistance heat = 1420.7 + 90.7 – 299.1

= 1212.3 kWh

ΔkWH heat pump heat = (((1/0.945– 1/2.0) \* 17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 – 60.9) \* 1.0) / 3412) + kWh\_cooling - kWh\_heating

kWh\_cooling = 90.7 kWh

kWh\_heating= (((17.6 \* 2.53 \* 365.25 \* 8.33 \* (125-60.9) \* 1.0) / 3412) - ((1/2.0 \* 17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 - 60.9) \* 1.0) / 3412)) \* 0.5 \* 0.47) / 2.0

= 149.5 kWh

ΔkWH heat pump heat = 1420.7 + 90.7 – 149.5

= 1361.9 kWh

ΔkWH fossil fuel heat = (((1/0.945– 1/2.0) \* 17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 – 60.9) \* 1.0) / 3412) + kWh\_cooling - kWh\_heating

kWh\_cooling = 90.7

kWh\_heating= 0

ΔkWH fossil fuel heat = 1420.7 + 90.7 - 0

= 1511.4 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = 0.17 kW *[[484]](#footnote-484)*

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = - ((((GPD \* Household \* 365.25 \* γWater \* (TOUT – TIN) \* 1.0) / 3412) – (((1/ EFNEW *\** GPD \* Household \* 365.25 \* γWater \* (TOUT – TIN) \* 1.0) / 3412))) \* LF \* 47% \* 0.003412) / (ηHeat \* % Natural Gas)

*Where:*

*ΔMMBtu = Heating cost from conversion of heat in home to water heat for homes with Natural Gas heat.[[485]](#footnote-485)*

*0.003412 = conversion factor (MMBtu per kWh)*

*ηHeat = Efficiency of heating system*

*= Actual.[[486]](#footnote-486) If not available use 72%.[[487]](#footnote-487)*

*% Natural Gas = Factor dependent on heating fuel:*

|  |  |
| --- | --- |
| ***Heating System*** | ***%Natural Gas*** |
| *Electric resistance or heat pump* | *0%* |
| *Natural Gas* | *100%* |
| *Unknown heating fuel[[488]](#footnote-488)* | *62.5%* |

*Other factors as defined above*

Prescriptive savings based on defaults provided above:

ΔMMBtu for fossil fuel heated homes:

ΔMMBtu = - (((17.6 \* 2.53 \* 365.25 \* 8.33 \* (125-60.9) \* 1.0) / 3412) - ((1/2.0 \* 17.6 \* 2.53 \* 365.25 \* 8.33 \* (125 - 60.9) \* 1.0) / 3412)) \* 0.5 \* 0.47 \* 0.003412) / (0.72 \* 1.0)

= - 1.41MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental capital cost for this measure is $1,000, for a HPWH with an energy factor of 2.0.[[489]](#footnote-489)

**Measure Life**

The expected measure life is assumed to be 13 years.[[490]](#footnote-490)

**Operation and Maintenance Impacts**

n/a

**Thermostatic Restrictor Shower Valve**

**Unique Measure Code: RS\_HV\_TOS\_GSHPS\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

The measure is the installation of a thermostatic restrictor shower valve in a single or multi-family household. This is a valve attached to a residential showerhead which restricts hot water flow through the showerhead once the water reaches a set point (generally 95F or lower).

This measure was developed to be applicable to the following program types: RF, NC, DI. If applied to other program types, the measure savings should be verified.

**Definition of Baseline Condition**

The baseline equipment is the residential showerhead without the restrictor valve installed.

**Definition of Efficient Condition**

To qualify for this measure the installed equipment must be a thermostatic restrictor shower valve installed on a residential showerhead.

**Annual Energy Savings Algorithm**

ΔkWh = %ElectricDHW \* ((GPM\_base\_S \* L\_showerdevice) \* Household \* SPCD \* 365.25 / SPH) \* EPG\_electric

*Where:*

*%ElectricDHW = proportion of water heating supplied by electric resistance heating*

|  |  |
| --- | --- |
| ***DHW fuel*** | ***%ElectricDHW*** |
| *Electric* | *100%* |
| *Natural Gas* | *0%* |
| *Unknown* | *24%[[491]](#footnote-491)* |

*GPM\_base\_S = Flow rate of the basecase showerhead, or actual if available*

|  |  |
| --- | --- |
| ***Program*** | ***GPM*** |
| *Direct-install, device only* | *2.5 [[492]](#footnote-492)* |
| *New Construction or direct install of device and low flow showerhead* | *Rated or actual flow of program-installed showerhead* |

*L\_showerdevice = Hot water waste time avoided due to thermostatic restrictor valve*

*= 0.89 minutes[[493]](#footnote-493)*

*Household = Average number of people per household*

*= 2.56 [[494]](#footnote-494)*

*SPCD = Showers Per Capita Per Day*

*= 0.6[[495]](#footnote-495)*

*365.25 = Days per year, on average.*

*SPH = Showerheads Per Household so that per-showerhead savings fractions can be determined*

*= 1.6 [[496]](#footnote-496)*

*EPG\_electric = Energy per gallon of hot water supplied by electric*

*= (8.33 \* 1.0 \* (ShowerTemp - SupplyTemp)) / (RE\_electric \* 3412)*

*= (8.33 \* 1.0 \* (105 – 60.9)) / (0.98 \* 3412)*

*= 0.11kWh/gal*

*8.33 = Specific weight of water (lbs/gallon)*

*1.0 = Heat Capacity of water (btu/lb-°)*

*ShowerTemp = Assumed temperature of water*

*= 105F [[497]](#footnote-497)*

*SupplyTemp = Assumed temperature of water entering house*

*= 60.9 [[498]](#footnote-498)*

*RE\_electric = Recovery efficiency of electric water heater*

*= 98% [[499]](#footnote-499)*

*3412 = Constant to convert Btu to kWh*

Illustrative Example - do not use as default assumption

For example, a direct installed valve in a home with electric DHW:

ΔkWh = 1.0 \* (2.5 \* 0.89 \* 2.56 \* 0.6 \* 365.25 / 1.6) \* 0.11

= 86 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/Hours \* CF

*Where:*

*Hours = Annual electric DHW recovery hours for wasted showerhead use prevented by device*

*= ((GPM\_base\_S \* L\_showerdevice) \* Household \* SPCD \* 365.25 / SPH ) \* 0.746[[500]](#footnote-500) / GPH*

*GPH = Gallons per hour recovery of electric water heater calculated for 59.1 temp rise (120-60.9), 98% recovery efficiency, and typical 4.5kW electric resistance storage tank.*

*= 30.0*

*Hours = ((2.5 \* 0.89) \* 2.56 \* 0.6 \* 365.25 / 1.6) \* 0.746 / 30*

*= 19.4 hours*

*CF = Coincidence Factor for electric load reduction*

*= 0.0015[[501]](#footnote-501)*

Illustrative example – do not use as default assumption

For example, a direct installed valve in a home with electric DHW:

ΔkW = 86 / 19.4 \* 0.0015

= 0.007 kW

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = %FossilDHW \* ((GPM\_base\_S \* L\_showerdevice)\* Household \* SPCD \* 365.25 / SPH) \* EPG\_gas

*Where:*

*%FossilDHW = proportion of water heating supplied by Natural Gas heating*

| ***DHW fuel*** | ***%Fossil\_DHW*** |
| --- | --- |
| *Electric* | *0%* |
| *Natural Gas* | *100%* |
| *Unknown* | *76%[[502]](#footnote-502)* |

*EPG\_gas = Energy per gallon of Hot water supplied by gas*

*= (8.33 \* 1.0 \* (ShowerTemp - SupplyTemp)) / (RE\_gas \* 1,000,000)*

*= 0.00065 MMBTu/gal*

*RE\_gas = Recovery efficiency of gas water heater*

*= 75% For SF homes[[503]](#footnote-503)*

*1,000,000 = Converts Btus to MMBtu*

*Other variables as defined above.*

Illustrative example – do not use as default assumption

For example, a direct installed valve in a home with gas DHW:

ΔMMBTu = 1.0 \* ((2.5 \* 0.89) \* 2.56 \* 0.6 \* 365.25 / 1.6) \* 0.00065

= 0.51 MMBtu

**Water impact Descriptions and calculations**

ΔCCF = ((GPM\_base\_S \* L\_showerdevice) \* Household \* SPCD \* 365.25 / SPH) / 748

*Where:*

*748 = Constant to convert from gallons to CCF*

*Other variables as defined above*

Illustrative example – do not use as default assumption

For example, a direct installed valve:

ΔCCF = ((2.5 \* 0.89) \* 2.56 \* 0.6 \* 365.25 / 1.6) / 748

= 1.0 CCF

**Measure Life**

The expected measure life is assumed to be 10 years. [[504]](#footnote-504)

**Deemed Measure Cost**

The incremental cost of the measure should be the actual program cost or $30[[505]](#footnote-505) if not available.

**Operation and Maintenance Impacts**

N/A

**Water Heater Temperature Setback\***

**Unique Measure Code: RS\_WT\_RTR\_WHTSB\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to turning down an existing hot water tank thermostat setting that is at 130 degrees or higher. Savings are provided to account for the resulting reduction in standby losses. This is a retrofit measure.

**Definition of Baseline Equipment**

The baseline condition is a hot water tank with a thermostat setting that is 130 degrees or higher. Note if there are more than one DHW tanks in the home at or higher than 130 degrees and they are all turned down, then the savings per tank can be multiplied by the number of tanks.

**Definition of Efficient Equipment**

The efficient condition is a hot water tank with the thermostat reduced to no lower than 120 degrees.

**Annual Energy Savings Algorithm**

For homes with electric DHW tanks:

ΔkWh[[506]](#footnote-506)= (UA \* (Tpre – Tpost) \* Hours) / (3412 \* RE\_electric)

*Where:*

*U = Overall heat transfer coefficient of tank (Btu/Hr-°F-ft­­2).*

*= Actual if known. If unknown assume R-12, U = 0.083*

*A = Surface area of storage tank (square feet)*

*= Actual if known. If unknown use table below based on capacity of tank. If capacity unknown assume 50 gal tank; A = 24.99ft2*

|  |  |
| --- | --- |
| ***Capacity (gal)*** | ***A (ft2)[[507]](#footnote-507)*** |
| *30* | *19.16* |
| *40* | *23.18* |
| *50* | *24.99* |
| *80* | *31.84* |

*Tpre = Actual hot water setpoint prior to adjustment.*

*= 135 degrees default*

*Tpost = Actual new hot water setpoint, which may not be lower than 120 degrees*

*= 120 degrees default*

*Hours = Number of hours in a year (since savings are assumed to be constant over year).*

*= 8760*

*3412 = Conversion from Btu to kWh*

*RE\_electric = Recovery efficiency of electric hot water heater*

*= 0.98 [[508]](#footnote-508)*

The deemed savings assumption, where site specific assumptions are not available would be as follows:

ΔkWh= (UA \* (Tpre – Tpost) \* Hours) / (3412 \* RE\_electric)

= (((0.083 \* 24.99) \* (135 – 120) \* 8760) / (3412 \* 0.98)

= 81.5 kWh

**Summer Coincident Peak kW Savings Algorithm**

∆kW**=** ∆kWh/ Hours

*Where:*

*Hours = 8760*

The deemed savings assumption, where site specific assumptions are not available would be as follows:

ΔkW = (81.5/ 8760)

= 0.0093 kW

**Annual Fossil Fuel Savings Algorithm**

For homes with gas water heaters:

ΔMMBtu= (U \* A \* (Tpre – Tpost) \* Hours) / (1,000,000 \* RE\_gas)

*Where*

*1,000,000 = Converts Btus to MMbtu (btu/MMBtu)*

*RE\_gas = Recovery efficiency of gas water heater*

*= 0.75 [[509]](#footnote-509)*

The deemed savings assumption, where site specific assumptions are not available would be as follows:

ΔMMBtu= (U \* A \* (Tpre – Tpost) \* Hours) / (1,000,000 \* RE\_gas)

= (0.083 \* 24.99 \* (135 – 120) \* 8760) / (1,000,000 \* 0.75)

= 0.36 MMBtu

**Annual Water Savings Algorithm**

N/A

**Incremental Cost**

The incremental cost of the setback is assumed to be $5 for contractor time.

**Deemed Lifetime of Efficient Equipment**

The assumed lifetime of the measure is 2 years.

**Operation and Maintenance Impacts**

N/A

*Appliance End Use*

**Clothes Washer**

**Unique Measure Code(s): RS\_LA\_TOS\_CWASHES\_0415, RS\_LA\_TOS\_CWASHT2\_0415, RS\_LA\_TOS\_CWASHT3\_0415, RS\_LA\_TOS\_CWASHME\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the purchase (time of sale) and installation of a clothes washer exceeding either the ENERGY STAR/CEE Tier 1, ENERGY STAR Most Efficient/CEE Tier 2 or CEE Tier 3 minimum qualifying efficiency standards presented below:

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Efficiency Level** | **Integrated Modified Energy Factor (IMEF)** | | **Integrated Water Factor**  **(IWF)** | |
| **Front Loading** | **Top Loading** | **Front Loading** | **Top Loading** |
| ENERGY STAR, CEE Tier 1 | >= 2.38 | >= 2.06 | <= 3.7 | <= 4.3 |
| ENERGY STAR Most Efficient, CEE TIER 2 | >= 2.74 | >= 2.76 | <= 3.2 | <= 3.5 |
| CEE TIER 3 | >= 2.92 | n/a | <= 3.2 | n/a |

The Integrated Modified Energy Factor (IMEF) measures energy consumption of the total laundry cycle (washing and drying). It indicates how many cubic feet of laundry can be washed and dried with one kWh of electricity and the per-cycle standby and off mode energy consumption; the higher the number, the greater the efficiency.

The Integrated Water Factor (IWF) is the number of gallons needed for each cubic foot of laundry. A lower number indicates lower consumption and more efficient use of water.

**Definition of Baseline Condition**

The baseline efficiency is determined according to the Integrated Modified Energy Factor (IMEF) that takes into account the energy and water required per clothes washer cycle, including energy required by the clothes dryer per clothes washer cycle and standby/off mode consumption. The Federal baseline IMEF as of March 2015 is 1.84 for front loading units and 1.29 for top loading units.

**Definition of Efficient Condition**

The efficient condition is a clothes washer meeting either the ENERGY STAR/CEE Tier 1, ENERGY STAR Most Efficient/CEE Tier 2 or CEE TIER 3 efficiency criteria presented above.

**Annual Energy Savings Algorithm**

(see ‘2015 Mid Atlantic CW Analysis.xls’ for detailed calculation)

∆kWh = [(Capacity \* 1/IMEFbase \* Ncycles) \* (%CWbase + (%DHWbase \* %Electric\_DHW) + (%Dryerbase \* %Electric\_Dryer)] - [(Capacity \* 1/IMEFeff \* Ncycles) \* (%CWeff + (%DHWeff \* %Electric\_DHW) + (%Dryereff \* %Electric\_Dryer)]

*Where*

*Capacity = Clothes Washer capacity (cubic feet)*

*= Actual. If capacity is unknown assume average 3.45 cubic feet[[510]](#footnote-510)*

*IMEFbase = Integrated Modified Energy Factor of baseline unit*

*= Values provided in table below*

*IMEFeff = Integrated Modified Energy Factor of efficient unit*

*= Actual. If unknown assume average values provided below.*

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
| **Efficiency Level** | **Integrated Modified Energy Factor (IMEF)** | | | **Weighting Percentages[[511]](#footnote-511)** | |
| **Front Loading** | **Top Loading** | **Weighted Average** | **Front Loading** | **Top Loading** |
| Federal Standard | >= 1.84 | >= 1.29 | >= 1.66 | 67% | 33% |
| ENERGY STAR, CEE Tier 1 | >= 2.38 | >= 2.06 | >= 2.26 | 62% | 38% |
| ENERGY STAR Most Efficient, CEE TIER 2 | >= 2.74 | >= 2.76 | >= 2.74 | 98% | 2% |
| CEE TIER 3 | >= 2.92 | n/a | >= 2.92 | 100% | 0% |

*Ncycles = Number of Cycles per year*

*= 254[[512]](#footnote-512)*

*%CW = Percentage of total energy consumption for Clothes Washer operation*

*%DHW = Percentage of total energy consumption used for water heating*

*%Dryer = Percentage of total energy consumption for dryer operation*

*(dependent on efficiency level – see table below)*

|  |  |  |  |
| --- | --- | --- | --- |
|  | **Percentage of Total Energy Consumption**[[513]](#footnote-513) | | |
|  | **%CW** | **%DHW** | **%Dryer** |
| Federal Standard | 8% | 31% | 61% |
| ENERGY STAR, CEE Tier 1 | 8% | 23% | 69% |
| ENERGY STAR Most Efficient, CEE TIER 2 | 14% | 10% | 76% |
| CEE TIER 3 | 14% | 10% | 77% |

*%Electric\_DHW = Percentage of DHW savings assumed to be electric*

|  |  |
| --- | --- |
| **DHW fuel** | **%Electric\_DHW** |
| Electric | 100% |
| Fossil Fuel | 0% |
| Unknown | 65%[[514]](#footnote-514) |

*%Electric\_Dryer = Percentage of dryer savings assumed to be electric*

|  |  |
| --- | --- |
| **Dryer fuel** | **%Electric\_Dryer** |
| Electric | 100% |
| Fossil Fuel | 0% |
| Unknown | 79%[[515]](#footnote-515) |

The prescriptive kWH savings based on values provided above where DHW and Dryer fuels are unknown is provided below[[516]](#footnote-516):

| **Efficiency Level** | **ΔkWH** | | |
| --- | --- | --- | --- |
| **Front** | **Top** | **Weighted Average** |
| ENERGY STAR, CEE Tier 1 | 112.7 | 84.2 | 102.2 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 145.0 | 162.2 | 145.4 |
| CEE TIER 3 | 160.9 | n/a | 160.9 |

The unit specific kWh savings when DHW and Dryer fuels are known is provided below:

| **Efficiency Level** | **Dryer/DHW Gas Combo** | **ΔkWH** | | |
| --- | --- | --- | --- | --- |
| **Front** | **Top** | **Weighted Average** |
| ENERGY STAR, CEE Tier 1 | Electric Dryer/Electric DHW | 160.0 | 104.9 | 140.1 |
| Electric Dryer/Gas DHW | 59.8 | 79.7 | 66.3 |
| Gas Dryer/Electric DHW | 101.7 | 47.8 | 82.6 |
| Gas Dryer/Gas DHW | 1.5 | 22.5 | 8.8 |
| ENERGY STAR Most Efficient, CEE TIER 2 | Electric Dryer/Electric DHW | 208.4 | 210.7 | 208.5 |
| Electric Dryer/Gas DHW | 74.5 | 138.3 | 76.0 |
| Gas Dryer/Electric DHW | 129.7 | 99.1 | 129.1 |
| Gas Dryer/Gas DHW | -4.1 | 26.7 | -3.5 |
| CEE TIER 3 | Electric Dryer/Electric DHW | 228.1 | n/a | 228.1 |
| Electric Dryer/Gas DHW | 92.4 | n/a | 92.4 |
| Gas Dryer/Electric DHW | 134.4 | n/a | 134.4 |
| Gas Dryer/Gas DHW | -1.4 | n/a | -1.4 |

Note, utilities may consider whether it is appropriate to claim kWh savings from the reduction in water consumption arising from this measure. The kWh savings would be in relation to the pumping and wastewater treatment. See water savings for characterization.

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/Hours \* CF

*Where:*

*Hours = Assumed Run hours of Clothes Washer*

*= 265 [[517]](#footnote-517)*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.029 [[518]](#footnote-518)*

The prescriptive kW savings based on values provided above where DHW and Dryer fuels are unknown is provided below:

|  |  |  |  |
| --- | --- | --- | --- |
| **Efficiency Level** | **ΔkW** | | |
| **Front** | **Top** | **Weighted Average** |
| ENERGY STAR, CEE Tier 1 | 0.012 | 0.009 | 0.011 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 0.016 | 0.018 | 0.016 |
| CEE TIER 3 | 0.018 | n/a | 0.018 |

The unit specific kW savings when DHW and Dryer fuels are known is provided below:

| **Efficiency Level** | **Dryer/DHW Fuel Combo** | **ΔkW** | | |
| --- | --- | --- | --- | --- |
| **Front** | **Top** | **Weighted Average** |
| ENERGY STAR, CEE Tier 1 | Electric Dryer/Electric DHW | 0.018 | 0.011 | 0.015 |
| Electric Dryer/Fuel DHW | 0.007 | 0.009 | 0.007 |
| Fuel Dryer/Electric DHW | 0.011 | 0.005 | 0.009 |
| Fuel Dryer/Fuel DHW | 0.000 | 0.002 | 0.001 |
| ENERGY STAR Most Efficient, CEE TIER 2 | Electric Dryer/Electric DHW | 0.023 | 0.023 | 0.023 |
| Electric Dryer/Fuel DHW | 0.008 | 0.015 | 0.008 |
| Fuel Dryer/Electric DHW | 0.014 | 0.011 | 0.014 |
| Fuel Dryer/Fuel DHW | 0.000 | 0.003 | 0.000 |
| CEE TIER 3 | Electric Dryer/Electric DHW | 0.025 | n/a | 0.025 |
| Electric Dryer/Fuel DHW | 0.010 | n/a | 0.010 |
| Fuel Dryer/Electric DHW | 0.015 | n/a | 0.015 |
| Fuel Dryer/Fuel DHW | 0.000 | n/a | 0.000 |

**Annual Fossil Fuel Savings Algorithm**

∆MMBtu = [(Capacity \* 1/MEFbase \* Ncycles) \* ((%DHWbase \* %Natural Gas\_DHW \* R\_eff) + (%Dryerbase \* %Gas \_Dryer)] - [(Capacity \* 1/MEFeff \* Ncycles) \* ((%DHWeff \* %Natural Gas\_DHW \* R\_eff) + (%Dryereff \* %Gas\_Dryer)] \* MMBtu\_convert

*Where:*

*R\_eff = Recovery efficiency factor*

*= 1.26[[519]](#footnote-519)*

*MMBtu \_convert = Convertion factor from kWh to MMBtu*

*= 0.003413*

*%Natural Gas\_DHW = Percentage of DHW savings assumed to be Natural Gas*

|  |  |
| --- | --- |
| **DHW fuel** | **%Natural Gas\_DHW** |
| Electric | 0% |
| Natural Gas | 100% |
| Unknown | 35%[[520]](#footnote-520) |

*%Gas\_Dryer = Percentage of dryer savings assumed to be Natural Gas*

|  |  |
| --- | --- |
| **Dryer fuel** | **%Gas\_Dryer** |
| Electric | 0% |
| Natural Gas | 100% |
| Unknown | 6%[[521]](#footnote-521) |

*Other factors as defined above*

The prescriptive MMBtu savings based on values provided above where DHW and Dryer fuels are unknown is provided below:

| **Efficiency Level** | **ΔMMBtu** | | |
| --- | --- | --- | --- |
| **Front** | **Top** | **Weighted Average** |
| ENERGY STAR, CEE Tier 1 | 0.16 | 0.05 | 0.12 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 0.22 | 0.13 | 0.22 |
| CEE TIER 3 | 0.22 | n/a | 0.22 |

The unit specific MMBtu savings when DHW and Dryer fuels are known is provided below:

| **Efficiency Level** | **Configuration** | **ΔMMBtu** | | |
| --- | --- | --- | --- | --- |
| **Front** | **Top** | **Weighted Average** |
| ENERGY STAR, CEE Tier 1 | Electric Dryer/Electric DHW | 0.00 | 0.00 | 0.00 |
| Electric Dryer/Gas DHW | 0.43 | 0.11 | 0.32 |
| Gas Dryer/Electric DHW | 0.20 | 0.19 | 0.20 |
| Gas Dryer/Gas DHW | 0.63 | 0.30 | 0.51 |
| ENERGY STAR Most Efficient, CEE TIER 2 | Electric Dryer/Electric DHW | 0.00 | 0.00 | 0.00 |
| Electric Dryer/Gas DHW | 0.58 | 0.31 | 0.57 |
| Gas Dryer/Electric DHW | 0.27 | 0.38 | 0.27 |
| Gas Dryer/Gas DHW | 0.84 | 0.69 | 0.84 |
| CEE TIER 3 | Electric Dryer/Electric DHW | 0.00 | n/a | 0.00 |
| Electric Dryer/Gas DHW | 0.58 | n/a | 0.58 |
| Gas Dryer/Electric DHW | 0.32 | n/a | 0.32 |
| Gas Dryer/Gas DHW | 0.90 | n/a | 0.90 |

**Annual Water Savings Algorithm**

∆Water (CCF) = (Capacity \* (IWFbase - IWFeff)) \* Ncycles

*Where*

*IWFbase = Integrated Water Factor of baseline clothes washer*

*= Values provided below*

*IWFeff = Integrated Water Factor of efficient clothes washer*

*= Actual. If unknown assume average values provided below.*

|  |  |  |  |
| --- | --- | --- | --- |
| **Efficiency Level** | **IWF**[[522]](#footnote-522) | | |
| **Front Loading** | **Top Loading** | **Weighted Average** |
| Federal Standard | 4.7 | 8.4 | 5.92 |
| ENERGY STAR, CEE Tier 1 | 3.7 | 4.3 | 3.93 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 3.2 | 3.5 | 3.21 |
| CEE TIER 3 | 3.2 | n/a | 3.2 |

The prescriptive water savings for each efficiency level are presented below:

| **Efficiency Level** | **∆Water (ccf per year)** | | |
| --- | --- | --- | --- |
| **Front Loading** | **Top Loading** | **Weighted Average** |
| ENERGY STAR, CEE Tier 1 | 2.6 | 1.9 | 2.3 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 3.2 | 2.8 | 3.2 |
| CEE TIER 3 | 3.2 | 6.9 | 3.2 |

kWh Savings from Water Reduction

The kWh savings from the waste reduction characterized above is now estimated. Please note that utilities’ must be careful not to double count the monetary benefit of these savings within cost effectiveness testing if the avoided costs of water already include the associated electric benefit.

ΔkWhwater[[523]](#footnote-523) = 2.07 kWh \* ∆Water (CCF)

Using the default assumptions provided above, the prescriptive water savings for each efficiency level are presented below:

|  |  |  |  |
| --- | --- | --- | --- |
| **Efficiency Level** | **ΔkWhwater** | | |
| **Front** | **Top** | **Weighted Average** |
| ENERGY STAR, CEE Tier 1 | 5.4 | 3.9 | 4.8 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 6.6 | 5.9 | 6.6 |
| CEE TIER 3 | 6.6 | 14.4 | 6.6 |

**Incremental Cost**

The incremental cost for this measure is provided in the table below[[524]](#footnote-524):

|  |  |
| --- | --- |
| **Efficiency Level** | **Market Opportunity**  **Incremental Cost** |
| ENERGY STAR, CEE Tier 1 | $48 |
| ENERGY STAR Most Efficient, CEE TIER 2 | $269 |
| CEE TIER 3 | $297 |

**Measure Life**

The measure life is assumed to be 14 years [[525]](#footnote-525).

**Operation and Maintenance Impacts**

n/a

**Clothes Washer Early Replacement**

**Unique Measure Code(s): RS\_LA\_RTR\_CWASHES\_0415, RS\_LA\_ RTR\_CWASHT2\_0415, RS\_LA\_ RTR\_CWASHT3\_0415, RS\_LA\_ RTR\_CWASHME\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the early removal of an existing inefficient clothes washer from service, prior to its natural end of life, and replacement with a new unit exceeding either the ENERGY STAR/CEE Tier 1, ENERGY STAR Most Efficient / CEE Tier 2 or CEE Tier 3 minimum qualifying efficiency standards presented below.

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Efficiency Level** | **Integrated Modified Energy Factor (IMEF)** | | **Integrated Water Factor (IWF)** | |
|  | Front Loading | Top Loading | Front Loading | Top Loading |
| ENERGY STAR, CEE Tier 1 | >= 2.38 | >= 2.06 | <= 3.7 | <= 4.3 |
| ENERGY STAR Most Efficient, CEE TIER 2 | >= 2.74 | >= 2.76 | <= 3.2 | <= 3.5 |
| CEE TIER 3 | >= 2.92 | n/a | <= 3.2 | n/a |

The Integrated modified energy factor (MEF) measures energy consumption of the total laundry cycle (washing and drying). It indicates how many cubic feet of laundry can be washed and dried with one kWh of electricity and the per-cycle standby and off mode energy consumption; the higher the number, the greater the efficiency.

The Integrated Water Factor (IWF) is the number of gallons needed for each cubic foot of laundry. A lower number indicates lower consumption and more efficient use of water.

Savings are calculated between the existing unit and the new efficient unit consumption during the assumed remaining life of the existing unit, and between a hypothetical new baseline unit and the efficient unit consumption for the remainder of the measure life.

This is a retrofit measure.

**Definition of Baseline Condition**

The baseline condition is the existing inefficient clothes washer for the remaining assumed useful life of the unit, assumed to be 5 years[[526]](#footnote-526), and then for the remainder of the measure life (next 9 years) the baseline becomes a new replacement unit meeting the minimum federal efficiency standard presented above.

The existing unit efficiency is assumed to be 1.0 IMEF for front loaders and 0.84 IMEF for top loaders. This is based on the Federal Standard for clothes washers from 2004 - 2015; 1.26 MEF converted to IMEF using an ENERGY STAR conversion tool copied in to the reference calculation spreadsheet “2015 Mid Atlantic Early Replacement Clothes Washer Analysis.xls”. The Integrated Water Factor is assumed to be 8.2 IWF for front loaders and 8.4 for top loaders, based on a similar conversion of the 2004 Federal Standard 7.93WF.

The new baseline unit is consistent with the Time of Sale measure.

The baseline assumptions are provided below:

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Efficiency Level** | **Integrated Modified Energy Factor (IMEF)** | | **Integrated Water Factor (IWF)** | |
| **Front Loading** | **Top Loading** | **Front Loading** | **Top Loading** |
| Existing unit | 1.0 | 0.84 | 8.2 | 8.4 |
| Federal Standard | 1.84 | 1.29 | 4.7 | 8.4 |

**Definition of Efficient Condition**

The efficient condition is a clothes washer meeting either the exceeding ENERGY STAR/ CEE Tier 1, ENERGY STAR Most Efficient / CEE Tier 2 or CEE Tier 3 standards as of 1/1/2015 as presented in the measure description.

**Annual Energy Savings Algorithm**

(see ‘2015 Mid Atlantic Early Replacement Clothes Washer Analysis.xls’ for detailed calculation)

∆kWh = [(Capacity \* 1/IMEFbase \* Ncycles) \* (%CWbase + (%DHWbase \* %Electric\_DHW) + (%Dryerbase \* %Electric\_Dryer)] - [(Capacity \* 1/IMEFeff \* Ncycles) \* (%CWeff + (%DHWeff \* %Electric\_DHW) + (%Dryereff \* %Electric\_Dryer)]

*Where*

*Capacity = Clothes Washer capacity (cubic feet)*

*= Actual. If capacity is unknown assume average 3.45 cubic feet[[527]](#footnote-527)*

*IMEFbase = Integrated Modified Energy Factor of baseline unit*

*= Values provided in table below*

*IMEFeff = Integrated Modified Energy Factor of efficient unit*

*= Actual. If unknown assume average values provided below.*

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
| **Efficiency Level** | **Integrated Modified Energy Factor (IMEF)** | | | **Weighting Percentages**[[528]](#footnote-528) | |
| **Front Loading** | **Top Loading** | **Weighted Average** | **Front Loading** | **Top Loading** |
| Existing Unit[[529]](#footnote-529) | 1.0 | 0.84 | n/a[[530]](#footnote-530) | n/a | n/a |
| Federal Standard | >= 1.84 | >= 1.29 | >= 1.66 | 67% | 33% |
| ENERGY STAR, CEE Tier 1 | >= 2.38 | >= 2.06 | >= 2.26 | 62% | 38% |
| ENERGY STAR Most Efficient, CEE TIER 2 | >= 2.74 | >= 2.76 | >= 2.74 | 98% | 2% |
| CEE TIER 3 | >= 2.92 | n/a | >= 2.92 | 100% | 0% |

*Ncycles = Number of Cycles per year*

*= 254[[531]](#footnote-531)*

*%CW = Percentage of total energy consumption for Clothes Washer operation*

*%DHW = Percentage of total energy consumption used for water heating*

*%Dryer = Percentage of total energy consumption for dryer operation*

*(dependent on efficiency level – see table below)*

|  | **Percentage of Total Energy Consumption**[[532]](#footnote-532) | | |
| --- | --- | --- | --- |
|  | **%CW** | **%DHW** | **%Dryer** |
| Federal Standard | 8% | 31% | 61% |
| ENERGY STAR, CEE Tier 1 | 8% | 23% | 69% |
| ENERGY STAR Most Efficient, CEE TIER 2 | 14% | 10% | 76% |
| CEE TIER 3 | 14% | 10% | 77% |

*%Electric\_DHW = Percentage of DHW savings assumed to be electric*

|  |  |
| --- | --- |
| **DHW fuel** | **%Electric\_DHW** |
| Electric | 100% |
| Fossil Fuel | 0% |

*%Electric\_Dryer = Percentage of dryer savings assumed to be electric*

|  |  |
| --- | --- |
| **Dryer fuel** | **%Electric\_Dryer** |
| Electric | 100% |
| Fossil Fuel | 0% |

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below.

|  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- |
| **Efficiency Level** | **Dryer/DHW Fuel Combo** | **Remaining life of existing unit**  **(first 5 years)**  **ΔkWH** | | **Remaining measure life (next 9 years)**  **ΔkWH** | **Mid Life Adjustment** | | **Equivalent Weighted Average Annual Savings** | |
| **Front** | **Top** | **Weighted Average** | **Front** | **Top** | **Front** | **Top** |
| ENERGY STAR, CEE TIER 1 | Electric Dryer/Electric DHW | 488.7 | 655.6 | 140.1 | 29% | 21% | 292.6 | 365.6 |
| Electric Dryer/Gas DHW | 316.3 | 397.0 | 66.3 | 21% | 17% | 175.6 | 210.9 |
| Gas Dryer/Electric DHW | 208.4 | 305.1 | 82.6 | 40% | 27% | 137.6 | 180.0 |
| Gas Dryer/Gas DHW | 36.0 | 46.5 | 8.8 | 25% | 19% | 20.7 | 25.3 |
| ENERGY STAR Most Efficient, CEE TIER 2 | Electric Dryer/Electric DHW | 556.5 | 723.4 | 208.5 | 37% | 29% | 360.7 | 433.7 |
| Electric Dryer/Gas DHW | 325.5 | 406.2 | 76.0 | 23% | 19% | 185.1 | 220.4 |
| Gas Dryer/Electric DHW | 254.6 | 351.4 | 129.1 | 51% | 37% | 184.0 | 226.3 |
| Gas Dryer/Gas DHW | 23.6 | 34.2 | -3.5 | -15% | -10% | 8.4 | 13.0 |
| CEE TIER 3 | Electric Dryer/Electric DHW | 576.1 | 743.0 | 228.1 | 40% | 31% | 380.3 | 453.3 |
| Electric Dryer/Gas DHW | 341.9 | 422.6 | 92.4 | 27% | 22% | 201.5 | 236.8 |
| Gas Dryer/Electric DHW | 259.9 | 356.7 | 134.4 | 52% | 38% | 189.3 | 231.6 |
| Gas Dryer/Gas DHW | 25.7 | 36.3 | -1.4 | -5% | -4% | 10.4 | 15.1 |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/Hours \* CF

*Where:*

*Hours = Assumed Run hours of Clothes Washer*

*= 265 [[533]](#footnote-533)*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.029 [[534]](#footnote-534)*

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below.

|  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- |
| **Efficiency Level** | **Dryer/DHW Fuel Combo** | **Remaining life of existing unit (first 5 years) ΔkW** | | **Remaining measure life (next 9 years) ΔkW** | **Mid Life Adjustment** | | **Equivalent Weighted Average Annual Savings** | |
| **Front** | **Top** | **Weighted Average** | **Front** | **Top** | **Front** | **Top** |
| ENERGY STAR, CEE Tier 1 | Electric Dryer/Electric DHW | 0.053 | 0.072 | 0.015 | 29% | 21% | 0.033 | 0.042 |
| Electric Dryer/Fuel DHW | 0.035 | 0.043 | 0.007 | 21% | 17% | 0.020 | 0.024 |
| Fuel Dryer/Electric DHW | 0.023 | 0.033 | 0.009 | 40% | 27% | 0.016 | 0.021 |
| Fuel Dryer/Fuel DHW | 0.004 | 0.005 | 0.001 | 25% | 19% | 0.002 | 0.003 |
| ENERGY STAR Most Efficient, CEE TIER 2 | Electric Dryer/Electric DHW | 0.061 | 0.079 | 0.023 | 37% | 29% | 0.041 | 0.050 |
| Electric Dryer/Fuel DHW | 0.036 | 0.044 | 0.008 | 23% | 19% | 0.021 | 0.025 |
| Fuel Dryer/Electric DHW | 0.028 | 0.038 | 0.014 | 51% | 37% | 0.021 | 0.026 |
| Fuel Dryer/Fuel DHW | 0.003 | 0.004 | 0.000 | -15% | -10% | 0.001 | 0.001 |
| CEE TIER 3 | Electric Dryer/Electric DHW | 0.063 | 0.081 | 0.025 | 40% | 31% | 0.043 | 0.052 |
| Electric Dryer/Fuel DHW | 0.037 | 0.046 | 0.010 | 27% | 22% | 0.023 | 0.027 |
| Fuel Dryer/Electric DHW | 0.028 | 0.039 | 0.015 | 52% | 38% | 0.022 | 0.026 |
| Fuel Dryer/Fuel DHW | 0.003 | 0.004 | 0.000 | -5% | -4% | 0.001 | 0.002 |

**Annual Fossil Fuel Savings Algorithm**

Break out savings calculated in Step 1 of electric energy savings (MEF savings) and extract Natural Gas DHW and Natural Gas dryer savings from total savings:

∆MMBtu = [(Capacity \* 1/IMEFbase \* Ncycles) \* ((%DHWbase \* %Natural Gas\_DHW \* R\_eff) + (%Dryerbase \* %Gas \_Dryer)] - [(Capacity \* 1/IMEFeff \* Ncycles) \* ((%DHWeff \* %Natural Gas\_DHW \* R\_eff) + (%Dryereff \* %Gas\_Dryer)] \* MMBtu\_convert

*Where:*

*R\_eff = Recovery efficiency factor*

*= 1.26[[535]](#footnote-535)*

*MMBtu \_convert = Convertion factor from kWh to MMBtu*

*= 0.003413*

*%Natural Gas\_DHW = Percentage of DHW savings assumed to be Natural Gas*

|  |  |
| --- | --- |
| **DHW fuel** | **%Natural Gas\_DHW** |
| Electric | 0% |
| Natural Gas | 100% |

*%Gas\_Dryer = Percentage of dryer savings assumed to be Natural Gas*

|  |  |
| --- | --- |
| **Dryer fuel** | **%Gas\_Dryer** |
| Electric | 0% |
| Natural Gas | 100% |

*Other factors as defined above*

Using the default assumptions provided above, the prescriptive savings for each configuration are presented below.

|  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- |
| **Efficiency Level** | **Configuration** | **Remaining life of existing unit (first 5 years)  ΔMMBtu** | | **Remaining measure life (next 9 years) ΔMMBtu** | **Mid Life Adjustment** | | **Equivalent Weighted Average Annual Savings** | |
| **Front** | **Top** | **Weighted Average** | **Front** | **Top** | **Front** | **Top** |
| ENERGY STAR, CEE Tier 1 | Electric Dryer/Electric DHW | 0.00 | 0.00 | 0.00 | n/a | n/a | 0.00 | 0.00 |
| Electric Dryer/Gas DHW | 0.74 | 1.11 | 0.32 | 43% | 29% | 0.50 | 0.66 |
| Gas Dryer/Electric DHW | 0.96 | 1.20 | 0.20 | 20% | 16% | 0.53 | 0.63 |
| Gas Dryer/Gas DHW | 1.70 | 2.31 | 0.51 | 30% | 22% | 1.03 | 1.30 |
| ENERGY STAR Most Efficient, CEE TIER 2 | Electric Dryer/Electric DHW | 0.00 | 0.00 | 0.00 | n/a | n/a | 0.00 | 0.00 |
| Electric Dryer/Gas DHW | 0.99 | 1.36 | 0.57 | 57% | 42% | 0.76 | 0.92 |
| Gas Dryer/Electric DHW | 1.03 | 1.27 | 0.27 | 26% | 21% | 0.60 | 0.71 |
| Gas Dryer/Gas DHW | 2.02 | 2.63 | 0.84 | 42% | 32% | 1.36 | 1.62 |
| CEE TIER 3 | Electric Dryer/Electric DHW | 0.00 | n/a | 0.00 | n/a | n/a | 0.00 | 0.00 |
| Electric Dryer/Gas DHW | 1.01 | 1.38 | 0.58 | 58% | 42% | 0.77 | 0.93 |
| Gas Dryer/Electric DHW | 1.08 | 1.32 | 0.32 | 30% | 24% | 0.65 | 0.76 |
| Gas Dryer/Gas DHW | 2.09 | 2.70 | 0.90 | 43% | 34% | 1.42 | 1.69 |

**Annual Water Savings Algorithm**

∆Water (CCF) = (Capacity \* (IWFbase - IWFeff)) \* Ncycles

*Where*

*WFbase = Integrated Water Factor of baseline clothes washer*

*= Values provided below*

*WFeff = Integrated Water Factor of efficient clothes washer*

*= Actual. If unknown assume average values provided below.*

|  |  |  |  |
| --- | --- | --- | --- |
| **Efficiency Level** | **IWF**[[536]](#footnote-536) | | |
| **Front Loading** | **Top Loading** | **Weighted Average** |
| Existing[[537]](#footnote-537) | 8.2 | 8.4 | n/a[[538]](#footnote-538) |
| Federal Standard | 4.7 | 8.4 | 5.92 |
| ENERGY STAR, CEE Tier 1 | 3.7 | 4.3 | 3.9 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 3.2 | 3.5 | 3.21 |
| CEE TIER 3 | 3.2 | n/a | 3.2 |

Using the default assumptions provided above, the prescriptive water savings for each efficiency level are presented below[[539]](#footnote-539):

|  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- |
| **Efficiency Level** | **Remaining life of existing unit (first 5 years)  ∆Water (ccf per year)** | | **Remaining measure life (next 9 years)  ∆Water (ccf per year)** | **Mid Life Adjustment** | | **Equivalent Weighted Average Annual Savings** | |
| **Front** | **Top** | **Weighted Average** | **Front** | **Top** | **Front** | **Top** |
| Existing | n/a | n/a | n/a | n/a | n/a | 0.00 | 0.00 |
| Federal Standard | n/a | n/a | n/a | n/a | n/a | 0.00 | 0.00 |
| ENERGY STAR, CEE Tier 1 | 5.0 | 5.2 | 2.3 | 47% | 44% | 3.5 | 3.6 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 5.8 | 6.1 | 3.2 | 54% | 52% | 4.3 | 4.4 |
| CEE TIER 3 | 5.9 | 6.1 | 3.2 | 54% | 52% | 4.4 | 4.5 |

kWh Savings from Water Reduction

The kWh savings from the waste reduction characterized above is now estimated. Please note that utilities’ must be careful not to double count the monetary benefit of these savings within cost effectiveness testing if the avoided costs of water already include the associated electric benefit.

ΔkWhwater[[540]](#footnote-540) = 2.07 kWh \* ∆Water (CCF)

Using the default assumptions provided above, the prescriptive water savings for each efficiency level are presented below:

|  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- |
| **Efficiency Level** | **Remaining life of existing unit (first 5 years)  ∆Water (ccf per year)** | | **Remaining measure life (next 9 years)  ∆Water (ccf per year)** | **Mid Life Adjustment** | | **Equivalent Weighted Average Annual Savings** | |
| **Front** | **Top** | **Weighted Average** | **Front** | **Top** | **Front** | **Top** |
| Existing | n/a | n/a | n/a | n/a | n/a | n/a | n/a |
| Federal Standard | n/a | n/a | n/a | n/a | n/a | n/a | n/a |
| ENERGY STAR, CEE Tier 1 | 10.4 | 10.8 | 4.8 | 47% | 44% | 7.2 | 7.5 |
| ENERGY STAR Most Efficient, CEE TIER 2 | 12.1 | 12.6 | 6.6 | 54% | 52% | 9.0 | 9.2 |
| CEE TIER 3 | 12.1 | 12.6 | 6.6 | 54% | 52% | 9.0 | 9.2 |

**Incremental Cost**

The full measure cost assumption is provided below[[541]](#footnote-541):

|  |  |
| --- | --- |
| **Efficiency Level** | **Early Replacement Full Install Cost** |
| ENERGY STAR, CEE Tier 1 | $879 |
| ENERGY STAR Most Efficient, CEE TIER 2 | $1100 |
| CEE TIER 3 | $1128 |

For early replacement measures, the deferred baseline replacement cost that would have been incurred after 3 years had the existing unit not been replaced is assumed to be $831.

**Measure Life**

The measure life is assumed to be 14 years [[542]](#footnote-542) and the existing unit is assumed to have a remaining life of 5 years[[543]](#footnote-543).

**Operation and Maintenance Impacts**

n/a

**Dehumidifier**

**Unique Measure Code(s): RS\_AP\_TOS\_DEHUMID\_0113**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the purchase (time of sale) and installation of a dehumidifier meeting the minimum qualifying efficiency standard established by the current ENERGY STAR (Version 3.0)[[544]](#footnote-544) in place of a unit that meets the minimum federal standard efficiency.

**Definition of Baseline Condition**

The baseline for this measure is defined as a new dehumidifier that meets the Federal Standard efficiency standards as defined below:

|  |  |
| --- | --- |
| **Capacity**  **(pints/day)** | **Federal Standard Criteria**  **(L/kWh)**[[545]](#footnote-545) |
| Up to 35 | ≥1.35 |
| > 35 to ≤45 | ≥1.50 |
| > 45 to ≤ 54 | ≥1.60 |
| > 54 to ≤ 75 | ≥1.70 |
| > 75 to ≤ 185 | ≥2.50 |

**Definition of Efficient Condition**

To qualify for this measure, the new dehumidifier must meet the ENERGY STAR standards effective 10/1/2012[[546]](#footnote-546) as defined below:

|  |  |
| --- | --- |
| **Capacity**  **(pints/day)** | **ENERGY STAR Criteria**  **(L/kWh)** |
| <75 | ≥1.85 |
| 75 to ≤185 | ≥2.80 |

Qualifying units shall be equipped with an adjustable humidistat control or shall require a remote humidistat control to operate.

**Annual Energy Savings Algorithm**

ΔkWh = (((Capacity \* 0.473) / 24) \* Hours) \* (1 / (L/kWh\_Base)– 1 / (L/kWh\_Eff))

*Where:*

*Capacity = Capacity of the unit (pints/day)*

*0.473 = Constant to convert Pints to Liters*

*24 = Constant to convert Liters/day to Liters/hour*

*Hours = Run hours per year*

*= 1632 [[547]](#footnote-547)*

*L/kWh = Liters of water per kWh consumed, as provided in tables above*

Annual kWh results for each capacity class are presented below using the average of the capacity range. If the capacity of installed units is collected, the savings should be calculated using the algorithm. If the capacity is unknown, a default average value is provided:

|  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- |
|  |  |  |  | **Annual kWh** | | |
| **Capacity** | **[[548]](#footnote-548)Capacity Used** | **Federal Standard Criteria** | **ENERGY STAR Criteria** | **Federal Standard** | **ENERGY STAR** | **Savings** |
| **(pints/day) Range** | **(≥ L/kWh)** | **(≥ L/kWh)** |
| ≤25 | 20 | 1.35 | 1.85 | 477 | 348 | 129 |
| > 25 to ≤35 | 30 | 1.35 | 1.85 | 715 | 522 | 193 |
| > 35 to ≤45 | 40 | 1.5 | 1.85 | 858 | 695 | 162 |
| > 45 to ≤ 54 | 50 | 1.6 | 1.85 | 1005 | 869 | 136 |
| > 54 to ≤ 75 | 65 | 1.7 | 1.85 | 1230 | 1130 | 100 |
| > 75 to ≤ 185 | 130 | 2.5 | 2.8 | 1673 | 1493 | 179 |
| Average | 46 | 1.51 | 1.85 | 983 | 800 | 183 |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/Hours \* CF

*Where:*

*Hours = Annual operating hours*

*= 1632 hours[[549]](#footnote-549)*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.37 [[550]](#footnote-550)*

|  |  |
| --- | --- |
| **Capacity** | **ΔkW** |
| **(pints/day) Range** |
| ≤25 | 0.029 |
| > 25 to ≤35 | 0.044 |
| > 35 to ≤45 | 0.037 |
| > 45 to ≤ 54 | 0.031 |
| > 54 to ≤ 75 | 0.023 |
| > 75 to ≤ 185 | 0.041 |
| Average | 0.042 |

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The assumed incremental capital cost for this measure is $45[[551]](#footnote-551).

**Measure Life**

The measure life is assumed to be 12 years. [[552]](#footnote-552)

**Operation and Maintenance Impacts**

n/a

**ENERGY STAR Air Purifier/Cleaner**

**Unique Measure Code(s): RS\_AP\_TOS\_AIRPUR\_0414**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

An air purifier (cleaner) is a portable electric appliance that removes dust and fine particles from indoor air. This measure characterizes the purchase and installation of a unit meeting the efficiency specifications of ENERGY STAR in place of a baseline model.

**Definition of Baseline Condition**

The baseline equipment is assumed to be a conventional non-ENERGY STAR unit with consumption estimates based upon EPA research on available models, 2011[[553]](#footnote-553).

**Definition of Efficient Condition**

The efficient equipment is defined as an air purifier meeting the efficiency specifications of ENERGY STAR as provided below.

* Must produce a minimum 50 Clean Air Delivery Rate (CADR) for Dust[[554]](#footnote-554) to be considered under this specification.
* Minimum Performance Requirement: = 2.0 CADR/Watt (Dust)
* Standby Power Requirement: = 2.0 Watts Qualifying models that perform secondary consumer functions (e.g. clock, remote control) must meet the standby power requirement.
* UL Safety Requirement: Models that emit ozone as a byproduct of air cleaning must meet UL Standard 867 (ozone production must not exceed 50ppb)

**Annual Energy Savings Algorithm**

ΔkWh = kWhBase- kWhESTAR

Where:

kWhBASE *=* Baseline kWh consumption per year[[555]](#footnote-555)

= see table below

kWhESTAR *=* ENERGY STAR kWh consumption per year[[556]](#footnote-556)

= see table below

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Clean Air Delivery Rate (CADR)** | **CADR used in calculation** | **Baseline Unit Energy Consumption (kWh/year)** | **ENERGY STAR Unit Energy Consumption (kWh/year)** | **ΔkWH** |
| CADR 51-100 | 75 | 441 | 148 | 293 |
| CADR 101-150 | 125 | 733 | 245 | 488 |
| CADR 151-200 | 175 | 1025 | 342 | 683 |
| CADR 201-250 | 225 | 1317 | 440 | 877 |
| CADR Over 250 | 275 | 1609 | 537 | 1072 |

**Summer Coincident Peak kW Savings Algorithm**

∆kW*=* ∆kWh/Hours \* CF

Where:

∆kWh = Gross customer annual kWh savings for the measure

Hours = Average hours of use per year

= 5840 hours[[557]](#footnote-557)

CF = Summer Peak Coincidence Factor for measure

= 0.67[[558]](#footnote-558)

|  |  |
| --- | --- |
| **Clean Air Delivery Rate** | **ΔkW** |
| CADR 51-100 | 0.034 |
| CADR 101-150 | 0.056 |
| CADR 151-200 | 0.078 |
| CADR 201-250 | 0.101 |
| CADR Over 250 | 0.123 |

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is $0.[[559]](#footnote-559)

**Measure Life**

The measure life is assumed to be 9 years[[560]](#footnote-560).

**Operation and Maintenance Impacts**

There are no operation and maintenance cost adjustments for this measure.[[561]](#footnote-561)

**Clothes Dryer**

**Unique Measure Code(s): RS\_AP\_TOS\_DISHWAS\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a residential clothes dryer meeting the ENERGY STAR criteria. ENERGY STAR qualified clothes dryers save energy through a combination of more efficient drying and reduced runtime of the drying cycle. More efficient drying is achieved through increased insulation, modifying operating conditions such as air flow and/or heat input rate, improving air circulation through better drum design or booster fans, and improving efficiency of motors. Reducing the runtime of dryers through automatic termination by temperature and moisture sensors is believed to have the greatest potential for reducing energy use in clothes dryers[[562]](#footnote-562). ENERGY STAR provides criteria for both gas and electric clothes dryers.

**Definition of Baseline Condition**

The baseline condition is a clothes dryer meeting the minimum federal requirements for units manufactured on or after January 1, 2015.

**Definition of Efficient Condition**

Clothes dryer must meet the ENERGY STAR criteria, as required by the program.

**Annual Energy Savings Algorithm**

∆kWh = (Load/CEFbase – Load/CEFeff) \* Ncycles \* %Electric

*Where:*

*Load = The average total weight (lbs) of clothes per drying cycle. If dryer size is unknown, assume standard.*

|  |  |
| --- | --- |
| ***Dryer Size*** | ***Load (lbs.)[[563]](#footnote-563)*** |
| *Standard* | *8.45* |
| *Compact* | *3* |

*CEFbase = Combined energy factor (CEF) (lbs/kWh) of the baseline unit is based on existing federal standards energy factor and adjusted to CEF as performed in the ENERGY STAR analysis[[564]](#footnote-564). If product class unknown, assume electric, standard.*

| ***Product Class*** | ***CEFbase (lbs/kWh)*** |
| --- | --- |
| *Vented Electric, Standard (≥ 4.4 ft3)* | *3.11* |
| *Vented Electric, Compact (120V) (< 4.4 ft3)* | *3.01* |
| *Vented Electric, Compact (240V) (<4.4 ft3)* | *2.73* |
| *Ventless Electric, Compact (240V) (<4.4 ft3)* | *2.13* |
| *Vented Gas* | *2.84[[565]](#footnote-565)* |

*CEFeff = CEF (lbs/kWh) of the ENERGY STAR unit based on ENERGY STAR requirements.[[566]](#footnote-566) If product class unknown, assume electric, standard.*

| ***Product Class*** | ***CEFeff (lbs/kWh)*** |
| --- | --- |
| *Vented or Ventless Electric, Standard (≥ 4.4 ft3)* | *3.93* |
| *Vented or Ventless Electric, Compact (120V) (< 4.4 ft3)* | *3.80* |
| *Vented Electric, Compact (240V) (< 4.4 ft3)* | *3.45* |
| *Ventless Electric, Compact (240V) (< 4.4 ft3)* | *2.68* |
| *Vented Gas* | *3.48[[567]](#footnote-567)* |

*Ncycles = Number of dryer cycles per year*

*= 311 cycles per year.[[568]](#footnote-568)*

*%Electric = The percent of overall savings coming from electricity*

| ***Clothes Dryer Fuel Type*** | ***%Electric [[569]](#footnote-569)*** |
| --- | --- |
| *Electric* | *100%* |
| *Gas* | *16%* |

|  |  |  |
| --- | --- | --- |
| **Product Class** | **Algorithm** | **ΔkWh** |
| *Vented or Ventless Electric, Standard (≥ 4.4 ft3)* | = ((8.45/3.11 – 8.45/3.93) \* 311 \* 100%) | 176.3 |
| *Vented or Ventless Electric, Compact (120V) (< 4.4 ft3)* | = ((3/3.01 – 3/3.80) \* 311 \* 100%) | 64.4 |
| *Vented Electric, Compact (240V) (< 4.4 ft3)* | = ((3/2.73 – 3/3.45) \* 311 \* 100%) | 71.3 |
| *Ventless Electric, Compact (240V) (< 4.4 ft3)* | = ((3/2.13 – 3/2.68) \* 311 \* 100%) | 89.9 |
| *Vented Gas* | = ((8.45/2.84 – 8.45/3.48) \* 311 \* 16%) | 27.2 |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/Hours \* CF

*Where:*

*ΔkWh = Energy Savings as calculated above*

*Hours = Annual run hours of clothes dryer.*

*=290 hours per year.[[570]](#footnote-570)*

*CF = Summer Peak Coincidence Factor for measure*

*= 2.9%[[571]](#footnote-571)*

|  |  |  |
| --- | --- | --- |
| **Product Class** | **Algorithm** | **ΔkW** |
| *Vented or Ventless Electric, Standard (≥ 4.4 ft3)* | = 176.3/290 \* 0.029 | 0.018 |
| *Vented or Ventless Electric, Compact (120V) (< 4.4 ft3)* | = 64.4/290 \* 0.029 | 0.006 |
| *Vented Electric, Compact (240V) (< 4.4 ft3)* | = 71.3/290 \* 0.029 | 0.007 |
| *Ventless Electric, Compact (240V) (< 4.4 ft3)* | = 89.9/290 \* 0.029 | 0.009 |
| *Vented Gas* | = 27.2/290 \* 0.029 | 0.003 |

**Annual Fossil Fuel Savings Algorithm**

Natural gas savings only apply to ENERGY STAR vented gas clothes dryers.

∆MMBtu = (Load/CEFbase – Load/CEFeff) \* Ncycles \* MMBtu\_convert \* %Gas

*Where:*

*MMBtu\_convert = Conversion factor from kWh to MMBtu*

*= 0.003413*

*%Gas = Percent of overall savings coming from gas*

| ***Clothes Dryer Fuel Type*** | ***%Gas*** *[[572]](#footnote-572)* |
| --- | --- |
| *Electric* | *0%* |
| *Gas* | *84%* |

|  |  |  |
| --- | --- | --- |
| **Product Class** | **Algorithm** | **ΔMMBtu** |
| *Vented or Ventless Electric, Standard (≥ 4.4 ft3)* | n/a | 0 |
| *Vented or Ventless Electric, Compact (120V) (< 4.4 ft3)* | n/a | 0 |
| *Vented Electric, Compact (240V) (< 4.4 ft3)* | n/a | 0 |
| *Ventless Electric, Compact (240V) (< 4.4 ft3)* | n/a | 0 |
| *Vented Gas* | =(8.45/2.84 – 8.45/3.48) \* 311 \* 0.003413 \* 0.84 | 0.49 |

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for an ENERGY STAR clothes dryer is assumed to be $152[[573]](#footnote-573)

**Measure Life**

The expected measure life is assumed to be 14 years[[574]](#footnote-574).

**Operation and Maintenance Impacts**

n/a

**Dishwasher**

**Unique Measure Code(s): RS\_AP\_TOS\_DISHWAS\_0415**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

A dishwasher meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard. This measure is only for standard dishwashers, not compact dishwashers. A compact dishwasher is a unit that holds less than eight place settings with six serving pieces.

The ENERGY STAR Dishwasher specification for Dishwashers is in the process of being revised from version 5.2 to version 6.0. The version 6.0 specification will become effective on January 29, 2016. Savings for both specification version 5.2 and 6.0 are contained in this measure characterization.

**Definition of Baseline Condition**

The baseline for this measure is defined as a new dishwasher that meets the Federal Standard efficiency standards as defined below[[575]](#footnote-575):

|  |  |  |
| --- | --- | --- |
| **Dishwasher Type** | **Maximum kWh/year** | **Maximum gallons/cycle** |
| Standard | 307 | 5.0 |

**Definition of Efficient Condition**

To qualify for this measure, the new dishwasher must meet the ENERGY STAR standards effective 01/20/2012[[576]](#footnote-576) for version 5.2 and 01/29/2016[[577]](#footnote-577) for version 6.0 as defined below:

|  |  |  |  |
| --- | --- | --- | --- |
| **ENERGY STAR Version** | **Dishwasher Type** | **Maximum kWh/year** | **Maximum gallons/cycle** |
| 5.2 | Standard | 295 | 4.25 |
| 6.0 | Standard | 270 | 3.50 |

**Annual Energy Savings Algorithm**

ΔkWh[[578]](#footnote-578) = ((kWhBase- kWhESTAR) \* (%kWh\_op + (%kWh\_heat \* %Electric\_DHW )))

*Where:*

*kWhBASE = Baseline kWh consumption per year*

*= 307 kWh*

*kWhESTAR = ENERGY STAR kWh annual consumption*

|  |  |
| --- | --- |
| **ENERGY STAR Version** | **Maximum kWh/year** |
| 5.2 | 295 |
| 6.0 | 270 |

*%kWh\_op = Percentage of dishwasher energy consumption used for unit operation*

*= 1 - 56%[[579]](#footnote-579)*

*= 44%*

*%kWh\_heat = Percentage of dishwasher energy consumption used for water heating*

*= 56%[[580]](#footnote-580)*

*%Electric\_DHW = Percentage of DHW savings assumed to be electric*

|  |  |
| --- | --- |
| ***DHW fuel*** | ***%Electric\_DHW*** |
| *Electric* | *100%* |
| *Natural Gas* | *0%* |
| *Unknown* | *65%[[581]](#footnote-581)* |

|  |  |  |  |
| --- | --- | --- | --- |
| **ENERGY STAR Specification** | **DHW Fuel** | **Algorithm** | **ΔkWh** |
| 5.2 | Electric | =((307- 295) \* (0.44 + (0.56 \* 1.0))) | 12.0 |
| 5.2 | Unknown | = ((307- 295) \* (0.44 + (0.56 \* 0.65))) | 9.6 |
| 6.0 | Electric | = ((307- 270) \* (0.44 + (0.56 \* 1.0))) | 37 |
| 6.0 | Unknown | = ((307- 270) \* (0.44 + (0.56 \* 0.65))) | 29.7 |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/Hours \* CF

*Where:*

*Hours = Annual operating hours[[582]](#footnote-582)*

*= 210 hours*

*CF = Summer Peak Coincidence Factor*

*= 2.6% [[583]](#footnote-583)*

|  |  |  |  |
| --- | --- | --- | --- |
| **ENERGY STAR Specification** | **DHW Fuel** | **Algorithm** | **ΔkW** |
| 5.2 | Electric | = 12/210 \* 0.026 | 0.0015 |
| 5.2 | Unknown | = 9.65/210 \* 0.026 | 0.0012 |
| 6.0 | Electric | = 37/210 \* 0.026 | 0.0046 |
| 6.0 | Unknown | = 29.75/210 \* 0.02 | 0.0037 |

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = (kWhBase- kWhESTAR) \* %kWh\_heat \* %Natural Gas\_DHW \* R\_eff \*0.003413

*Where*

*%kWh\_heat = % of dishwasher energy used for water heating*

*= 56%*

*%Natural Gas\_DHW = Percentage of DHW savings assumed to be Natural Gas*

|  |  |
| --- | --- |
| ***DHW fuel*** | ***%Natural Gas\_DHW*** |
| *Electric* | *0%* |
| *Natural Gas* | *100%* |
| *Unknown* | *35%[[584]](#footnote-584)* |

*R\_eff = Recovery efficiency factor*

*= 1.*

*26[[585]](#footnote-585)*

*0.003413 = factor to convert from kWh to MMBtu*

|  |  |  |  |
| --- | --- | --- | --- |
| **ENERGY STAR Specification** | **DHW Fuel** | **Algorithm** | **ΔMMBtu** |
| 5.2 | Gas | = (307- 295) \* 0.56 \* 1.0 \* 1.26 \* 0.003413 | 0.03 |
| 5.2 | Unknown | = (307- 295) \* 0.56 \* 0.35 \* 1.26 \* 0.003413 | 0.01 |
| 6.0 | Gas | = (307- 270) \* 0.56 \* 1.0 \* 1.26 \* 0.003413 | 0.09 |
| 6.0 | Unknown | = (307- 270) \* 0.56 \* 0.35 \* 1.26 \* 0.003413 | 0.03 |

**Annual Water Savings Algorithm**

ΔCCF = (WaterBase- WaterEFF) \* GalToCCF

*Where*

*WaterBase = water consumption of conventional unit*

*= 700 gallons[[586]](#footnote-586)*

*WaterEFF = annualwater consumption of efficient unit:*

|  |  |
| --- | --- |
| **ENERGY STAR Specification** | **WaterEFF (gallons)** |
| 5.2 | 595[[587]](#footnote-587) |
| 6.0 | 490[[588]](#footnote-588) |

GalToCCF = factor to convert from gallons to CCF

= 0.001336

|  |  |  |
| --- | --- | --- |
| **ENERGY STAR Specification** | **Algorithm** | **ΔCCF** |
| 5.2 | = (700– 595) \* 0.001336 | 0.14 |
| 6.0 | = (700– 490) \* 0.001336 | 0.28 |

**Incremental Cost**

The assumed incremental capital cost for this measure is $50[[589]](#footnote-589).

**Measure Life**

The measure life is assumed to be 10 years[[590]](#footnote-590).

**Operation and Maintenance Impacts**

n/a

*Shell Savings End Use*

**Air sealing\***

**Unique Measure Code: RS\_SL\_RTR\_AIRSLG\_0711**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure characterization provides a method of claiming both heating and cooling (where appropriate) savings from the improvement of a residential building’s air-barrier, which together with its insulation defines the thermal boundary of the conditioned space.

The measure assumes that a trained auditor, contractor or utility staff member is on location, and will measure and record the existing and post air-leakage rate using a blower door in accordance with industry best practices[[591]](#footnote-591). Where possible, the efficiency of the heating and cooling system used in the home should be recorded, but default estimates are provided if this is not available.

This is a retrofit measure.

**Definition of Baseline Condition**

The existing air leakage prior to any air sealing work should be determined using a blower door.

**Definition of Efficient Condition**

Air sealing materials and diagnostic testing should meet all program eligibility qualification criteria. The post air sealing leakage rate should then be determined using a blower door.

**Annual Energy Savings Algorithm**

Cooling savings from reduction in Air Conditioning Load:

ΔkWh = [(((CFM50Exist – CFM50New) / N-cool) \*60 \* CDH \* DUA \* 0.018) / 1,000 / ηCool] \* LM

*Where:*

*CFM50exist = Blower Door result (CFM50) prior to air sealing*

*= actual*

*CFMnew = Blower Door result (CFM50) after air sealing*

*= actual*

*N-cool = conversion from CFM50 to CFMNatural[[592]](#footnote-592)*

*= dependent on* location and number of stories:[[593]](#footnote-593)

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Location** | **N\_cool (by # of stories)** | | | |
| **1** | **1.5** | **2** | **3** |
| *Wilmington, DE* | 38.4 | 34.0 | 31.2 | 27.6 |
| *Baltimore, MD* | 38.4 | 34.0 | 31.2 | 27.6 |
| *Washington, DC* | 40.3 | 35.7 | 32.7 | 29.0 |

*CDH = Cooling Degree Hours[[594]](#footnote-594)*

*= dependent on location:*

|  |  |
| --- | --- |
| ***Location*** | ***Cooling Degree Hours***  ***(75°F set point)*** |
| *Wilmington, DE* | *7,514* |
| *Baltimore, MD* | *9,616* |
| *Washington, DC* | *13,178* |

*DUA = Discretionary Use Adjustment[[595]](#footnote-595)*

*= 0.75*

*0.018 = The volumetric heat capacity of air (Btu/ft3°F)*

*ηCool = Efficiency in SEER of Air Conditioning equipment*

*= actual. If not available use[[596]](#footnote-596):*

|  |  |
| --- | --- |
| ***Age of Equipment*** | ***SEER Estimate*** |
| *Before 2006* | *10* |
| *After 2006* | *13* |

*LM = Latent Multiplier to account for latent cooling demand*[[597]](#footnote-597)

|  |  |
| --- | --- |
| **Location** | **LM** |
|
| *Wilmington, DE* | 4.09 |
| *Baltimore, MD* | 3.63 |
| *Washington, DC* | 3.63 |

Illustrative example – do not use as default assumption

A single story home in Wilmington, DE with a 12 SEER Air Conditioning unit, has pre and post blower door test results of 3,400 and 2,250.

ΔkWh = [(((3,400 – 2,250) / 38.4) \*60 \* 7,514 \* 0.75 \* 0.018) / 1,000 / 12] \* 4.09

= 62.1 kWh

Heating savings for homes with electric heat (Heat Pump or resistance):

ΔkWh = ((((CFM50Exist – CFM50New) / N-heat) \* 60 \* 24 \* HDD \* 0.018) / 1,000,000 / ηHeat) \* 293.1

*Where:*

*N-heat = conversion from CFM50 to CFMNatural*

*= Based on location and* number of stories[[598]](#footnote-598)*:*

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Location** | **N\_heat (by # of stories)** | | | |
| **1** | **1.5** | **2** | **3** |
| *Wilmington, DE* | 24.5 | 21.7 | 19.9 | 17.6 |
| *Baltimore, MD* | 25.1 | 22.3 | 20.4 | 18.1 |
| *Washington, DC* | 25.7 | 22.7 | 20.8 | 18.5 |

*HDD = Heating Degree Days*

*= dependent on location[[599]](#footnote-599)*

|  |  |
| --- | --- |
| ***Location*** | ***Heating Degree Days***  ***(60°F set point)*** |
| *Wilmington, DE* | *3,275* |
| *Baltimore, MD* | *3,457* |
| *Washington, DC* | *2,957* |

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[600]](#footnote-600):*

|  |  |  |  |
| --- | --- | --- | --- |
| ***System Type*** | ***Age of Equipment*** | ***HSPF Estimate*** | ***COP Estimate****[[601]](#footnote-601)* |
| *Heat Pump* | *Before 2006* | *6.8* | *2.00* |
| *After 2006* | *7.7* | *2.26* |
| *Resistance* | *n/a* | *n/a* | *1.00* |

293.1 = Converts MMBtu to kWh

Illustrative example – do not use as default assumption

A two storyhome in Wilmington, DE with a heat pump with COP of 2.5, has pre and post blower door test results of 3,400 and 2,250.

ΔkWh = [(((3,400 – 2,250) / 24.5) \*60 \* 24 \* 3,275 \* 0.018) / 1,000,000 / 2.5] \* 293.1

= 467.1 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh / FLHcool \* CF

*Where:*

*FLHcool = Full Load Cooling Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | **FLHcool** |
| *Wilmington, DE* | *524 [[602]](#footnote-602)* |
| *Baltimore, MD* | *542 [[603]](#footnote-603)* |
| *Washington, DC* | *681* |

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[604]](#footnote-604)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[605]](#footnote-605)*

Illustrative example – do not use as default assumption

A single storyhome in Wilmington, DE with a 12 SEER Air Conditioning unit, has pre and post blower door test results of 3,400 and 2,250.

ΔkW = 62.1 / 524 \* 0.69

= 0.08 kW

**Annual Fossil Fuel Savings Algorithm**

For homes with Fossil Fuel Heating:

ΔMMBTU = (((CFM50Exist – CFM50New) / N-heat) \*60 \* 24 \* HDD \* 0.018) / 1,000,000 / ηHeat

*Where:*

N-heat *= conversion from CFM50 to CFMNatural*

*= Based on location and* number of stories[[606]](#footnote-606)*:*

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Location** | **N\_heat (by # of stories)** | | | |
| **1** | **1.5** | **2** | **3** |
| *Wilmington, DE* | 24.5 | 21.7 | 19.9 | 17.6 |
| *Baltimore, MD* | 25.1 | 22.3 | 20.4 | 18.1 |
| *Washington, DC* | 25.7 | 22.7 | 20.8 | 18.5 |

*HDD = Heating Degree Days*

*= dependent on location[[607]](#footnote-607)*

|  |  |
| --- | --- |
| ***Location*** | ***Heating Degree Days***  ***(60°F set point)*** |
| *Wilmington, DE* | *3,275* |
| *Baltimore, MD* | *3,457* |
| *Washington, DC* | *2,957* |

*ηHeat = Efficiency of Heating equipment (equipment efficiency \* distribution efficiency)*

*= actual[[608]](#footnote-608). If not available use 84% for equipment efficiency and 78% for distribution efficiency to give 66%[[609]](#footnote-609).*

Illustrative example – do not use as default assumption

A single story home in Wilmington, DE with a 70% heating system efficiency, has pre and post blower door test results of 3,400 and 2,250.

ΔMMBtu = (((3,400 – 2,250) / 24.5) \*60 \* 24 \* 3,275 \* 0.018) / 1,000,000 / 0.7

= 5.7 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure should be the actual installation and labor cost to perform the air sealing work.

**Measure Life**

The measure life is assumed to be 15 yrs[[610]](#footnote-610).

**Operation and Maintenance Impacts**

n/a

**Attic/ceiling/roof insulation\***

**Unique Measure Code: RS\_SL\_RTR\_ATTICI\_0711**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure characterization is for the installation of new insulation in the attic/roof/ceiling of a residential building. The measure assumes that an auditor, contractor or utility staff member is on location, and will measure and record the existing and new insulation depth and type (to calculate R-values), the surface area of insulation added, and where possible the efficiency of the heating and cooling system used in the home.

This is a retrofit measure.

**Definition of Baseline Condition**

The existing insulation R-value should include the total attic floor / roof assembly. An R-value of 5 should be assumed for the roof assembly plus the R-value of any existing insulation[[611]](#footnote-611). Therefore if there is no insulation currently present, the R-value of 5 should be used.

**Definition of Efficient Condition**

The new insulation should meet any qualification criteria required for participation in the program. The new insulation R-value should include the total attic floor /roof assembly and include the effective R-value of any existing insulation that is left in situ.

**Annual Energy Savings Algorithm**

Savings from reduction in Air Conditioning Load:

ΔkWh = ((1/Rexist – 1/Rnew) \* CDH \* DUA \* Area) / 1,000 / ηCool \* Adjcool

*Where:*

*Rexist = R-value of roof assembly plus any existing insulation*

*= actual (minimum of R-5)*

*Rnew = R-value of roof assembly plus new insulation*

*= actual*

*CDH = Cooling Degree Hours[[612]](#footnote-612)*

*= dependent on location:*

|  |  |
| --- | --- |
| ***Location*** | ***Cooling Degree Hours***  ***(75°F set point)*** |
| *Wilmington, DE* | *7,514* |
| *Baltimore, MD* | *9,616* |
| *Washington, DC* | *13,178* |

*DUA = Discretionary Use Adjustment[[613]](#footnote-613)*

*= 0.75*

*Area = square footage of area covered by new insulation*

*= actual*

*ηCool = Efficiency in SEER of Air Conditioning equipment*

*= actual. If not available use[[614]](#footnote-614):*

|  |  |
| --- | --- |
| ***Age of Equipment*** | ***SEER Estimate*** |
| *Before 2006* | *10* |
| *After 2006* | *13* |

Adjcool = 0.8[[615]](#footnote-615)

Illustrative example – do not use as default assumption

Insulating 1200 square feet of attic from R-5 to R-30 in a home with a 12 SEER central Air Conditioning unit in Baltimore, MD.

ΔkWh = ((1/5 – 1/30) \* 9,616 \* 0.75 \* 1,200) / 1,000 / 12 \* 0.8

= 96 kWh

Savings for homes with electric heat (Heat Pump or resistance):

ΔkWh = (((1/Rexist – 1/Rnew) \* HDD \* 24 \* Area) / 1,000,000 / ηHeat) \* 293.1 \* Adjheat

*HDD = Heating Degree Days*

*= dependent on location[[616]](#footnote-616)*

|  |  |
| --- | --- |
| ***Location*** | ***Heating Degree Days***  ***(60°F set point)*** |
| *Wilmington, DE* | *3,275* |
| *Baltimore, MD* | *3,457* |
| *Washington, DC* | *2,957* |

1,000,000 = Converts Btu to MMBtu

*ηHeat = Efficiency in COP of Heating equipment*

*= actual. If not available use[[617]](#footnote-617):*

|  |  |  |  |
| --- | --- | --- | --- |
| ***System Type*** | ***Age of Equipment*** | ***HSPF Estimate*** | ***COP Estimate*** |
| *Heat Pump* | *Before 2006* | *6.8* | *2.00* |
| *After 2006* | *7.7* | *2.26* |
| *Resistance* | *n/a* | *n/a* | *1.00* |

293.1 = Converts MMBtu to kWh

Adjheat = 0.6[[618]](#footnote-618)

Illustrative example – do not use as default assumption

Insulating 1200 square feet of attic from R-5 to R-30 in a home with a 2.5COP Heat Pump in Baltimore, MD.

ΔkWh = (((1/5 – 1/30) \* 3457 \* 24 \* 1,200) / 1,000,000 / 2.5) \* 293.1 \* 0.6

= 1,167 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh / FLHcool \* CF

*Where:*

*FLHcool = Full Load Cooling Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | **FLHcool** |
| *Wilmington, DE* | *524 [[619]](#footnote-619)* |
| *Baltimore, MD* | *542 [[620]](#footnote-620)* |
| *Washington, DC* | *681* |

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[621]](#footnote-621)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[622]](#footnote-622)*

Illustrative example – do not use as default assumption

Insulating 1200 square feet of attic from R-5 to R-30 in a home with a 12 SEER central Air Conditioning unit in Baltimore, MD.

ΔkW = 96 / 542 \* 0.69

= 0.12 kW

**Annual Fossil Fuel Savings Algorithm**

ΔMMBTU = ((1/Rexist – 1/Rnew) \* HDD \* 24 \* Area) / 1,000,000 / ηHeat \* Adjheat

*Where:*

*HDD = Heating Degree Days*

*= dependent on location[[623]](#footnote-623)*

|  |  |
| --- | --- |
| ***Location*** | ***Heating Degree Days***  ***(60°F set point)*** |
| *Wilmington, DE* | *3,275* |
| *Baltimore, MD* | *3,457* |
| *Washington, DC* | *2,957* |

*ηHeat = Efficiency of Heating equipment (equipment efficiency \* distribution efficiency)*

*= actual[[624]](#footnote-624). If not available use 84% for equipment efficiency and 78% for distribution efficiency to give 66%[[625]](#footnote-625).*

Adjheat = 0.60[[626]](#footnote-626)

Illustrative example – do not use as default assumption

Insulating 1200 square feet of attic from R-5 to R-30 in a home with a 75% efficiency heating system in Baltimore, MD.

ΔMMBtu =((1/5 – 1/30) \* 3457 \* 24 \* 1,200) / 1,000,000 / 0.75 \* .60

= 13 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure should be the actual installation and labor cost to perform the insulation work.

**Measure Life**

The measure life is assumed to be 25 years[[627]](#footnote-627).

**Operation and Maintenance Impacts**

n/a

**Efficient Windows - Energy Star Time of sale**

**Unique Measure Code(s): RS\_SL\_TOS\_WINDOW\_0510**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes the purchase of Energy Star Windows (u-0.32; SHGC-0.40 minimum requirement for North Central region) at natural time of replacement or new construction outside of the Energy Star Homes program. This does not relate to a window retrofit program. Measure characterization assumes electric heat- either resistance or heat pump.

**Definition of Baseline Condition**

The baseline condition is a standard double pane window with vinyl sash, (u- 0.49 SHGC-0.58).

**Definition of Efficient Condition**

The efficient condition is an ENERGY STAR window (u-0.32; SHGC-0.40 minimum requirement for North Central region).

**Annual Energy Savings Algorithm** [[628]](#footnote-628)

Heating kWh Savings (Electric Resistance) = 356 kWh per 100 square feet window area

Heating kWh Savings (Heat Pump COP 2.0) = 194 kWh per 100 square feet window area

Cooling kWh Savings (SEER 10) = 205 kWh per 100 square feet window area

**Summer Coincident Peak kW Savings Algorithm**

ΔkWcooling = ΔkWREM \* CF

*Where:*

*ΔkWREM = Delta kW calculated in REMRate model*

*= 0.12 kW per 100 square feet window area*

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[629]](#footnote-629)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[630]](#footnote-630)*

ΔkW*SSP* cooling = 0.12 \* 0.69

= 0.083 kW per 100 square feet of windows

ΔkW*PJM* cooling = 0.12 \* 0.66

= 0.079 kW per 100 square feet of windows

**Annual Fossil Fuel Savings Algorithm**

n/a for homes with electric heat.

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $150 per 100 square feet of windows.[[631]](#footnote-631)

**Measure Life**

The measure life is assumed to be 25 years.[[632]](#footnote-632)

**Operation and Maintenance Impacts**

n/a

**Crawl Space Insulation and Encapsulation\*\***

**Unique Measure Code(s): TBD**

**Effective Date: TBD**

**End Date: TBD**

**Measure Description**

This measure relates to the insulation and/or encapsulation to a crawl space under a single family home. This measure also allows for the possibility that the crawl space will be encapsulated. This encapsulation in effect changes the crawlspace from an unconditioned space to a conditioned space, thus eliminating losses from any duct work that may run through the space.

**Definition of Baseline Condition**

The baseline depends on site specific conditions. However, it is most likely to be an unencapsulated, uninsulated crawlspace.

**Definition of Efficient Condition**

The efficient condition is a crawlspace that is insulated and/or encapsulated.

**Annual Energy Savings Algorithm**[[633]](#footnote-633)

∆kWh = kWhcooling + kWhheating + kWhducts

*Where:*

*kWhcooling = reduction in cooling requirement. Only applicable to homes with central cooling*

*= ((1 / R\_Old\_AG – 1/(R\_Old\_AG + R\_Added\_AG)) \* L\_Basement\_Wall \* H\_Basement\_Wall\_AG \* (1-Framing\_Factor) \* CDH \* DUA) / (1000 \* ηCool) \* AdjBasementcool*

*Where:*

*R\_Old\_AG = R\_Value of foundation wall above grade*

*= Actual, if unknown assume 1.0[[634]](#footnote-634)*

*R\_Added\_AG = R-Value of additional insulation*

*L\_Basement\_Wall = Length of basement wall around the insulated perimeter*

*H\_Basement\_Wall\_AG = Height of basement wall above grade*

*Framing\_Factor = Adjustment to account for area of framing if cavity insulation*

*= 0% if spray foam or rigid foam*

*=25% if studs and cavity insulation[[635]](#footnote-635)*

*24 = converts days to hours*

*CDH= Cooling Degree Hours[[636]](#footnote-636)*

*= dependent on location:*

|  |  |
| --- | --- |
| ***Location*** | ***Cooling Degree Hours***  ***(75°F set point)*** |
| *Wilmington, DE* | *7,514* |
| *Baltimore, MD* | *9,616* |
| *Washington, DC* | *13,178* |

*DUA = Discretionary Use Adjustment, to account for the fact that people do not always operate AC when conditions call for it.*

*=0.75[[637]](#footnote-637)*

*ηCool = Efficiency in SEER of Cooling Equipment.*

*= Actual. If unknown use[[638]](#footnote-638):*

|  |  |
| --- | --- |
| ***Age of Equipment*** | ***SEER Estimate*** |
| *Before 2006* | *10* |
| *After 2006* | *13* |

*AdjBasementcool = Adjustment to take into account prescriptive algorithms overclaiming savings*

*= 80%[[639]](#footnote-639)*

*kWhheating = Reduction inannual heating requirement, if electric heat (resistance or heat pump)*

*= (kWhAG + kWhBG) \* AdjBasement*

*Where:*

*kWhAG = Savings from insulation on walls or crawlspaces above grade*

*=((1/R\_Old\_AG – 1/(R\_Old\_AG + R\_Added)) \* L\_Basement\_Wall \* H\_Basement\_Wall\_AG \* (1-Framing\_Factor) \* HDD \* 24) / (3412 \* ηHeat)*

*kWhBG = Savings from insulation on walls or crawlspaces below grade*

*= ((1/R\_Old\_BG – 1/(R\_Old\_BG + R\_Added)) \* L\_Basement\_Wall \* H\_Basement\_Wall\_BG \* (1-Framing\_Factor) \* HDD \* 24) / (3412 \* ηHeat)*

*Where:*

*HDD = Heating Degree Days*

*= Dependent on location:[[640]](#footnote-640)*

|  |  |
| --- | --- |
| ***Location*** | ***Heating Degree Days***  ***(60°F set point)*** |
| *Wilmington, DE* | *3,275* |
| *Baltimore, MD* | *3,457* |
| *Washington, DC* | *2,957* |

*3412 = Converts kWh to Btu*

*ηHeat = Efficiency of Heating system, in COP. If not available, use[[641]](#footnote-641):*

|  |  |  |  |
| --- | --- | --- | --- |
| ***System Type*** | ***Age of Equipment*** | ***HSPF Estimate*** | ***COP Estimate*** |
| *Heat Pump* | *Before 2006* | *6.8* | *2.00* |
| *After 2006* | *7.7* | *2.26* |
| *Resistance* | *n/a* | *n/a* | *1.00* |

*R\_Old\_BG = R-Value of Wall below Grade*

*= Dependent on depth of foundation[[642]](#footnote-642)*

|  |  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- | --- |
| ***Depth below grade (ft)*** | ***0*** | ***1*** | ***2*** | ***3*** | ***4*** | ***5*** | ***6*** | ***7*** | ***8*** |
| Earth R-value | 2.44 | 4.5 | 6.3 | 8.4 | 10.44 | 12.66 | 14.49 | 17 | 20 |
| Average Earth R-value | 2.44 | 3.16 | 3.79 | 4.40 | 4.97 | 5.53 | 6.07 | 6.60 | 7.13 |
| Total Below Grade R-value (earth + R-1.0 foundation) default | 3.44 | 4.47 | 5.41 | 6.41 | 7.42 | 8.46 | 9.46 | 10.53 | 11.69 |

*H\_Basement\_Wall\_BG = Height of basement wall below grade*

*AdjBasementheat = Adjustment to account for prescriptive algorithms overclaiming savings*

*= 60%[[643]](#footnote-643)*

*kWhducts* = electric savings from loss of duct leaks, if more than 50% of ducts are in a conditioned area

*= kWhduct\_cool* + kWhduct\_heat

*And:*

*kWhduct\_cool = Hours\_Cool \* Btu/Hour \* (1 / SEER) \* Duct\_Factor / 1000*

*kWhduct\_heat = Hours\_Heat \* Btu/Hour \* (1/HSPF) \* Duct\_Factor / 1,000*

*Where:*

*Hours\_Cool = Full load cooling hours*

*Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | ***Run Hours*** |
| *Wilmington, DE* | *524 [[644]](#footnote-644)* |
| *Baltimore, MD* | *542 [[645]](#footnote-645)* |
| *Washington, DC* | *681* |

*Btu/Hour = Size of equipment in Btu/hour (note 1 ton = 12,000Btu/hour)*

*= Actual installed*

*SEER = Seasonal Efficiency of conditioning equipment*

*= actual installed*

*Duct\_Factor =Factor to account for elimination of duct losses from encapsulation*

*=0.05*

*Hours\_Heat = Full Load Heating Hours*

*= Dependent on location as below:*

|  |  |
| --- | --- |
| ***Location*** | **FLHheat** |
| *Wilmington, DE* | *935[[646]](#footnote-646)* |
| *Baltimore, MD* | *866[[647]](#footnote-647)* |
| *Washington, DC* | *822* |

*HSPF = Heating Seasonal Performance Factorof heating equipment*

*= Actual*

Illustrative examples – do not use as default assumption

*A single family home in Wilmington is getting its crawlspace insulated with R-13 spray foam and encapsulated. The crawlspace currently has an R-value of 2.25, and a significant portion of the home’s ductwork runs through the crawlspae. The house has a 20x25 footprint, and the crawl space walls are 7 feet tall, 3 of which are above grade.The HVAC unit is a heat pump with 13 SEER and 2.26 COP.*

∆kWh = kWhcooling + kWhheating + kWhducts

kWhcooling = ((1/2.25 – 1/ (2.25 +13)) \* (20\*2 + 25\*2) \* 3 \* (1-0) \* 7514 \* 0.75) / (1,000 \* 13) \* 0.8

= 35 kWh

kWhheating = ([((1/2.25 – 1/(2.25+13)) \* (20\*2 + 25\*2) \*3 \* (1-0) \* 3275 \* 24 ) / (3412 \* 2.26) ] + [ ((1/(6.42+2.25) – 1/(6.42 + 2.25 + 13)) \* (20\*2+25\*2) \* 4 \* (1-0) \* 3275 \* 24) / (3412 \* 2.26) ]) \* 0*.6*

= 722 kWh

kWhducts = 524 \* 36,000 \* (1/13) \* 0.05 / 1000 + 935 \* 36,000 \* (1/8) \* 0.05 / 1,000

= 283 kWh

∆kWh = 35 + 722 + 283

= 1,040 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = kWhcooling / Hours\_Cool \* CF

Where:

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.69 [[648]](#footnote-648)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.66 [[649]](#footnote-649)*

Illustrative examples – do not use as default assumption

For the house described above:

ΔkW = 35 / 524 \* 0.69

= 0.046 kW

**Annual Fossil Fuel Savings Algorithm**

If Natural Gas heating:

Δtherms *= (thermsAG + thermsBG) \* AdjBasement +thermsduct*

*Where:*

*thermsAG = Savings from insulation on walls or crawlspaces above grade*

*=((1/R\_Old\_AG – 1/(R\_Old\_AG + R\_Added)) \* L\_Basement\_Wall \* H\_Basement\_Wall\_AG \* (1-Framing\_Factor) \* HDD \* 24) / (100,067 \* ηHeat)*

*thermsBG = Savings from insulation on walls or crawlspaces below grade*

*= ((1/R\_Old\_BG – 1/(R\_Old\_BG + R\_Added)) \* L\_Basement\_Wall \* H\_Basement\_Wall\_BG \* (1-Framing\_Factor) \* HDD \* 24) / (100,067 \* ηHeat)*

*thermsduct = Hours\_Heat \* Btu/Hour \* AFUE \* Duct\_Factor / 100,000*

*Where:*

*Hours\_heat = Equivalent Full Load Heating Hours*

|  |  |
| --- | --- |
| ***Location*** | ***EFLH*** |
| *Wilmington, DE* | *848[[650]](#footnote-650)* |
| *Baltimore, MD* | *620[[651]](#footnote-651)* |
| *Washington, DC* | *528*[[652]](#footnote-652) |

*ηHeat = Efficiency of Heating equipment (equipment efficiency \* distribution efficiency)*

*= actual[[653]](#footnote-653). If not available use 84% for equipment efficiency and 78% for distribution efficiency to give 66%[[654]](#footnote-654).*

*Other factors as defined above*

Illustrative examples – do not use as default assumption

For the house described above, but with a central furnace:

Δtherms *= (thermsAG + thermsBG) \* AdjBasement +thermsduct*

*thermsAG = ((1/2.25 – 1/(2.25+13)) \* (20\*2+25\*2) \* 3 \* (1-0) \* 3275 \* 24 ) / (100,067 \* 0.66)*

*= 122 therms*

*thermsBG = ((1/(2.25+6.42)-1/(2.25+6.42+13)) \* (20\*2+25\*2) \* 4 \* (1-0) \* 3275 \* 24 ) / (100,067 \* 0.66)*

*= 30 therms*

*thermsduct = 848 \* 100,000 \* .84 \* 0.05 / 100,000*

*= 36 therms*

Δtherms = (122 + 30) \*0.6 + 36

= 127

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost of this measure is to be calculated on a site specific basis

**Measure Life**

The expected measure life is assumed to be 25 years.[[655]](#footnote-655)

**Operation and Maintenance Impacts**

n/a

*Pool Pump End Use*

**Pool pump-two speed**

**Unique Measure Code: RS\_PP\_TOS\_PPTWO\_0711**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes the purchase of a two speed swimming pool pump capable of running at 50% speed and being run twice as many hours to move the same amount of water through the filter. The measure could be installed in either an existing or new swimming pool. The installation is assumed to occur during a natural time of sale.

**Definition of Baseline Condition**

The baseline condition is a standard efficiency, 1.36 kW electric pump operating 5.18 hours per day.

**Definition of Efficient Condition**

The efficient condition is an identically sized two speed pump operating at 50% speed (50% flow) for 10.36 hours per day.

**Annual Energy Savings Algorithm**

ΔkWh = kWhBase – kWhTwo Speed [[656]](#footnote-656)

*Where:*

*kWhBase = typical consumption of a single speed motor in a cool climate (assumes 100 day pool season)*

*= 707 kWh*

*kWhTwo Speed = typical consumption for an efficient two speed pump motor*

*= 177 kWh*

ΔkWh = 707 – 177

= 530 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (kWBase – kWTwo Speed) \* CF [[657]](#footnote-657)

*Where:*

*kWBase = Connected load of baseline motor*

*= 1.3 kW*

*kWTwo Speed = Connected load of two speed motor*

*= 0.171 kW*

*CFSSP  = Summer System Peak Coincidence Factor for pool pumps (hour ending 5pm on hottest summer weekday)*

*= 0.20[[658]](#footnote-658)*

*CFPJM = PJM Summer Peak Coincidence Factor for pool pumps (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.27[[659]](#footnote-659)*

ΔkW *SSP* = (1.3-0.171) \* 0.20

= 0.23 kW

ΔkW *SSP* = (1.3-0.171) \* 0.27

= 0.31 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $175 for a two speed pool pump motor[[660]](#footnote-660).

**Measure Life**

The measure life is assumed to be 10 yrs[[661]](#footnote-661).

**Operation and Maintenance Impacts**

n/a

**Pool pump-variable speed\***

**Unique Measure Code: RS\_PP\_TOS\_PPVAR\_0711**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes the purchase of a variable speed swimming pool pump capable of running at 40% speed and being run two and a half times as many hours to move the same amount of water through the filter. The measure could be installed in either an existing or new swimming pool. The installation is assumed to occur during a natural time of sale.

**Definition of Baseline Condition**

The baseline condition is a standard efficiency, 1.36 kW electric pump operating 5.18 hours per day.

**Definition of Efficient Condition**

The efficient condition is an identically sized variable speed pump operating at 40% speed (50% flow) for 13 hours per day.

**Annual Energy Savings Algorithm**

ΔkWh = kWhBase – *kWhVariable* Speed [[662]](#footnote-662)

*Where:*

*kWhBase = typical consumption of a single speed motor in a cool climate (assumes 100 day pool season)*

*= 707 kWh*

*kWhVariable Speed = typical consumption for an efficient variable speed pump motor*

*= 113 kWh*

ΔkWh = 707 – 113

= 594 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (kWBase – kWTwo Speed) \* CF [[663]](#footnote-663)

*Where:*

*kWBase = Connected load of baseline motor*

*= 1.3 kW*

*kWTwo Speed = Connected load of variable speed motor*

*= 0.087 kW*

*CFSSP  = Summer System Peak Coincidence Factor for pool pumps (hour ending 5pm on hottest summer weekday)*

*= 0.20[[664]](#footnote-664)*

*CFPJM = PJM Summer Peak Coincidence Factor for pool pumps (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.27[[665]](#footnote-665)*

ΔkW *SSP* = (1.3-0.087) \* 0.20

= 0.24 kW

ΔkW *SSP* = (1.3-0. 087) \* 0.27

= 0.34 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $750 for a variable speed pool pump motor[[666]](#footnote-666).

**Measure Life**

The measure life is assumed to be 10 yrs[[667]](#footnote-667).

**Operation and Maintenance Impacts**

n/a

*Plug Load End Use*

**Advanced Power Strip\***

**Unique Measure Code: RS\_PL\_TOS\_APS\_0711**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes savings associated with the purchase and use of a Current-Sensing Master/Controlled Advanced Power Strip (APS). These multi-plug power strips have the ability to automatically disconnect specific connected loads depending upon the power draw of a control load, also plugged into the strip. Power is disconnected from the switched (controlled) outlets when the control load power draw is reduced below a certain adjustable threshold, thus turning off the appliances plugged into the switched outlets. By disconnecting, the standby load of the controlled devices, the overall load of a centralized group of equipment (i.e. entertainment centers and home office) can be reduced.

This measure characterization provides a single prescriptive savings assumption based on office and entertainment savings from a 2011 NYSERDA Advanced Power Strip Research Report and weightings and in service rates based on EmPower evaluations.

**Definition of Baseline Condition**

The assumed baseline is a standard power strip that does not control any of the connected loads.

**Definition of Efficient Condition**

The efficient case is the use of a Current-Sensing Master/Controlled Advanced Power Strip.

**Annual Energy Savings Algorithm**

ΔkWh = (kWhoffice \* WeightingOffice + kWhEnt \* WeightingEnt) \* ISR

Where:

kWhoffice = Estimated energy savings from using an APS in a home office

= 31.0 kWh[[668]](#footnote-668)

WeightingOffice = Relative penetration of computers

= 41%[[669]](#footnote-669)

kWhEnt = Estimated energy savings from using an APS in a home entertainment system

= 75.1 kWh[[670]](#footnote-670)

WeightingEnt = Relative penetration of televisions

= 59%[[671]](#footnote-671)

ISR = In service rate

= 89%[[672]](#footnote-672)

ΔkWh = (31 \* 41% + 75.1 \* 59%) \* 83.2%

= 47.4 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh / Hours \* CF

*Where:*

*Hours = Annual hours when controlled standby loads are turned off*

*= 6,351[[673]](#footnote-673)*

*CF = Coincidence Factor*

*= 0.8[[674]](#footnote-674)*

ΔkW = (47.4/6,351) \* 0.8

= 0.0060 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $35[[675]](#footnote-675).

**Measure Life**

The measure life is assumed to be 4 years[[676]](#footnote-676).

**Operation and Maintenance Impacts**

n/a

*Retail Products Program*

**ENERGY STAR +50% Soundbar\*\***

**Unique Measure Code(s): TBD**

**Effective Date: TBD**

**End Date: TBD**

**Measure Description**

This measure relates to the upstream promotion of residential soundbar meeting the ENERGY STAR criteria through the Energy Star Retail Products Program. This measure assumes a more stringent requirement than ENERGY STAR Version 3.0.[[677]](#footnote-677)

**Definition of Baseline Condition**

The baseline condition is assumed to be a standard soundbar.

**Definition of Efficient Condition**

The efficient condition is assumed to be an ENERGY STAR +50% soundbar. The more stringent requirement was developed by decreasing the power requirements and increasing the efficiency requirements by 50%.

**Annual Energy Savings Algorithm[[678]](#footnote-678)**

∆kWh = kWhbase – kWheff

*Where:*

*kWhbase = Baseline unit energy consumption*

*= Assumed to be 69 kWh/year[[679]](#footnote-679)*

*kWheff  = Efficient unit energy consumption*

*= Assumed to be 25 kWh/year[[680]](#footnote-680)*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = 0.0005[[681]](#footnote-681)

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is $0[[682]](#footnote-682).

**Measure Life**

The expected measure life is assumed to be 7 years.[[683]](#footnote-683)

**Operation and Maintenance Impacts**

n/a

**Freezer\*\***

**Unique Measure Code(s): TBD**

**Effective Date: TBD**

**End Date: TBD**

**Measure Description**

This measure relates to the upstream promotion of residential freezers meeting the ENERGY STAR criteria through the Energy Star Retail Products Program. In the measure, a freezer meeting the efficiency specifications of ENERGY STAR is installed in place of a model meeting the federal standard (NAECA). Energy usage specifications are defined in the table below (note, AV is the freezer Adjusted Volume and is calculated as 1.73\*Total Volume):[[684]](#footnote-684)

|  |  |  |
| --- | --- | --- |
| **Product**  **Category** | **Federal Baseline Maximum Energy Usage in kWh/year[[685]](#footnote-685)** | **ENERGY STAR Maximum Energy Usage in kWh/year[[686]](#footnote-686)** |
| Upright Freezers | 8.62\*AV+228.3 | 7.76\*AV+205.5 |
| Chest Freezers | 7.29\*AV+107.8 | 6.56\*AV+97.0 |

**Definition of Baseline Condition**

The baseline equipment is assumed to be a model that meets the federal minimum standard for energy efficiency. The standard varies depending on the type of the freezer (chest or upright freezer) and is defined in the table above.

**Definition of Efficient Condition**

The efficient equipment is defined as a freezer meeting the efficiency specifications of ENERGY STAR, as calculated above.

**Annual Energy Savings Algorithm**

ΔkWh = kWhBase- kWhESTAR

*Where:*

*kWhBASE = Baseline kWh consumption per year*

*= As calculated in the table below*

*kWhESTAR = ENERGY STAR kWh consumption per year*

*=As calculated in the table below*

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
| **Product Category[[687]](#footnote-687)** | **Adj. Volume Use** | **kWhBASE** | **kWhESTAR** | **kWh Savings** | **Weighting for unknown configuration** |
| Upright Freezer | 24.4 | 439 | 395 | 43.78 | 36.74% |
| Chest  Freezer | 18.0 | 239 | 215 | 23.97 | 63.26% |
| Weighted Average |  | 313 | 281 | 31.25 | 100% |

If product category is unknown assume weighted average values[[688]](#footnote-688).

**Summer Coincident Peak kW Savings Algorithm**

∆kW*=* (ΔkWh/8760) \* TAF \* LSAF

*Where:*

TAF = Temperature Adjustment Factor

= 1.23 [[689]](#footnote-689)

LSAF = Load Shape Adjustment Factor

= 1.15 [[690]](#footnote-690)

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is $12.14 for an upright freezer and $6.62 for a chest freezer[[691]](#footnote-691).

**Measure Life**

The measure life is assumed to be 11 years[[692]](#footnote-692).

**Operation and Maintenance Impacts**

n/a

**Clothes Dryer\*\***

**Unique Measure Code(s): TBD**

**Effective Date: TBD**

**End Date: TBD**

**Measure Description**

This measure relates to the upstream promotion of residential clothes dryer meeting the ENERGY STAR criteria through the Energy Star Retail Products Program. ENERGY STAR qualified clothes dryers save energy through a combination of more efficient drying and reduced runtime of the drying cycle. More efficient drying is achieved through increased insulation, modifying operating conditions such as air flow and/or heat input rate, improving air circulation through better drum design or booster fans, and improving efficiency of motors. Reducing the runtime of dryers through automatic termination by temperature and moisture sensors is believed to have the greatest potential for reducing energy use in clothes dryers[[693]](#footnote-693). ENERGY STAR provides criteria for both gas and electric clothes dryers.

**Definition of Baseline Condition**

The baseline condition is a clothes dryer meeting the minimum federal requirements for units manufactured on or after June 1, 2015.

**Definition of Efficient Condition**

Clothes dryer must meet the ENERGY STAR criteria, as required by the program.

**Annual Energy Savings Algorithm**

ΔkWh[[694]](#footnote-694) = kWhBase- kWhESTAR

*Where:*

*kWhBASE = Baseline kWh consumption per year*

*= As presented in the table below*

*kWhESTAR = ENERGY STAR kWh consumption per year*

*=As presented in the table below*

|  |  |  |  |
| --- | --- | --- | --- |
| **Product Category[[695]](#footnote-695)** | **kWhBASE** | **kWhESTAR** | **kWh Savings** |
| Vented Gas Dryer | 42.10 | 34.36 | 7.74 |
| Ventless or Vented Electric Dryer | 768.92 | 608.49 | 160.44 |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/Hours \* CF

*Where:*

*ΔkWh = Energy Savings as calculated above*

*Hours = Annual run hours of clothes dryer.*

*=290 hours per year.[[696]](#footnote-696)*

*CF = Summer Peak Coincidence Factor for measure*

*= 2.9%[[697]](#footnote-697)*

**Annual Fossil Fuel Savings Algorithm**

Natural gas savings only apply to ENERGY STAR vented gas clothes dryers.

ΔMMBtu= MMBtuBase- MMBtuSTAR

*Where:*

*MMBtuBASE = Baseline MMBtu consumption per year*

*= As presented in the table below*

*MMBtuESTAR = ENERGY STAR MMBtu consumption per year*

*=As presented in the table below*

|  |  |  |  |
| --- | --- | --- | --- |
| **Product Category[[698]](#footnote-698)** | **MMBtuBASE** | **MMBtuESTAR** | **MMBtu Savings** |
| Vented Gas Dryer | 2.72 | 2.22 | 0.50 |

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for an ENERGY STAR clothes dryer is assumed to be $270.16 for a vented gas dryer and $224.91 for a ventless or vented electric, standard clothes dryer.[[699]](#footnote-699)

**Measure Life**

The expected measure life is assumed to be 12 years[[700]](#footnote-700).

**Operation and Maintenance Impacts**

n/a

**ENERGY STAR Air Cleaner\*\***

**Unique Measure Code(s): TBD**

**Effective Date: TBD**

**End Date: TBD**

**Measure Description**

An air cleaner is a portable electric appliance that removes dust and fine particles from indoor air. This measure characterizes the purchase and installation of a unit meeting the efficiency specifications of ENERGY STAR in place of a baseline model.

**Definition of Baseline Condition**

The baseline equipment is assumed to be a standard non-ENERGY STAR unit.

**Definition of Efficient Condition**

The efficient equipment is defined as an air cleaner meeting the efficiency specifications of ENERGY STAR as provided below[[701]](#footnote-701).

* Clean Air Delivery Rate (CADR)/Watt Requirement: Must be equal to or greater than 2.0 CADR/Watt (Dust).
* UL Safety Requirements for Ozone Emitting Models: Measured ozone shall not exceed 50 parts per billion.
* Standby Power Requirements: Measured standby power shall not exceed 2 Watts.

**Annual Energy Savings Algorithm**

ΔkWh[[702]](#footnote-702) = kWhBase- kWhESTAR

Where:

kWhBASE *=* Baseline kWh consumption per year

= see table below

kWhESTAR *=* ENERGY STAR kWh consumption per year

= see table below

|  |  |  |
| --- | --- | --- |
| **kWhBASE** | **kWhESTAR** | **kWh Savings** |
| 530.98 | 317.10 | 213.88 |

**Summer Coincident Peak kW Savings Algorithm**

∆kW*=* ∆kWh/Hours \* CF

Where:

∆kWh = Gross customer annual kWh savings for the measure

Hours = Average hours of use per year

= 5840 hours[[703]](#footnote-703)

CF = Summer Peak Coincidence Factor for measure

= 0.67[[704]](#footnote-704)

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is $56.[[705]](#footnote-705)

**Measure Life**

The measure life is assumed to be 9 years[[706]](#footnote-706).

**Operation and Maintenance Impacts**

There are no operation and maintenance cost adjustments for this measure.[[707]](#footnote-707)

**Room Air Conditioners (Upstream)\*\***

**Unique Measure Code(s): TBD**

**Effective Date: TBD**

**End Date: TBD**

**Measure Description**

This measure relates to the purchase (time of sale) and installation of a room air conditioning unit that meets the ENERGY STAR minimum qualifying efficiency specifications presented below:

|  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- |
| **Product Type and Class (Btu/hour)** | | **Federal Standard with louvered sides**  **(EER)** | **Federal Standard without louvered sides**  **(EER)** | **ENERGY STAR with louvered sides (EER)** | **ENERGY STAR without louvered sides (EER)** |
| Without Reverse Cycle | < 6,000 | 11.0 | 10.0 | 12.1 | 11.0 |
| 6,000 to 7,999 | 11.0 | 10.0 | 12.1 | 11.0 |
| 8,000 to 13,999 | 10.9 | 9.6 | 12.0 | 10.6 |
| 14,000 to 19,999 | 10.7 | 9.5 | 12.0 | 10.5 |
| 20,000 to 24,999 | 9.4 | 9.3 | 10.3 | 10.2 |
| >=25,000 | 9.0 | 9.4 | 9.9 | 10.3 |
| With Reverse Cycle | <14,000 | n/a | 9.3 | n/a | 10.2 |
| >=14,000 | n/a | 8.7 | n/a | 9.6 |
| <20,000 | 9.8 | n/a | 10.8 | n/a |
| >=20,000 | 9.3 | n/a | 10.2 | n/a |
| Casement only | | 9.5 | | 10.5 | |
| Casement-Slider | | 10.4 | | 11.4 | |

**Definition of Baseline Condition**

The baseline condition is a window AC unit that meets the minimum federal efficiency standards as of June 1, 2014 presented above. [[708]](#footnote-708)

**Definition of Efficient Condition**

The baseline condition is a window AC unit that meets the ENERGY STAR v4.0 as of October 26, 2015 presented above.[[709]](#footnote-709)

**Annual Energy Savings Algorithm**

ΔkWh[[710]](#footnote-710) = kWhBase- kWhESTAR

Where:

kWhBASE *=* Baseline kWh consumption per year

= see table below for calculated values

kWhESTAR *=* ENERGY STAR kWh consumption per year

= see table below for calculated values

|  |  |  |
| --- | --- | --- |
| **Location** | **Full-Load Cooling Hours** | **Savings (kWh/year)** |
| Wlimington, DE | 1,015 | 74.72 |
| Baltimore, MD | 1,050 | 77.30 |
| Washington, DC | 1,320 | 97.18 |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = Btu/hour \* (1/EERbase - 1/EERee))/1000 \* CF

*Where:*

*CF = Summer Peak Coincidence Factor for measure*

*CFSSP  = Summer System Peak Coincidence Factor for Central A/C (hour ending 5pm on hottest summer weekday)*

*= 0.31 [[711]](#footnote-711)*

*CFPJM  = PJM Summer Peak Coincidence Factor for Central A/C (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.3[[712]](#footnote-712)*

Using deemed values above:

ΔkWENERGY STAR SSP

= (8500 \* (1/10.9 – 1/11.3)) / 1000 \* 0.31

= 0.009 kW

ΔkWCEE TIER 1 SSP

= (8500 \* (1/10.9 – 1/11.8)) / 1000 \* 0.31

= 0.018 kW

ΔkWENERGY STAR PJM

= (8500 \* (1/10.9 – 1/11.3)) / 1000 \* 0.30

= 0.008 kW

ΔkWCEE TIER 1 PJM

= (8500 \* (1/10.9 – 1/11.8)) / 1000 \* 0.30

= 0.018 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $114.45 for an ENERGY STAR unit.[[713]](#footnote-713)

**Measure Life**

The measure life is assumed to be 9 years.[[714]](#footnote-714)

**Operation and Maintenance Impacts**

n/a

**COMMERCIAL & INDUSTRIAL MARKET SECTOR**

*Lighting End Use*

**General Purpose CFL Screw base, Retail – Commercial\***

**Unique Measure Code(s): CI\_LT\_TOS\_CFLSCR\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure characterizes the installation of a general purpose compact fluorescent light bulb (CFL) in place of an incandescent bulb. The measure provides assumptions based on a Time of Sale implementation strategy. Direct Install assumptions are presented with the residential characterization. This characterization is for a general purpose medium screw based CFL bulb (A-lamps/twists/spirals), and not a specialty bulb (e.g., reflector (PAR) lamp, globes <= 40 watts, candelabras <= 40 watts, 3-ways, etc.).

**Definition of Baseline Condition**

The baseline is the installation of a halogen incandescent light bulb meeting the standards described in the Energy Independence and Security Act of 2007.[[715]](#footnote-715)

**Definition of Efficient Condition**

The efficient condition is the installation of a compact fluorescent light bulb.

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) /1000) \* HOURS \* ISR \* WHFe

*Where:*

WattsBASE *= Based on lumens of CFL bulb*[[716]](#footnote-716):

For non-decorative bulbs:

| **Minimum Lumens** | **Maximum Lumens** | **WattsBase** |
| --- | --- | --- |
| 4000 | 6000 | 300 |
| 3001 | 3999 | 200 |
| 2601 | 3000 | 150 |
| 1490 | 2600 | 72 |
| 1050 | 1489 | 53 |
| 750 | 1049 | 43 |
| 310 | 749 | 29 |
| 250 | 309 | 25 |

For decorative bulbs and non-G40 globes greater than 40 watts:

|  |  |  |  |
| --- | --- | --- | --- |
|  | **Minimum Lumens** | **Maximum Lumens** | **WattsBASE** |
| **Decorative** | 500 | 699 | 43 |
| **Non-G40 globe** | 500 | 574 | 43 |
| 575 | 649 | 53 |
| 650 | 1099 | 72 |
| 1100 | 1300 | 150 |

WattsEE *=* *Actual wattage of CFL purchased / installed*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[717]](#footnote-717)*

*ISR = In Service Rate or percentage of units rebated that are installed and operational*

*= 1.00 [[718]](#footnote-718)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

Illustrative examples – do not use as default assumption

For example, a 19W, 1,200 lumen CFL is purchased and installed in a conditoned office building with gas heat in BGE service territory:

ΔkWh = ((53 - 19) / 1000) \* 2,969 \* 1.00 \* 1.10

= 111 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) / 1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, a 19W, 1,200 lumen CFL is purchased and installed in a conditoned office building with gas heat in BGE service territory and estimating PJM summer peak coincidence:

ΔkW = ((53 - 19) / 1000) \* 1.00 \* 1.32 \* 0.69

= 0.03 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= (-ΔkWh / WHFe) \* 0.00073

*Where:*

*0.7 = Aspect ratio [[719]](#footnote-719)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[720]](#footnote-720)*

*0.75 = Assumed heating system efficiency [[721]](#footnote-721)*

Illustrative examples – do not use as default assumption

For example, assuming a 19W CFL is purchased and installed in a conditoned office building with gas heat in BGE service territory:

ΔMMBTU = (-111 / 1.10) \* 0.00073

= -0.07 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

For the Retail (Time of Sale) measure, the incremental capital cost is $1.80.[[722]](#footnote-722)

**Measure Life**

The measure life by building type is presented in the table below.[[723]](#footnote-723)

| **Building Type** | **Measure Life (Years)** |
| --- | --- |
| Grocery | 1.4 |
| Health | 2.6 |
| Office | 3.4 |
| Other | 2.2 |
| Retail | 2.0 |
| School | 3.9 |
| Warehouse/ Industrial | 2.4 |
| Unknown | 2.6 |

**Operation and Maintenance Impacts**

For convenience, the levelized baseline replacement cost over the lifetime of the CFL is presented below (see MidAtlantic Lighting Adjustments and O&M\_042015.xls). The key assumptions used in this calculation are documented below:

| **Attribute** | **Halogen Incandescent** |
| --- | --- |
| Replacement Lamp Cost | $1.40[[724]](#footnote-724) |
| Replacement Labor Cost | $1.54[[725]](#footnote-725) |
| Component Life (Hours) | 1,000[[726]](#footnote-726) |

The calculated net present value of the baseline replacement costs for CFL type and installation year are presented below[[727]](#footnote-727):

| **Building Type** | **NPV of Baseline Replacement Costs** |
| --- | --- |
| Grocery | $26.18 |
| Health | $25.35 |
| Office | $24.82 |
| Other | $25.76 |
| Retail | $25.76 |
| School | $24.69 |
| Warehouse/ Industrial | $25.63 |
| Unknown | $25.35 |

**Four-Foot Fluorescent T8 Replacement Lamps\***

**Unique Measure Code(s): CI\_LT\_TOS\_HPT8\_0516 and CI\_LT\_RTR\_HPT8\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

This measure promotes the installation of high performance fluorescent T8 4-ft replacement lamps that have higher lumens per watt and longer life than standard 4-ft T8 systems. This results in lamp/ballast systems that produce equal or greater lumens than standard T8 systems, while using fewer watts. The Consortium for Energy Efficiency (CEE) maintains specifications and a list for qualifying high performance 4-ft T8 replacement lamps. The list is updated frequently and is available at: <http://library.cee1.org/content/commercial-lighting-qualifying-products-lists>.

In November 2014, federal minimum standards for ballasts increased to meet CEE performance levels for HPT8 systems. In response, in January 2015, CEE published an updated Commercial Lighting Systems Initiative that discontinued the qualified ballast list and transitioned the T8 lamp specification to solely a replacement lamp strategy.[[728]](#footnote-728) This new strategy is not technology dependent; both conventional fluorescent lamps and LED linear fluorescent replacement lamps (TLEDs) are qualified under the new requirements. CEE no longer maintains a list of ballasts that meet the previous HPT8 ballast specifications because all ballasts manufactured after November 2014 meet those requirements; however, an archived list of qualifying ballasts can be viewed at the aforementioned website.

**Definition of Baseline Condition**

The baseline condition is assumed to be the existing lighting fixture in retrofit applications.[[729]](#footnote-729) For time of sale applications, the baseline condition will vary depending upon the specific characteristics of the fixtures installed (e.g. number of lamps), federal minimum standards (e.g., based on correlated color temperature), and applicable building energy codes. For illustrative purposes, the following examples are provided but they are *not to be used as default assumptions*:

Illustrative examples – do not use as default assumption

Time of Sale: A 3-lamp standard performance 4-ft F32 800-series, 18,000 hour T8 fixture with normal output electronic ballast compliant with 2014 federal standards with a fixture input wattage of 89W.[[730]](#footnote-730)

T12 Retrofit implemented June 30, 2017 and earlier: A 3-lamp 4-ft F34 T12 fixture with magnetic ballast with an input wattage of 122W.

T12 Retrofit implemented July 1, 2017 and later: Due to the federal standards change described in the Measure Life section below, an existing 3-lamp 4-ft F34 T12 fixture with magnetic ballast would be treated like a standard performance 4-ft F32 800-series T8 fixture with normal output electronic ballast compliant with 2014 federal standards with a fixture input wattage of 89W, which is about a 65% reduction in baseline wattage.

**Definition of Efficient Condition**

The efficient conditions for the time of sale and retrofit applications are a qualifying high performance T8 fixture and lamp/ballast combination, respectively. For illustrative purposes the following high efficiency conditions for the corresponding baselines are described:

Illustrative examples – do not use as default assumption

Time of Sale: A 3-lamp CEE Qualified Replacement Lamp T8 fixture with electronic, normal output type ballast compliant with 2014 federal appliance standards with a fixture input wattage of 72W.

Any Retrofit: Relamp and reballast with 3 CEE Qualified T8 Replacement Lamps and electronic, normal output type ballast compliant with 2014 federal standards with a fixture input wattage of 72W.

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE – WattsEE) / 1000) \* HOURS \* ISR \* WHFe

*Where:*

*WattsBASE = Connected load of baseline fixture (for “Time of Sale” measures)*

*Or = Connected load of existing fixture (for “Retrofit” measures)*

*WattsEE = Connected load of efficient fixture*

*HOURS = Average hours of use per year, no change in control type.*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[731]](#footnote-731)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[732]](#footnote-732)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

Illustrative examples – do not use as default assumption

For example, assuming installation in a conditoned (cooled and heated) large office building with gas heat in BGE service territory, the per fixture savings are:

Time of Sale:

ΔkWh = ((89 - 72) / 1000) \* 2,969 \* 1.00 \* 1.10

= 56 kWh per fixture for 15 years.

T12 Retrofit implemented June 30, 2017 and earlier:

ΔkWh = ((122 - 72) / 1000) \* 2,969 \* 1.00 \* 1.10

= 163 kWh per fixture for 5.2 years[[733]](#footnote-733)

T12 Retrofit implemented July 1, 2017 and later:

ΔkWh = ((89 - 72) / 1000) \* 2,969 \* 1.00 \* 1.10

= 56 kWh per fixture for 15 years.

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE – WattsEE) / 1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, assuming installation in a conditoned (cooled and heated) large office building with gas heat in BGE service territory and estimating PJM summer peak demand:

Time of Sale:

ΔkW = ((89 - 72) / 1000) \* 1.00 \* 1.32 \* 0.69

= 0.015 kW per fixture for 15 years

T12 Retrofit (June 30, 2017 and earlier):

ΔkW = ((122 - 72) / 1000) \* 1.00 \* 1.32 \* 0.69

= 0.046 kW per fixture for 5.2 years

T12 Retrofit (July 1, 2017 and later):

ΔkW = ((89 - 72) / 1000) \* 1.00 \* 1.32 \* 0.69

= 0.015 kW per fixture for 15 years

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= -ΔkWh \* 0.00065

*Where:*

*0.7 = Aspect ratio [[734]](#footnote-734)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[735]](#footnote-735)*

*0.75 = Assumed heating system efficiency [[736]](#footnote-736)*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

Incremental costs will vary by specific equipment installed. The incremental costs for the example measures are assumed to be $25 for time of sale and $60 for retrofit.[[737]](#footnote-737)

**Measure Life**

The measure life is assumed to be 15 years for “Time of Sale” measures.[[738]](#footnote-738) For “Retrofit” measures replacing existing, federal standards compliant T12 lamps, see the adjusted effective useful life in the table below. Otherwise, 15 years should be used provided that the existing equipment will be available for the assumed measure life (i.e., there will be no “baseline shift”).

**Measure Life for Retrofit Measures with T12 Baseline[[739]](#footnote-739)**

|  |  |  |
| --- | --- | --- |
| **Year** | **July 1, 2016 to June 30, 2017** | **July 1, 2017 and beyond 2017** |
| **Measure Life** | 5.2 | No T12 baseline replaced by Standard 800-series T8 |

**T12 Baseline Phase-out, Adjusted Measure Life**

On June 26, 2009, the U.S. Department of Energy issued a final rule establishing new energy conservation standards for general service fluorescent lamps. These standards cover the most common types of linear fluorescent lamps including all 4-foot T12 and T8 lamps. Beginning July 14, 2012, the manufacture of T12 linear fluorescent lamps and the lowest efficiency 700-series T8 lamps was largely banned, leaving 800-series standard T8 lamps as the default. Some manufacturers have continued to produce an exempted type of T12 lamp with greater than 87 CRI.[[740]](#footnote-740) While it was once thought that the shortage and high cost of the rare earth metals necessary to meet the CRI exemption would make the exempted T12 lamps cost-prohibitive for ongoing replacements and drive more users to upgrade to T8 systems, costs for the exempted T12s have since dropped to levels comparable to 800-series T8s. Therefore, the measure lives in the table above are only intended to apply to existing T12 lamp/ballast systems that are not exempt from the current federal standards (e.g., those lamps with CRI less than 87).

If a customer relamped an existing fixture with T12s the day the standard took effect, an assumption can be made that they would likely need to upgrade to, at a minimum, 800-series T8s in less than 6 years’ time. This assumes the T12s installed have a typical rated life of 20,000 hours and are operated for 3,500 hours annually. Certainly, it is not realistic that everyone would wait until the final moment to relamp with T12s. Therefore the more likely scenario would be a gradual shift in the baseline to 800-series T8s over the 6 year timeframe. To simplify this assumption, it is recommended that the assumed measure life be gradually reduced between 2012 and 2017. *Note: Adjusted measures lives take into account the savings that would result over the duration of the unadjusted measure life relative to new baseline 800-series T8 fixtures once T12s are no longer available.*

**T12 Baseline Replaced by Standard T8.**

As illustrated in previous examples, this means that after June 30, 2017, an existing 3‑lamp 4-ft F34 T12 fixture with magnetic ballast (122 W) would be treated like a standard performance 4-ft F32 800-series T8 fixture with normal output electronic ballast compliant with 2014 federal standards with a fixture input wattage of 89W. The T12 fixture may remain in a standard fixture wattage tables used by program implementers, but the assumed fixture wattage for the T12 fixture would need to be reduced from 122W to 89W. The fixture description should also be changed accordingly to indicate the adjusted wattage.

**Operation and Maintenance Impacts**

Due to differences in costs and lifetimes of replacement lamps and ballasts between the efficient and baseline cases, there are significant operation and maintenance impacts associated with this measure. Actual operation and maintenance costs will vary by specific equipment installed/replaced. For the selected examples presented in the “Definition of Baseline Condition” and “Definition of Efficient Condition” sections:[[741]](#footnote-741)

Illustrative examples – do not use as default assumption

**Retrofit[[742]](#footnote-742)**

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
|  | **Baseline Linear Fluorescent (Standard 800-series T8)** | | **Efficient Linear Fluorescent (High Performance T8)** | |
|  | **Lamp (each)** | **Ballast** | **Lamp (each)** | **Ballast** |
| Replacement Cost | $5.17 | $35 | $7.67 | $47.50 |
| Component Life[[743]](#footnote-743) (years) | 5.71[[744]](#footnote-744) | 20[[745]](#footnote-745) | 8.57[[746]](#footnote-746) | 20[[747]](#footnote-747) |

**Time of Sale**

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
|  | **Baseline Linear Fluorescent (Standard 800-series T8)** | | **Efficient Linear Fluorescent (High Performance T8)** | |
|  | **Lamp (each)** | **Ballast** | **Lamp (each)** | **Ballast** |
| Replacement Cost | $5.17 | $47.50 | $7.67 | $47.50 |
| Component Life[[748]](#footnote-748) (years) | 5.71[[749]](#footnote-749) | 20[[750]](#footnote-750) | 8.57[[751]](#footnote-751) | 20[[752]](#footnote-752) |

The calculated net present value of the net replacement costs by market are presented below**[[753]](#footnote-753)**:

|  |  |
| --- | --- |
| **Application** | **NPV of Net Replacement Costs** |
| **2016** |
| Retrofit | $52.08 |
| Time of Sale | $5.65 |

**T5 Lighting**

**Unique Measure Code(s): CI\_LT\_TOS\_T5\_0614 and CI\_LT\_RTR\_T5\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes the installation of high-bay T5 lamp/ballast systems.

**Definition of Baseline Condition**

The baseline condition is a metal-halide fixture.

**Definition of Efficient Condition**

The efficient condition is a four Lamp T5 High Output fixture.

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) / 1000) \* HOURS \* ISR \* WHFe

*Where:*

*WattsBASE = Actual Connected load of baseline fixture*

*WattsEE = Actual Connected load of T5 fixture*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[754]](#footnote-754)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[755]](#footnote-755)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

Illustrative examples – do not use as default assumption

For example, a 240W T5 fixture installed in place of a 455W metal-halide in a conditoned warehouse with gas heat in BGE service territory in 2014:

ΔkWh = ((455 - 240) / 1000) \* 4,116 \* 1.00 \* 1.02

= 902.6 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) /1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, a 240W T5 fixture installed in place of a 455W metal-halide in a warehouse and estimating PJM summer peak coincidence:

ΔkW = ((455 - 240) / 1000) \* 1.00 \* 1.24 \* 0.72

= 0.19 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= -ΔkWh \* 0.00065

*Where:*

*0.7 = Aspect ratio [[756]](#footnote-756)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[757]](#footnote-757)*

*0.75 = Assumed heating system efficiency [[758]](#footnote-758)*

Illustrative examples – do not use as default assumption

For example, a 240W T5 fixture installed in place of a 455W metal-halide in a conditoned warehouse with gas heat in 2014:

ΔMMBTU = -902.6 \* 0.00065

= -0.59 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $300.[[759]](#footnote-759)

**Measure Life**

The measure life is assumed to be 15 years.[[760]](#footnote-760)

**Operation and Maintenance Impacts**

n/a

**LED Exit Sign**

**Unique Measure Code(s): CI\_LT\_RTR\_LEDEXI\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of an exit sign illuminated with light emitting diodes (LED). This measure should be limited to retrofit installations.

**Definition of Baseline Condition**

The baseline condition is an exit sign with a non-LED light-source.

**Definition of Efficient Condition**

The efficient condition is an exit sign illuminated with light emitting diodes (LED).

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) / 1000) \* HOURS \* ISR \* WHFe

*Where:*

*WattsBASE = Actual Connected load of existing exit sign. If connected load of existing exit sign is unknown, assume 16 W.[[761]](#footnote-761)*

*WattsEE = Actual Connected load of LED exit sign*

*HOURS = Average hours of use per year*

*= 8,760 [[762]](#footnote-762)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[763]](#footnote-763)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

Illustrative examples – do not use as default assumption

For example a 5W LED lamp in place of a 16W CFL in a conditoned office building with gas heat in BGE service territory in 2014:

ΔkWh = ((16 - 5) / 1000) \* 8,760 \* 1.00 \* 1.10

= 106.0 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (WattsBASE - WattsEE) / 1000 \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= 1.0 [[764]](#footnote-764)*

Illustrative examples – do not use as default assumption

For example, a 5W LED lamp in place of a 16W CFL installed in a conditoned office building with gas heat in BGE service territory in 2014 and estimating PJM summer peak coincidence:

ΔkW = ((16 - 5) / 1000) \* 1.00 \* 1.32 \* 1.0

= 0.015 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= -ΔkWh \* 0.00065

*Where:*

*0.7 = Aspect ratio [[765]](#footnote-765)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[766]](#footnote-766)*

*0.75 = Assumed heating system efficiency [[767]](#footnote-767)*

Illustrative examples – do not use as default assumption

For example, a 5W LED lamp in place of a 16W CFL installed in a conditoned office building with gas heat in BGE service territory in 2014:

ΔMMBTU = -106 \* 0.00065

= -0.069 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $35.[[768]](#footnote-768)

**Measure Life**

The measure life is assumed to be 7 years.[[769]](#footnote-769)

**Operation and Maintenance Impacts**

|  |  |
| --- | --- |
|  | **Baseline** |
|  | **CFL** |
| Replacement Cost | $12[[770]](#footnote-770) |
| Component Life (years) | 1.14[[771]](#footnote-771) |

The calculated net present value of the baseline replacement costs are presented below**[[772]](#footnote-772)**:

|  |  |
| --- | --- |
| **Baseline** | **NPV of Baseline Replacement Costs** |
| **2014** |
| CFL | $62.59 |

**Solid State Lighting (LED) Recessed Downlight Luminaire**

**Unique Measure Code: CI\_LT\_TOS\_SSLDWN\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure describes savings from the purchase and installation of a Solid State Lighting (LED) Recessed Downlight luminaire in place of an incandescent downlight lamp (i.e. time of sale). The SSL downlight should meet the ENERGY STAR Luminaires Version 2.0 specification[[773]](#footnote-773). The characterization of this measure should not be applied to other types of LEDs.

Note, this measure assumes the baseline is a Bulged Reflector (BR) lamp. This lamp type is generally the cheapest and holds by far the largest market share for this fixture type. They currently are *not* subject to EISA regulations and so this characterization does not include the baseline shift provided in other lighting measures.

**Definition of Baseline Condition**

The baseline is the purchase and installation of a standard BR30-type incandescent downlight light bulb.

**Definition of Efficient Condition**

The efficient condition is the purchase and installation of an ENERGY STAR Solid State Lighting (LED) Recessed Downlight luminaire.

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) / 1,000) \* ISR \* HOURS \* WHFe

*Where:*

*WattsBASE = Connected load of baseline lamp*

*= Actual if retrofit, if LED lumens is known – find the equivalent baseline wattage from the table below[[774]](#footnote-774), if unknown assume 65W[[775]](#footnote-775)*

| **Bulb Type** | **Lower Lumen Range** | **Upper Lumen Range** | **WattsBase** |
| --- | --- | --- | --- |
| **Reflector with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649 | 50 |
| 650 | 1199 | 65 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter >2.5" (\*see exceptions below)** | 640 | 739 | 40 |
| 740 | 849 | 45 |
| 850 | 1179 | 50 |
| 1180 | 1419 | 65 |
| 1420 | 1789 | 75 |
| 1790 | 2049 | 90 |
| 2050 | 2579 | 100 |
| 2580 | 3429 | 120 |
| 3430 | 4270 | 150 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter > 2.26'' and ≤ 2.5" (\*see exceptions below)** | 540 | 629 | 40 |
| 630 | 719 | 45 |
| 720 | 999 | 50 |
| 1000 | 1199 | 65 |
| 1200 | 1519 | 75 |
| 1520 | 1729 | 90 |
| 1730 | 2189 | 100 |
| 2190 | 2899 | 120 |
| 2900 | 3850 | 150 |
| **\*ER30, BR30, BR40, or ER40** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649-1179[[776]](#footnote-776) | 50 |
| **\*BR30, BR40, or ER40** | 650 | 1419 | 65 |
| **\*R20** | 400 | 449 | 40 |
| 450 | 719 | 45 |
| **\*All reflector lamps**  **below lumen ranges specified above** | 200 | 299 | 20 |
| 300 | 399-639[[777]](#footnote-777) | 30 |

*WattsEE = Connected load of efficient lamp*

*= Actual. If unknown assume 9.2W [[778]](#footnote-778)*

*ISR = In Service Rate or percentage of units rebated that get installed.*

*= 1.0[[779]](#footnote-779)*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[780]](#footnote-780)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBase - WattsEE) /1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= (-ΔkWh / WFHe) \* 0.00073

*Where:*

*0.7 = Aspect ratio [[781]](#footnote-781)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[782]](#footnote-782)*

*0.75 = Assumed heating system efficiency [[783]](#footnote-783)*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

If the implementation strategy allows the collection of actual costs, or an appropriate average, then that should be used. If not, the incremental cost for this measure is assumed to be $36[[784]](#footnote-784).

**Measure Life**

The measure life is assumed to be 14.2 years for downlights featuring inseparable components and 7.1 years for downlights with replaceable parts [[785]](#footnote-785).

**Operation and Maintenance Impacts**

The levelized baseline replacement cost over the lifetime of the SSL is calculated (see ‘MidAtlantic Lighting Adjustments and O&M\_041816.xls’). The key assumptions used in this calculation are documented below:

|  |  |
| --- | --- |
|  | **BR-type Incandescent** |
| Replacement Lamp Cost | $4.00 |
| Replacement Labor Cost | $2.56[[786]](#footnote-786) |
| Component Life (years) | 0.57[[787]](#footnote-787) |

The calculated net present value of the baseline replacement costs is $116 for downlights featuring inseparable components and $67 for downlights with replaceable parts.[[788]](#footnote-788)

**Delamping**

**Unique Measure Code(s): CI\_LT\_ERT\_DELAMP\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the permanent removal of a lamp and the associated electrical sockets (or “tombstones”) from a fixture.

**Definition of Baseline Condition**

The baseline conditions will vary dependent upon the characteristics of the existing fixture. For illustrative purposes, a baseline three lamp 4ft T8 Fixture with input wattage of 89W is assumed.

**Definition of Efficient Condition**

The efficient condition will vary depending on the existing fixture and the number of lamps removed. For illustrative purposes, a two lamp 4ft T8 Fixture on a three lamp ballast (67W) is assumed.

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) / 1000) \* HOURS \* WHFe

*Where:*

*WattsBASE = Actual Connected load of baseline fixture*

*WattsEE = Actual Connected load of delamped fixture*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[789]](#footnote-789)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) /1000) \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, one lamp of a three lamp 4ft T8 Fixture (89W) is removed (leaving 67W) in a conditoned office building with gas heat in BGE service territory in 2014 and estimating PJM summer peak coincidence:

ΔkW = ((89 - 67) / 1000) \* 1.32 \* 0.69

= 0.020 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= -ΔkWh \* 0.00065

*Where:*

*0.7 = Aspect ratio [[790]](#footnote-790)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[791]](#footnote-791)*

*0.75 = Assumed heating system efficiency [[792]](#footnote-792)*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $10.8 per fixture.[[793]](#footnote-793)

**Measure Life**

The measure life is assumed to be 15 years.[[794]](#footnote-794)

**Operation and Maintenance Impacts**

Delamping reduces the number of periodic lamp replacements required, saving $1.25/year.

**Occupancy Sensor – Wall-, Fixture-, or Remote-Mounted**

**Unique Measure Code(s): CI\_LT\_TOS\_OSWALL\_0614, CI\_LT\_TOS\_OSFIX/REM\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure defines the savings associated with installing a wall-, fixture, or remote-mounted occupancy sensor that switches lights off after a brief delay when it does not detect occupancy.

**Definition of Baseline Condition**

The baseline condition is lighting that is controlled with a manual switch.

**Definition of Efficient Condition**

The efficient condition is lighting that is controlled with an occupancy sensor.

**Annual Energy Savings Algorithm**

ΔkWh = kWconnected \* HOURS \* SVGe \* ISR \* WHFe

*Where:*

*kWconnected = Assumed kW lighting load connected to control.*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[795]](#footnote-795)*

*SVGe = Percentage of annual lighting energy saved by lighting control; determined on a site-specific basis or using default below.*

*= 0.28 [[796]](#footnote-796)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[797]](#footnote-797)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = kWconnected \* SVGd \* ISR \* WHFd \* CF

*Where:*

*SVGd = Percentage of lighting demand saved by lighting control; determined on a site-specific basis or using default below.*

*= 0.14 [[798]](#footnote-798)*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, a 400W connected load being controlled in a conditoned office building with gas heat in BGE service territory in 2014 and estimating PJM summer peak coincidence:

ΔkW = 0.4 \* 0.14 \* 1.00 \* 1.32 \* 0.69

= 0.051 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= -ΔkWh \* 0.00065

*Where:*

*0.7 = Aspect ratio [[799]](#footnote-799)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[800]](#footnote-800)*

*0.75 = Assumed heating system efficiency [[801]](#footnote-801)*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $115 per control for wall occupancy sensors, $200 per control for fixture-mounted and remote-mounted occupancy sensors.[[802]](#footnote-802)

**Measure Life**

The measure life is assumed to be 10 years.[[803]](#footnote-803)

**Operation and Maintenance Impacts**

n/a

**Daylight Dimming Control**

**Unique Measure Code(s): CI\_LT\_TOS\_DDIM\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure defines the savings associated with installing a daylighting dimming control system to reduce electric lighting levels during periods of high natural light. Systems typical include daylight sensors, control electronics, and, if necessary, dimmable ballasts.

**Definition of Baseline Condition**

The baseline condition is lighting that is controlled with a manual switch.

**Definition of Efficient Condition**

The efficient condition is lighting that is controlled with a daylight dimming system capable of continuous dimming to reduce electric lighting to the lowest possible levels during periods of adequate natural light.

**Annual Energy Savings Algorithm**

ΔkWh = kWconnected x HOURS x SVG x ISR x WHFe

*Where:*

*kWconnected = Assumed kW lighting load connected to control.*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[804]](#footnote-804)*

*SVG = Percentage of annual lighting energy saved by lighting control; determined on a site-specific basis or using default below.*

*= 0.28 [[805]](#footnote-805)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[806]](#footnote-806)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

**Summer Coincident Peak kW Savings Algorithm[[807]](#footnote-807)**

ΔkW = kWconnected x SVG x ISR x WHFd x CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, a 400W connected load being controlled in a conditoned office building with gas heat in BGE service territory in 2014 and estimating PJM summer peak coincidence:

ΔkW = 0.4 \* 0.28 \* 1.00 \* 1.32 \* 0.69

= 0.10 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= -ΔkWh \* 0.00065

*Where:*

*0.7 = Aspect ratio [[808]](#footnote-808)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[809]](#footnote-809)*

*0.75 = Assumed heating system efficiency [[810]](#footnote-810)*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $100 per ballast controlled for both fixture-mounted and remote-mounted daylight sensors.[[811]](#footnote-811)

**Measure Life**

The measure life is assumed to be 10 years.[[812]](#footnote-812)

**Operation and Maintenance Impacts**

n/a

**Advanced Lighting Design – Commercial**

**Unique Measure Code(s): CI\_LT\_TOS\_ADVLTNG\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

Advanced lighting design refers to the implementation of various lighting design principles aimed at creating a quality and appropriate lighting experience while reducing unnecessary light usage. This is often done by a professional in a new construction situation. Advanced lighting design uses techniques like maximizing task lighting and efficient fixtures to create a system of optimal energy efficiency and functionality to ultimately reduce the wattage required per square foot while maintaining acceptable lumen levels.

This measure characterization is intended for use in new construction or in existing buildings where significant lighting renovations are taking place and energy code requirements must be met.

**Definition of Baseline Condition**

The baseline condition assumes compliance with lighting power density requirements as mandated by jurisdiction: Maryland Building Performance Standards (2015 International Energy Conservation Code); Title 16, Chapter 76 of the Delaware Code (2012 International Energy Conservation Code); and District of Columbia Construction Codes Supplement of 2013 (2012 International Energy Conservation Code). Because lighting power density requirements differ by jurisdiction, this measure entry presents two different baseline conditions to be used in each of the three relevant jurisdictions. For completeness, the lighting power density requirements for both the Building Area Method and the Space-by-Space Method are presented.[[813]](#footnote-813)

**Definition of Efficient Condition**

The efficient condition assumes lighting systems that achieve lighting power densities below the maximum lighting power densities required by the relevant jurisdictional energy codes as described above. Actual lighting power densities should be determined on a site-specific basis.

**Annual Energy Savings Algorithm[[814]](#footnote-814)**

ΔkWh = ((LPDBASE – LPDEE) / 1000) \* AREA \* HOURS \* WHFe

*Where:*

*LPDBASE = Baseline lighting power density for building or space type (W/ft2). See tables below for values by jurisdiction and method.*

*LPDEE = Efficient lighting power density (W/ft2)*

*= Actual calculated*

*AREA = Building or space area (ft2)*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[815]](#footnote-815)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

**Building Area Method Baseline LPD Requirements by Jurisdiction[[816]](#footnote-816)**

|  | **Lighting Power Density (W/ft2)** | |
| --- | --- | --- |
| **Building Area Type** | **Washington, D.C. and Delaware** | **Maryland** |
| Automotive Facility | 0.90 | 0.80 |
| Convention Center | 1.20 | 1.01 |
| Court House | 1.20 | 1.01 |
| Dining: Bar Lounge/Leisure | 1.30 | 1.01 |
| Dining: Cafeteria/Fast Food | 1.40 | 0.90 |
| Dining: Family | 1.60 | 0.95 |
| Dormitory | 1.00 | 0.57 |
| Exercise Center | 1.00 | 0.84 |
| Fire Station | 0.80 | 0.67 |
| Gymnasium | 1.10 | 0.94 |
| Healthcare-Clinic | 1.00 | 0.90 |
| Hospital | 1.20 | 1.05 |
| Hotel | 1.00 | 0.87 |
| Library | 1.30 | 1.19 |
| Manufacturing Facility | 1.30 | 1.17 |
| Motel | 1.00 | 0.87 |
| Motion Picture Theatre | 1.20 | 0.76 |
| Multi-Family | 0.70 | 0.51 |
| Museum | 1.10 | 1.02 |
| Office | 0.90 | 0.82 |
| Parking Garage | 0.30 | 0.21 |
| Penitentiary | 1.00 | 0.81 |
| Performing Arts Theatre | 1.60 | 1.39 |
| Police Station | 1.00 | 0.87 |
| Post Office | 1.10 | 0.87 |
| Religious Building | 1.30 | 1.00 |
| Retail | 1.40 | 1.26 |
| School/University | 1.20 | 0.87 |
| Sports Arena | 1.10 | 0.91 |
| Town Hall | 1.10 | 0.89 |
| Transportation | 1.00 | 0.70 |
| Warehouse | 0.60 | 0.66 |
| Workshop | 1.40 | 1.19 |

**Space-by-Space Method Baseline LPD Requirements for Washington, D.C. and Delaware[[817]](#footnote-817)**

|  |  |
| --- | --- |
| **Common Space-By-Space Types** | **Lighting Power Density (W/ft2)** |
| Atrium - First 40 feet in height | 0.03 per ft. ht. |
| Atrium - Above 40 feet in height | 0.02 per ft. ht. |
| Audience/seating area - Permanent |  |
| For auditorium | 0.9 |
| For performing arts theater | 2.6 |
| For motion picture theater | 1.2 |
| Classroom/lecture/training | 1.3 |
| Conference/meeting/multipurpose | 1.2 |
| Corridor/transition | 0.7 |
| Dining Area |  |
| Bar/lounge/leisure dining | 1.4 |
| Family dining area | 1.4 |
| Dressing/fitting room performing arts theater | 1.1 |
| Electrical/mechanical | 1.1 |
| Food preparation | 1.2 |
| Laboratory for classrooms | 1.3 |
| Laboratory for medical/industrial/research | 1.8 |
| Lobby | 1.1 |
| Lobby for performing arts theater | 3.3 |
| Lobby for motion picture theater | 1.0 |
| Locker room | 0.8 |
| Lounge recreation | 0.8 |
| Office – enclosed | 1.1 |
| Office - open plan | 1.0 |
| Restroom | 1.0 |
| Sales area | 1.6 |
| Stairway | 0.7 |
| Storage | 0.8 |
| Workshop | 1.6 |
| Courthouse/police station/penitentiary |  |
| Courtroom | 1.9 |
| Confinement cells | 1.1 |
| Judge chambers | 1.3 |
| Penitentiary audience seating | 0.5 |
| Penitentiary classroom | 1.3 |
| Penitentiary dining | 1.1 |
| **Building Specific Space-By-Space Types** | **Lighting Power Density (W/ft2)** |
| Automobile – service/repair | 0.7 |
| Bank/office - banking activity area | 1.5 |
| Dormitory living quarters | 1.1 |
| Gymnasium/fitness center |  |
| Fitness area | 0.9 |
| Gymnasium audience/seating | 0.4 |
| Playing area | 1.4 |
| Healthcare clinic/hospital |  |
| Corridor/transition | 1.0 |
| Exam/treatment | 1.7 |
| Emergency | 2.7 |
| Public and staff lounge | 0.8 |
| Medical supplies | 1.4 |
| Nursery | 0.9 |
| Nurse station | 1.0 |
| Physical therapy | 0.9 |
| Patient Room | 0.7 |
| Pharmacy | 1.2 |
| Radiology/imaging | 1.3 |
| Operating room | 2.2 |
| Recovery | 1.2 |
| Lounge/recreation | 0.8 |
| Laundry - washing | 0.6 |
| Hotel |  |
| Dining area | 1.3 |
| Guest rooms | 1.1 |
| Hotel lobby | 2.1 |
| Highway lodging dining | 1.2 |
| Highway lodging guest rooms | 1.1 |
| Library |  |
| Stacks | 1.7 |
| Card file and cataloging | 1.1 |
| Reading area | 1.2 |
| Manufacturing |  |
| Corridor/transition | 0.4 |
| Detailed manufacturing | 1.3 |
| Equipment room | 1.0 |
| Extra high bay (>50-foot floor-ceiling height) | 1.1 |
| High bay (25-50-foot floor-ceiling height) | 1.2 |
| Low bay (<25-foot floor-ceiling height) | 1.2 |
| Museum |  |
| General exhibition | 1.0 |
| Restoration | 1.7 |
| Parking garage – garage areas | 0.2 |
| Convention center |  |
| Exhibit space | 1.5 |
| Audience/seating area | 0.9 |
| Fire stations |  |
| Engine room | 0.8 |
| Sleeping quarters | 0.3 |
| Post office – sorting area | 0.9 |
| Religious building |  |
| Fellowship hall | 0.6 |
| Audience seating | 2.4 |
| Worship pulpit/choir | 2.4 |
| Retail |  |
| Dressing/fitting area | 0.9 |
| Mall concourse | 1.6 |
| Sales area | 1.6 |
| Sports arena |  |
| Audience seating | 0.4 |
| Court sports area - Class 4 | 0.7 |
| Court sports area - Class 3 | 1.2 |
| Court sports area - Class 2 | 1.9 |
| Court sports area - Class 1 | 3.0 |
| Ring sports arena | 2.7 |
| Transportation |  |
| Airport/train/bus baggage area | 1.0 |
| Airport concourse | 0.6 |
| Terminal - ticket counter | 1.5 |
| Warehouse |  |
| Fine material storage | 1.4 |
| Medium/bulky material | 0.6 |

**Space-by-Space Method Baseline LPD Requirements for Maryland[[818]](#footnote-818)**

|  |  |
| --- | --- |
| **Common Space-By-Space Types** | **Lighting Power Density (W/ft2)** |
| Atrium |  |
| Less than 40 feet in height | 0.03 per foot in total height |
| Greater than 40 feet in height | 0.40 + 0.02 per foot in total height |
| Audience seating area |  |
| In an auditorium | 0.63 |
| In a convention center | 0.82 |
| In a gymnasium | 0.65 |
| In a motion picture theater | 1.14 |
| In a penitentiary | 0.28 |
| In a performing arts theater | 2.43 |
| In a religious building | 1.53 |
| In a sports arena | 0.43 |
| Otherwise | 0.43 |
| Banking activity area | 1.01 |
| Breakroom (See Lounge/Breakroom) |  |
| Classroom/lecture hall/training room |  |
| In a penitentiary | 1.34 |
| Otherwise | 1.24 |
| Conference/meeting/multipurpose room | 1.23 |
| Copy/print room | 0.72 |
| Corridor |  |
| In a facility for the visually impaired (and not used primarily by staff) | 0.92 |
| In a hospital | 0.79 |
| In a manufacturing facility | 0.41 |
| Otherwise | 0.66 |
| Courtroom | 1.72 |
| Computer room | 1.71 |
| Dining area |  |
| In a penitentiary | 0.96 |
| In a facility for the visually impaired (and not used primarily by staff) | 1.9 |
| In bar/lounge or leisure dining | 1.07 |
| In cafeteria or fast food dining | 0.65 |
| In family dining | 0.89 |
| Otherwise | 0.65 |
| Electrical/mechanical room | 0.95 |
| Emergency vehicle garage | 0.56 |
| Food preparation area | 1.21 |
| Guest room | 0.47 |
| Laboratory |  |
| In or as a classroom | 1.43 |
| Otherwise | 1.81 |
| Laundry/washing area | 0.6 |
| Loading dock, interior | 0.47 |
| Lobby |  |
| In a facility for the visually impaired (and not used primarily by the staff) | 1.8 |
| For an elevator | 0.64 |
| In a hotel | 1.06 |
| In a motion picture theater | 0.59 |
| In a performing arts theater | 2.0 |
| Otherwise | 0.9 |
| Locker room | 0.75 |
| Lounge/breakroom |  |
| In a healthcare facility | 0.92 |
| Otherwise | 0.73 |
| Office |  |
| Enclosed | 1.11 |
| Open plan | 0.98 |
| Parking area, interior | 0.19 |
| Pharmacy area | 1.68 |
| Restroom |  |
| In a facility for the visually impaired (and not used primarily by the staff) | 1.21 |
| Otherwise | 0.98 |
| Sales area | 1.59 |
| Seating area, general | 0.54 |
| Stairway (See space containing stairway) |  |
| Stairwell | 0.69 |
| Storage room | 0.63 |
| Vehicular maintenance area | 0.67 |
| Workshop | 1.59 |
| **Building Type Specific Space Types** | **Lighting Power Density (W/ft2)** |
| Facility for the visually impaired |  |
| In a chapel (and not used primarily by the staff) | 2.21 |
| In a recreation room (and not used primarily by the staff) | 2.41 |
| Automotive (See Vehicular Maintenance Area above) |  |
| Convention Center – exhibit space | 1.45 |
| Dormitory – living quarters | 0.38 |
| Fire Station – sleeping quarters | 0.22 |
| Gymnasium/fitness center |  |
| In an exercise area | 0.72 |
| In a playing area | 1.2 |
| Healthcare facility |  |
| In an exam/treatment room | 1.66 |
| In an imaging room | 1.51 |
| In a medical supply room | 0.74 |
| In a nursery | 0.88 |
| In a nurse’s station | 0.71 |
| In an operating room | 2.48 |
| In a patient room | 0.62 |
| In a physical therapy room | 0.91 |
| In a recovery room | 1.15 |
| Library |  |
| In a reading area | 1.06 |
| In the stacks | 1.71 |
| Manufacturing facility |  |
| In a detailed manufacturing facility | 1.29 |
| In an equipment room | 0.74 |
| In an extra high bay area (greater than 50’ floor-to-ceiling height) | 1.05 |
| In a high bay area (25’-50’ floor-to-ceiling height) | 1.23 |
| In a low bay area (less than 25’ floor-to-ceiling height) | 1.19 |
| Museum |  |
| In a general exhibition area | 1.05 |
| In a restoration room | 1.02 |
| Performing arts theater – dressing room | 0.61 |
| Post Office – Sorting Area | 0.94 |
| Religious buildings |  |
| In a fellowship hall | 0.64 |
| In a worship/pulpit/choir area | 1.53 |
| Retail facilities |  |
| In a dressing/fitting room | 0.71 |
| In a mall concourse | 1.1 |
| Sports arena – playing area |  |
| For a Class I facility | 3.68 |
| For a Class II facility | 2.4 |
| For a Class III facility | 1.8 |
| For a Class IV facility | 1.2 |
| Transportation facility |  |
| In a baggage/carousel area | 0.53 |
| In an airport concourse | 0.36 |
| At a terminal ticket counter | 0.8 |
| Warehouse – storage area |  |
| For medium to bulky, palletized items | 0.58 |
| For smaller, hand-carried items | 0.95 |

Illustrative examples – do not use as default assumption

For example, assuming a 15,000 ft2 conditoned office building with gas heat in in DE using the Building Area Method with an LPDEE of 0.75:

ΔkWh = ((0.9 - 0.75) / 1000) \* 15,000 \* 2,969 \* 1.10

= 7,348 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((LPDBASE – LPDEE) / 1000) \* AREA \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, assuming a 15,000 ft2 conditoned office building with gas heat in DE using the Building Area Method with an LPDEE of 0.75 and estimating PJM summer peak coincidence:

ΔkWh = ((0.9 - 0.75) / 1000) \* 15,000 \* 1.32 \* 0.69

= 2.05 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= (-ΔkWh / WHFe) \* 0.00073

*Where:*

*0.7 = Aspect ratio [[819]](#footnote-819)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[820]](#footnote-820)*

*0.75 = Assumed heating system efficiency [[821]](#footnote-821)*

Illustrative examples – do not use as default assumption

For example, assuming a 15,000 ft2 conditoned office building with gas heat in DE using the Building Area Method with an LPDEE of 0.75:

ΔkWh = (-7,348 / 1.10) \* 0.00073

= -4.88 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

Incremental costs will vary greatly from project to project depending on the advanced lighting design principles and lighting technologies used. Incremental costs should be estimated on a case-by-case basis.

**Measure Life**

The measure life is assumed to be 15 years.[[822]](#footnote-822)

**Operation and Maintenance Impacts**

Due to differences in costs and lifetimes of the efficient and baseline replacement components, there may be significant operation and maintenance impacts associated with this measure. Actual operation and maintenance costs should be estimated on a case-by-case basis.

**LED Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Lighting Luminaires and Retrofit Kits**

**Unique Measure Code(s): CI\_LT\_TOS\_LEDODPO\_0615 and CI\_LT\_RTR\_LEDODPO\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of an LED outdoor pole/arm- or wall-mounted luminaire or retrofit kit for parking lot, street, or general area illumination in place of a high-intensity discharge light source. Eligible applications include time of sale or new construction and retrofit applications.

**Definition of Baseline Condition**

The baseline condition is defined as an outdoor pole/arm- or wall-mounted luminaire with a high intensity discharge light-source. Typical baseline technologies include metal halide (MH) and high pressure sodium (HPS) lamps.

**Definition of Efficient Condition**

The efficient condition is defined as an LED outdoor pole/arm- or wall-mounted luminaire or retrofit kit. Eligible fixtures and retrofit kits must be listed on the DesignLights Consortium Qualified Products List[[823]](#footnote-823).

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) / 1000) \* HOURS

*Where:*

*WattsBASE = Actual Connected load of baseline fixture*

*= If the actual baseline fixture wattage is unknown, use the default values presented in the “Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Lighting Baseline and Efficient Wattage” table below.*

*WattsEE = Actual Connected load of the LED fixture*

*= If the actual LED fixture wattage is unknown, use the default values presented in the “Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Lighting Baseline and Efficient Wattage” table below based on the appropriate baseline description.*

**Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Lighting Baseline and Efficient Wattage[[824]](#footnote-824)**

| **Measure Category** | **Baseline Description** | **WattsBASE** | **Efficient Description** | **WattsEE** |
| --- | --- | --- | --- | --- |
| LED Outdoor Area Fixture replacing up to 175W HID | 175W or less base HID | 171 | DLC Qualified LED Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Luminaires | 99 |
| LED Outdoor Area Fixture replacing 176-250W HID | 176W up to 250W base HID | 288 | DLC Qualified LED Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Luminaires | 172 |
| LED Outdoor Area Fixture replacing 251-400W HID | 251W up to 400W base HID | 452 | DLC Qualified LED Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Luminaires | 293 |
| LED Outdoor Area Fixture replacing 401-1000W HID | 401W up to 1000W base HID | 1075 | DLC Qualified LED Outdoor Pole/Arm- or Wall-Mounted Area and Roadway Luminaires | 663 |

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, assume 3,338 [[825]](#footnote-825). Otherwise, use site specific annual operating hours information.[[826]](#footnote-826)*

Illustrative examples – do not use as default assumption

For example, a 250W metal halide fixture is replaced with an LED fixture:

ΔkWh = ((288 – 172) / 1000) \* 3,338

= 387 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) / 1000) \* CF

*Where:*

*CF = Summer Peak Coincidence Factor for measure*

*= 0 [[827]](#footnote-827)*

Illustrative examples – do not use as default assumption

For example, a 250W metal halide fixture is replaced with an LED fixture:

ΔkW = ((288 - 172) / 1000) \* 0

= 0 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost[[828]](#footnote-828)**

| **Measure Category** | **Installed Cost** | **Incremental Cost** |
| --- | --- | --- |
| LED Outdoor Pole/Arm Area and Roadway Luminaires | | |
| Fixture replacing up to 175W HID | $460 | $195 |
| Fixture replacing 176-250W HID | $620 | $310 |
| Fixture replacing 251+ HID | $850 | $520 |
| LED Wall-Mounted Area Luminaires | | |
| All Fixtures | $250 | $120 |

**Measure Life**

The measure life is assumed to be 18 years.[[829]](#footnote-829)

**Operation and Maintenance Impacts[[830]](#footnote-830)**

Due to differences in costs and lifetimes of fixture components between the efficient and baseline cases, there are significant operation and maintenance impacts associated with this measure. Estimated O&M savings and the component cost and lifetime assumptions are presented in the table below.

| **Measure Category** | **Baseline Description** | **Lamp Life (Hours)** | **Lamp Cost** | **Lamp Rep. Labor/Disposal Cost** | **Ballast Life (Hours)** | **Ballast Cost** | **Ballast Rep. Labor/Disposal Cost** | **NPV O&M Savings** |
| --- | --- | --- | --- | --- | --- | --- | --- | --- |
| LED Outdoor Area Fixture replacing up to 175W HID | 175W or less base HID | 10000 | $31.00 | $2.92 | 40000 | $95.85 | $27.50 | $180.37 |
| LED Outdoor Area Fixture replacing 176-250W HID | 176W up to 250W base HID | 10000 | $21.00 | $2.92 | 40000 | $87.75 | $27.50 | $147.44 |
| LED Outdoor Area Fixture replacing 251-400W HID | 251W up to 400W base HID | 10000 | $11.00 | $2.92 | 40000 | $60.46 | $27.50 | $114.52 |
| LED Outdoor Area Fixture replacing 401-1000W HID | 401W up to 1000W base HID | 10000 | $23.00 | $2.92 | 40000 | $100.09 | $27.50 | $154.03 |

**LED High-Bay Luminaires and Retrofit Kits**

**Unique Measure Code(s): CI\_LT\_TOS\_LEDHB\_0615 and CI\_LT\_RTR\_LEDHB\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of an LED high-bay luminaire or retrofit kit for general area illumination in place of a high-intensity discharge or fluorescent light source. Eligible applications include time of sale or new construction luminaires and retrofit kits installed at a minimum height of 20 feet. Because of the improved optical control afforded by LED luminaires and retrofit kits, LED lighting systems can typically reduce total lumen output while maintaining required illuminance on work surfaces. Therefore, illuminance calculations should be performed in the process of selecting LED luminaires.

**Definition of Baseline Condition**

The baseline condition is defined as a high-bay luminaire with a high intensity discharge or fluorescent light-source. Typical baseline technologies include pulse-start metal halide (PSMH) and fluorescent T5 high-output fixtures. For time of sale applications, the baseline condition will vary depending upon the specific characteristics of the fixtures installed (e.g. light source technology, number of lamps). For retrofit applications, the baseline is the existing fixture.

**Definition of Efficient Condition**

The efficient condition is defined as an LED high-bay luminaire. Eligible fixtures must be listed on the DesignLights Consortium Qualified Products List[[831]](#footnote-831).

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) / 1000) \* HOURS \* ISR \* WHFe

*Where:*

*WattsBASE = Actual Connected load of baseline fixture*

*WattsEE = Actual Connected load of the LED fixture*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[832]](#footnote-832)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[833]](#footnote-833)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

Illustrative examples – do not use as default assumption

For example, a 250W pulse start metal halide fixture delivering 16,000 mean system lumens is replaced with an LED fixture drawing 178W in a warehouse with gas heat in BGE service territory:[[834]](#footnote-834)

ΔkWh = ((288 - 178) / 1000) \* 4,116 \* 1.00 \* 1.02

= 462 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) / 1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, a 250W metal halide fixture delivering 16,000 mean system lumens is replaced with an LED fixture drawing 178W in a warehouse with gas heat in BGE service territory and estimating PJM summer peak coincidence:

ΔkW = ((288 - 178) / 1000) \* 1.00 \* 1.24 \* 0.72

= 0.10 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= -ΔkWh \* 0.00065

*Where:*

*0.7 = Aspect ratio [[835]](#footnote-835)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[836]](#footnote-836)*

*0.75 = Assumed heating system efficiency [[837]](#footnote-837)*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

Incremental costs should be determined on a site-specific basis depending on the actual baseline and efficient equipment.

Illustrative examples – do not use as default assumption

For the illustrative example, the incremental cost is approximately $200.[[838]](#footnote-838)

**Measure Life**

The measure life is assumed to be 12 years for both luminaires and retrofit kits.[[839]](#footnote-839)

**Operation and Maintenance Impacts**

Due to differences in costs and lifetimes of fixture components between the efficient and baseline cases, there are significant operation and maintenance impacts associated with this measure. Estimated O&M savings should be calculated on a site-specific basis depending on the actual baseline and efficient equipment.

**LED 1x4, 2x2, and 2x4 Luminaires and Retrofit Kits**

**Unique Measure Code(s): CI\_LT\_TOS\_LED1x4\_0615, CI\_LT\_TOS\_LED2x2\_0615, CI\_LT\_TOS\_LED2x4\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of an LED 1x4, 2x2, or 2x4 luminaire or retrofit kits for general area illumination in place of a fluorescent light source. These luminaires and retrofit kits are typically recessed, suspended, or surface-mounted and intended to provide ambient lighting in settings such as office spaces, schools, retail stores, and other commercial environments. Eligible applications include time of sale or new construction and retrofits applications. Because of the improved optical control afforded by LED luminaires and retrofit kits, LED lighting systems can typically reduce total lumen output while maintaining required illuminance on work surfaces. Therefore, illuminance calculations should be performed in the process of selecting LED luminaires and retrofit kits.

**Definition of Baseline Condition**

The baseline condition is defined as a 1x4, 2x2, or 2x4 fixture with a fluorescent light-source. Typical baseline technologies include fluorescent T8 fixtures. For time of sale applications, the baseline condition will vary depending upon the specific characteristics of the fixtures installed (e.g. number of lamps).

**Definition of Efficient Condition**

The efficient condition is defined as an LED high-bay luminaire. Eligible fixtures must be listed on the DesignLights Consortium Qualified Products List[[840]](#footnote-840).

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE – WattsEE) / 1000) \* HOURS \* ISR \* WHFe

*Where:*

*WattsBASE = Actual Connected load of baseline fixture*

*WattsEE = Actual Connected load of the LED fixture*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D.[[841]](#footnote-841) Otherwise, use site specific annual operating hours information.[[842]](#footnote-842)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[843]](#footnote-843)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

Illustrative examples – do not use as default assumption

For example, a 1x4 fixture with 4ft F32 T8 2-Lamp and electronic ballast delivering 4,600 mean system lumens is replaced with an LED luminaire drawing 43W in a conditoned office building with gas heat in BGE service territory:[[844]](#footnote-844)

ΔkWh = ((53 - 43) / 1000) \* 2,969 \* 1.00 \* 1.10

= 32.7 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) / 1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative examples – do not use as default assumption

For example, a 1x4 fixture with 4ft F32 T8 2-Lamp and electronic ballast delivering 4,600 mean system lumens is replaced with an LED luminaire drawing 43W in a conditoned office building with gas heat in BGE service territory and estimating PJM summer peak coincidence:

ΔkW = ((53 - 43) / 1000) \* 1.00 \* 1.32 \* 0.69

= 0.01 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= -ΔkWh \* 0.00065

*Where:*

*0.7 = Aspect ratio [[845]](#footnote-845)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[846]](#footnote-846)*

*0.75 = Assumed heating system efficiency [[847]](#footnote-847)*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

Incremental costs should be determined on a site-specific basis depending on the actual baseline and efficient equipment.

Illustrative examples – do not use as default assumption

For examples, the incremental costs are approximately $100 for 1x4 (4,600 mean system lumens), $75 for 2x2 (4,100 mean system lumens), and $125 for 2x4 (6,900 mean system lumens) luminaires.[[848]](#footnote-848)

**Measure Life**

The measure life is assumed to be 14 years.[[849]](#footnote-849)

**Operation and Maintenance Impacts**

Due to differences in costs and lifetimes of fixture components between the efficient and baseline cases, there are significant operation and maintenance impacts associated with this measure. Estimated O&M savings should be calculated on a site-specific basis depending on the actual baseline and efficient equipment.

**LED Parking Garage/Canopy Luminaires and Retrofit Kits**

**Unique Measure Code(s): CI\_LT\_TOS\_LEDODPG\_0615 and CI\_LT\_RTR\_LEDODPG\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of an LED parking garage or fuel pump canopy luminaire or retrofit kit in place of a high-intensity discharge light source. Eligible applications include time of sale or new construction and retrofit applications.

**Definition of Baseline Condition**

The baseline condition is defined as a parking garage or canopy luminaire with a high intensity discharge light-source. Typical baseline technologies include metal halide (MH) and high pressure sodium (HPS) lamps.

**Definition of Efficient Condition**

The efficient condition is defined as an LED parking garage or canopy luminaire or retrofit kit. Eligible luminaires and retrofit kits must be listed on the DesignLights Consortium Qualified Products List[[850]](#footnote-850).

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) / 1000) \* HOURS \* ISR

*Where:*

*WattsBASE = Actual Connected load of baseline fixture*

*= If the actual baseline fixture wattage is unknown, use the default values presented in the “Parking Garage or Canopy Fixture Baseline and Efficient Wattage” table below.*

*WattsEE = Actual Connected load of the LED fixture*

*= If the actual LED fixture wattage is unknown, use the default values presented in the “Parking Garage or Canopy Fixture Baseline and Efficient Wattage” table below based on the based on the appropriate baseline description.*

**Parking Garage or Canopy Fixture Baseline and Efficient Wattage[[851]](#footnote-851)**

| **Measure Category** | **Baseline Description** | **WattsBASE** | **Efficient Description** | **WattsEE** |
| --- | --- | --- | --- | --- |
| LED Parking Garage/Canopy Fixture replacing up to 175W HID | 175W or less base HID | 171 | DLC Qualified LED Parking Garage and Canopy Luminaires | 94 |
| LED Parking Garage/Canopy Fixture replacing 176-250W HID | 176W up to 250W base HID | 288 | DLC Qualified LED Parking Garage and Canopy Luminaires | 162 |
| LED Parking Garage/Canopy Fixture replacing 251 and above HID | 251W and above base HID | 452 | DLC Qualified LED Parking Garage and Canopy Luminaires | 248 |

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, assume 3,338 for canopy applications and 8,760 for parking garage applications[[852]](#footnote-852). Otherwise, use site specific annual operating hours information.[[853]](#footnote-853)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[854]](#footnote-854)*

Illustrative examples – do not use as default assumption

For example, a 250W parking garage standard metal halide fixture is replaced with an LED fixture:

ΔkWh = ((288 - 162) / 1000) \* 8,760 \* 1.00

= 1104 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) / 1000) \* ISR \* CF

*Where:*

*CF = Summer Peak Coincidence Factor for measure*

*= 0 for canopy applications and 1.0 for parking garage applications [[855]](#footnote-855)*

Illustrative examples – do not use as default assumption

For example, a 250W parking garage standard metal halide fixture is replaced with an LED fixture:

ΔkW = ((288 - 162) / 1000) \* 1.00 \* 1.00

= 0.13 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost[[856]](#footnote-856)**

|  |  |  |
| --- | --- | --- |
| **Measure Category** | **Installed Cost** | **Incremental Cost** |
| Average of All Categories | $585 | $343 |

**Measure Life**

The measure life is assumed to be 21 years for canopy applications and 8 years for parking garage applications.[[857]](#footnote-857)

**Operation and Maintenance Impacts[[858]](#footnote-858)**

Due to differences in costs and lifetimes of fixture components between the efficient and baseline cases, there are significant operation and maintenance impacts associated with this measure. Estimated O&M savings and the component cost and lifetime assumptions are presented in the table below.

| **Measure Category** | **Baseline Description** | **Lamp Life (Hours)** | **Lamp Cost** | **Lamp Rep. Labor/Disposal Cost** | **Ballast Life (Hours)** | **Ballast Cost** | **Ballast Rep. Labor/Disposal Cost** | **NPV O&M Savings (Canopy/Parking Garage)** |
| --- | --- | --- | --- | --- | --- | --- | --- | --- |
| LED Parking Garage/Canopy Fixture replacing up to 175W HID | 175W or less base HID | 10000 | $31.00 | $2.92 | 40000 | $95.85 | $27.50 | $194.46 / $156.09 |
| LED Parking Garage/Canopy Fixture replacing 176-250W HID | 176W up to 250W base HID | 10000 | $21.00 | $2.92 | 40000 | $87.75 | $27.50 | $142.93 / $133.33 |
| LED Parking Garage/Canopy Fixture replacing 251 and above HID | 251W and above base HID | 10000 | $11.00 | $2.92 | 40000 | $60.46 | $27.50 | $94.81 / $94.78 |

**ENERGY STAR Integrated Screw Based SSL (LED) Lamp – Commercial\***

**Unique Measure Code: CI\_LT\_TOS\_SSLDWN\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

This measure describes savings from the purchase and installation of an ENERGY STAR Integrated Screw Based SSL (LED) Lamp in place of an incandescent lamp. Note: In December 2015, ENERGY STAR published V2.0 of the Product Specification for Lamps (Light Bulbs). Products that certify to both specifications are available until January 2, 2017, when only V2.0 products can carry the ENERGY STAR mark. Product brand owners may have products certified to V2.0 as early as December 31, 2015. Therefore, where applicable, this measure includes parameters for both the ENERGY STAR Product Specification for Lamps (Light Bulbs) V1.1 (i.e., the current version of the specification) and V2.0. Beginning January 2, 2017, the savings assumptions for the V1.1 specification will no longer be effective.

**Definition of Baseline Condition**

For time of sale replacement, the baseline wattage is assumed to be an incandescent or EISA complaint (where applicable) bulb installed in a screw-base socket.[[859]](#footnote-859) Note that the baseline will be EISA compliant for all categories to which EISA applies. If the in situ lamp wattage is known and lower than the EISA mandated maximum wattage (where applicable), the baseline wattage should be assumed equal to the in situ lamp wattage.

**Definition of Efficient Condition**

The high efficiency wattage is assumed to be an ENERGY STAR qualified Integrated Screw Based SSL (LED) Lamp. As noted in the measure description, eligible products may be certified to either V1.1 or V2.0. The ENERGY STAR specifications can be viewed here: http://1.usa.gov/1QJFLgT

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBase - WattsEE) /1000) \* HOURS \* ISR \* WHFe

*Where:*

*WattsBase = Use equivalent baseline wattage from the table below, based on actual LED lamp lumens[[860]](#footnote-860)*

*WattsEE = Actual LED lamp watts.*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D. Otherwise, use site specific annual operating hours information.[[861]](#footnote-861)*

*ISR = In Service Rate or percentage of units rebated that are installed and operational*

*= 1.00 [[862]](#footnote-862)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

| **Bulb Type** | **Lower Lumen Range** | **Upper Lumen Range** | **WattsBase** |
| --- | --- | --- | --- |
| **Standard A-Type** | 250 | 449 | 25 |
| 450 | 799 | 29 |
| 800 | 1099 | 43 |
| 1100 | 1599 | 53 |
| 1600 | 1999 | 72 |
| 2000 | 2599 | 72 |
| 2600 | 3000 | 150 |
| 3001 | 3999 | 200 |
| 4000 | 6000 | 300 |
| **3-Way (Highest Setting), bug, marine, rough service, infrared** | 250 | 449 | 25 |
| 450 | 799 | 40 |
| 800 | 1099 | 60 |
| 1100 | 1599 | 75 |
| 1600 | 1999 | 100 |
| 2000 | 2549 | 125 |
| 2550 | 2999 | 150 |
| **Globe**  **(medium and intermediate bases less than 750 lumens)** | 90 | 179 | 10 |
| 180 | 249 | 15 |
| 250 | 349 | 25 |
| 350 | 749 | 40 |
| **Decorative**  **(Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)** | 70 | 89 | 10 |
| 90 | 149 | 15 |
| 150 | 299 | 25 |
| 300 | 749 | 40 |
| **Globe**  **(candelabra bases less than 1050 lumens)** | 90 | 179 | 10 |
| 180 | 249 | 15 |
| 250 | 349 | 25 |
| 350 | 499 | 40 |
| 500 | 1049 | 60 |
| **Decorative**  **(Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)** | 70 | 89 | 10 |
| 90 | 149 | 15 |
| 150 | 299 | 25 |
| 300 | 499 | 40 |
| 500 | 1049 | 60 |
| **Reflector with medium screw bases w/ diameter <=2.25"** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649 | 50 |
| 650 | 1199 | 65 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter >2.5" (\*see exceptions below)** | 640 | 739 | 40 |
| 740 | 849 | 45 |
| 850 | 1179 | 50 |
| 1180 | 1419 | 65 |
| 1420 | 1789 | 75 |
| 1790 | 2049 | 90 |
| 2050 | 2579 | 100 |
| 2580 | 3429 | 120 |
| 3430 | 4270 | 150 |
| **R, PAR, ER, BR, BPAR or similar bulb shapes with medium screw bases w/ diameter > 2.26'' and ≤ 2.5" (\*see exceptions below)** | 540 | 629 | 40 |
| 630 | 719 | 45 |
| 720 | 999 | 50 |
| 1000 | 1199 | 65 |
| 1200 | 1519 | 75 |
| 1520 | 1729 | 90 |
| 1730 | 2189 | 100 |
| 2190 | 2899 | 120 |
| 2900 | 3850 | 150 |
| **\*ER30, BR30, BR40, or ER40** | 400 | 449 | 40 |
| 450 | 499 | 45 |
| 500 | 649-1179[[863]](#footnote-863) | 50 |
| **\*BR30, BR40, or ER40** | 650 | 1419 | 65 |
| **\*R20** | 400 | 449 | 40 |
| 450 | 719 | 45 |
| **\*All reflector lamps**  **below lumen ranges specified above** | 200 | 299 | 20 |
| 300 | 399-639[[864]](#footnote-864) | 30 |

Illustrative example – do not use as default assumption

For example, a 10W 550 lumen LED directional lamp with medium screw base diameter <=2.25" is installed in a conditoned office building with gas heat in BGE service territory in 2015.

ΔkWh = ((50 - 10)/ 1,000) \* 2,969 \* 1.00 \* 1.10

= 131 kWh

**Baseline Adjustment**

Currently the EISA legislation only applies to omnidirectional bulbs, with decorative <40 watts and directional being exempted. If additional legislation is passed, this TRM will be adjusted accordingly.

To account for these new standards, the savings for this measure should be reduced to account for the higher baselines in 2020. The following table shows the calculated adjustments. The calculated energy savings for omnidirectional lamps should be multiplied by the appropriate factor from the table below for years 2020 and beyond[[865]](#footnote-865):

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Lower Lumen Range** | **Upper Lumen Range** | **Mid-life Adjustment in 2020** | | |
| **ENERGY STAR V1.1** | **ENERGY STAR V2.0** | |
| **CRI>=90** | **CRI<90** |
| 200 | 449 | 100% | 100% | 100% |
| 450 | 799 | 100% | 100% | 100% |
| 800 | 1099 | 9% | 16% | 19% |
| 1,100 | 1599 | 11% | 20% | 24% |
| 1,600 | 1999 | 21% | 23% | 27% |
| 2,000 | 2599 | 23% | 26% | 30% |
| 2,600 | 3000 | 100% | 100% | 100% |
| 3001 | 3999 | 100% | 100% | 100% |
| 4000 | 6000 | 100% | 100% | 100% |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBase - WattsEE) /1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

Illustrative example – do not use as default assumption

For example, a 10W 550 lumen LED directional lamp with medium screw base diameter <=2.25" is installed in a conditoned office building with gas heat in BGE service territory and estimating PJM summer peak coincidence.

ΔkW = ((50 - 10)/ 1,000) \* 1.0 \* 0.69 \* 1.32

= 0.036 kW

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= (-ΔkWh / WHFe) \* 0.00073

*Where:*

*0.7 = Aspect ratio [[866]](#footnote-866)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[867]](#footnote-867)*

*0.75 = Assumed heating system efficiency [[868]](#footnote-868)*

Illustrative example – do not use as default assumption

For example, a 10W 550 lumen LED directional lamp with medium screw base diameter <=2.25" is installed a conditoned office building with gas heat in BGE service territory.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.00073

= - 0.087 MMBtu

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

If the implementation strategy allows the collection of actual costs, or an appropriate average, then that should be used. If not, the incremental cost for this measure is presented below:[[869]](#footnote-869)

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Lamp Type** | **LED Wattage** | **Lamp Costs** | | **Incremental Cost** |
| **Efficient** | **Baseline** |
| **LED** | **Incandescent or EISA compliant** | **Incandescent or EISA compliant** |
| **Omni-directional** | <15W | $6.11 | $1.50 | $4.61 |
| >=15W | $6.81 | $1.50 | $5.31 |
| **Decorative** | <15W | $8.00 | $1.00 | $7.00 |
| <=15 to <25W | $25.00 | $1.00 | $24.00 |
| >=25W | $25.00 | $1.00 | $24.00 |
| **Directional** | <20W | $17.63 | $5.00 | $12.63 |
| >=20W | $70.78 | $5.00 | $65.78 |

**Measure Life**

The table below shows the assumed measure life for ENERGY STAR Versions 1.1 and 2.0:

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Lamp Type** | **ENERGY STAR V1.1**[[870]](#footnote-870) | | **ENERGY STAR V2.0**[[871]](#footnote-871) | |
| **[[872]](#footnote-872)Rated Life (Hours)** | **Measure Life (Years)** | **Rated Life (Hours)** | **Measure Life (Years)** |
| **Commercial Interior** | **Commercial Interior** |
| **Omnidirectional** | 25,000 | 7 | 15,000 | 4 |
| **Decorative** | 15,000 | 4 | 15,000 | 4 |
| **Directional** | 25,000 | 7 | 25,000 | 7 |

**Operation and Maintenance Impacts**

For Decorative and Directional bulbs, without a baseline shift, the following component costs and lifetimes should be used to calculate O&M savings:

|  |  |  |
| --- | --- | --- |
| **Lamp Type** | **Baseline Lamp Cost** | **Lamp Lifetime**[[873]](#footnote-873) |
| **Commercial Interior** |
| **Decorative** | $1.00 | 0.29 |
| **Directional <15W** | $5.00 | 0.29 |
| **Directional >=15W** | $5.00 | 0.29 |

For Omni-directional bulbs, to account for the shift in baseline due to the Federal Legislation, the baseline replacement cost over the lifetime of the LED should be adjusted based on the key assumptions documented below:

|  |  |  |
| --- | --- | --- |
|  | **EISA**  **2012-2014 Compliant** | **EISA 2020 Compliant** |
| **Replacement Cost <10W** | $1.50 | $2.86 |
| **Replacement Cost >=10W** | $1.50 | $3.19 |
| **Component Life (hours)** | 1000 | 10,000 |

**LED Refrigerated Case Lighting**

**Unique Measure Code(s): CI\_LT\_TOS\_LEDRCL\_0615 and CI\_LT\_RTR\_LEDRCL\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of LED luminaries in vertical and horizontal refrigerated display cases replacing T8 or T12HO linear fluorescent lamp technology. Savings characterizations are provided for both coolers and freezers. Specified LED luminaires should meet v2.1 DesignLights Consortium Product Qualification Criteria for either the “Vertical Refrigerated Case Luminaire” or “Horizontal Refrigerated Case Luminares” category. LED luminaires not only provide the same light output with lower connected wattages, but also produce less waste heat which decreases the cooling load on the refrigeration system and energy needed by the refrigeration compressor. Savings and assumptions are based on a per linear foot of installed lighting basis.

**Definition of Baseline Condition**

The baseline equipment is assumed to be T8 or T12HO linear fluorescent lamps.

**Definition of Efficient Condition**

The efficient equipment is assumed to be DesignLights Consortium qualified LED vertical or horizontal refrigerated case luminaires.

**Annual Energy Savings Algorithm**

ΔkWh = (WattsPerLFBASE – WattsPerLFEE) / 1000 \* LF \* HOURS \* WHFe

*Where:*

WattsPerLFBASE = Connected wattage per linear foot of the baseline fixtures; see table below for default values.[[874]](#footnote-874)

WattsPerLFEE = Connected wattage per linear foot of the LED fixtures.[[875]](#footnote-875)

= Actual installed. If actual installed wattage is unknown, see table below for default values.

|  |  |  |  |
| --- | --- | --- | --- |
| **Efficient Lamp** | **Baseline Lamp** | **Efficient Fixture Wattage (WattsPerLFEE)** | **Baseline Fixture Watts (WattsPerLFBASE)** |
| LED Case Lighting System | T8 Case Lighting System | 7.6 | 15.2 |
| LED Case Lighting System | T12HO Case Lighting System | 7.7 | 18.7 |

LF = Linear feet of installed LED luminaires.

= Actual installed

HOURS = Annual operating hours; assume 6,205 operating hours per year if actual operating hours are unknown.[[876]](#footnote-876)

WHFe = Waste heat factor for energy to account for refrigeration savings from efficient lighting. For prescriptive refrigerated lighting measures, the default value is 1.41 for refrigerated cases and 1.52 for freezer cases.[[877]](#footnote-877)

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (WattsPerLFBASE – WattsPerLFEE) / 1000 \* LF \* WHFd \* CF

*Where:*

*WHFd = Waste heat factor for demand to account for refrigeration savings from efficient lighting. For prescriptive refrigerated lighting measures, the default value is 1.40 for refrigerated cases and 1.51 for freezer cases.[[878]](#footnote-878)*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.96 (lighting in Grocery).[[879]](#footnote-879)*

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost[[880]](#footnote-880)**

|  |  |
| --- | --- |
| **Efficient Measure Incremental Cost (TOS, NC)** | |
| **Application** | **Cost per Foot ($/ft.)** |
| Vertical – Center | $28.43 |
| Vertical – End | $21.10 |
| Horizontal | $21.55 |

|  |  |
| --- | --- |
| **Efficient Measure Full Cost (Retrofit)** | |
| **Application** | **Cost per Foot ($/ft.)** |
| Vertical – Center | $37.76 |
| Vertical – End | $30.54 |
| Horizontal | $31.15 |

**Measure Life[[881]](#footnote-881)**

The expected measure life is assumed to be 8 years.

**Operation and Maintenance Impacts**

LED case lighting is expected to have a longer service life than the baseline T8 and T12HO fluorescent lighting systems. Estimated O&M savings should be calculated on a site-specific basis depending on the actual baseline and efficient equipment.

**Exterior LED Flood and Spot Luminaires**

**Unique Measure Code(s): CI\_LT\_TOS\_LEDFLS\_0615 and CI\_LT\_RTR\_LEDFLS\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of an exterior LED flood or spot luminaire for landscape or architectural illumination applications in place of a halogen incandescent or high-intensity discharge light source. Eligible applications include time of sale and new construction.

**Definition of Baseline Condition**

The baseline condition is defined as an exterior flood or spot fixture with a high intensity discharge light-source. Typical baseline technologies include halogen incandescent parabolic aluminized reflector (PAR) lamps and metal halide (MH) luminaires.

**Definition of Efficient Condition**

The efficient condition is defined as an LED flood or spot luminaire. Eligible luminaires must be listed on the DesignLights Consortium Qualified Products List[[882]](#footnote-882).

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE - WattsEE) / 1000) \* HOURS

*Where:*

*WattsBASE = Actual Connected load of baseline fixture*

*= If the actual baseline fixture wattage is unknown, use the actual LED lumens to find equivalent baseline wattage from the table below.[[883]](#footnote-883)*

| **Bulb Type** | **Lower Lumen Range** | **Upper Lumen Range** | **WattsBase** |
| --- | --- | --- | --- |
| **PAR38** | 500 | 1000 | 52.5 |
| 1000 | 4000 | 108.7 |
| **Metal Halide** | 4000 | 15000[[884]](#footnote-884) | 205.0 |

*WattsEE = Actual Connected load of the LED luminaire*

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, assume 3,338 [[885]](#footnote-885). Otherwise, use site specific annual operating hours information.[[886]](#footnote-886)*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) / 1000) \* CF

*Where:*

*CF = Summer Peak Coincidence Factor for measure*

*= 0 [[887]](#footnote-887)*

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost of the LED luminaire is presented by lumen output in the table below.[[888]](#footnote-888)

| **Lower Lumen Range** | **Upper Lumen Range** | **Incremental Cost** |
| --- | --- | --- |
| 500 | 1000 | $150 |
| 1000 | 4000 | $245 |
| 4000 | 15000 | $315 |

**Measure Life**

The measure life is assumed to be 15 years.[[889]](#footnote-889)

**Operation and Maintenance Impacts**

Due to differences in costs and lifetimes of fixture components between the efficient and baseline cases, there are significant operation and maintenance impacts associated with this measure. O&M impacts should be determined on a case-by-case basis.[[890]](#footnote-890)

**LED Four-Foot Linear Replacement Lamps**

**Unique Measure Code(s): CI\_LT\_RTR\_LEDTUBE\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the replacement of four-foot linear fluorescent lamps with tubular, LED four-foot linear replacement lamps. Depending on the specific LED replacement lamp product, this measure may require changing the electrical wiring, replacing the ballast with an external driver, or altering the existing lamp holders (or “tombstones”) to accommodate the new lamp. Eligible applications are limited to retrofits. LED replacement lamp types are described in the table below:[[891]](#footnote-891)

|  |  |
| --- | --- |
| **LED Replacement Lamp Type** | **Description** |
| Type A | The Type A lamp is designed with an internal driver that allows the lamp to operate directly from the existing linear fluorescent ballast. Most of these products are designed to work with T12, T8 and T5 ballasts. |
| Type B | The Type B lamp operates with an internal driver; however, the driver is powered directly from the main voltage supplied to the existing linear fluorescent fixture. |
| Type C | The Type C lamp operates with a remote driver that powers the LED linear lamp, rather than an integrated  driver. The Type B lamp involves electrical modification to the existing fixture, but the low-voltage outputs of the driver are connected to the sockets instead of line voltage. |

Measure eligibility is limited to “Type A” products that are powered by a new compatible T8 fluorescent electronic ballast installed at the same time as the LED replacement lamp or “Type C” products with an external LED driver.

All of the EmPOWER Maryland Utilities, no longer provide incentives for linear LED lamps with an internal driver connected directly to the line voltage (commonly referred to as “Type B.”) This is due to the wide variety of installation characteristics of these types of lamps and the inherent safety concerns with these being powered directly from 120 – 277 voltage.

**Definition of Baseline Condition**

The baseline condition is defined as an existing four-foot linear fluorescent fixture.

**Definition of Efficient Condition**

The efficient condition is defined as an as an four-foot linear fluorescent fixture retrofit with LED four-foot linear replacement lamp(s) and, if required, external driver. Eligible LED replacement lamp fixture wattage must be less than the baseline fixture wattage and listed on the DesignLights Consortium Qualified Products List[[892]](#footnote-892).

**Annual Energy Savings Algorithm**

ΔkWh = ((WattsBASE – WattsEE) / 1000) \* HOURS \* ISR \* WHFe

*Where:*

*WattsBASE = Actual connected load of baseline fixture*

*= If actual baseline wattage is unknown, assume the “Delta Watts” from the table below based on existing lamp/ballast system.*

*WattsEE = Actual connected load of the fixture with LED replacement lamps.*

*= If actual baseline wattage is unknown, assume the “Delta Watts” from the table below based on existing lamp/ballast system.*

**Default Baseline and Efficient Lamp Wattage Assumptions[[893]](#footnote-893)**

|  |  |  |  |
| --- | --- | --- | --- |
| **Baseline Lamp/Ballast System** | **Baseline Lamp Wattage (WattsBASE)** | **Replacement Wattage (WattsEE)** | **Delta Watts** |
| 32W T8 IS NLO | 29.5 | 23 | 6.5 |
| 28W T8 Premium PRS NLO | 25 | 19 | 6 |
| 25W T8 Premium PRS NLO | 22 | 16 | 6 |

*HOURS = Average hours of use per year*

*= If annual operating hours are unknown, see table “C&I Interior Lighting Operating Hours by Building Type” in Appendix D.[[894]](#footnote-894) Otherwise, use site specific annual operating hours information.[[895]](#footnote-895)*

*ISR = In Service Rate or percentage of units rebated that get installed*

*= 1.00 [[896]](#footnote-896)*

*WHFe = Waste Heat Factor for Energy to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ((WattsBASE - WattsEE) / 1000) \* ISR \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand to account for cooling and heating impacts from efficient lighting.*

*= Varies by utility, building type, and HVAC equipment type. If HVAC type is known, see table “Waste Heat Factors for C&I Lighting – Known HVAC Types” in Appendix D. If HVAC type is unknown or the space is unconditioned, assume WHFe = WHFd = 1.0.*

*CF = Summer Peak Coincidence Factor for measure*

*= See table “C&I Interior Lighting Coincidence Factors by Building Type” in Appendix D.*

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBTU = (-ΔkWh / WHFe) \* 0.70 \* 0.003413 \* 0.23 / 0.75

= (-ΔkWh / WHFe) \* 0.00073

*Where:*

*0.7 = Aspect ratio [[897]](#footnote-897)*

*0.003413 = Constant to convert kWh to MMBTU*

*0.23 = Fraction of lighting heat that contributes to space heating [[898]](#footnote-898)*

*0.75 = Assumed heating system efficiency [[899]](#footnote-899)*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental costs (equipment and labor) LED linear replacement lamps are as follows:[[900]](#footnote-900)

Type A: $22.67 per LED replacement lamp, $47.50 for the ballast.

Type C: $22.67 per LED replacement lamp, $15.07 for the external driver.

**Measure Life**

The measure life is assumed to be 14 years.[[901]](#footnote-901)

**Operation and Maintenance Impacts**

Due to differences in costs and lifetimes of fixture components between the efficient and baseline cases, there are significant operation and maintenance impacts associated with this measure. O&M impacts should be determined on a case-by-case basis.[[902]](#footnote-902)

*Heating Ventilation and Air Conditioning (HVAC) End Use*

**Unitary HVAC Systems\***

**Unique Measure Code(s): CI\_HV\_TOS\_HVACSYS\_0516, CI\_HV\_EREP\_HVACSYS\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

This measure documents savings associated with the installation of new heating, ventilating, and air conditioning systems exceeding baseline efficiency criteria in place of an existing system or a new standard efficiency system of the same capacity. This measure covers air conditioners (including unitary air conditioners and packaged terminal AC) and heat pumps (air source and packaged terminal heat pumps). It does not cover ductless mini-split units. This measure applies to time of sale, new construction, and early replacement opportunities.

**Definition of Baseline Condition**

**Time of Sale or New Construction:** The baseline condition is a new system meeting minimum efficiency standards as presented in the 2012 International Energy Conservation Code (IECC 2012) and the 2015 International Energy Conservation Code (IECC 2015) (see table “Baseline Efficiencies by System Type and Unit Capacity” below)[[903]](#footnote-903) or federal standards where more stringent than local energy codes.

**Early Replacement:** The baseline condition for the Early Replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit, and the new baseline as defined above for the remainder of the measure life.

**Definition of Efficient Condition**

The efficient condition is an HVAC system of the same type as the baseline system exceeding baseline efficiency levels.

**Baseline Efficiencies by System Type and Unit Capacity**

| **Size Category (Cooling Capacity)** | **Subcategory** | **Baseline Condition (IECC 2012)** | **Baseline Condition (IECC 2015)** |
| --- | --- | --- | --- |
| **Air Conditioners, Air Cooled** |  |  |  |
| <65,000 Btu/h | Split system | 13.0 SEER | 13.0 SEER |
|  | Single package | 13.0 SEER | 14.0 SEER |
| ≥65,000 Btu/h and <135,000 Btu/h | Split system and single package | 11.2 EER  11.4 IEER | 11.2 EER  12.8 IEER |
| ≥135,000 Btu/h and <240,000 Btu/h | Split system and single package | 11.0 EER  11.2 IEER | 11.0 EER  12.4 IEER |
| ≥240,000 Btu/h and <760,000 Btu/h | Split system and single package | 10.0 EER  10.1 IEER | 10.0 EER  11.6 IEER |
| ≥760,000 Btu/h | Split system and single package | 9.7 EER  9.8 IEER | 9.7 EER  11.2 IEER |
| **Air Conditioners, Water Cooled** |  |  |  |
| <65,000 Btu/h | Split system and single package | 12.1 EER  12.3 IEER | 12.1 EER  12.3 IEER |
| ≥65,000 Btu/h and <135,000 Btu/h | Split system and single package | 12.1 EER  12.3 IEER | 12.1 EER  13.9 IEER |
| ≥135,000 Btu/h and <240,000 Btu/h | Split system and single package | 12.5 EER  12.7 IEER | 12.5 EER  13.9 IEER |
| ≥240,000 Btu/h and <760,000 Btu/h | Split system and single package | 12.4 EER  12.6 IEER | 12.4 EER  13.6 IEER |
| ≥760,000 Btu/h | Split system and single package | 12.0 EER  12.4 IEER | 12.2 EER  13.5 IEER |
| **Air Conditioners, Evaporatively Cooled** |  |  |  |
| <65,000 Btu/h | Split system and single package | 12.1 EER  12.3 IEER | 12.1 EER  12.3 IEER |
| ≥65,000 Btu/h and <135,000 Btu/h | Split system and single package | 12.1 EER  12.3 IEER | 12.1 EER  12.3 IEER |
| ≥135,000 Btu/h and <240,000 Btu/h | Split system and single package | 12.0 EER  12.2 IEER | 12.0 EER  12.2 IEER |
| ≥240,000 Btu/h and <760,000 Btu/h | Split system and single package | 11.9 EER  12.1 IEER | 11.9 EER  12.1 IEER |
| ≥760,000 Btu/h | Split system and single package | 11.7 EER  11.9 IEER | 11.7 EER  11.9 IEER |
| **Heat Pumps, Air Cooled**[[904]](#footnote-904) |  |  |  |
| <65,000 Btu/h | Split System | 13.0 SEER  7.7 HSPF | 14.0 SEER  8.2 HSPF |
|  | Single Package | 13.0 SEER  7.7 HSPF | 14.0 SEER  8.0 HSPF |
| ≥65,000 Btu/h and <135,000 Btu/h | Split system and single package | 11.0 EER  11.2 IEER  3.3 COP | 11.0 EER  12.0 IEER  3.3 COP |
| ≥135,000 Btu/h and <240,000 Btu/h | Split system and single package | 10.6 EER  10.7 IEER  3.2 COP | 10.6 EER  11.6 IEER  3.2 COP |
| ≥240,000 Btu/h and <760,000 Btu/h | Split system and single package | 9.5 EER  9.6 IEER  3.2 COP | 9.5 EER  10.6 IEER  3.2 COP |

| **Size Category (Cooling Capacity)** | **Subcategory** | **Baseline Condition (Federal Standards)[[905]](#footnote-905)** |
| --- | --- | --- |
| **Packaged Terminal Air Conditioners[[906]](#footnote-906),[[907]](#footnote-907)** |  |  |
| All Capacities | New Construction (Standard Size)[[908]](#footnote-908) | 14.0 – (0.300 \* Cap/1000) EER |
| All Capacities | Replacement (Non-Standard Size) | 10.9 – (0.213 \* Cap/1000) EER |
| **Packaged Terminal Heat Pumps[[909]](#footnote-909),[[910]](#footnote-910)** |  |  |
| All Capacities | New Construction (Standard Size) | 14.0 – (0.300 \* Cap/1000) EER  3.7 – (0.052 \* Cap/1000) COP |
| All Capacities | Replacement (Non-Standard Size) | 10.8 – (0.213 \* Cap/1000) EER  2.9 – (0.026 \* Cap/1000) COP |

Notes: 1) All cooling mode efficiency ratings in the table above assume electric resistance heating section type (or none). Subtract 0.2 from each baseline efficiency rating value if unit has heating section other than electric resistance.

**Annual Energy Savings Algorithm**

**Air Conditioners (includes air-, water-, and evaporatively-cooled unitary ACs and PTACs)**

Time of Sale:

For units with capacities less than 65,000 Btu/h and all PTACs, the energy savings are calculated using the Seasonal Energy Efficiency Ratio (SEER) as follows:

ΔkWh = (Btu/h/1000) \* ((1/SEERBASE) – (1/SEEREE)) \* HOURS

For units with capacities greater than or equal to 65,000 Btu/h, the energy savings are calculated using the Integrated Energy Efficiency Ratio (EER) as follows:

ΔkWh = (Btu/h/1000) \* ((1/IEERBASE) – (1/IEEREE)) \* HOURS

Early Replacement[[911]](#footnote-911):

For units with capacities less than 65,000 Btu/h and all PTACs, the energy savings are calculated using the Seasonal Energy Efficiency Ratio (SEER) as follows:

ΔkWh for remaining life of existing unit (i.e., measure life less the age of the existing equipment):

= (Btu/h/1000) \* ((1/SEEREXIST) – (1/SEEREE)) \* HOURS

ΔkWh for remaining measure life (i.e., measure life less the remaining life of existing unit):

= (Btu/h/1000) \* ((1/SEERBASE) – (1/SEEREE)) \* HOURS

For units with capacities greater than or equal to 65,000 Btu/h, the energy savings are calculated using the Integrated Energy Efficiency Ratio (IEER) as follows:

ΔkWh for remaining life of existing unit (i.e., measure life less the age of the existing equipment):

= (Btu/h/1000) \* ((1/IEEREXIST) – (1/IEEREE)) \* HOURS

ΔkWh for remaining measure life (i.e., measure life less the remaining life of existing unit):

= (Btu/h/1000) \* ((1/IEERBASE) – (1/IEEREE)) \* HOURS

**Heat Pumps (includes air-source HPs and PTHPs)**

Time of Sale:

For units with capacities less than 65,000 Btu/h (except PTHPs), the energy savings are calculated using the Seasonal Energy Efficiency Ratio (SEER) and Heating Season Performance (HSPF) as follows:

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/SEERBASE) – (1/SEEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/1000) \* ((1/HSPFBASE) – (1/HSPFEE)) \* HOURSHEAT

For units with capacities greater than or equal to 65,000 Btu/h (except PTHPs), the energy savings are calculated using the Integrated Energy Efficiency Ratio (IEER) and Coefficient of Performance (COP) as follows:

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/IEERBASE) – (1/IEEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/3412) \* ((1/COPBASE) – (1/COPEE)) \* HOURSHEAT

For all PTHPs, the energy savings are calculated using the Energy Efficiency Ratio (EER) and Coefficient of Performance (COP) as follows:

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/EERBASE) – (1/EEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/3412) \* ((1/COPBASE) – (1/COPEE)) \* HOURSHEAT

Early Replacement[[912]](#footnote-912):

For units with capacities less than 65,000 Btu/h, the energy savings are calculated using the Seasonal Energy Efficiency Ratio (SEER) and Heating Season Performance (HSPF) as follows:

ΔkWh for remaining life of existing unit (i.e., measure life less the age of the existing equipment):

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/SEEREXIST) – (1/SEEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/1000) \* ((1/HSPFEXIST) – (1/HSPFEE)) \* HOURSHEAT

ΔkWh for remaining measure life (i.e., measure life less the remaining life of existing unit):

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/SEERBASE) – (1/SEEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/1000) \* ((1/HSPFBASE) – (1/HSPFEE)) \* HOURSHEAT

For units with capacities greater than or equal to 65,000 Btu/h, the energy savings are calculated using the Integrated Energy Efficiency Ratio (EER) and Coefficient of Performance (COP) as follows:

ΔkWh for remaining life of existing unit (i.e., measure life less the age of the existing equipment):

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/IEEREXIST) – (1/IEEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/3412) \* ((1/COPEXIST) – (1/COPEE)) \* HOURSHEAT

ΔkWh for remaining measure life (i.e., measure life less the remaining life of existing unit):

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/IEERBASE) – (1/IEEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/3412) \* ((1/COPBASE) – (1/COPEE)) \* HOURSHEAT

For all PTHPs, the energy savings are calculated using the Energy Efficiency Ratio (EER) and Coefficient of Performance (COP) as follows:

ΔkWh for remaining life of existing unit (i.e., measure life less the age of the existing equipment):

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/EEREXIST) – (1/EEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/3412) \* ((1/COPEXIST) – (1/COPEE)) \* HOURSHEAT

ΔkWh for remaining measure life (i.e., measure life less the remaining life of existing unit):

ΔkWh = ΔkWhCOOL + ΔkWhHEAT

ΔkWhCOOL = (Btu/hCOOL/1000) \* ((1/EERBASE) – (1/EEREE)) \* HOURSCOOL

ΔkWhHEAT = (Btu/hHEAT/3412) \* ((1/COPBASE) – (1/COPEE)) \* HOURSHEAT

*Where:*

ΔkWhCOOL = Annual cooling season electricity savings (kWh)

ΔkWhHEAT *=* Annual heating season electricity savings (kWh)

*Btu/hCOOL = Cooling capacity of equipment in Btu/hour*

*= Actual Installed*

*Btu/hHEAT = Heating capacity of equipment in Btu/hour*

*= Actual Installed*

*SEEREE = SEER of efficient unit*

*= Actual Installed*

*SEERBASE = SEER of baseline unit*

*= Based on IECC 2012 or IECC 2015 for the installed capacity. See table above.*

*SEEREXIST = SEER of the existing unit.*

*= Actual*

*HSPFEE = HSPF of efficient unit*

*= Actual Installed*

*HSPFBASE = HSPF of baseline unit*

*= Based on IECC 2012 or IECC 2015 for the installed capacity. See table above.*

*HSPFEXIST = HSPF of the existing unit.*

*= Actual*

*IEEREE = IEER of efficient unit*

*= Actual Installed*

*IEERBASE = IEER of baseline unit*

*= Based on IECC 2012 or IECC 2015 for the installed capacity. See table above.*

*IEEREXIST = IEER of the existing unit.*

*= Actual*

*COPEE = COP of efficient unit*

*= Actual Installed*

*COPBASE = COP of baseline unit*

*= Based on IECC 2012 or IECC 2015 for the installed capacity. See table above.*

*COPEXIST = COP of the existing unit.*

*= Actual*

*EERBASE = EER of baseline unit*

*= Based on IECC 2012 or 2015 for the installed capacity. See table above.*

*EEREE = EER of efficient unit (If the actual EER is unknown, it may be approximated by using the following equation: EER = SEER/1.2)*

*= Actual installed*

*EEREXIST = EER of existing unit*

*= Actual*

*3412 = Conversion factor (Btu/kWh)*

*HOURSCOOL = Full load cooling hours[[913]](#footnote-913)*

*= If actual full load cooling hours are unknown, see table “Full Load Cooling Hours by Location and Equipment Capacity” below. Otherwise, use site specific full load cooling hours information.*

*HOURSHEAT = Full load heating hours*

*= If actual full load heating hours are unknown, see table “Full Load Heating Hours by Location and Building Type” below. Otherwise, use site specific full load heating hours information.*

**Full Load Cooling Hours by Location and Equipment Capacity[[914]](#footnote-914)**

| **City, State** | **HOURSCool by Equipment Capacity** | |
| --- | --- | --- |
| **< 135 kBtu/h** | **>= 135 kBtu/h** |
| Dover, DE | 910 | 1,636 |
| Wilmington, DE | 980 | 1,762 |
| Baltimore, MD | 1,014 | 1,823 |
| Hagerstown, MD | 885 | 1,591 |
| Patuxent River, MD | 1,151 | 2,069 |
| Salisbury, MD | 1,008 | 1,812 |
| Washington D.C. | 1,275 | 2,292 |

**Full Load Heating Hours by Location and Building Type** [[915]](#footnote-915)

| **Building Type** | **Dover, DE** | **Wilmington, DE** | **Baltimore, MD** | **Hagerstown, MD** | **Patuxent River, MD** | **Salisbury, MD** | **Washington D.C.** |
| --- | --- | --- | --- | --- | --- | --- | --- |
| Assembly | 676 | 692 | 620 | 657 | 451 | 507 | 559 |
| Auto Repair | 2,292 | 2,344 | 2,106 | 2,229 | 1,543 | 1,728 | 1,901 |
| Big Box Retail | 286 | 298 | 241 | 271 | 107 | 151 | 192 |
| Fast Food Restaurant | 957 | 983 | 866 | 926 | 590 | 681 | 766 |
| Full Service Restaurant | 988 | 1,016 | 891 | 956 | 597 | 694 | 784 |
| Grocery | 286 | 298 | 241 | 271 | 107 | 151 | 192 |
| Light Industrial | 867 | 885 | 803 | 845 | 608 | 672 | 732 |
| Motel | 659 | 667 | 632 | 650 | 547 | 575 | 601 |
| Primary School | 978 | 993 | 926 | 960 | 767 | 819 | 868 |
| Religious Worship | 750 | 754 | 737 | 746 | 698 | 711 | 723 |
| Small Office | 511 | 524 | 466 | 496 | 329 | 374 | 416 |
| Small Retail | 657 | 674 | 595 | 636 | 410 | 471 | 528 |
| Warehouse | 556 | 576 | 487 | 533 | 278 | 347 | 411 |
| Other | 805 | 823 | 739 | 783 | 541 | 606 | 667 |

For example, a 5 ton unitary AC, split system with SEER rating of 15.0 installed in Baltimore:

ΔkWh = (60,000/1000) \* (1/13 - 1/15) \* 1014

= 624 kWh

**Summer Coincident Peak kW Savings Algorithm**

Time of Sale:

ΔkW = (Btu/hCOOL/1000) \* ((1/EERBASE) - (1/EEREE)) \* CF

Early Replacement:

ΔkW for remaining life of existing unit (i.e., measure life less the age of the existing equipment):

= (Btu/hCOOL/1000) \* ((1/EEREXIST) – (1/EEREE)) \* CF

ΔkW for remaining measure life (i.e., measure life less the remaining life of existing unit):

= (Btu/hCOOL/1000) \* ((1/EERBASE) – (1/EEREE)) \* CF

*Where:*

*CFPJM = PJM Summer Peak Coincidence Factor (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.360 for units <135 kBtu/h and 0.567 for units ≥135 kBtu/h[[916]](#footnote-916)*

*CFSSP = Summer System Peak Coincidence Factor (hour ending 5pm on hottest summer weekday)*

*= 0.588 for units <135 kBtu/h and 0.874 for units ≥135 kBtu/h[[917]](#footnote-917)*

For example, a 5 ton unitary AC, split system with EER rating of 12.5 installed in Baltimore estimating PJM summer peak coincidence:[[918]](#footnote-918)

ΔkW = (60,000/1000) \* (1/10.8 – 1/12.5) \* 0.360

= 0.27 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost[[919]](#footnote-919)**

The incremental costs are shown in the tables below for time of sale and new construction scenarios. If the measure is an early replacement, the full installed cost of the efficient unit should be used as the incremental cost and determined on a site-specific basis. For the purposes of cost-effectiveness screening, there can also be a deferred cost credit given at the end of the existing equipment’s remaining life to account for when the customer would have had to purchase new equipment if they had not performed the early replacement.[[920]](#footnote-920)

**Air-Cooled Unitary Air Conditioners[[921]](#footnote-921)**

| **Size Category** | **Efficient Condition (CEE Tier 1)** | **Efficient Condition (CEE Tier 2)** |
| --- | --- | --- |
| >=65,000 Btu/h and <135,000 | $62.96/ton | $125.92/ton |
| >=135,000 Btu/h and <240,000 Btu/h | $62.96/ton | $125.92/ton |
| >=240,000 Btu/h and <760,000 Btu/h | $18.78/ton | $37.56/ton |

**Air-Source Unitary Heat Pumps[[922]](#footnote-922)**

| **Size Category** | **Efficient Condition (CEE Tier 1)** | **Efficient Condition (CEE Tier 2)** |
| --- | --- | --- |
| <65,000 Btu/h | $443/unit | $886/ton |
| >=65,000 Btu/h and <135,000 | $62.96/ton | $125.92/ton |
| >=135,000 Btu/h and <240,000 Btu/h | $62.96/ton | $125.92/ton |
| >=240,000 Btu/h and <760,000 Btu/h | $18.78/ton | $37.56/ton |

**Measure Life**

The measure life is assumed to be 15 years.[[923]](#footnote-923)

**Operation and Maintenance Impacts**

n/a

**Ductless Mini-Split Heat Pump (DMSHP)**

**Unique Measure Code(s): CI\_HV\_TOS\_DMSHP\_0615, CI\_HV\_EREP\_DMSHP\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of new ENERGY STAR rated ductless “mini-split” heat pump(s) (DMSHP). A ductless mini-split heat pump is a type of heat pump with an outdoor condensing unit connected via refrigerant line to one or more indoor evaporator coils. Ductless mini-split heat pumps deliver cooling at the same or higher efficiency as standard central AC units, but can also deliver heat. Further, since the units do not require ductwork, they avoid duct losses.

**Definition of Baseline Condition**

This measure assumes installation in a small commercial space.

**Time of Sale or New Construction:** Since the efficient unit is unducted, it is assumed that the baseline equipment will also be unducted. In such cases, or if the baseline condition for an early replacement is unknown, it is assumed that the baseline equipment is a window AC unit with a gas hot water boiler feeding hot water baseboards. The assumed baseline efficiency is that of equipment minimally compliant federal efficiency standards.

**Early Replacement:** The baseline condition for the Early Replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit, and the new baseline as defined above for the remainder of the measure life.[[924]](#footnote-924) If the space is currently uncooled, it is assumed that the building owner would have installed cooling by other means and should therefore be treated as a lost opportunity measure with a window AC baseline.

**Definition of Efficient Condition**

The efficient equipment is assumed to be an ENERGY STAR qualified ductless mini-split heat pump, with a minimum 14.5 SEER, 12.0 EER, and 8.2 HSPF. If the rated efficiency of the actual unit is higher than the ENERGY STAR minimum requirements, the actual efficiency ratings should be used in the calculation.

**Baseline and Efficient Levels by Unit Capacity**

If the measure is a retrofit, the actual efficiencies of the baseline heating and cooling equipment should be used. If it is a market opportunity, the baseline efficiency should be selected from the tables below.

**Baseline Window AC Efficiency[[925]](#footnote-925)**

| **Equipment Type** | **Capacity (Btu/h)** | **Federal Standard with louvered sides (CEER)** | **Federal Standard without louvered sides (CEER)** |
| --- | --- | --- | --- |
| Without Reverse Cycle | < 8,000 | 11.0 | 10.0 |
| 8,000 to 10,999 | 10.9 | 9.6 |
| 11,000 to 13,999 | 10.9 | 9.5 |
| 14,000 to 19,999 | 10.7 | 9.3 |
| 20,000 to 24,999 | 9.4 | 9.4 |
| With Reverse Cycle | <14,000 | 9.8 | 9.3 |
| 14,000 to 19,999 | 9.8 | 8.7 |
| >=20,000 | 9.3 | 8.7 |
| Casement-Only | All | 9.5 | |
| Casement-Slider | All | 10.4 | |

**Baseline Central AC Efficiency**

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Equipment Type** |  | **Capacity (Btu/h)** | **SEER** | **EER** |
| Split System Air Conditioners[[926]](#footnote-926) | | All | 13 | 11 |
| Packaged Air Conditioners[[927]](#footnote-927) | | All | 14 | 11.5 |
| Packaged Air Source Heat Pumps[[928]](#footnote-928) | | All | 14 | 11.5 |

**Baseline Heating System Efficiency**

| **Equipment Type** | **Efficiency Metric** | **Efficiency** |
| --- | --- | --- |
| Gas Boiler[[929]](#footnote-929) | AFUE | 82% |
| Air Source Heat Pump – Split System[[930]](#footnote-930) | HSPF | 8.2 |
| Air Source Heat Pump - Packaged | HSPF | 8.0 |
| Electric Resistance[[931]](#footnote-931) | HSPF | 3.41 |

**Annual Energy Savings Algorithm**

ΔkWhtotal = ΔkWhcool + ΔkWhheat

ΔkWhcool = CCAP x (1/SEERbase – 1/SEERee) x EFLHcool

ΔkWhheat [[932]](#footnote-932)=HCAP x (ELECHEAT/HSPFbase – 1/HSPFee) x EFLHheat

*Where:*

*CCAP = Cooling capacity of DMSHP unit, in kBtu/hr*

*SEERbase = SEER of baseline unit. If unknown, use 9.8[[933]](#footnote-933).*

*SEERee = SEER of actual DMSHP. If unknown, use ENERGY STAR minimum of 14.5.*

*EFLHcool = Full load hours for cooling equipment. See table below for default values.*

*HCAP = Heating capacity of DMSHP unit, in kBtu/hr*

*ELECHEAT = 1 if the baseline is electric heat, 0 otherwise. If unknown, assume the baseline is a gas boiler, so ELECHEAT = 0.*

*HSPFbase  = HSPF of baseline equipment. See table above[[934]](#footnote-934).*

*HSPFee  = HSPF of actual DMSHP. If unknown, use ENERGY STAR minimum of 8.2.*

*EFLHheat = Full load hours for heating equipment. See table below for default values.*

**Full Load Cooling Hours by Location and Equipment Capacity[[935]](#footnote-935)**

| **City, State** | **HOURS by Equipment Capacity** | |
| --- | --- | --- |
| **< 135 kBtu/h** | **>= 135 kBtu/h** |
| Dover, DE | 910 | 1,636 |
| Wilmington, DE | 980 | 1,762 |
| Baltimore, MD | 1,014 | 1,823 |
| Hagerstown, MD | 885 | 1,591 |
| Patuxent River, MD | 1,151 | 2,069 |
| Salisbury, MD | 1,008 | 1,812 |
| Washington D.C. | 1,275 | 2,292 |

**Heating Full Load Hours**[[936]](#footnote-936)

| **Building Type** | **Dover, DE** | **Wilmington, DE** | **Baltimore, MD** | **Hagerstown, MD** | **Patuxent River, MD** | **Salisbury, MD** | **Washington D.C.** |
| --- | --- | --- | --- | --- | --- | --- | --- |
| Assembly | 676 | 692 | 620 | 657 | 451 | 507 | 559 |
| Auto Repair | 2,292 | 2,344 | 2,106 | 2,229 | 1,543 | 1,728 | 1,901 |
| Big Box Retail | 286 | 298 | 241 | 271 | 107 | 151 | 192 |
| Fast Food Restaurant | 957 | 983 | 866 | 926 | 590 | 681 | 766 |
| Full Service Restaurant | 988 | 1,016 | 891 | 956 | 597 | 694 | 784 |
| Grocery | 286 | 298 | 241 | 271 | 107 | 151 | 192 |
| Light Industrial | 867 | 885 | 803 | 845 | 608 | 672 | 732 |
| Motel | 659 | 667 | 632 | 650 | 547 | 575 | 601 |
| Primary School | 978 | 993 | 926 | 960 | 767 | 819 | 868 |
| Religious Worship | 750 | 754 | 737 | 746 | 698 | 711 | 723 |
| Small Office | 511 | 524 | 466 | 496 | 329 | 374 | 416 |
| Small Retail | 657 | 674 | 595 | 636 | 410 | 471 | 528 |
| Warehouse | 556 | 576 | 487 | 533 | 278 | 347 | 411 |
| Other | 805 | 823 | 739 | 783 | 541 | 606 | 667 |

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = CCAP x (1/EERbase – 1/EERee) x CF

*Where:*

*EERbase = EER of baseline unit. If unknown, use 9.8[[937]](#footnote-937).*

*EERee = EER of actual DMSHP. If unknown, use ENERGY STAR minimum of 12.0.*

*CFPJM = PJM Summer Peak Coincidence Factor (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.360 for units <135 kBtu/h and 0.567 for units ≥135 kBtu/h[[938]](#footnote-938)*

*CFSSP = Summer System Peak Coincidence Factor (hour ending 5pm on hottest summer weekday)*

*= 0.588 for units <135 kBtu/h and 0.874 for units ≥135 kBtu/h[[939]](#footnote-939)*

**Annual Fossil Fuel Savings Algorithm**

Note: Only applies if retrofit space is heated with fossil fuels. Negative value denotes *increased* fossil fuel consumption.

ΔMMBtu = HCAP x EFLHheat / AFUE / 1,000

*Where:*

*EFLHheat = Full load hours for heating equipment. See table above.*

*AFUE = AFUE of baseline equipment. If unknown use 82%[[940]](#footnote-940).*

**Incremental Cost**

The full installed cost of the ductless mini-split system is shown below[[941]](#footnote-941).

| **Capacity (kBtu/h)** | **Efficiency** | | | |
| --- | --- | --- | --- | --- |
| **13 SEER** | **18 SEER** | **21 SEER** | **26 SEER** |
| 9 | $2,733 | $3,078 | $3,236 | $3,460 |
| 12 | $2,803 | $3,138 | $3,407 | $3,363 |
| 18 | $3,016 | $3,374 | $3,640 | N/A |
| 24 | $3,273 | $3,874 | N/A | N/A |

The full installed cost of the baseline equipment is shown below.

| **Unit** | **Cost** |
| --- | --- |
| Window AC[[942]](#footnote-942) | $170/unit |
| Gas furnace[[943]](#footnote-943) | $1,606/unit |
| Electric Baseboard[[944]](#footnote-944) | $0 |

If the measure is a time of sale or new construction project, subtract the costs of the baseline heating and cooling equipment from the appropriate cost of the DSMHP, as shown in the first table above. If the measure is an early replacement, use the full installed cost of the DMSHP as the incremental cost. For the purposes of cost-effectiveness screening, there can also be a deferred cost credit given at the end of the existing equipment’s remaining life to account for when the customer would have had to purchase new equipment if they had not performed the early replacement.

**Measure Life**

The measure life for a DSMHP is 18 years.[[945]](#footnote-945)

**Operation and Maintenance Impacts**

n/a

**Variable Frequency Drive (VFD) for HVAC\***

**Unique Measure Code(s): CI\_MO\_RTR\_VFDRIVE\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

This measure defines savings associated with installing a variable frequency drive on a motor of 200 hp or less for the following HVAC applications: supply fans, return fans, exhaust fans, chilled water pumps, and boiler feedwater pumps. The fan or pump speed will be controlled to maintain the desired system pressure. The application must have a load that varies and proper controls (i.e., Two–way valves, VAV boxes) must be installed.

**Definition of Baseline Condition**

The baseline condition is a motor, 200 hp or less, without a VFD control.

**Definition of Efficient Condition**

The efficient condition is a motor, 200 hp or less, with a VFD control.

**Annual Energy Savings Algorithm[[946]](#footnote-946)**

**HVAC Fan Applications**

*Where:*

*∆kWhFAN = Fan-only annual energy savings*

*IEENERGY = HVAC interactive effects factor for energy*

*= Assume 0%[[947]](#footnote-947)*

*∆kWhFAN = Baseline annual energy consumption (kWh/yr)*

*∆kWhRETRO = Retrofit annual energy consumption (kWh/yr)*

*0.746 = Conversion factor for hp to kWh*

*HP = Nominal horsepower of controlled motor*

*= Actual*

*LF = Load Factor; Motor Load at Fan Design CFM*

*= If actual load factor is unknown, assume 65%.*

*ηMOTOR = Installed nominal/nameplate motor efficiency*

*= Actual efficiency*

*RHRSBASE = Annual operating hours for fan motor based on building type*

*= If actual hours are unknown, assume defaults in VFD Operating Hours by Application and Building Type table below.*

*%FF = Percentage of run-time spent within a given flow fraction range*

*= If actual values unknown, see Default Fan Duty Cycle table below for default values*

**Default Fan Duty Cycle**

|  |  |
| --- | --- |
| **Flow Fraction (% of design cfm)** | **Percent of Time at Flow Fraction (%FF)** |
| 0% to 10% | 0.0% |
| 10% to 20% | 1.0% |
| 20% to 30% | 5.5% |
| 30% to 40% | 15.5% |
| 40% to 50% | 22.0% |
| 50% to 60% | 25.0% |
| 60% to 70% | 19.0% |
| 70% to 80% | 8.5% |
| 80% to 90% | 3.0% |
| 90% to 100% | 0.5% |

*PLRBASE = Part load ratio for a given flow fraction range based on the baseline flow control type*

*PLRRETRO = Part load ratio for a given flow fraction range based on the retrofit flow control type*

**Part Load Ratios by Control and Fan Type and Flow Fraction (PLR)**

|  |  |  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- |
| **Control Type** | **Flow Fraction** | | | | | | | | | |
| **10%** | **20%** | **30%** | **40%** | **50%** | **60%** | **70%** | **80%** | **90%** | **100%** |
| No Control or Bypass Damper | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 |
| Discharge Dampers | 0.46 | 0.55 | 0.63 | 0.70 | 0.77 | 0.83 | 0.88 | 0.93 | 0.97 | 1.00 |
| Outlet Damper, BI & Airfoil Fans | 0.53 | 0.53 | 0.57 | 0.64 | 0.72 | 0.80 | 0.89 | 0.96 | 1.02 | 1.05 |
| Inlet Damper Box | 0.56 | 0.60 | 0.62 | 0.64 | 0.66 | 0.69 | 0.74 | 0.81 | 0.92 | 1.07 |
| Inlet Guide Vane, BI & Airfoil Fans | 0.53 | 0.56 | 0.57 | 0.59 | 0.60 | 0.62 | 0.67 | 0.74 | 0.85 | 1.00 |
| Inlet Vane Dampers | 0.38 | 0.40 | 0.42 | 0.44 | 0.48 | 0.53 | 0.60 | 0.70 | 0.83 | 0.99 |
| Outlet Damper, FC Fans | 0.22 | 0.26 | 0.30 | 0.37 | 0.45 | 0.54 | 0.65 | 0.77 | 0.91 | 1.06 |
| Eddy Current Drives | 0.17 | 0.20 | 0.25 | 0.32 | 0.41 | 0.51 | 0.63 | 0.76 | 0.90 | 1.04 |
| Inlet Guide Vane, FC Fans | 0.21 | 0.22 | 0.23 | 0.26 | 0.31 | 0.39 | 0.49 | 0.63 | 0.81 | 1.04 |
| VFD with duct static pressure controls | 0.09 | 0.10 | 0.11 | 0.15 | 0.20 | 0.29 | 0.41 | 0.57 | 0.76 | 1.01 |
| VFD with low/no duct static pressure (<1" w.g.) | 0.05 | 0.06 | 0.09 | 0.12 | 0.18 | 0.27 | 0.39 | 0.55 | 0.75 | 1.00 |

**HVAC Pump Applications**

ΔkWh = ((HP \* 0.746 \* LF) / ηMOTOR) \* RHRSBASE \* ESF

*Where:*

*HP = Nominal horsepower of controlled motor*

*= Actual*

*0.746 = Conversion factor for hp to kWh*

*LF = Load Factor; Motor Load at Fan Design CFM*

*= If actual load factor is unknown, assume 65%.*

*ηMOTOR = Installed nominal/nameplate motor efficiency*

*= Actual efficiency*

*RHRSBASE = Annual operating hours for fan motor based on building type*

*= If actual hours are unknown, assume defaults in VFD Operating Hours by Application and Building Type table below.*

*ESF = Energy Savings Factor (see table “Energy and Demand Savings Factors” below)*

**Summer Coincident Peak kW Savings Algorithm**

**HVAC Fan Applications**

ΔkW = ΔkWFAN \* (1 + IEDEMAND)

ΔkWFAN = ΔkWBASE - ΔkWRETRO

ΔkWBASE = (0.746 \* HP \* LF / ηMOTOR) \* PLRBASE, PEAK

ΔkWRETRO = (0.746 \* HP \* LF / ηMOTOR) \* PLRRETRO, PEAK

*Where:*

*∆kWFAN = Fan-only annual demand savings (kW)*

*IEENERGY = HVAC interactive effects factor for demand*

*= If unknown, assume 15.7%*

*∆kWFAN = Baseline summer coincident peak demand (kW)*

*∆kWRETRO = Retrofit summer coincident peak demand (kW)*

*PLRBASE, PEAK = PLR for the average flow fraction during summer peak period for baseline flow control type (default average flow fraction during peak period = 90%)*

*PLRRETRO, PEAK = PLR for the average flow fraction during summer peak period for retrofit flow control type (default average flow fraction during peak period = 90%)*

**HVAC Pump Applications**

ΔkW = ((HP \* 0.746 \* LF) / *ηMOTOR*) \* DSF \* CF

*Where:*

*DSF = Demand Savings Factor (see table “Energy and Demand Savings Factors” below)*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.55 [[948]](#footnote-948)*

**VFD Operating Hours by Application and Building Type (RHRSBASE)[[949]](#footnote-949)**

| **Facility Type** | **Fan Motor Hours** | **Chilled Water Pumps** | **Heating Pumps** |
| --- | --- | --- | --- |
| Auto Related | 4,056 | 1,878 | 5,376 |
| Bakery | 2,854 | 1,445 | 5,376 |
| Banks, Financial Centers | 3,748 | 1,767 | 5,376 |
| Church | 1,955 | 1,121 | 5,376 |
| College – Cafeteria | 6,376 | 2,713 | 5,376 |
| College - Classes/Administrative | 2,586 | 1,348 | 5,376 |
| College - Dormitory | 3,066 | 1,521 | 5,376 |
| Commercial Condos | 4,055 | 1,877 | 5,376 |
| Convenience Stores | 6,376 | 2,713 | 5,376 |
| Convention Center | 1,954 | 1,121 | 5,376 |
| Court House | 3,748 | 1,767 | 5,376 |
| Dining: Bar Lounge/Leisure | 4,182 | 1,923 | 5,376 |
| Dining: Cafeteria / Fast Food | 6,456 | 2,742 | 5,376 |
| Dining: Family | 4,182 | 1,923 | 5,376 |
| Entertainment | 1,952 | 1,120 | 5,376 |
| Exercise Center | 5,836 | 2,518 | 5,376 |
| Fast Food Restaurants | 6,376 | 2,713 | 5,376 |
| Fire Station (Unmanned) | 1,953 | 1,121 | 5,376 |
| Food Stores | 4,055 | 1,877 | 5,376 |
| Gymnasium | 2,586 | 1,348 | 5,376 |
| Hospitals | 7,674 | 3,180 | 5,376 |
| Hospitals / Health Care | 7,666 | 3,177 | 5,376 |
| Industrial - 1 Shift | 2,857 | 1,446 | 5,376 |
| Industrial - 2 Shift | 4,730 | 2,120 | 5,376 |
| Industrial - 3 Shift | 6,631 | 2,805 | 5,376 |
| Laundromats | 4,056 | 1,878 | 5,376 |
| Library | 3,748 | 1,767 | 5,376 |
| Light Manufacturers | 2,857 | 1,446 | 5,376 |
| Lodging (Hotels/Motels) | 3,064 | 1,521 | 5,376 |
| Mall Concourse | 4,833 | 2,157 | 5,376 |
| Manufacturing Facility | 2,857 | 1,446 | 5,376 |
| Medical Offices | 3,748 | 1,767 | 5,376 |
| Motion Picture Theatre | 1,954 | 1,121 | 5,376 |
| Multi-Family (Common Areas) | 7,665 | 3,177 | 5,376 |
| Museum | 3,748 | 1,767 | 5,376 |
| Nursing Homes | 5,840 | 2,520 | 5,376 |
| Office (General Office Types) | 3,748 | 1,767 | 5,376 |
| Office/Retail | 3,748 | 1,767 | 5,376 |
| Parking Garages & Lots | 4,368 | 1,990 | 5,376 |
| Penitentiary | 5,477 | 2,389 | 5,376 |
| Performing Arts Theatre | 2,586 | 1,348 | 5,376 |
| Police / Fire Stations (24 Hr) | 7,665 | 3,177 | 5,376 |
| Post Office | 3,748 | 1,767 | 5,376 |
| Pump Stations | 1,949 | 1,119 | 5,376 |
| Refrigerated Warehouse | 2,602 | 1,354 | 5,376 |
| Religious Building | 1,955 | 1,121 | 5,376 |
| Residential (Except Nursing Homes) | 3,066 | 1,521 | 5,376 |
| Restaurants | 4,182 | 1,923 | 5,376 |
| Retail | 4,057 | 1,878 | 5,376 |
| School / University | 2,187 | 1,205 | 5,376 |
| Schools (Jr./Sr. High) | 2,187 | 1,205 | 5,376 |
| Schools (Preschool/Elementary) | 2,187 | 1,205 | 5,376 |
| Schools (Technical/Vocational) | 2,187 | 1,205 | 5,376 |
| Small Services | 3,750 | 1,768 | 5,376 |
| Sports Arena | 1,954 | 1,121 | 5,376 |
| Town Hall | 3,748 | 1,767 | 5,376 |
| Transportation | 6,456 | 2,742 | 5,376 |
| Warehouse (Not Refrigerated) | 2,602 | 1,354 | 5,376 |
| Waste Water Treatment Plant | 6,631 | 2,805 | 5,376 |
| Workshop | 3,750 | 1,768 | 5,376 |

**Energy and Demand Savings Factors[[950]](#footnote-950)**

|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- |
| |  |  |  | | --- | --- | --- | | HVAC Pump VFD Savings Factors | | | | **System** | **ESF** | **DSF** | | Chilled Water Pump | 0.633 | 0.460 | | Hot Water Pump | 0.652 | 0.000 | |  |  |

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure varies by controlled motor hp. See table “VFD Incremental Costs” below.

**VFD Incremental Costs[[951]](#footnote-951)**

|  |  |
| --- | --- |
| **Rated Motor Horsepower (HP)** | **Total Installed Costs** |
| 5 | $2,125 |
| 15 | $3,193 |
| 25 | $4,260 |
| 50 | $6,448 |
| 75 | $8,407 |
| 100 [[952]](#footnote-952) | $10,493 |
| 200 [[953]](#footnote-953) | $17,266 |

**Measure Life**

The measure life is assumed to be 15 years for HVAC applications.[[954]](#footnote-954)

**Operation and Maintenance Impacts**

n/a

**Electric Chillers**

**Unique Measure Code: CI\_HV\_TOS\_ELCHIL\_0615, CI\_HV\_EREP\_ELCHIL\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a new high-efficiency electric water chilling package in place of an existing chiller or a new standard efficiency chiller of the same capacity. This measure applies to time of sale, new construction, and early replacement opportunities.

**Definition of Baseline Condition**

**Time of Sale or New Construction**: For Washington, D.C. and Delaware, the baseline condition is a standard efficiency water chilling package equal to the requirements presented in the International Energy Conservation Code 2012 (IECC 2012), Table C403.2.3(7). For Maryland, the baseline condition is a standard efficiency water chilling package equal to the requirements presented in the International Energy Conservation Code 2015 (IECC 2015), Table C403.2.3(7).

**Early Replacement:** The baseline condition for the Early Replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit, and the new baseline as defined above for the remainder of the measure life.

**Definition of Efficient Condition**

For Washington, D.C. and Delaware, the efficient condition is a high-efficiency electric water chilling package exceeding the requirements presented in the International Energy Conservation Code 2012 (IECC 2012), Table C403.2.3(7). For Maryland, the efficient condition is a high-efficiency electric water chilling package exceeding the requirements presented in the International Energy Conservation Code 2015 (IECC 2015), Table C403.2.3(7).

**Annual Energy Savings Algorithm**

Time of Sale and New Construction:

ΔkWh = TONS \* (IPLVbase - IPLVee) \* HOURS

Early Replacement[[955]](#footnote-955):

ΔkWh for remaining life of existing unit (i.e., measure life less the age of the existing equipment):

= TONS \* (IPLVexist - IPLVee) \* HOURS

ΔkWh for remaining measure life (i.e., measure life less the remaining life of existing unit):

= TONS \* (IPLVbase - IPLVee) \* HOURS

*Where:*

*TONS = Total installed capacity of the water chilling package[tons]*

*= Actual Installed*

*IPLVexist = Integrated Part Load Value (IPLV)[[956]](#footnote-956) of the existing equipment [kW/ton]*

*IPLVbase = Integrated Part Load Value (IPLV) of the new baseline equipment [kW/ton]*

*= Varies by equipment type and capacity. See “Time of Sale Baseline Equipment Efficiency” table in the “Reference Tables” section below[[957]](#footnote-957)*

*IPLVee = Integrated Part Load Value (IPLV) of the efficient equipment [kW/ton]*

*= Actual Installed*

*HOURS = Full load cooling hours*

*= If actual full load cooling hours are unknown, assume values presented in table “Default Electric Chiller Full Load Cooling Hours” in the “Reference Tables” section below. Otherwise, use site specific full load cooling hours information.*

**Summer Coincident Peak kW Savings Algorithm**

Time of Sale and New Construction:

ΔkW = TONS \* (Full\_Loadbase - Full\_Loadee) \* CF

Early replacement:

ΔkW for remaining life of existing unit (i.e., measure life less the age of the existing equipment):

= TONS \* (Full\_Loadexist - Full\_Loadee) \* CF

ΔkW for remaining measure life (i.e., measure life less the remaining life of existing unit):

= TONS \* (Full\_Loadbase - Full\_Loadee) \* CF

*Where:*

*Full\_Loadexist = Full load efficiency of the existing equipment [kW/ton]*

*Full\_Loadbase = Full load efficiency of the baseline equipment [kW/ton]*

*= Varies by equipment type and capacity. See “Time of Sale Baseline Equipment Efficiency” table in the “Reference Tables” section below[[958]](#footnote-958)*

*Full\_Loadee = Full load efficiency of the efficient equipment*

*= Actual Installed [kW/ton]*

*CFPJM = PJM Summer Peak Coincidence Factor (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.808[[959]](#footnote-959)*

*CFSSP = Summer System Peak Coincidence Factor (hour ending 5pm on hottest summer weekday)*

*= 0.923[[960]](#footnote-960)*

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental costs for chillers are shown in the tables below for time of sale and new construction scenarios.[[961]](#footnote-961) Because of differences in baselines due to differing code requirements by jurisdiction, the incremental costs vary by jurisdiction. If the measure is an early replacement, the full installed cost of the efficient unit should be used as the incremental cost and determined on a site-specific basis. For the purposes of cost-effectiveness screening, there can also be a deferred cost credit given at the end of the existing equipment’s remaining life to account for when the customer would have had to purchase new equipment if they had not performed the early replacement.

**Air-Cooled Chiller Incremental Costs ($/Ton) for Washington, D.C. and Delaware**

| **Capacity (Tons)** | **Baseline EER** | **Efficient EER** | | | |
| --- | --- | --- | --- | --- | --- |
| **9.9** | **10.2** | **10.52** | **10.7** |
| 50 | 9.562 | $258 | $486 | $730 | $867 |
| 100 | 9.562 | $128 | $243 | $364 | $433 |
| 150 | 9.562 | $86 | $162 | $244 | $289 |
| 200 | 9.562 | $53 | $99 | $149 | $177 |
| 400 | 9.562 | $26 | $50 | $74 | $88 |

**Air-Cooled Chiller Incremental Costs ($/Ton) for Maryland**

| **Capacity (Tons)** | **Baseline EER** | **Efficient EER** | | | |
| --- | --- | --- | --- | --- | --- |
| **9.9** | **10.2** | **10.52** | **10.7** |
| 50 | 10.1 | N/A | $76 | $320 | $457 |
| 100 | 10.1 | N/A | $38 | $159 | $228 |
| 150 | 10.1 | N/A | $25 | $107 | $152 |
| 200 | 10.1 | N/A | $15 | $65 | $93 |
| 400 | 10.1 | N/A | $8 | $32 | $46 |

**Water-Cooled Scroll/Screw Chiller Incremental Costs ($/Ton) for Washington, D.C. and Delaware**

| **Capacity (Tons)** | **Baseline kW/ton** | **Efficient kW/ton** | | | |
| --- | --- | --- | --- | --- | --- |
| **0.72** | **0.68** | **0.64** | **0.60** |
| 50 | 0.78 | $114 | $164 | N/A | N/A |
| 100 | 0.775 | $52 | $77 | N/A | N/A |
| 150 | 0.68 | N/A | N/A | N/A | N/A |
| 200 | 0.68 | N/A | N/A | $61 | $122 |
| 400 | 0.62 | N/A | N/A | N/A | $16 |

**Water-Cooled Scroll/Screw Chiller Incremental Costs ($/Ton) for Maryland**

| **Capacity (Tons)** | **Baseline kW/ton** | **Efficient kW/ton** | | | |
| --- | --- | --- | --- | --- | --- |
| **0.72** | **0.68** | **0.64** | **0.60** |
| 50 | 0.75 | $57 | $107 | N/A | N/A |
| 100 | 0.72 | $0 | $25 | N/A | N/A |
| 150 | 0.66 | N/A | N/A | N/A | N/A |
| 200 | 0.66 | N/A | N/A | $31 | $92 |
| 400 | 0.61 | N/A | N/A | N/A | $8 |

**Water-Cooled Centrifugal Chiller Incremental Costs ($/Ton) for Washington, D.C. and Delaware**

| **Capacity (Tons)** | **Baseline kW/ton** | **Efficient kW/ton** | | |
| --- | --- | --- | --- | --- |
| **0.6** | **0.58** | **0.54** |
| 100 | 0.634 | $62 | $99 | $172 |
| 150 | 0.634 | $42 | $66 | $115 |
| 200 | 0.634 | $31 | $49 | $86 |
| 300 | 0.576 | N/A | N/A | $55 |
| 600 | 0.57 | N/A | N/A | $22 |

**Water-Cooled Centrifugal Chiller Incremental Costs ($/Ton) for Maryland**

| **Capacity (Tons)** | **Baseline kW/ton** | **Efficient kW/ton** | | |
| --- | --- | --- | --- | --- |
| **0.6** | **0.58** | **0.54** |
| 100 | 0.61 | $18 | $55 | $128 |
| 150 | 0.61 | $12 | $36 | $85 |
| 200 | 0.61 | $9 | $27 | $64 |
| 300 | 0.56 | N/A | N/A | $31 |
| 600 | 0.56 | N/A | N/A | $15 |

**Measure Life**

The measure life is assumed to be 23 years[[962]](#footnote-962).

**Operation and Maintenance Impacts**

n/a

**Reference Tables**

**Time of Sale Baseline Equipment Efficiency for Washington, D.C. and Delaware**[[963]](#footnote-963)

| **Equipment Type** | **Size Category** | **Units** | **Path Aa** | | **Path Ba** | |
| --- | --- | --- | --- | --- | --- | --- |
| **Full Load** | **IPLV** | **Full Load** | **IPLV** |
| Air-Cooled Chillers | <150 tons | EER | ≥9.562 | ≥12.500 | NA | NA |
| ≥150 tons | EER | ≥9.562 | ≥12.750 | NA | NA |
| Water Cooled, Electrically Operated, Positive Displacement | <75 tons | kW/ton | ≤0.780 | ≤0.630 | ≤0.800 | ≤0.600 |
| ≥75 tons and <150 tons | kW/ton | ≤0.775 | ≤0.615 | ≤0.790 | ≤0.586 |
| ≥150 tons and <300 tons | kW/ton | ≤0.680 | ≤0.580 | ≤0.718 | ≤0.540 |
| ≥300 tons | kW/ton | ≤0.620 | ≤0.540 | ≤0.639 | ≤0.490 |
| Water Cooled, Electrically Operated, Centrifugal | <150 tons | kW/ton | ≤0.634 | ≤0.596 | ≤0.639 | ≤0.450 |
| ≥150 tons and <300 tons | kW/ton | ≤0.634 | ≤0.596 | ≤0.639 | ≤0.450 |
| ≥300 tons and <600 tons | kW/ton | ≤0.576 | ≤0.549 | ≤0.600 | ≤0.400 |
| ≥600 tons | kW/ton | ≤0.570 | ≤0.539 | ≤0.590 | ≤0.400 |

a. Compliance with IECC 2012 can be obtained by meeting the minimum requirements of Path A or B. However, both the full load and IPLV must be met to fulfill the requirements of Path A or B.

**Time of Sale Baseline Equipment Efficiency for Maryland**[[964]](#footnote-964)

| **Equipment Type** | **Size Category** | **Units** | **Path Aa** | | **Path Ba** | |
| --- | --- | --- | --- | --- | --- | --- |
| **Full Load** | **IPLV** | **Full Load** | **IPLV** |
| Air-Cooled Chillers | <150 tons | EER | ≥10.100 | ≥13.700 | ≥9.700 | ≥15.800 |
| ≥150 tons | EER | ≥10.100 | ≥14.000 | ≥9.700 | ≥16.100 |
| Water Cooled, Electrically Operated, Positive Displacement | <75 tons | kW/ton | ≤0.750 | ≤0.600 | ≤0.780 | ≤0.500 |
| ≥75 tons and <150 tons | kW/ton | ≤0.720 | ≤0.560 | ≤0.750 | ≤0.490 |
| ≥150 tons and <300 tons | kW/ton | ≤0.660 | ≤0.540 | ≤0.680 | ≤0.440 |
| ≥300 tons and <600 tons | kW/ton | ≤0.610 | ≤0.520 | ≤0.625 | ≤0.410 |
| ≥600 tons | kW/ton | ≤0.560 | ≤0.500 | ≤0.585 | ≤0.380 |
| Water Cooled, Electrically Operated, Centrifugal | <150 tons | kW/ton | ≤0.610 | ≤0.550 | ≤0.695 | ≤0.440 |
| ≥150 tons and <300 tons | kW/ton | ≤0.610 | ≤0.550 | ≤0.635 | ≤0.400 |
| ≥300 tons and <400 tons | kW/ton | ≤0.560 | ≤0.520 | ≤0.595 | ≤0.390 |
| ≥400 tons and <600 tons | kW/ton | ≤0.560 | ≤0.500 | ≤0.585 | ≤0.380 |
| ≥600 tons | kW/ton | ≤0.560 | ≤0.500 | ≤0.585 | ≤0.380 |

a. Compliance with IECC 2015 can be obtained by meeting the minimum requirements of Path A or B. However, both the full load and IPLV must be met to fulfill the requirements of Path A or B.

**Default Electric Chiller Full Load Cooling Hours**[[965]](#footnote-965)

| **Building Type** | **System Typea** | **Dover, DE** | **Wilmington, DE** | **Baltimore, MD** | **Hagerstown, MD** | **Patuxent River, MD** | **Salisbury, MD** | **Washington D.C.** |
| --- | --- | --- | --- | --- | --- | --- | --- | --- |
| Community College | CAV w/o economizer | 1,010 | 1,048 | 1,121 | 1,044 | 1,202 | 1,117 | 1,274 |
| Community College | CAV w/ economizer | 752 | 781 | 836 | 777 | 897 | 833 | 952 |
| Community College | VAV w/ economizer | 585 | 607 | 649 | 605 | 695 | 647 | 736 |
| High School | CAV w/o economizer | 819 | 830 | 851 | 829 | 875 | 850 | 896 |
| High School | CAV w/ economizer | 428 | 440 | 463 | 439 | 489 | 462 | 511 |
| High School | VAV w/ economizer | 306 | 316 | 336 | 315 | 359 | 335 | 379 |
| Hospital | CAV w/o economizer | 2,094 | 2,135 | 2,213 | 2,130 | 2,302 | 2,210 | 2,379 |
| Hospital | CAV w/ economizer | 1,307 | 1,341 | 1,406 | 1,338 | 1,479 | 1,403 | 1,543 |
| Hospital | VAV w/ economizer | 1,142 | 1,165 | 1,208 | 1,162 | 1,257 | 1,206 | 1,300 |
| Hotel | CAV w/o economizer | 3,166 | 3,165 | 3,163 | 3,165 | 3,161 | 3,163 | 3,159 |
| Hotel | CAV w/ economizer | 2,972 | 2,972 | 2,971 | 2,972 | 2,971 | 2,971 | 2,971 |
| Hotel | VAV w/ economizer | 2,953 | 2,958 | 2,967 | 2,957 | 2,977 | 2,966 | 2,986 |
| Large Retail | CAV w/o economizer | 1,719 | 1,730 | 1,750 | 1,729 | 1,772 | 1,749 | 1,792 |
| Large Retail | CAV w/ economizer | 987 | 1,011 | 1,057 | 1,009 | 1,109 | 1,055 | 1,155 |
| Large Retail | VAV w/ economizer | 817 | 838 | 877 | 835 | 921 | 875 | 959 |
| Office Building | CAV w/o economizer | 2,162 | 2,193 | 2,252 | 2,189 | 2,318 | 2,249 | 2,377 |
| Office Building | CAV w/ economizer | 700 | 710 | 729 | 709 | 750 | 728 | 768 |
| Office Building | VAV w/ economizer | 670 | 685 | 716 | 684 | 749 | 714 | 779 |
| University | CAV w/o economizer | 1,103 | 1,135 | 1,198 | 1,132 | 1,267 | 1,194 | 1,329 |
| University | CAV w/ economizer | 796 | 822 | 871 | 819 | 925 | 868 | 974 |
| University | VAV w/ economizer | 626 | 645 | 682 | 643 | 724 | 680 | 760 |

a. “CAV” refers to constant air volume systems whereas “VAV” refers to variable air volume systems.

**Gas Boiler**

**Unique Measure Code: CI\_HV\_TOS\_GASBLR\_0614 and CI\_HV\_RTR\_GASBLR\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a high efficiency gas boiler in the place of a standard efficiency gas boiler. This measure applies to time of sale and new construction opportunities.

**Definition of Baseline Condition**

**Time of Sale**: The baseline condition is a gas boiler with efficiency equal to the current federal standards. See the “Time of Sale Baseline Equipment Efficiency” table in the “Reference Tables” section.

**Definition of Efficient Condition**

The efficient condition is a high-efficiency gas boiler of at least 85% AFUE for units <300 kBtu/h and 85% Et for units >300 kBtu/h. See the “Time of Sale Baseline Equipment Efficiency” table in the “Reference Tables” section.

**Annual Energy Savings Algorithm**

n/a

**Summer Coincident Peak kW Savings Algorithm**

n/a

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = CAP \* HOURS \* (1/EFFbase - 1/EFFee) / 1,000,000

*Where:*

*CAP = Equipment capacity [Btu/h]*

*= Actual Installed*

HOURS *= Full Load Heating Hours*

*= See “Heating Full Load Hours” table in the “Reference Tables” section below[[966]](#footnote-966)*

*EFFbase = The efficiency of the baseline equipment; Can be expressed as thermal efficiency (Et), combustion efficiency (Ec), or Annual Fuel Utilization Efficiency (AFUE), depending on equipment type and capacity.*

*= For time of sale: See “Time of Sale Baseline Equipment Efficiency” table in the “Reference Tables” section below[[967]](#footnote-967)*

*equipment*

*EFFee = The efficiency of the efficient equipment; Can be expressed as thermal efficiency (Et), combustion efficiency (Ec), or Annual Fuel Utilization Efficiency (AFUE), depending on equipment type and capacity.*

*= Actual Installed*

*1,000,000 = Btu/MMBtu unit conversion factor*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure varies by size category and efficiency level. See the “Time of Sale Incremental Costs” table in the “Reference Tables” section below.

**Measure Life**

The measure life is assumed to be 20 years[[968]](#footnote-968).

**Operation and Maintenance Impacts**

n/a

**Reference Tables**

**Time of Sale Baseline Equipment Efficiency**[[969]](#footnote-969)

| **Equipment Type** | **Size Category** | **Subcategory or Rating Condition** | **Minimum Efficiency** |
| --- | --- | --- | --- |
| Boilers, Gas-fired | <300,000 Btu/h | Hot water | 82% AFUE |
| Steam | 80% AFUE |
| >=300,000 Btu/h and <=2,500,000 Btu/h | Hot water | 80% Et |
| Steam – all, except natural draft | 79.0% Et |
| Steam – natural draft | 77.0% Et |
| >2,500,000 Btu/h | Hot water | 82.0% Ec |
| Steam – all, except natural draft | 79.0% Et |
| Steam – natural draft | 77.0% Et |

**Time of Sale Incremental Costs[[970]](#footnote-970)**

|  |  |  |  |
| --- | --- | --- | --- |
| **Size Category (kBtu/h)** | **Incremental Cost** | | **Efficiency Metric** |
| **>=85% and <90% Efficiency** | **>=90% Efficiency** |
| <300 | $934 | $1481 | AFUE |
| 300 | $572 | $3,025 | Et |
| 500 | $1,267 | $3,720 | Et |
| 700 | $1,962 | $4,414 | Et |
| 900 | $2,657 | $5,109 | Et |
| 1,100 | $3,352 | $5,804 | Et |
| 1,300 | $4,047 | $6,499 | Et |
| 1,500 | $4,742 | $7,194 | Et |
| 1,700 | $5,436 | $7,889 | Et |
| 2,000 | $6,479 | $8,931 | Et |
| >=2200 | $7,174 | $9,626 | Et |

**Heating Full Load Hours**[[971]](#footnote-971)

|  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- |
| **Building Type** | **Dover, DE** | **Wilmington, DE** | **Baltimore, MD** | **Hagerstown, MD** | **Patuxent River, MD** | **Salisbury, MD** | **Washington D.C.** |
| Assembly | 676 | 692 | 620 | 657 | 451 | 507 | 559 |
| Auto Repair | 2,292 | 2,344 | 2,106 | 2,229 | 1,543 | 1,728 | 1,901 |
| Big Box Retail | 286 | 298 | 241 | 271 | 107 | 151 | 192 |
| Fast Food Restaurant | 957 | 983 | 866 | 926 | 590 | 681 | 766 |
| Full Service Restaurant | 988 | 1,016 | 891 | 956 | 597 | 694 | 784 |
| Grocery | 286 | 298 | 241 | 271 | 107 | 151 | 192 |
| Light Industrial | 867 | 885 | 803 | 845 | 608 | 672 | 732 |
| Motel | 659 | 667 | 632 | 650 | 547 | 575 | 601 |
| Primary School | 978 | 993 | 926 | 960 | 767 | 819 | 868 |
| Religious Worship | 750 | 754 | 737 | 746 | 698 | 711 | 723 |
| Small Office | 511 | 524 | 466 | 496 | 329 | 374 | 416 |
| Small Retail | 657 | 674 | 595 | 636 | 410 | 471 | 528 |
| Warehouse | 556 | 576 | 487 | 533 | 278 | 347 | 411 |
| Other | 805 | 823 | 739 | 783 | 541 | 606 | 667 |

**Gas Furnace**

**Unique Measure Code: CI\_HV\_TOS\_GASFUR\_0615, CI\_HV\_RTR\_GASFUR\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a high efficiency gas furnace with capacity less than 225,000 Btu/h with an electronically commutated fan motor (ECM) in the place of a standard efficiency gas furnace. This measure applies to time of sale and new construction opportunities.

**Definition of Baseline Condition**

**Time of Sale**: The baseline condition is a gas furnace with an Annual Fuel Utilization Efficiency (AFUE) of 80% with a standard efficiency furnace fan.

**Definition of Efficient Condition**

The efficient condition is a high-efficiency gas furnace with an AFUE of 90% or higher. This characterization only applies to furnaces with capacities less than 225,000 Btu/h with an electronically commutated fan motor (ECM).

**Annual Energy Savings Algorithm**[[972]](#footnote-972)

ΔkWh = 733 kWh[[973]](#footnote-973)

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = 0.19 kW[[974]](#footnote-974)

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = CAP \* HOURS \* ((1/AFUEbase) - (1/AFUEee)) / 1,000,000

*Where:*

*CAP = Capacity of the high-efficiency equipment [Btu/h]*

*= Actual Installed*

*HOURS = Full Load Heating Hours*

*= See “Heating Full Load Hours” table in the “Reference Tables” section below[[975]](#footnote-975)*

*AFUEbase = Annual Fuel Utilization Efficiency of the baseline equipment*

*= For time of sale: 0.80[[976]](#footnote-976)*

*AFUEee = Annual Fuel Utilization Efficiency of the efficient equipment*

*= Actual Installed.*

*1,000,000 = Btu/MMBtu unit conversion factor*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is provided below[[977]](#footnote-977):

|  |  |
| --- | --- |
| **Efficiency of Furnace (AFUE)** | **Incremental Cost** |
| 90% | $630 |
| 92% | $802 |
| 96% | $1,747 |

**Measure Life**

The measure life is assumed to be 18 years[[978]](#footnote-978).

**Operation and Maintenance Impacts**

n/a

**Reference Tables**

**Heating Full Load Hours**[[979]](#footnote-979)

|  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- |
| **Building Type** | **Dover, DE** | **Wilmington, DE** | **Baltimore, MD** | **Hagerstown, MD** | **Patuxent River, MD** | **Salisbury, MD** | **Washington D.C.** |
| Assembly | 676 | 692 | 620 | 657 | 451 | 507 | 559 |
| Auto Repair | 2,292 | 2,344 | 2,106 | 2,229 | 1,543 | 1,728 | 1,901 |
| Big Box Retail | 286 | 298 | 241 | 271 | 107 | 151 | 192 |
| Fast Food Restaurant | 957 | 983 | 866 | 926 | 590 | 681 | 766 |
| Full Service Restaurant | 988 | 1,016 | 891 | 956 | 597 | 694 | 784 |
| Grocery | 286 | 298 | 241 | 271 | 107 | 151 | 192 |
| Light Industrial | 867 | 885 | 803 | 845 | 608 | 672 | 732 |
| Motel | 659 | 667 | 632 | 650 | 547 | 575 | 601 |
| Primary School | 978 | 993 | 926 | 960 | 767 | 819 | 868 |
| Religious Worship | 750 | 754 | 737 | 746 | 698 | 711 | 723 |
| Small Office | 511 | 524 | 466 | 496 | 329 | 374 | 416 |
| Small Retail | 657 | 674 | 595 | 636 | 410 | 471 | 528 |
| Warehouse | 556 | 576 | 487 | 533 | 278 | 347 | 411 |
| Other | 805 | 823 | 739 | 783 | 541 | 606 | 667 |

**Dual Enthalpy Economizer**

**Unique Measure Code: CI\_HV\_RTR\_DEECON\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure involves the installation of a dual enthalpy economizer to provide free cooling during the appropriate ambient conditions. Enthalpy refers to the total heat content of the air. A dual enthalpy economizer uses two sensors — one measuring return air enthalpy and one measuring outdoor air enthalpy. Dampers are modulated for optimum and lowest enthalpy to be used for cooling. This measure applies only to retrofits.

**Definition of Baseline Condition**

The baseline condition is the existing HVAC system with no economizer.

**Definition of Efficient Condition**

The efficient condition is the HVAC system with dual enthalpy controlled economizer.

**Annual Energy Savings Algorithm**

ΔkWh = TONS \* SF

*Where:*

*TONS = Actual Installed*

*SF = Savings factor for the installation of dual enthalpy economizer control [kWh/ton],*

*= See “Savings Factors” table in “Reference Tables” section below[[980]](#footnote-980)*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = 0 kW[[981]](#footnote-981)

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental costs for this measure are presented in the “Dual Enthalpy Economizer Incremental Costs” table below.

**Dual Enthalpy Economizer Incremental Costs[[982]](#footnote-982)**

|  |  |
| --- | --- |
| **HVAC System Capacity (Tons)** | **Incremental Cost** |
| 5 | $943 |
| 15 | $1,510 |
| 25 | $2,077 |
| 40 | $2,927 |
| 70 | $4,628 |

**Measure Life**

The measure life is assumed to be 10 years[[983]](#footnote-983).

**Operation and Maintenance Impacts**

n/a

**Reference Tables**

**Savings Factors**[[984]](#footnote-984)

|  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- |
| **Savings Factors (kWh/ton)** | **Dover, DE** | **Wilmington, DE** | **Baltimore, MD** | **Hagerstown, MD** | **Patuxent River, MD** | **Salisbury, MD** | **Washington D.C.** |
| Assembly | 26 | 22 | 25 | 29 | 25 | 27 | 25 |
| Big Box Retail | 144 | 125 | 143 | 165 | 141 | 155 | 139 |
| Fast Food | 37 | 32 | 37 | 42 | 36 | 40 | 36 |
| Full Service Restaurant | 29 | 25 | 29 | 34 | 29 | 32 | 28 |
| Light Industrial | 24 | 21 | 23 | 27 | 23 | 25 | 23 |
| Primary School | 40 | 34 | 39 | 45 | 39 | 43 | 39 |
| Small Office | 177 | 153 | 175 | 201 | 173 | 189 | 171 |
| Small Retail | 90 | 78 | 89 | 103 | 88 | 97 | 87 |
| Religious | 6 | 5 | 6 | 6 | 6 | 6 | 6 |
| Warehouse | 2 | 2 | 2 | 2 | 2 | 2 | 2 |
| Other | 58 | 50 | 57 | 66 | 57 | 62 | 56 |

**AC Tune-Up**

**Unique Measure Code(s): CI\_HV\_RET\_ACTUNE\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

This measure is for a “tune-up” for a commercial central AC. This measure only applies to residential-style central AC systems of 5.4 tons (65,000 Btu/h) or less. Tune-ups for larger units, including units with variable air volume and air handling units, should be treated as custom measures. A recent California evaluation suggests that tune-ups on these larger systems may be better handled by breaking up the overall tune-up into a series of specific activities performed – for example, refrigerant charge correction, economizer repair, leak sealing, etc[[985]](#footnote-985). For smaller units, tuning measures may include:

* Refrigerant charge correction
* Cleaning the condensate drain line
* Clean and straighten coils and fans
* Replace air filter
* Repair damaged insulation

**Definition of Baseline Condition**

The baseline condition is a pre-tune-up air conditioner. Where possible, spot measurements should be used to estimate the baseline EER. An HVAC system is eligible for a tune-up once every five years.

**Definition of Efficient Condition**

The efficient condition is a post-tune-up air conditioner. Where possible, spot measurements should be used to estimate the EER post-tune-up.

**Annual Energy Savings Algorithm**

ΔkWh= CCAP x EFLH x 1/SEERpre x %\_impr

*Where:*

*CCAP = Cooling capacity of existing AC unit, in kBtu/hr*

*SEERpre = SEER of actual unit, before the tune-up. If testing is not done on the baseline condition, use the nameplate SEER.*

*EFLH = Full load hours for cooling equipment. See table below*

*%\_impr = Percent improvement based on measured EERs pre- and post-tune-up. Calculated as (EERpost – EERpre)/EERpost,where subscripts “pre” and “post” refer to the EER before and after the tune-up, respectively. If onsite testing data is not available, assume %\_impr = 0.05.[[986]](#footnote-986)*

**Full Load Cooling Hours by Location and Equipment Capacity[[987]](#footnote-987)**

|  |  |
| --- | --- |
| **City, State** | **HOURS by Equipment Capacity** |
| **< 135 kBtu/h** |
| Dover, DE | 910 |
| Wilmington, DE | 980 |
| Baltimore, MD | 1,014 |
| Hagerstown, MD | 885 |
| Patuxent River, MD | 1,151 |
| Salisbury, MD | 1,008 |
| Washington D.C. | 1,275 |

S**ummer Coincident Peak kW Savings Algorithm**

ΔkW = CCAP x 1/EERpre x %\_impr x CF

*Where:*

*CCAP = Cooling capacity of DMSHP unit, in kBtu/hr*

*EERpre = EER of actual unit, before the tune-up. If testing is not done on the baseline condition, use the nameplate EER.*

*%\_impr = Percent improvement based on measured EERs pre and post tune-up. Calculated as (EERpost – EERpre)/EERpost. If onsite testing data is not available, assumed %\_impr = 0.05.[[988]](#footnote-988)*

*CFPJM = PJM Summer Peak Coincidence Factor (June to August weekdays between 2 pm and 6 pm) valued at peak weather*

*= 0.360 for units <135 kBtu/h and 0.567 for units ≥135 kBtu/h[[989]](#footnote-989)*

*CFSSP = Summer System Peak Coincidence Factor (hour ending 5pm on hottest summer weekday)*

*= 0.588 for units <135 kBtu/h and 0.874 for units ≥135 kBtu/h[[990]](#footnote-990)*

**Annual Fossil Fuel Savings Algorithm**

n/a

**Incremental Cost**

Use the actual cost of the tune-up. If this is unknown, use a default of $35/ton[[991]](#footnote-991).

**Measure Life**

The measure life for an AC tune-up is 5 years.[[992]](#footnote-992)

**Operation and Maintenance Impacts**

n/a

*Refrigeration End Use*

**ENERGY STAR Commercial Freezers**

**Unique Measure Code(s): CI\_RF\_TOS\_FREEZER\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes the installation of an ENERGY STAR qualified, high-efficiency packaged commercial freezer intended for food product storage.

**Definition of Baseline Condition**

The baseline condition is a standard-efficiency packaged commercial freezer meeting, but not exceeding, federal energy efficiency standards.

**Definition of Efficient Condition**

The efficient condition is a high-efficiency packaged commercial freezer meeting ENERGY STAR Version 2.1 requirements[[993]](#footnote-993).

**Annual Energy Savings Algorithm**

ΔkWh = (kWhBASEdailymax - kWhEEdailymax) \* 365

*Where:*

*kWhBASEdailymax [[994]](#footnote-994) = See table below.*

|  |  |
| --- | --- |
| **Product Volume (in cubic feet)** | **kWhBASEdailymax** |
| Solid Door Cabinets | 0.40V + 1.38 |
| Glass Door Cabinets | 0.75V + 4.10 |

*Where V = Association of Home Appliances Manufacturers (AHAM) volume*

*kWhEEdailymax [[995]](#footnote-995) = See table below.*

|  |  |
| --- | --- |
| **Product Volume (in cubic feet)** | **kWhEEdailymax** |
| **Vertical Configuration** | |
| Solid Door Cabinets |  |
| 0 < V < 15 | ≤ 0.250V + 1.250 |
| 15 ≤ V < 30 | ≤ 0.400V – 1.000 |
| 30 ≤ V < 50 | ≤ 0.163V + 6.125 |
| 50 ≤ V | ≤ 0.158V + 6.333 |
| Glass Door Cabinets |  |
| 0 < V < 15 | ≤ 0.607V + 0.893 |
| 15 ≤ V < 30 | ≤ 0.733V – 1.000 |
| 30 ≤ V < 50 | ≤ 0.250V + 13.500 |
| 50 ≤ V | ≤ 0.450V + 3.500 |
| **Chest Configuration** | |
| Solid or Glass Door Cabinets | ≤ 0.270V + 0.130 |

*Where V = Association of Home Appliances Manufacturers (AHAM) volume*

Illustrative examples – do not use as default assumption

For example, for a 50 ft2 vertical configuration, solid door freezer:

ΔkWh = ((0.4 \* 50 + 1.38) - (0.158 \* 50 + 6.333)) \* 365

= 2,608.7 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (ΔkWh/HOURS) x CF

*Where:*

*HOURS = Full load hours*

*= 5858 [[996]](#footnote-996)*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.772 [[997]](#footnote-997)*

Illustrative examples – do not use as default assumption

For example, for a 50 ft2 vertical configuration, solid door freezer:

ΔkW = (2,608.7 / 5858) \* 0.772

= 0.34 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost [[998]](#footnote-998)**

The incremental cost for this measure is assumed to be $25 for solid door freezers and $256 for glass door freezers.

**Measure Life**

The measure life is assumed to be 12 years.[[999]](#footnote-999)

**Operation and Maintenance Impacts**

n/a

**ENERGY STAR Commercial Refrigerator**

**Unique Measure Code(s): CI\_RF\_TOS\_REFRIG\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure describes the installation of an ENERGY STAR qualified, high-efficiency packaged commercial refrigerator intended for food product storage.

**Definition of Baseline Condition**

The baseline condition is a standard-efficiency packaged commercial refrigerator meeting, but not exceeding, federal energy efficiency standards.

**Definition of Efficient Condition**

The efficient condition is a high-efficiency packaged commercial refrigerator meeting ENERGY STAR Version 2.1 requirements[[1000]](#footnote-1000).

**Annual Energy Savings Algorithm**

ΔkWh = (kWhBASEdailymax - kWhEEdailymax) \* 365

*Where:*

*kWhBASEdailymax [[1001]](#footnote-1001) = See table below.*

|  |  |
| --- | --- |
| **Product Volume (in cubic feet)** | **kWhBASEdailymax** |
| Solid Door Cabinets | 0.10V + 2.04 |
| Glass Door Cabinets | 0.12V + 3.34 |

*Where V = Association of Home Appliances Manufacturers (AHAM) volume*

*kWhEEdailymax [[1002]](#footnote-1002) = See table below.*

|  |  |
| --- | --- |
| **Product Volume (in cubic feet)** | **kWhEEdailymax** |
| **Vertical Configuration** |  |
| Solid Door Cabinets |  |
| 0 < V < 15 | ≤ 0.089V + 1.411 |
| 15 ≤ V < 30 | ≤ 0.037V + 2.200 |
| 30 ≤ V < 50 | ≤ 0.056V + 1.635 |
| 50 ≤ V | ≤ 0.060V + 1.416 |
| Glass Door Cabinets |  |
| 0 < V < 15 | ≤ 0.118V + 1.382 |
| 15 ≤ V < 30 | ≤ 0.140V + 1.050 |
| 30 ≤ V < 50 | ≤ 0.088V + 2.625 |
| 50 ≤ V | ≤ 0.110V + 1.500 |
| **Chest Configuration** |  |
| Solid or Glass Door Cabinets | ≤ 0.125V + 0.475 |

*Where V = Association of Home Appliances Manufacturers (AHAM) volume*

Illustrative examples – do not use as default assumption

For example, for a 50 ft2 vertical configuration, solid door refrigerator:

ΔkWh = ((0.1 \* 50 + 2.04) - (0.06 \* 50 + 1.416)) \* 365

= 957.8 kWh

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = (ΔkWh/HOURS) \* CF

*Where:*

*HOURS = Full load hours*

*= 5858 [[1003]](#footnote-1003)*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.772 [[1004]](#footnote-1004)*

Illustrative examples – do not use as default assumption

For example, for a 50 ft2 vertical configuration, solid door refrigerator:

ΔkW = (957.8 / 5858) \* 0.772

= 0.13 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost [[1005]](#footnote-1005)**

The incremental cost for this measure is assumed to be $0 for solid door refrigerators and $158 for glass door refrigerators.

**Measure Life**

The measure life is assumed to be 12 years.[[1006]](#footnote-1006)

**Operation and Maintenance Impacts**

n/a

**Night Covers for Refrigerated Cases**

**Unique Measure Code(s): CI\_RF\_TOS\_NTCOV\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

By covering refrigerated cases, the heat gain due to the spilling of refrigerated air and convective mixing with room air is reduced at the case opening. Continuous curtains can be pulled down overnight while the store is closed, yielding significant energy savings.

**Definition of Baseline Condition**

In order for this characterization to apply, the baseline equipment is assumed to be a refrigerated case without a night cover.

**Definition of Efficient Condition**

In order for this characterization to apply, the efficient equipment is assumed to be a refrigerated case with a continuous cover deployed during overnight periods. Characterization assumes covers are deployed for six hours daily.

**Annual Energy Savings Algorithm**

ΔkWh = (LOAD / 12,000) \* FEET \* (3.516) / COP \* ESF \* 8,760

ΔkWh = 346.5 \* FEET / COP

*Where:*

*LOAD = average refrigeration load per linear foot of refrigerated case without night covers deployed*

*= 1,500 Btu/h[[1007]](#footnote-1007) per linear foot*

*FEET = linear (horzontal) feet of covered refrigerated case*

*12,000 = conversion factor - Btu per ton cooling.*

*3.516 = conversion factor – Coefficient of Performance (COP) to kW per ton.*

*COP = Coefficient of Performance of the refrigerated case.*

*= assume 2.2[[1008]](#footnote-1008), if actual value is unknown.*

*ESF = Energy Savings Factor; reflects the percent reduction in refrigeration load due to the deployment of night covers.*

*= 9%[[1009]](#footnote-1009)*

*8,760 = assumed annual operating hours of the refrigerated case*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = 0[[1010]](#footnote-1010)

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental capital cost for this measure is $42 per linear foot of cover installed including material and labor.[[1011]](#footnote-1011)

**Measure Life**

The expected measure life is assumed to be 5 years [[1012]](#footnote-1012).

**Operation and Maintenance Impacts**

n/a

**Anti-Sweat Heater Controls\***

**Unique Measure Code(s): CI\_RF\_TOS\_ASHC\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

Anti-sweat door heaters (ASDH) prevent condensation from forming on cooler and freezer doors. By installing a control device to turn off door heaters when there is little or no risk of condensation, significant energy savings can be realized. There are two commercially available control strategies – (1) ON/OFF controls and (2) micropulse controls – that respond to a call for heating, which is typically determined using either a door moisture sensor or an indoor air temperature and humidity sensor to calculate the dew point. In the first strategy, the ON/OFF controls turn the heaters on and off for minutes at a time, resulting in a reduction in run time. In the second strategy, the micropulse controls pulse the door heaters for fractions of a second, in response to the call for heating.

Both of these strategies result in energy and demand savings. Additional savings come from refrigeration interactive effects. When the heaters run less, they introduce less heat into the refrigerated spaces and reduce the cooling load.

**Definition of Baseline Condition**

In order for this characterization to apply, the baseline condition is assumed to be a commercial glass door cooler or refrigerator with a standard heated door running 24 hours a day, seven days per week (24/7) with no controls installed.

**Definition of Efficient Condition**

In order for this characterization to apply, the efficient equipment is assumed to be a door heater control on a commercial glass door cooler or refrigerator utilizing either ON/OFF or micropulse controls.

**Annual Energy Savings Algorithm**

ΔkWh = kWd \* (%ON*NONE* - %ON*CONTROL*) \* NUMdoors \* HOURS \* WHFe

*Where:*

*kWd = connected load kW per connected door*

*= If actual kWd is unknown, assume 0.13 kW[[1013]](#footnote-1013).*

*%ONNONE = Effective run time of uncontrolled ASDH*

*= assume 90.7%[[1014]](#footnote-1014).*

*%ONCONTROL = Effective run time of ASDH with controls*

*= assume 58.9% for ON/OFF controls and 42.8% for micropulse controls[[1015]](#footnote-1015).*

*NUMdoors = number of reach-in refrigerator or freezer doors controlled by sensor*

*= Actual number of doors controlled by sensor*

*HOURS = Hours of operation*

*= 8,760*

*WHFe = Waste Heat Factor for Energy; represents the increased savings due to reduced waste heat from motors that must be rejected by the refrigeration equipment.*

*= assume 1.25 for cooler and 1.50 for freezer applications[[1016]](#footnote-1016).*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = kWd \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand; represents the increased savings due to reduced waste heat from motors that must be rejected by the refrigeration equipment.*

*= assume 1.25 for cooler and 1.50 for freezer*

*CF = Summer Peak Coincidence Factor*

*= If site specific CFs are unkown, use deemed estimates in the table below[[1017]](#footnote-1017).*

**Coincidence Factors by Control Type**

|  |  |
| --- | --- |
| **Control Type** | **CF** |
| On/Off Controls | 0.32 |
| Micropulse Controls | 0.45 |

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental capital cost is $994 for a door heater controller, $123 for a cooler door, and $219 for a freezer door[[1018]](#footnote-1018). Values include labor costs.

**Measure Life**

The expected measure life is assumed to be 12 years.[[1019]](#footnote-1019)

**Operation and Maintenance Impacts**

n/a

**Evaporator Fan Electronically-Commutated Motor (ECM) Retrofit\*\***

**Unique Measure Code(s): CI\_RF\_RET\_ECMFAN\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

Evaporator fans circulate air in refrigerated spaces by drawing air across the evaporator coil and into the space. Fans are found in both reach-in and walk-in coolers and freezers. Energy and demand savings for this measure are achieved by reducing motor operating power. Additional savings come from refrigeration interactive effects. Because electronically-commutated motors (ECMs) are more efficient and use less power, they introduce less heat into the refrigerated space compared to the baseline motors and result in a reduction in cooling load on the refrigeration system.

**Definition of Baseline Condition**

In order for this characterization to apply, the baseline condition is assumed to be an evaporator fan powered by a shaded pole (SP) motor that runs 24 hours a day, seven days per week (24/7) with no controls.

**Definition of Efficient Condition**

In order for this characterization to apply, the efficient equipment is assumed to be an evaporator fan powered by an ECM that runs 24/7 with no controls.

**Annual Energy Savings Algorithm**

ΔkWh = kWhp \* HP \* %∆P \* %ONUC \* HOURS \* WHFe

*Where:*

*kWhp = ECM connected load kW per horsepower*

*= If actual kWhp is unknown, assume 0.758 kW/hp[[1020]](#footnote-1020).*

*HP = Horsepower of ECM*

*= Actual horsepower of ECM.*

*%∆P = Percent change in power relative to ECM kW, calculated as the kW of the SP motor minus the kW of the ECM, divided by the kW of the ECM*

*= If actual %∆P is unkown, assume 157%[[1021]](#footnote-1021).*

*%ONUC = Effective run time of uncontrolled motors*

*= If actual %ONUC is unknown, assume 97.8% [[1022]](#footnote-1022).*

*HOURS = Hours of operation*

*= 8,760*

*WHFe = Waste Heat Factor for Energy; represents the increased savings due to reduced waste heat from motors that must be rejected by the refrigeration equipment.*

*= assume 1.38 for cooler and 1.76 for freezer applications [[1023]](#footnote-1023).*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = kWhp \* HP \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand; represents the increased savings due to reduced waste heat from motors that must be rejected by the refrigeration equipment.*

*= assume 1.38 for cooler and 1.76 for freezer applications [[1024]](#footnote-1024).*

*CF = Summer Peak Coincidence Factor*

*= If site specific CFs are unkown, use 1.53[[1025]](#footnote-1025).*

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental capital cost is $60. Values include labor costs.[[1026]](#footnote-1026)

**Measure Life**

The expected measure life is assumed to be 15 years.[[1027]](#footnote-1027)

**Operation and Maintenance Impacts**

n/a

**Evaporator Fan Motor Controls\*\***

**Unique Measure Code(s): CI\_RF\_RET\_EFCTRL\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

Evaporator fans circulate cool air in refrigerated spaces by drawing air across the evaporator coil and into the space. Uncontrolled, evaporator fans run 24 hours a day, seven days per week (24/7). Evaporator fan controls reduce fan run time or speed depending on the call for cooling, and therefore provide an opportunity for energy and demand savings. There are two commercially available strategies – (1) ON/OFF controls and (2) multispeed controls – that respond to a call for cooling. In the first strategy, the ON/OFF controls turn the motors on and off in response to the call for cooling, generating energy and demand savings as a result of a reduction in run time. In the second strategy, the multispeed controls change the speed of the motors in response to the call for cooling, saving energy and reducing demand by reductingoperating power and run time (multispeed controls can also turn the motor off).

Additional savings come from the refrigeration interactive effects. Because fan controls reduce motor operating power and/or run time, they introduce less heat into the refrigerated space compared to uncontrolled motors and result in a reduction in cooling load on the refrigeration system.

**Definition of Baseline Condition**

In order for this characterization to apply, the baseline condition is assumed to be an evaporator fan powered by an uncontrolled ECM or SP motor that runs 24/7.

**Definition of Efficient Condition**

In order for this characterization to apply, the efficient equipment is assumed to be an evaporator fan powered by an ECM or SP motor utilizing either ON/OFF or multispeed controls.

**Annual Energy Savings Algorithm**

ΔkWh = kWhp \* HP \* (%ONUC - %ONCONTROL)\* HOURS \* WHFe

*Where:*

*kWhp = connected load kW per horsepower of motor*

*= If actual kWhp is unknown, assume 0.758 kW/hp for ECM and 2.088 kW/hp for SP motor[[1028]](#footnote-1028).*

*HP = Horespower of ECM or SP motor*

*= Actual horsepower of ECM or SP motor.*

*%ONUC = Effective run time of uncontrolled motor*

*= If actual %ONUC is unkown, assume 97.8% [[1029]](#footnote-1029).*

*%ONCONTROL = Effective run time of motor with controls*

*= Assume 63.6% for ON/OFF style controls and 69.2% for multi-speed style controls [[1030]](#footnote-1030).*

*HOURS = Hours of operation*

*= 8,760*

*WHFe = Waste Heat Factor for Energy; represents the increased savings due to reduced waste heat from motors that must be rejected by the refrigeration equipment.*

*= assume 1.38 for cooler and 1.76 for freezer applications[[1031]](#footnote-1031).*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = kWhp \* HP \* WHFd \* CF

*Where:*

*WHFd = Waste Heat Factor for Demand; represents the increased savings due to reduced waste heat from motors that must be rejected by the refrigeration equipment.*

*= assume 1.38 for cooler and 1.76 for freezer applications[[1032]](#footnote-1032).*

*CF = Summer Peak Coincidence Factor*

*= If site specific CFs are unkown, use 0.26[[1033]](#footnote-1033).*

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental capital cost is $532 for multispeed controls[[1034]](#footnote-1034). Value includes labor costs.

The actual measure installation cost for ON/OFF controls should be used (including materials and labor)[[1035]](#footnote-1035).

**Measure Life**

The expected measure life is assumed to be 10 years.[[1036]](#footnote-1036)

**Operation and Maintenance Impacts**

n/a

*Hot Water End Use*

**C&I Heat Pump Water Heater**

**Unique Measure Code(s): CI\_WT\_TOS\_HPCIHW\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a Heat Pump water heater in place of a standard electric water heater. This measure applies to time of sale and new construction opportunities.

**Definition of Baseline Condition**

The baseline condition is a standard electric water heater.

**Definition of Efficient Condition**

The efficient condition is a heat pump water heater.

**Annual Energy Savings Algorithm**

ΔkWh = (kBtu\_req / 3.413) \* ((1/EFbase) - (1/EFee))

*Where:*

*kBtu\_req (Office) = Required annual heating output of office (kBtu)*

*= 6,059 [[1037]](#footnote-1037)*

*kBtu\_req (School) = Required annual heating output of school (kBtu)*

*= 22,191 [[1038]](#footnote-1038)*

*3.413 = Conversion factor from kBtu to kWh*

*EFee = Energy Factor of Heat Pump domestic water heater*

*= 2.0* *[[1039]](#footnote-1039)*

*EFbase = Energy Factor of baseline domestic water heater*

*= 0.904 [[1040]](#footnote-1040)*

ΔkWh Office = (6,059 / 3.413) \* ((1/0.904) – (1/2.0))

= 1076.2 kWh

ΔkWh School = (22,191 / 3.413) \* ((1/0.904) – (1/2.0))

= 3941.4 kWh

If the deemed “kBtu\_req” estimates are not applicable, the following equation can be used to estimate annual water heating energy requirements:

*kBtu\_req = GPD \* 8.33 \* 1.0 \* WaterTempRise \* 365*

*Where:*

*GDP = Average daily hot water requirements (gallons/day)*

*= Actual usage (Note: days when the building is unoccupied must be included in the averaging calculation)*

*8.33 = Density of water (lb/gallon)*

*1.0 = Specific heat of water (Btu/lb-°F)*

*WaterTempRise = Difference between average temperature of water delivered to site and water heater setpoint (°F)*

*365 = Days per year*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh / Hours \* CF

*Where:*

*Hours (Office) = Run hours in office*

*= 5885* *[[1041]](#footnote-1041)*

*Hours (School) = Run hours in school*

*= 2218 [[1042]](#footnote-1042)*

*CF (Office) = Summer Peak Coincidence Factor for office measure*

*= 0.630 [[1043]](#footnote-1043)*

*CF (School) = Summer Peak Coincidence Factor for school measure*

*= 0.580 [[1044]](#footnote-1044)*

ΔkW Office = (1076.2 / 5885) \* 0.630

= 0.12 kW

ΔkW School = (3941.4 / 3.413) \* 0.580

= 1.03 kW

If annual operating hours and CF estimates are unknown, use deemed HOURS and CF estimates above. Otherwise, use site specific values.

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $925.[[1045]](#footnote-1045)

**Measure Life**

The measure life is assumed to be 10 years.[[1046]](#footnote-1046)

**Operation and Maintenance Impacts**

n/a

**Pre-Rinse Spray Valves**

**Unique Measure Code(s): CI\_WT\_TOS\_PRSPRY\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

All pre-rinse valves use a spray of water to remove food waste from dishes prior to cleaning in a dishwasher. They reduce water consumption, water heating cost, and waste water (sewer) charges. Pre-rinse spray valves include a nozzle, squeeze lever, and dish guard bumper. The spray valves usually have a clip to lock the handle in the “on” position. Pre-rinse valves are inexpensive and easily interchangeable with different manufacturers’ assemblies. The primary impacts of this measure are water savings. Energy savings depend on the facility’s water heating fuel - if the facility does not have electric water heating, there are no electric savings for this measure; if the facility does not have fossil fuel water heating, there are no MMBtu savings for this measure.

**Definition of Baseline Condition**

The baseline equipment is assumed to be a spray valve with a flow rate of 3 gallons per minute.

**Definition of Efficient Condition**

The efficient equipment is assumed to be a pre-rinse spray valve with a flow rate of 1.6 gallons per minute, and with a cleanability performance of 26 seconds per plate or less.

**Annual Energy Savings Algorithm**

ΔkWh = ΔWater x HOT% x 8.33 x (ΔT) x (1/EFF) / 3413

*Where:*

*ΔWater = Water savings (gallons); see calculation in “Water Impact” section below.*

*HOT% = The percentage of water used by the pre-rinse spray valve that is heated*

*= 69%[[1047]](#footnote-1047)*

*8.33 = The energy content of heated water (Btu/gallon/°F)*

*ΔT = Temperature rise through water heater (°F)*

*= 70[[1048]](#footnote-1048)*

*EFF = Water heater thermal efficiency*

*= 0.97[[1049]](#footnote-1049)*

*3413 = Factor to convert Btu to kwh*

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = 0

**Annual Fossil Fuel Savings Algorithm**

ΔMMBtu = ΔWater x HOT% x 8.33 x (ΔT) x (1/EFF) x 10-6

*Where:*

*EFF = Water heater thermal efficiency*

*= 0.80[[1050]](#footnote-1050)*

*10-6 = Factor to convert Btu to MMBtu*

**Annual Water Savings Algorithm**

ΔWater = (FLObase – FLOeff) x 60 x HOURSday x 365

*Where :*

*ΔWater = Annual water savings (gal)*

*FLObase = The flow rate of the baseline spray nozzle*

*= 3 gallons per minute*

*FLOeff  = The flow rate of the efficient equipment*

*= 1.6 gallons per minute*

*60 = minutes per hour*

*365 = days per year*

*HOURS = Hours used per day – depends on facility type as below:[[1051]](#footnote-1051)*

|  |  |
| --- | --- |
| **Facility Type** | **Hours of Pre-Rinse Spray Valve Use per**  **Day (HOURS)** |
| Full Service Restaurant | 4 |
| Other | 2 |
| Limited Service (Fast Food ) Restaurant | 1 |

**Incremental Cost**

The actual measure installation cost should be used (including material and labor).

**Measure Life**

The measure life is assumed to be 5 years.[[1052]](#footnote-1052)

**Operation and Maintenance Impacts**

n/a

*Appliance End Use*

**Commercial Clothes Washer\*\***

**Unique Measure Code(s): CI\_LA\_TOS\_CCWASH\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

This measure relates to the purchase (time of sale) and installation of a commercial clothes washer (i.e., soft-mounted front-loading or soft-mounted top-loading clothes washer that is designed for use in applications in which the occupants of more than one household will be using the clothes washer, such as multi-family housing common areas and coin laundries) exceeding the ENERGY STAR minimum qualifying efficiency standards presented below:[[1053]](#footnote-1053)

|  |  |  |
| --- | --- | --- |
| **Efficiency Level** | **Modified Energy Factor (MEF)** | **Water Factor**  **(WF)** |
| ENERGY STAR | >= 2.2 | <= 4.5 |

The Modified Energy Factor (MEF) measures energy consumption of the total laundry cycle (washing and drying). It indicates how many cubic feet of laundry can be washed and dried with one kWh of electricity; the higher the number, the greater the efficiency.

The Water Factor (WF) is the number of gallons needed for each cubic foot of laundry. A lower number indicates lower consumption and more efficient use of water.

**Definition of Baseline Condition**

The baseline efficiency is determined according to the Modified Energy Factor (MEF) that takes into account the energy and water required per clothes washer cycle, including energy required by the clothes dryer per clothes washer cycle. The federal baseline MEF as of May 2016 is 2.00 for front loading units and 1.60 for top loading units.

**Definition of Efficient Condition**

The efficient condition is a clothes washer meeting the ENERGY STAR efficiency criteria presented above.

**Annual Energy Savings Algorithm**

∆kWh = ∆kWhCW + ∆kWhDHW + ∆kWhDRYER

∆kWhCW = (kWhUNIT, BASE - kWhUNIT, EE) \* %CW

∆kWhDHW = (kWhUNIT, BASE - kWhUNIT, EE) \* %DHW \* DHWELEC

∆kWhDRYER = [(kWhTOTAL,BASE - kWhTOTAL,EE) - (kWhUNIT, BASE - kWhUNIT, EE)] \* %LOADSDRYED / DRYERUSAGE \* DRYERUSAGE\_MOD \* DRYERELEC

kWhUNIT,i = kWhUNIT\_RATED,i \* Ncycles / Ncycles\_ref

kWhTOTAL,i = Capacity / MEFi \* Ncycles

*Where*

*i = Subscript denoting either baseline (“BASE”) or efficient (“EE”) equipment*

*∆kWhCW = Clothes washer machine electric energy savings*

*∆kWhDHW = Water heating electric energy savings*

*∆kWhDRYER = Dryer electric energy savings*

*kWhUNIT, BASE = Conventional unit electricity consumption exclusive of required dryer energy*

*kWhUNIT, EE = ENERGY STAR unit electricity consumption exclusive of required dryer energy*

*kWhTOTAL, BASE  = Conventional unit electricity consumption inclusive of required dryer energy (assuming electric dryer)*

*kWhTOTAL, EE  = ENERGY STAR unit electricity consumption inclusive of required dryer energy (assuming electric dryer)*

*kWhUNIT\_RATED, BASE = Conventional rated unit electricity consumption*

*= If actual value unknown, assume 241 kWh/yr[[1054]](#footnote-1054)*

*kWhUNIT\_RATED, EE = Efficient rated unit electricity consumption*

*= If actual value unknown, assume 97 kWh/yr[[1055]](#footnote-1055)*

*%CW = Percentage of unit energy consumption unsed for clothes washer operation*

*= If unknown, assume 20%.[[1056]](#footnote-1056)*

*%DHW = Percentage of unit energy consumption used for water heating*

*= If unknown, assume 80%.[[1057]](#footnote-1057)*

*DHWELEC = 1 if electric water heating; 0 if gas water heating*

*MEFBASE = Modified Energy Factor of baseline unit*

*= Values provided in table below*

*MEFEE = Modified Energy Factor of efficient unit*

*= Actual. If unknown assume average values provided below.*

*Capacity = Clothes washer capacity (cubic feet)*

*= Actual. If capacity is unknown assume average 3.43 cubic feet[[1058]](#footnote-1058)*

|  |  |  |
| --- | --- | --- |
| **Efficiency Level** | **Modified Energy Factor (MEF)** | |
| **Front Loading** | **Top Loading** |
| Federal Standard | >= 2.00 | >= 1.60 |
| ENERGY STAR | >= 2.20 | |

*Ncycles = Number of cycles per year*

*= If actual value unknown, assume 1,241 for multifamily applications and 2,190 for landromats[[1059]](#footnote-1059)*

*Ncycles\_ref = Reference number of cycles per year*

*= 392[[1060]](#footnote-1060)*

*%LOADSDRYED = Percentage of washer loads dried in machine*

*= If actual value unknown, assume 100%*

*DRYERUSAGE = Dryer usage factor*

*= 0.84[[1061]](#footnote-1061)*

*DRYERUSAGE\_MOD = Dryer usage in buildings with dryer and washer*

*= 0.95[[1062]](#footnote-1062)*

*DRYERELEC = 1 if electric dryer; 0 if gas dryer*

Note, utilities may consider whether it is appropriate to claim kWh savings from the reduction in water consumption arising from this measure. The kWh savings would be in relation to the pumping and wastewater treatment. See water savings for characterization.

**Summer Coincident Peak kW Savings Algorithm**

ΔkW = ΔkWh/Hours \* CF

*Where:*

*Hours = Assumed Run hours of Clothes Washer*

*= 265 [[1063]](#footnote-1063)*

*CF = Summer Peak Coincidence Factor for measure*

*= 0.029 [[1064]](#footnote-1064)*

**Annual Fossil Fuel Savings Algorithm**

∆MMBtu = ∆MMBtuDHW + ∆MMBtuDRYER

∆MMBtuDHW = (kWhUNIT, BASE - kWhUNIT, EE) \* %DHW / DHWEFF \*

MMBtu \_convert \* DHWGAS

∆MMBtuDRYER = [(kWhTOTAL,BASE - kWhTOTAL,EE) - (kWhUNIT, BASE - kWhUNIT, EE)] \* MMBtu \_convert \* %LOADSDRYED / DRYERUSAGE \* DRYERUSAGE\_MOD \* DRYERGAS,CORR \* DRYERGAS

*Where:*

*∆MMBtuDHW = Water heating gas energy savings*

*∆MMBtuDRYER = Dryer gas energy savings*

*DHWEFF = Gas water heater efficiency*

*= If actual unknown, assume 75%*

*MMBtu \_convert = Convertion factor from kWh to MMBtu*

*= 0.003413*

*DHWGAS = 1 if gas water heating; 0 if electric water heating*

*DRYERGAS,CORR = Gas dryer correction factor; 1.12[[1065]](#footnote-1065)*

*DRYERGAS = 1 if gas dryer; 0 if electric dryer*

**Annual Water Savings Algorithm**

∆Water (CCF) = Capacity \* (WFBASE - WFEE) \* Ncycles / 748

*Where*

*WFBASE = Water Factor of baseline clothes washer*

*= Values provided below*

*WFEE = Water Factor of efficient clothes washer*

*= Actual. If unknown assume value provided below.*

*748 = Conversion factor from gallons to CCF*

|  |  |  |
| --- | --- | --- |
| **Efficiency Level** | **Water Factor (WF)** | |
| **Front Loading** | **Top Loading** |
| Federal Standard | <= 5.5 | <= 8.5 |
| ENERGY STAR | <= 4.5 | |

kWh Savings from Water Reduction

The kWh savings from the waste reduction characterized above is now estimated. Please note that utilities’ must be careful not to double count the monetary benefit of these savings within cost effectiveness testing if the avoided costs of water already include the associated electric benefit.

ΔkWhwater[[1066]](#footnote-1066) = 2.07 kWh/CCF \* ∆Water (CCF)

**Incremental Cost**

The incremental cost for this measure is $200[[1067]](#footnote-1067):

**Measure Life**

The measure life is assumed to be 7 years [[1068]](#footnote-1068).

**Operation and Maintenance Impacts**

n/a

*Plug Load End Use*

**Advanced Power Strip**

**Unique Measure Code: CI\_PL\_TOS\_APS\_0614**

**Effective Date: June 2014**

**End Date: TBD**

**Measure Description**

This measure relates to the installation of a Current-Sensing Master/Controlled Advanced Power Strip (APS)in place of a standard “power strip,” a device used to expand a single wall outlet into multiple outlets. This measure is assumed to be a time of sale installation.

**Definition of Baseline Condition**

The baseline condition is a standard “power strip”. This strip is simply a “plug multiplier” that allows the user to plug in multiple devices using a single wall outlet. Additionally, the baseline unit has no ability to control power flow to the connected devices.

**Definition of Efficient Condition**

The efficient condition is a Current-Sensing Master/Controlled Advanced Power Strip that functions as both a “plug multiplier” and also as a plug load controller. The efficient unit has the ability to essentially disconnect controlled devices from wall power when the APS detects that a controlling device, or master load, has been switched off. The efficient device effectively eliminates standby power consumption for all controlled devices[[1069]](#footnote-1069) when the master load is not in use.

**Annual Energy Savings Algorithm**

∆kWh = 26.9 kWh[[1070]](#footnote-1070)

**Summer Coincident Peak kW Savings Algorithm**

∆kW = 0 kW

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for this measure is assumed to be $16 for a 5-plug

$26 for a 7-plug[[1071]](#footnote-1071).

**Measure Life**

The measure life is assumed to be 4 years[[1072]](#footnote-1072).

**Operation and Maintenance Impacts**

n/a

*Commercial Kitchen Equipment End Use*

**Commercial Fryers\***

**Unique Measure Code(s): CI\_KE\_TOS\_FRY\_0516**

**Effective Date: May 2016**

**End Date: TBD**

**Measure Description**

Commercial fryers that have earned the ENERGY STAR offer shorter cook times and higher production rates through advanced burner and heat exchanger designs. Frypot insulation reduces standby losses resulting in a lower idle energy rate. This measure applies to both standard sized fryers and large vat fryers.[[1073]](#footnote-1073) Standard sized fryers that have earned the ENERGY STAR are up to 30% more efficient than non-qualified models; large vat fryers are 35% more efficient. This measure applies to time of sale opportunities.

**Definition of Baseline Condition**

The baseline equipment is assumed to be a standard efficiency electric fryer with a heavy load efficiency of 75% for standard sized equipment and 70% for large vat equipment or a gas fryer with heavy load efficiency of 35% for both standard sized and large vat equipment.

**Definition of Efficient Condition**

The efficient equipment is assumed to be an ENERGY STAR qualified electric or gas fryer.[[1074]](#footnote-1074)

**Annual Energy Savings Algorithm**

kWhi = (kWh\_Cookingi + kWh\_Idlei) x DAYS

kWh\_Cookingi = LB x EFOOD/EFFi

kWh\_Idlei = IDLEi x (HOURSDAY – LB/PCi)

kWhi = [LB x EFOOD/EFFi + IDLEi x (HOURSDAY – LB/PCi)] x DAYS

ΔkWh = kWhbase - kWheff

*Where:[[1075]](#footnote-1075)*

*i = either “base” or “eff” depending on whether the calculation of energy consumption is being performed for the baseline or efficient case, respectively.*

*kWh\_Cookingi = daily cooking energy consumption (kWh)*

*kWh\_Idlei = daily idle energy consumption (kWh)*

*kWhbase = the annual energy usage of the baseline equipment calculated using baseline values*

*kWheff = the annual energy usage of the efficient equipment calculated using efficient values*

*HOURSDAY = average daily operating hours*

*= if average daily operating hours are unknown, assume default of 16 hours/day for standard fryers and 12 hours/day for large vat fryers.*

*EFOOD = ASTM Energy to Food (kWh/lb); the amount of energy absorbed by the food during cooking, per pound of food*

*= 0.167*

*LB = Pounds of food cooked per day (lb/day)*

*= if average pounds of food cooked per day is unknown, assume default of 150 lbs/day.*

*DAYS = annual days of operation*

*= if annual days of operation are unknown, assume default of 365 days.*

*EFF = Heavy load cooking energy efficiency (%)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*IDLE = Idle energy rate (kW)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*PC = Production capacity (lb/hr)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

**Electric Fryer Performance Metrics: Baseline and Efficient Values**

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Parameter** | **Standard Size** | | **Large Vat** | |
| **Baseline Model** | **Energy Efficient Model** | **Baseline Model** | **Energy Efficient Model** |
| IDLE (kW) | 1.05 | 0.80 | 1.35 | 1.10 |
| EFF | 75% | 83% | 70% | 80% |
| PC | 65 | 70 | 100 | 110 |

**Summer Coincident Peak kW Savings Algorithm [[1076]](#footnote-1076)**

ΔkW = ΔkWh / (HOURSDAY x DAYS)

**Annual Fossil Fuel Savings Algorithm**

MMBtui = (MMBtu\_Cookingi + MMBtu\_Idlei) x DAYS

MMBtu\_Cookingi = LB x EFOOD/EFFi

MMBtu\_Idlei = IDLEi x (HOURSDAY – LB/PCi)

MMBtui = [LB x EFOOD/EFFi + IDLEi x (HOURSDAY – LB/PCi)] x DAYS

ΔMMBtu = MMBtubase - MMBtueff

*Where:[[1077]](#footnote-1077)*

*MMBtu\_Cookingi = daily cooking energy consumption (MMBtu)*

*MMBtu\_Idlei = daily idle energy consumption (MMBtu)*

*MMBtubase = the annual energy usage of the baseline equipment calculated using baseline values*

*MMBtueff = the annual energy usage of the efficient equipment calculated using efficient values*

*EFOOD = ASTM Energy to Food (MMBtu/lb); the amount of energy absorbed by the food during cooking, per pound of food*

*= 0.00057*

*IDLE = Idle energy rate (MMBtu/h)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

**Gas Fryer Performance Metrics: Baseline and Efficient Values**

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Parameter** | **Standard Size** | | **Large Vat** | |
| **Baseline Model** | **Energy Efficient Model** | **Baseline Model** | **Energy Efficient Model** |
| IDLE (MMBtu/h) | 0.014 | 0.009 | 0.016 | 0.012 |
| EFF | 35% | 50% | 35% | 50% |
| PC | 60 | 65 | 100 | 110 |

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

For electric fryers, the incremental cost is assumed to be $210 for standard sized equipment and $0 for large vat equipment.[[1078]](#footnote-1078) For gas fryers, the incremental cost is assumed to be $2,441 for both standard sized and large vat equipment[[1079]](#footnote-1079).

**Measure Life**

12 years[[1080]](#footnote-1080)

**Operation and Maintenance Impacts**

n/a

**Commercial Steam Cookers**

**Unique Measure Code(s): CI\_KE\_TOS\_STMR\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

Energy efficient steam cookers that have earned the ENERGY STAR label offer shorter cook times, higher production rates, and reduced heat loss due to better insulation and more efficient steam delivery system. This measure applies to time of sale opportunities.

**Definition of Baseline Condition**

The baseline condition assumes a standard efficiency electric or gas boiler-style steam cooker.

**Definition of Efficient Condition**

The efficient condition assumes the installation of an ENERGY STAR qualified electric or gas steam cooker.[[1081]](#footnote-1081)

**Annual Energy Savings Algorithm**

kWhi = (kWh\_Cookingi + kWh\_Idlei) x DAYS

kWh\_Cookingi = LB x EFOOD/EFFi

kWh\_Idlei = [(1 - PCTsteam) x IDLEi + PCTsteam x PCi x PANS x EFOOD /EFFi] x TIMEidle

TIMEidle = (HOURSDAY – LB/(PCi x PANS))

kWhi = [LB x EFOOD/EFFi + ((1 - PCTsteam) x IDLEi + PCTsteam x PCi x PANS x EFOOD /EFFi) x (HOURSDAY – LB/(PCi x PANS))] x DAYS

ΔkWh = kWhbase - kWheff

*Where: [[1082]](#footnote-1082)*

*i = either “base” or “eff” depending on whether the calculation of energy consumption is being performed for the baseline or efficient case, respectively.*

*kWh\_Cookingi = daily cooking energy consumption (kWh)*

*kWh\_Idlei = daily idle energy consumption (kWh)*

*Timeidle = daily idle time (h)*

*kWhbase = the annual energy usage of the baseline equipment calculated using baseline values*

*kWheff = the annual energy usage of the efficient equipment calculated using efficient values*

*DAYS = annual days of operation*

*= if annual days of operation are unknown, assume default of 365 days.*

*LB = Pounds of food cooked per day (lb/day)*

*= if average pounds of food cooked per day is unknown, assume default of 100 lbs/day.*

*EFOOD = ASTM Energy to Food (kWh/lb); the amount of energy absorbed by the food during cooking, per pound of food*

*= 0.0308*

*EFF = Heavy load cooking energy efficiency (%)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*PCTsteam = percent of time in constant steam mode (%)*

*= if percent of time in constant steam mode is unknown, assume default of 40%.*

*IDLE = Idle energy rate (kW/h)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*PC = Production capacity per pan (lb/hr)*

*= default baseline production capacity per pan is 23.3. If actual efficient production capacity per pan is unknown, assume default of 16.7.*

*PANS = number of pans per unit*

*= actual installed number of pans per unit*

*HOURSDAY = average daily operating hours*

*= if average daily operating hours are unknown, assume default of 12 hours/day.*

**Electric Steam Cooker Performance Metrics: Baseline and Efficient Values**

| **Parameter** | **No. of Pans** | **Baseline Model** | | **Energy Efficient Model** |
| --- | --- | --- | --- | --- |
| **Steam Generator** | **Boiler Based** | **All** |
| IDLE (kW) | 3 | 1.200 | 1.000 | 0.400 |
| 4 | 0.530 |
| 5 | 0.670 |
| 6+ | 0.800 |
| EFF | All | 30% | 26% | 50% |

**Summer Coincident Peak kW Savings Algorithm [[1083]](#footnote-1083)**

ΔkW = ΔkWh / (HOURSDAY x DAYS)

**Annual Fossil Fuel Savings Algorithm**

MMBtui = (MMBtu\_Cookingi + MMBtu\_Idlei) x DAYS

MMBtu\_Cookingi = LB x EFOOD/EFFi

MMBtu\_Idlei = [(1 - PCTsteam) x IDLEi + PCTsteam x PCi x PANS x EFOOD /EFFi] x TIMEidle

TIMEidle = (HOURSDAY – LB/(PCi x PANS))

MMBtui = [LB x EFOOD/EFFi + ((1 - PCTsteam) x IDLEi + PCTsteam x PCi x PANS x EFOOD /EFFi) x (HOURSDAY – LB/(PCi x PANS))] x DAYS

ΔMMBtu = MMBtubase - MMBtueff

*Where: [[1084]](#footnote-1084)*

*MMBtubase = the annual energy usage of the baseline equipment calculated using baseline values*

*MMBtueff = the annual energy usage of the efficient equipment calculated using efficient values*

*MMBtu\_Cookingi = daily cooking energy consumption (MMBtu)*

*MMBtu\_Idlei = daily idle energy consumption (MMBtu)*

*EFOOD = ASTM Energy to Food (MMBtu/lb); the amount of energy absorbed by the food during cooking, per pound of food*

*= 0.000105*

*IDLE = Idle energy rate (MMBtu/h)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*PC = Production capacity per pan (lb/hr)*

*= default baseline production capacity per pan is 23.3. If actual efficient production capacity per pan is unknown, assume default of 20.*

**Gas Steam Cooker Performance Metrics: Baseline and Efficient Values**

| **Parameter** | **No. of Pans** | **Baseline Model** | | **Energy Efficient Model** |
| --- | --- | --- | --- | --- |
| **Steam Generator** | **Boiler Based** | **All** |
| IDLE (MMBtu) | 3 | 0.018 | 0.015 | 0.00625 |
| 4 | 0.00835 |
| 5 | 0.01040 |
| 6+ | 0.01250 |
| EFF | All | 18% | 15% | 38% |

**Annual Water Savings Algorithm**

ΔWater = (GPHbase – GPHeff) x HOURSDAY x DAYS

*Where: [[1085]](#footnote-1085)*

*GPHbase = Water consumption rate (gal/h) of baseline equipment*

*= if water consumption rate of baseline equipment is unknown, assume default values from table below.*

*GPHeff = Water consumption rate (gal/h) of efficient equipment*

*= if water consumption rate of efficient equipment is unknown, assume default values from table below.*

| **Parameter** | **No. of Pans** | **Baseline Model** | **Energy Efficient Model** | | |
| --- | --- | --- | --- | --- | --- |
| **All** | **Steam Generator** | **Boiler Based** | **Boilerless** |
| GPH | All | 40 | 15 | 10 | 3 |

**Incremental Cost**[[1086]](#footnote-1086)

The incremental cost of an electric ENERGY STAR steam cooker is $630 for 3-pans, $1,210 for 4-pans, $0 for 5-pans, and $0 for 6-pans+. The incremental cost of a gas ENERGY STAR steam cooker is $260 for 3-pans, N/A for 4-pans, $0 for 5-pans, and $870 for 6-pans+.

**Measure Life**

12 years[[1087]](#footnote-1087)

**Operation and Maintenance Impacts**

n/a

**Commercial Hot Food Holding Cabinets**

**Unique Measure Code(s): CI\_KE\_TOS\_HFHC\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

Commercial insulated hot food holding cabinet models that meet ENERGY STAR requirements incorporate better insulation, reducing heat loss, and may also offer additional energy saving devices such as magnetic door gaskets, auto-door closures, or dutch doors. The insulation of the cabinet also offers better temperature uniformity within the cabinet from top to bottom. This means that qualified hot food holding cabinets are more efficient at maintaining food temperature while using less energy. This measure applies to time of sale opportunities.

**Definition of Baseline Condition**

The baseline equipment is assumed to be a standard efficiency hot food holding cabinet.

**Definition of Efficient Condition**

The efficient equipment is assumed to be an ENERGY STAR qualified hot food holding cabinet.[[1088]](#footnote-1088)

**Annual Energy Savings Algorithm**

ΔkWh = (IDLEbase - IDLEeff) / 1000 x HOURSDAY x DAYS

*Where:[[1089]](#footnote-1089)*

*IDLEbase = the idle energy rate of the baseline equpiment (W). See table below for calculation of default values.*

*IDLEeff = the idle energy rate of the efficient equipment (W). If actual efficient values are unknown, assume default values from table below.*

*1,000 = conversion of W to kW*

*HOURSDAY = average daily operating hours*

*= if average daily operating hours are unknown, assume default of 15 hours/day.*

*DAYS = annual days of operation*

*= if annual days of operation are unknown, assume default of 365 days.*

**Summer Coincident Peak kW Savings Algorithm [[1090]](#footnote-1090)**

ΔkW = (IDLEbase - IDLEeff) / 1000

**Hot Food Holding Cabinet Performance Metrics: Baseline and Efficient Values**

|  |  |  |
| --- | --- | --- |
| **VOLUME (Cubic Feet)** | **Product Idle Energy Consumption Rate (Watts)** | |
| **Baseline Model (IDLEbase)** | **Efficient Model (IDLEeff)** |
| 0 < VOLUME < 13 | 40 x VOLUME | 21.5 x VOLUME |
| 13 ≤ VOLUME < 28 | 40 x VOLUME | 2.0 x VOLUME + 254.0 |
| 28 ≤ VOLUME | 40 x VOLUME | 3.8 x VOLUME + 203.5 |

*Note: VOLUME = the internal volume of the holding cabinet (ft3).*

*= actual volume of installed unit*

**Annual Fossil Fuel Savings Algorithm**

n/a

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for ENERGY STAR hot food holding cabinets is assumed to be $0.[[1091]](#footnote-1091)

**Measure Life**

12 years[[1092]](#footnote-1092)

**Operation and Maintenance Impacts**

n/a

**Commercial Griddles**

**Unique Measure Code(s): CI\_KE\_TOS\_GRID\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

ENERGY STAR qualified commercial griddles have higher cooking energy efficiency and lower idle energy rates than standard equipment. The result is more energy being absorbed by the food compared with the total energy use, and less wasted energy when the griddle is in standby mode. This measure applies to time of sale opportunities.

**Definition of Baseline Condition**

The baseline equipment is assumed to be a standard efficiency electric griddle with a cooking energy efficiency of 65% or a gas griddle with a cooking efficiency of 32%.

**Definition of Efficient Condition**

The efficient equipment is assumed to be an ENERGY STAR qualified electric or gas griddle.[[1093]](#footnote-1093)

**Annual Energy Savings Algorithm**

kWhi = (kWh\_Cookingi + kWh\_Idlei) x DAYS

kWh\_Cookingi = LB x EFOOD/EFFi

kWh\_Idlei = IDLEi x SIZE x [HOURSDAY – LB/(PCi x SIZE)]

kWhi = [LB x EFOOD/EFFi + IDLEi x SIZE x (HOURSDAY – LB/(PCi x SIZE))] x DAYS

ΔkWh = kWhbase - kWheff

*Where:[[1094]](#footnote-1094)*

*i = either “base” or “eff” depending on whether the calculation of energy consumption is being performed for the baseline or efficient case, respectively.*

*kWh\_Cookingi = daily cooking energy consumption (kWh)*

*kWh\_Idlei = daily idle energy consumption (kWh)*

*kWhbase = the annual energy usage of the baseline equipment calculated using baseline values*

*kWheff = the annual energy usage of the efficient equipment calculated using efficient values*

*LB = Pounds of food cooked per day (lb/day)*

*= if average pounds of food cooked per day is unknown, assume default of 100 lbs/day.*

*EFOOD = ASTM Energy to Food (kWh/lb); the amount of energy absorbed by the food during cooking, per pound of food*

*= 0.139*

*EFF = Heavy load cooking energy efficiency (%)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*IDLE = Idle energy rate (kW/ft2)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*SIZE = size of the griddle surface (ft2)*

*HOURSDAY = average daily operating hours*

*= if average daily operating hours are unknown, assume default of 12 hours/day.*

*PC = Production capacity (lb/hr/ft2)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*DAYS = annual days of operation*

*= if annual days of operation are unknown, assume default of 365 days.*

**Efficient Griddle Performance Metrics: Baseline and Efficient Values**

|  |  |  |
| --- | --- | --- |
| **Parameter** | **Baseline Model** | **Efficient Model** |
| IDLE (kW/ft2) | 0.40 | 0.32 |
| EFF | 65% | 70% |
| PC | 5.83 | 6.67 |

**Summer Coincident Peak kW Savings Algorithm [[1095]](#footnote-1095)**

ΔkW = ΔkWh / (HOURSDAY x DAYS)

**Annual Fossil Fuel Savings Algorithm**

MMBtui = (MMBtu\_Cookingi + MMBtu\_Idlei) x DAYS

MMBtu\_Cookingi = LB x EFOOD/EFFi

MMBtu\_Idlei = IDLEi x SIZE x [HOURSDAY – LB/(PCi x SIZE)]

MMBtui = [LB x EFOOD/EFFi + IDLEi x SIZE x (HOURSDAY – LB/(PCi x SIZE))] x DAYS

ΔMMBtu = MMBtubase - MMBtueff

*Where:[[1096]](#footnote-1096)*

*MMBtu\_Cookingi = daily cooking energy consumption (MMBtu)*

*MMBtu\_Idlei = daily idle energy consumption (MMBtu)*

*MMBtubase = the annual energy usage of the baseline equipment calculated using baseline values*

*MMBtueff = the annual energy usage of the efficient equipment calculated using efficient values*

*EFOOD = ASTM Energy to Food (MMBtu/lb); the amount of energy absorbed by the food during cooking, per pound of food*

*= 0.000475*

*IDLE = Idle energy rate (MMBtu/h/ft2)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

**Gas Griddle Performance Metrics: Baseline and Efficient Values**

|  |  |  |
| --- | --- | --- |
| **Parameter** | **Baseline Model** | **Efficient Model** |
| IDLE (MMBtu/h/ft2) | 0.00350 | 0.00265 |
| EFF | 32% | 38% |
| PC | 4.17 | 7.50 |

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**[[1097]](#footnote-1097)

The incremental cost of an electric ENERGY STAR griddle is assumed to be $0. The incremental cost of a gas ENERGY STAR griddle is assumed to be $360.

**Measure Life**

12 years[[1098]](#footnote-1098)

**Operation and Maintenance Impacts**

n/a

**Commercial Convection Ovens**

**Unique Measure Code(s): CI\_KE\_TOS\_CONOV\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

Commercial convection ovens that are ENERGY STAR certified have higher heavy load cooking efficiencies and lower idle energy rates making them on average about 20 percent more efficient than standard models. This measure applies to time of sale opportunities.

**Definition of Baseline Condition**

The baseline equipment is assumed to be a standard efficiency convection oven with a heavy load efficiency of 65% for full size (i.e., a convection oven this is capable of accommodating full-size sheet pans measuring 18 x 26 x 1-inch) electric ovens, 68% for half size (i.e., a convection oven that is capable of accommodating half-size sheet pans measuring 18 x 13 x 1-inch) electric ovens, and 30% for gas ovens.

**Definition of Efficient Condition**

The efficient equipment is assumed to be an ENERGY STAR qualified electric or gas convection oven.[[1099]](#footnote-1099)

**Annual Energy Savings Algorithm**

kWhi = (kWh\_Cookingi + kWh\_Idlei) x DAYS

kWh\_Cookingi = LB x EFOOD/EFFi

kWh\_Idlei = IDLEi x (HOURSDAY – LB/PCi)

kWhi = [LB x EFOOD/EFFi + IDLEi x (HOURSDAY – LB/PCi)] x DAYS

ΔkWh = kWhbase - kWheff

*Where: [[1100]](#footnote-1100)*

*i = either “base” or “eff” depending on whether the calculation of energy consumption is being performed for the baseline or efficient case, respectively.*

*kWh\_Cookingi = daily cooking energy consumption (kWh)*

*kWh\_Idlei = daily idle energy consumption (kWh)*

*kWhbase = the annual energy usage of the baseline equipment calculated using baseline values*

*kWheff = the annual energy usage of the efficient equipment calculated using efficient values*

*HOURSDAY = average daily operating hours*

*= if average daily operating hours are unknown, assume default of 12 hours/day.*

*DAYS = annual days of operation*

*= if annual days of operation are unknown, assume default of 365 days.*

*EFOOD = ASTM Energy to Food (kWh/lb); the amount of energy absorbed by the food during cooking, per pound of food*

*= 0.0732*

*LB = Pounds of food cooked per day (lb/day)*

*= if average pounds of food cooked per day is unknown, assume default of 100 lbs/day.*

*EFF = Heavy load cooking energy efficiency (%)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*IDLE = Idle energy rate (kW)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*PC = Production capacity (lb/hr)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

**Electric Convection Oven Performance Metrics: Baseline and Efficient Values**[[1101]](#footnote-1101)

|  |  |  |  |  |
| --- | --- | --- | --- | --- |
| **Parameter** | **Half Size** | | **Full Size** | |
| **Baseline Model** | **Energy Efficient Model** | **Baseline Model** | **Energy Efficient Model** |
| IDLE (kW) | 1.03 | 1.00 | 2.00 | 1.60 |
| EFF | 68% | 71% | 65% | 71% |
| PC | 45 | 50 | 90 | 90 |

**Summer Coincident Peak kW Savings Algorithm [[1102]](#footnote-1102)**

ΔkW = ΔkWh / (HOURSDAY x DAYS)

**Annual Fossil Fuel Savings Algorithm**

MMBtui = (MMBtu\_Cookingi + MMBtu\_Idlei) x DAYS

MMBtu\_Cookingi = LB x EFOOD/EFFi

MMBtu\_Idlei = IDLEi x (HOURSDAY – LB/PCi)

MMBtui = [LB x EFOOD/EFFi + IDLEi x (HOURSDAY – LB/PCi)] x DAYS

ΔMMBtu = MMBtubase - MMBtueff

*Where:[[1103]](#footnote-1103)*

*MMBtu\_Cookingi = daily cooking energy consumption (MMBtu)*

*MMBtu\_Idlei = daily idle energy consumption (MMBtu)*

*MMBtubase = the annual energy usage of the baseline equipment calculated using baseline values*

*MMBtueff = the annual energy usage of the efficient equipment calculated using efficient values*

*EFOOD = ASTM Energy to Food (MMBtu/lb); the amount of energy absorbed by the food during cooking, per pound of food*

*= 0.000250*

*IDLE = Idle energy rate (MMBtu/h)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

**Gas Convection Oven Performance Metrics: Baseline and Efficient Values**

| **Parameter** | **Baseline Model** | **Energy Efficient Model** |
| --- | --- | --- |
| IDLE (MMBtu/h) | 0.0151 | 0.0120 |
| EFF | 44% | 46% |
| PC | 83 | 86 |

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost is assumed to be $0[[1104]](#footnote-1104) for electric commercial convection ovens and -$1,778[[1105]](#footnote-1105) for gas ovens.

**Measure Life**

12 years[[1106]](#footnote-1106)

**Operation and Maintenance Impacts**

n/a

**Commercial Combination Ovens**

**Unique Measure Code(s): CI\_KE\_TOS\_COMOV\_0615**

**Effective Date: June 2015**

**End Date: TBD**

**Measure Description**

A combination oven is a convection oven that includes the added capability to inject steam into the oven cavity and typically offers at least three distinct cooking modes. This measure applies to time of sale opportunities.

**Definition of Baseline Condition**

The baseline equipment is assumed to be a typical standard efficiency electric or gas combination oven.

**Definition of Efficient Condition**

The efficient equipment is assumed to be an ENERGY STAR qualified electric or gas combination oven.[[1107]](#footnote-1107)

**Annual Energy Savings Algorithm**

kWhi,j = (kWh\_Cookingi,j + kWh\_Idlei,j) x DAYS

kWh\_Cookingi,j = LB x EFOOD,j/EFFi,j x PCTj

kWh\_Idlei,j = IDLEi,j x (HOURSDAY – LB/PCi,j) x PCTj

kWhi,j = [LB x EFOOD,j/EFFi,j + IDLEi,j x (HOURSDAY – LB/PCi,j)] x PCTj x DAYS

kWhbase = kWhbase,conv + kWhbase,steam

kWheff = kWheff,conv + kWheff,steam

ΔkWh = kWhbase - kWheff

*Where:[[1108]](#footnote-1108)*

*i = either “base” or “eff” depending on whether the calculation of energy consumption is being performed for the baseline or efficient case, respectively.*

*j = cooking mode; either “conv” (i.e., convection) or “steam”*

*kWh\_Cookingi,j = daily cooking energy consumption (kWh)*

*kWh\_Idlei,j = daily idle energy consumption (kWh)*

*kWhbase = the annual energy usage of the baseline equipment calculated using baseline values*

*kWheff = the annual energy usage of the efficient equipment calculated using efficient values*

*HOURSDAY = average daily operating hours*

*= if average daily operating hours are unknown, assume default of 12 hours/day.*

*DAYS = annual days of operation*

*= if annual days of operation are unknown, assume default of 365 days.*

*EFOOD,conv = ASTM Energy to Food (kWh/lb); the amount of energy absorbed by the food during convention mode cooking, per pound of food*

*= 0.0732*

*EFOOD,steam = ASTM Energy to Food (kWh/lb); the amount of energy absorbed by the food during steam mode cooking, per pound of food*

*= 0.0308*

*LB = Pounds of food cooked per day (lb/day)*

*= if average pounds of food cooked per day is unknown, assume default of 200 lbs/day.*

*EFF = Heavy load cooking energy efficiency (%)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*IDLE = Idle energy rate (kW)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*PC = Production capacity (lb/hr)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

*PCTj = percent of food cooked in cooking mode j. Note: PCTconv + PCTsteam must equal 100%.*

*= if percent of food cooked in cooking mode j is unknown, assume default of PCTconv = PCTsteam = 50%.*

**Electric Combination Oven Performance Metrics: Baseline and Efficient Values**

| **Parameter** | **No. of Pans** | **Baseline Model** | | **Energy Efficient Model** | |
| --- | --- | --- | --- | --- | --- |
| **Convection Mode** | **Steam Mode** | **Convection Mode** | **Steam Mode** |
| IDLE (kW) | < 15 | 1.320 | 5.260 | 0.08 x PANS + 0.4989 | 0.133 x PANS + 0.64 |
| >= 15 | 2.280 | 8.710 |
| EFF | All | 72% | 49% | 76% | 55% |
| PC | < 15 | 79 | 126 | 119 | 177 |
| >= 15 | 166 | 295 | 201 | 349 |

*Note: PANS = The number of steam table pans the combination oven is able to accommodate as per the ASTM F-1495-05 standard specification.*

**Summer Coincident Peak kW Savings Algorithm [[1109]](#footnote-1109)**

ΔkW = ΔkWh / (HOURSDAY x DAYS)

**Annual Fossil Fuel Savings**

MMBtui = [LB x EFOOD/EFFi + IDLEi x (HOURSDAY – LB/PCi)] x DAYS

MMBtu\_Cookingi,j = LB x EFOOD,j/EFFi,j x PCTj

MMBtu\_Idlei,j = IDLEi,j x (HOURSDAY – LB/PCi,j) x PCTj

MMBtui,j = [LB x EFOOD,j/EFFi,j + IDLEi,j x (HOURSDAY – LB/PCi,j)] x PCTj x DAYS

MMBtubase = kWhbase,conv + kWhbase,steam

MMBtueff = kWheff,conv + kWheff,steam

ΔMMBtu = MMBtubase - MMBtueff

*Where:[[1110]](#footnote-1110)*

*MMBtu\_Cookingi = daily cooking energy consumption (MMBtu)*

*MMBtu\_Idlei = daily idle energy consumption (MMBtu)*

*MMBtubase = the annual energy usage of the baseline equipment calculated using baseline values*

*MMBtueff = the annual energy usage of the efficient equipment calculated using efficient values*

*EFOOD,conv = ASTM Energy to Food (MMBtu/lb); the amount of energy absorbed by the food during convention mode cooking, per pound of food*

*= 0.000250*

*EFOOD,steam = ASTM Energy to Food (MMBtu/lb); the amount of energy absorbed by the food during steam mode cooking, per pound of food*

*= 0.000105*

*LB = Pounds of food cooked per day (lb/day)*

*= if average pounds of food cooked per day is unknown, assume default of 250 lbs/day.*

*IDLE = Idle energy rate (MMBtu/h)*

*= see table below for default baseline values. If actual efficient values are unknown, assume default values from table below.*

**Gas Combination Oven Performance Metrics: Baseline and Efficient Values**

| **Parameter** | **No. of Pans** | **Baseline Model** | | **Energy Efficient Model** | |
| --- | --- | --- | --- | --- | --- |
| **Convection Mode** | **Steam Mode** | **Convection Mode** | **Steam Mode** |
| IDLE (MMBtu/h) | < 15 | 0.008747 | 0.018656 | 0.000150 x PANS + 0.005425 | 0.000200 x PANS + 0.006511 |
| >= 15 and < 30 | 0.007823 | 0.024562 |
| >= 30 | 0.013000 | 0.043300 |
| EFF | All | 52% | 39% | 56% | 41% |
| PC | < 15 | 125 | 195 | 124 | 172 |
| >= 15 and < 30 | 176 | 211 | 210 | 277 |
| >= 30 | 392 | 579 | 394 | 640 |

*Note: PANS = The number of steam table pans the combination oven is able to accommodate as per the ASTM F-1495-05 standard specification.*

**Annual Water Savings Algorithm**

n/a

**Incremental Cost**

The incremental cost for commercial combination ovens is assumed to be $0[[1111]](#footnote-1111)

**Measure Life**

12 years[[1112]](#footnote-1112)

**Operation and Maintenance Impacts**

n/a

**APPENDIX**

1. **Supporting Calculation Work Sheets**

For each of the embedded excel work sheets below, double click to open the file and review the calculations.

1. MidAtlantic Lighting Adjustments and O&M.xls – this contains 6 tabs; the first details the ISR and Measure Life adjustments, the second the CFL delta watts multiplier calculations, and the remaining tabs show the Operation and Maintenance calculations for RES CFL, RES Interior Fixture, RES Exterior Fixtures and C&I CFL.
2. **Recommendation for Process and Schedule for Maintenance and Update of TRM Contents**
3. **Description of Unique Measure Codes**
4. **Commercial & Industrial Lighting Operating Hours, Coincidence Factors, and Waste Heat Factors**
5. **Supporting Calculation Work Sheets**



1. **Recommendation for Process and Schedule for Maintenance and Update of TRM Contents**

Once developed, the Mid-Atlantic TRM will benefit from an objective and thoughtful update process. Defining a process that coordinates with the needs of users, evaluators, and regulators is critical. Below we outline our preliminary proposal for a process for the update of information and recommendations on the coordination of the timing of this process with other critical activities.

Proposed TRM Update Process

Once a TRM has been developed, it is vital that it is kept up to date, amended, and maintained in a timely and effective manner. There are three main points in time when a TRM is most likely to require changes:

1. New measure additions – As new technologies become cost effective, they will need to be characterized and added to the manual.
2. Existing measure updates – Updates will be required for a number of reasons. Examples include: the federal standard for efficiency of a measure is increased; the qualification criteria are altered; the measure cost falls; or a new evaluation provides a better value of an assumption for a variable. In such cases, the changes must be flagged and appropriate changes made to the TRM.
3. Retiring existing measures – When the economics of a measure become such that it is no longer cost effective, or the free rider rate is so high that it is not worth supporting, the measure should be retired.

It is important to maintain a record of changes made to the TRMs over time. It is therefore recommended to establish and maintain a Master Manual, containing all versions of each TRM in chronological order, and an abridged User Manual, in which only the current versions of active measures are included. Archived older information can be made available on a website or other accessible location.

The flowchart presented below outlines steps that will result in effective review and quality control for TRM updates.



***Key Roles and Responsibilities***

This process requires a number of different roles to ensure effectiveness, sufficient review, and independence. The specific parties who will hold these roles in the Mid-Atlantic TRM maintenance context will need to be identified by jurisdiction. The following list of key responsibilities is given as a starting place:

* Program administrators (utilities, MEA, SEU)
  + Identifies need for new or revised measure characterization (usually due to program changes or program/market feedback)
  + Researches and develops 1st draft measure characterizations when it identifies need
  + Develops 2nd draft measure characterizations following feedback on 1st draft from all parties
  + Feedback on draft measure characterizations from other parties
  + Participant in Technical Advisory Group (TAG) for formal discussion and dispute resolution when needed
  + Input to regulators if TAG process does not resolve all issues
* Independent TRM Manager (consultant or mutually agreed upon nominee)
  + Identifies need for revised measure characterization (usually based on knowledge of local or other relevant evaluation studies)
  + Researches and develops 1st draft measure characterizations when it identifies need
  + Feedback on 1st draft measure characterizations from other parties
  + Develops 2nd draft measure characterizations following feedback on 1st draft from all parties
  + Leads Technical Advisory Group (TAG) for formal discussion and dispute resolution when needed
  + Input to regulators if TAG process does not resolve all issues
  + Manages and updates TRM manuals
* Evaluators
  + Identifies need for revised measure characterization (usually based on local evaluation studies it has conducted or managed)
  + Input on draft measure characterizations developed by other parties
  + Participates in TAG meetings when appropriate
  + Performs program evaluation - includes statewide market assessment and baseline studies, savings impact studies (to measure the change in energy and / or demand use attributed to energy efficiency), and other energy efficiency program evaluation activities
  + Verifies annual energy and capacity savings claims of each program and portfolio
* Regulators/Commission staff
  + May serve as ultimate decision maker in any unresolved disputes between implementers, evaluators, and TRM Manager

Note that the process and responsibilities outlined above assume that the manager of the TRM is an entity independent from the program administrators. This is the approach the state of Ohio has recently adopted, with the Public Utilities Commission hiring a contractor to serve that function. Alternatively, the TRM could be managed by the Program Administrators themselves. That approach can also work very well as long as there is an independent party responsible for (1) reviewing and (2) either agreeing with proposed additions/changes or challenging such changes – with the regulators having final say regarding any disputes.

The process outlined above also assumes that there are several potential stages of “give and take” on draft modifications to the TRM. At a minimum, there is at least one round of informal feedback and comment between the program administrators and the independent reviewer (TRM manager or otherwise). Other parties could be invited to participate in this process as well. In the event that such informal discussions do not resolve all issues, the participants may find it beneficial to establish a Technical Advisory Group (TAG) to provide a more formal venue for resolution of technical disputes prior to any submission to the regulators. This group would include representation from the program administrators, the evaluators (when deemed useful), the TRM Manager, and Commission staff. The mission of such a group would be to discuss and reach agreement on any unresolved issues stemming from new measure proposals, savings verifications, or evaluations. They could also review and comment on the methodology and associated assumptions underlying measure savings calculations and provide an additional channel for transparency of information about the TRM and the savings assessment process.

Coordination with Other Savings Assessment Activities

Although the TRM will be a critically important tool for both DSM planning and estimation of actual savings, it will not, by itself, ensure that reported savings are the same as actual savings. There are two principal reasons for this:

1. **The TRM itself does not ensure appropriate estimation of savings**. One of the responsibilities of the Independent Program Evaluators will be to assess that the TRM has been used appropriately in the calculation of savings.
2. **The TRM may have assumptions or protocols that new information suggests are outdated**. New information that could inform the reasonableness of TRM assumptions or protocols can surface at any time, but they are particularly common as local evaluations or annual savings verification processes are completed. Obviously, the TRM should be updated to reflect such new information. However, it is highly likely that some such adjustments will be made too late to affect the annual savings estimate of a program administrator for the previous year. Thus, there may be a difference between savings estimates in annual compliance reports and the “actual savings” that may be considered acceptable from a regulatory perspective. However, such updates should be captured in as timely a fashion as possible.

These two issues highlight the fact that the TRM needs to be integrated into a broader process that has two other key components: an annual savings verification process and on-going evaluation.

In our view, an annual savings verification process should have several key features.

1. It should include a review of data tracking systems used to record information on efficiency measures that have been installed. Among other things, this review should assess whether data appear to have been appropriately and accurately entered into the system.
2. It should include a review of all deemed savings assumptions underlying the program administrators’ savings claims to ensure that they are consistent with the TRM.
3. It should include a detailed review of a statistically valid, random sample of custom commercial and industrial projects to ensure that custom savings protocols were appropriately applied. At a minimum, engineering reviews should be conducted; ideally, custom project reviews should involve some on-site assessments as well.
4. These reviews should be conducted by an independent organization with appropriate expertise.
5. The participants will need to have a process in place for quickly resolving any disputes between the utilities or program administrators on the one hand and the independent reviewer on the other.
6. The results of the independent review and the resolution of any disagreements should ideally be very transparent to stakeholders.

Such verification ensures that information is being tracked accurately and in a manner consistent with the TRM. However, as important as it is, verification does not ensure that reported savings are “actual savings”. TRMs are never and can never be perfect. Even when the verification process documents that assumptions have been appropriately applied, it can also highlight questions that warrant future analysis that may lead to changes to the TRM. Put another way, evaluation studies are and always will be necessary to identify changes that need to be made to the TRM. Therefore, in addition to annual savings verification processes, evaluations will periodically be made to assess or update the underlying assumption values for critical components of important measure characterizations.

In summary, there should be a strong, sometimes cyclical relationship between the TRM development and update process, annual compliance reports, savings verification processes, and evaluations. As such, we recommend coordinating these activities. An example of the timeline established from such a coordinated process is given below.

In this example, it assumed that updates to the TRM occur only in the second half of the year. One option is to establish two specific update deadlines: one at the end of August and the other at the end of December. The first would ensure that the best available data are available for utility planning for the following year. The second would ensure that best available assumptions are in place prior to the start of the new program year. The rationale for not updating the TRM during the first half of the year is that time is usually devoted, in part, to documenting, verifying and approving savings claims from the previous year. For example, the program administrator will likely require two months to produce its annual savings claim for the previous year. An independent reviewer will then require two to three months to review and probe that claim, with considerable back and forth between the two parties being very common. Typically, final savings estimates for the previous year are not finalized and approved until June.

Needless to say, the definitive schedule for savings verification and TRM updating will need to be developed with considerable input from state regulators. This plan and timeline will be also informed by each region’s Independent Program Evaluator and the EM&V plans they propose.

Annual Verification and TRM Update Timeline (example)

|  |  |  |  |  |  |  |  |  |  |  |  |  |
| --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- | --- |
|  | Jan | Feb | Mar | Apr | May | Jun | Jul | Aug | Sep | Oct | Nov | Dec |
| **Utility** | Draft annual savings report | | No TRM submittal during SV | | |  | Draft new or updated TRMs developed and submitted to TRM Manager, participate in TAG | | | | | |
|  | | | SV Response | | Prior year data finalized | Technical Advisory Group (TAG) negotiations and evaluation | | | | | |
| **Evaluator** |  | | Savings Verification (SV) | | |  |
|  | | No TRM review during SV | | |  | Refers need for TRM updates to TRM Manager, provides input on TRMs | | | | | |
| **TRM Manager/ Implementation staff** |  | |  | | Make final savings determination | | Draft new or updated TRMs developed, Review drafts provided by utilities, participate in TAG, propose new or updated TRMs | | | | | |

## Description of Unique Measure Codes

Each measure included in the TRM has been assigned a unique identification code. The code consists of a string of five descriptive categories connected by underscores, in the following format:

Sector\_End Use\_Program Type\_Measure\_MonthYear

A description of the abbreviations used in the codes is provided in the tables below:

|  |  |
| --- | --- |
| **SECTOR** | |
| **RS** | **Residential** |
| **CI** | **Commercial & Industrial** |
| **END USE** | |
| **LT** | **Lighting** |
| **RF** | **Refrigeration** |
| **HV** | **Heating, Ventilation, Air Conditioning** |
| **WT** | **Hot Water** |
| **LA** | **Laundry** |
| **SL** | **Shell (Building)** |
| **MO** | **Motors and Drives** |
| **KE** | **Commercial Kitchen Equipment** |
| **PL** | **Plug Load** |
| **PROGRAM TYPE** | |
| **TOS** | **Time of Sale** |
| **RTR** | **Retrofit** |
| **ERT** | **Early Retirement** |
| **INS** | **Direct Install** |
| **MEASURE** | |
| **CFLSCR** | **Compact Fluorescent Screw-In** |
| **CFLFIN** | **Compact Fluorescent Fixture, Interior** |
| **CFLFEX** | **Compact Fluorescent Fixture, Exterior** |
| **REFRIG** | **Refrigerator** |
| **FANMTR** | **Furnace Fan Motor** |
| **RA/CES** | **Window Air Conditioner Energy Star** |
| **RA/CT1** | **Window Air Conditioner Tier 1** |
| **CENA/C** | **Central Air Conditioner** |
| **SHWRHD** | **Low Flow Showerhead** |
| **FAUCET** | **Low Flow Faucet** |
| **HWWRAP** | **Water Tank Wrap** |
| **HPRSHW** | **Heat Pump Water Heater, Residential** |
| **CWASHES** | **Clothes Washer, Energy Star** |
| **CWASHT3** | **Clothes Washer, Tier 3** |
| **WINDOW** | **Window, Energy Star** |
| **HPT8** | **High Performance T8 Lighting** |
| **T5** | **T5 Lighting** |
| **MHFIN** | **Metal Halide Fixture, Interior** |
| **MHFEX** | **Metal Halide Fixture, Exterior** |
| **SODIUM** | **High Pressure Sodium Lighting** |
| **LECEXI** | **LED Exit Sign** |
| **DELAMP** | **Delamping** |
| **OSWALL** | **Occupancy Sensor, Wall box** |
| **UNIA/C** | **Unitary Air Conditioning system** |
| **EMOTOR** | **Efficient Motor** |
| **VFDRIVE** | **Variable Frequency Drive** |
| **FREEZER** | **Freezer** |
| **HPCIHW** | **Heat Pump Water Heater, Commercial** |

## Commercial & Industrial Lighting Operating Hours, Coincidence Factors, and Waste Heat Factors

**C&I Interior Lighting Operating Hours by Building Type***[[1113]](#footnote-1113)*

| **Building Type** | **Sector** | **HOURS** |
| --- | --- | --- |
| Grocery | Large Commercial/Industrial & Small Commercial | 7,134 |
| Health | Large Commercial/Industrial & Small Commercial | 3,909 |
| Office | Large Commercial/Industrial | 2,969 |
| Small Commercial | 2,950 |
| Other | Large Commercial/Industrial & Small Commercial | 4,573 |
| Retail | Large Commercial/Industrial | 4,920 |
| Small Commercial | 4,926 |
| School | Large Commercial/Industrial & Small Commercial | 2,575 |
| Warehouse/ Industrial | Large Commercial/Industrial | 4,116 |
| Small Commercial | 3,799 |
| Unknown[[1114]](#footnote-1114) | Large Commercial/Industrial | 2,575 |

Note: The “Other” building type should be used when the building type is known but not explicitly listed above. A description of the actual building type should be recorded in the project documentation.

**C&I Interior Lighting Coincidence Factors by Building Type***[[1115]](#footnote-1115)*

| **Building Type** | **Sector** | **CFSSP** | **CFPJM** |
| --- | --- | --- | --- |
| Grocery | Large Commercial/Industrial & Small Commercial | 0.96 | 0.96 |
| Health | Large Commercial/Industrial & Small Commercial | 0.8 | 0.79 |
| Office | Large Commercial/Industrial | 0.7 | 0.69 |
| Small Commercial | 0.67 | 0.67 |
| Other | Large Commercial/Industrial & Small Commercial | 0.66 | 0.67 |
| Retail | Large Commercial/Industrial | 0.96 | 0.94 |
| Small Commercial | 0.86 | 0.85 |
| School | Large Commercial/Industrial & Small Commercial | 0.50 | 0.42[[1116]](#footnote-1116) |
| Warehouse/ Industrial | Large Commercial/Industrial | 0.7 | 0.72 |
| Small Commercial | 0.68 | 0.7 |
| Unknown[[1117]](#footnote-1117) | Large Commercial/Industrial | 0.50 | 0.42 |

Note(s): 1) CFPJM refers to the PJM Summer Peak Coincidence Factor (June to August weekdays between 2 pm and 6 pm). CFSSP refers to Summer System Peak Coincidence Factor (hour ending 5pm on hottest summer weekday). 2) The “Other” building type should be used when the building type is known but not explicitly listed above. A description of the actual building type should be recorded in the project documentation.

**Waste Heat Factors for C&I Lighting – Known HVAC Types***[[1118]](#footnote-1118)*

| **State, Utility** | **Building Type** | **Demand Waste Heat Factor (WHFd)** | | **Annual Energy Waste Heat Factor by Cooling/Heating Type (WHFe)** | | | |
| --- | --- | --- | --- | --- | --- | --- | --- |
|  |  | **AC (Utility)** | **AC (PJM)** | **AC/ NonElec** | **AC/ ElecRes** | **Heat Pump** | **NoAC/ ElecRes[[1119]](#footnote-1119)** |
| Maryland, BGE | Office | 1.36 | 1.32 | 1.10 | 0.85 | 0.94 | 0.75 |
|  | Retail | 1.27 | 1.26 | 1.06 | 0.83 | 0.95 | 0.77 |
|  | School | 1.44 | 1.44 | 1.10 | 0.81 | 0.96 | 0.71 |
|  | Warehouse | 1.23 | 1.24 | 1.02 | 0.75 | 0.89 | 0.73 |
|  | Other | 1.35 | 1.33 | 1.08 | 0.82 | 0.93 | 0.74 |
| Maryland, SMECO | Office | 1.36 | 1.32 | 1.10 | 0.85 | 0.94 | 0.75 |
|  | Retail | 1.27 | 1.26 | 1.06 | 0.83 | 0.95 | 0.77 |
|  | School | 1.44 | 1.44 | 1.10 | 0.81 | 0.96 | 0.71 |
|  | Warehouse | 1.23 | 1.25 | 1.02 | 0.75 | 0.89 | 0.73 |
|  | Other | 1.35 | 1.33 | 1.08 | 0.82 | 0.93 | 0.74 |
| Maryland, Pepco | Office | 1.36 | 1.32 | 1.10 | 0.85 | 0.94 | 0.75 |
|  | Retail | 1.27 | 1.26 | 1.06 | 0.83 | 0.95 | 0.77 |
|  | School | 1.44 | 1.44 | 1.10 | 0.81 | 0.96 | 0.71 |
|  | Warehouse | 1.23 | 1.25 | 1.02 | 0.75 | 0.89 | 0.73 |
|  | Other | 1.35 | 1.33 | 1.08 | 0.82 | 0.93 | 0.74 |
| Maryland, DPL | Office | 1.35 | 1.32 | 1.10 | 0.85 | 0.94 | 0.75 |
|  | Retail | 1.27 | 1.26 | 1.06 | 0.83 | 0.95 | 0.77 |
|  | School | 1.44 | 1.44 | 1.10 | 0.81 | 0.96 | 0.71 |
|  | Warehouse | 1.22 | 1.23 | 1.02 | 0.75 | 0.89 | 0.73 |
|  | Other | 1.34 | 1.32 | 1.08 | 0.82 | 0.93 | 0.74 |
| Maryland, Potomac Edison | Office | 1.34 | 1.31 | 1.10 | 0.85 | 0.94 | 0.75 |
| Retail | 1.27 | 1.25 | 1.06 | 0.83 | 0.95 | 0.77 |
| School | 1.45 | 1.45 | 1.10 | 0.81 | 0.96 | 0.71 |
| Warehouse | 1.2 | 1.21 | 1.02 | 0.75 | 0.89 | 0.73 |
| Other | 1.33 | 1.31 | 1.08 | 0.82 | 0.93 | 0.74 |
| Washington, D.C., All | Office | 1.36 | 1.32 | 1.10 | 0.85 | 0.94 | 0.75 |
| Retail | 1.27 | 1.26 | 1.06 | 0.83 | 0.95 | 0.77 |
| School | 1.44 | 1.44 | 1.10 | 0.81 | 0.96 | 0.71 |
| Warehouse | 1.23 | 1.25 | 1.02 | 0.75 | 0.89 | 0.73 |
| Other | 1.35 | 1.33 | 1.08 | 0.82 | 0.93 | 0.74 |
| Delaware, All | Office | 1.35 | 1.32 | 1.10 | 0.85 | 0.94 | 0.75 |
| Retail | 1.27 | 1.26 | 1.06 | 0.83 | 0.95 | 0.77 |
| School | 1.44 | 1.44 | 1.10 | 0.81 | 0.96 | 0.71 |
| Warehouse | 1.22 | 1.23 | 1.02 | 0.75 | 0.89 | 0.73 |
| Other | 1.34 | 1.32 | 1.08 | 0.82 | 0.93 | 0.74 |

Note(s): The “Other” building type should be used when the building type is known but not explicitly listed above. A description of the actual building type should be recorded in the project documentation. If cooling and heating equipment types are unknown or the space is unconditioned, assume WHFd = WHFe = 1.0.

1. Typically, the algorithms provided contain a number of deemed underlying assumptions which when combined with some measure specific information (e.g. equipment capacity) produce deemed calculated savings values. [↑](#footnote-ref-1)
2. In some jurisdictions, this is called “replace on burn-out”. We use the term “time of sale” because not all new equipment purchases take place when an older existing piece of equipment reaches the end of its life. [↑](#footnote-ref-2)
3. For more discussion about the transferability of consumption data, see the EMV Forum Report: Cataloguing Available End-Use and Efficiency Measure Load Data, October 2009 at http://neep.org/emv-forum/forum-products-and-guidelines. [↑](#footnote-ref-3)
4. They are captured only for lighting measures. [↑](#footnote-ref-4)
5. The utilities might consider evaluating what percentage of retail sales end up in commercial locations, and apply the commercial CFL assumptions to that portion. In the absence of such data it is appropriate to use the Residential assumptions for all retail sales since they will represent a significant majority and result in an appropriately conservative estimate. [↑](#footnote-ref-5)
6. For text of Energy and Independence and Security Act, see http://www.gpo.gov/fdsys/pkg/PLAW-110publ140/pdf/PLAW-110publ140.pdf [↑](#footnote-ref-6)
7. Base wattage is based upon the post first phase of EISA wattage and wattage bins consistent with ENERGY STAR, v1.1; http://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Lamps%20V1%201\_Specification.pdf [↑](#footnote-ref-7)
8. Starting with a first year ISR of 0.80 (based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015) and a lifetime ISR of 0.97 (from Nexus Market Research, RLW Analytics and GDS Associates study; “New England Residential Lighting Markdown Impact Evaluation, January 20, 2009”). Assume that half of the bulbs not installed in year 1 are installed in year 2, and the other half in year 3. Using a discount rate of 5%, this gives a total ISR of 0.8 + 0.085 \* 0.95 + 0.85 \* 0.95^2 = 0.96. [↑](#footnote-ref-8)
9. Assumption is based on the EmPOWER \_EY5 Res Lighting Results Memo\_20Jan2015 DRAFT discussed above, but not adjusted upwards since those people removing bulbs after being installed in Direct Install program are likely to do so because they dislike them, not to use as replacements. Only evaluation we are aware of specifically for Direct Install installation (and persistence) rates is Megdal & Associates, 2003; “2002/2003 Impact Evaluation of LIPA's Clean Energy Initiative REAP Program”, which estimated 81%. [↑](#footnote-ref-9)
10. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 14. [↑](#footnote-ref-10)
11. Multifamily common area lighting assumption is 16.3 hours per day (5950 hours per year) based on Wisconsin’s Focus on Energy Evaluation, ACES Deemed Savings Desk Review, November 2010. This estimate is consistent with the Common Area “Non-Area Specific” assumption (16.2 hours per day or 5913 annually) from the Cadmus Group Inc., “Massachusetts Multifamily Program Impact Analysis”, July 2012, p 2-4. [↑](#footnote-ref-11)
12. Updated results from Nexus Market Research, “Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs”, Final Report, October 1, 2004, presented in 2005 memo; http://publicservice.vermont.gov/energy/ee\_files/efficiency/eval/marivtfinalresultsmemodelivered.pdf [↑](#footnote-ref-12)
13. For programs where the installation location is unknown (e.g. upstream lighting programs) the assumption is set conservatively to assume an interior residential bulb. [↑](#footnote-ref-13)
14. The value is estimated at 1.12 (calculated as 1 + (0.33 / 2.8)). Based on cooling loads decreasing by 33% of the lighting savings (average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 \* SEER2) + (1.12 \* SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP). [↑](#footnote-ref-14)
15. The value is estimated at 1.10 (calculated as 1 + (0.89\*(0.33 / 2.8)). Based on assumption that 89% of homes have central cooling (based onKEMA Maryland Energy Baseline Study. Feb 2011.). [↑](#footnote-ref-15)
16. Calculated using defaults; 1-((0.47/1.67) \* 0.375) = 0.894 [↑](#footnote-ref-16)
17. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-17)
18. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 and 2015 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-18)
19. Calculation assumes 59% Heat Pump and 41% Resistance which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey. Average efficiency of heat pump is based on assumption 50% are units from before 2006 and 50% after. [↑](#footnote-ref-19)
20. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-20)
21. The value is estimated at 1.24 (calculated as 1 + (0.66 / 2.8)). See footnote relating to WHFe for details. Note the 66% factor represents the Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load (i.e. consistent with the PJM coincident definition). [↑](#footnote-ref-21)
22. The value is estimated at 1.21 (calculated as 1 + (0.89 \* 0.66 / 2.8)). [↑](#footnote-ref-22)
23. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 16. [↑](#footnote-ref-23)
24. Ibid. [↑](#footnote-ref-24)
25. Consistent with “Lodging Common Area” coincidence factor in Commercial Screw base CFL measure characterization, based on ‘Development of Interior Lighting Hours of Use and Coincidence Factor Values for EmPOWER Maryland Commercial Lighting Program Evaluations, Itron, 2010’. [↑](#footnote-ref-25)
26. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. [↑](#footnote-ref-26)
27. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-27)
28. This has been estimated assuming typical efficiencies of existing heating systems weighted by percentage of homes with non-electric heating (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls). [↑](#footnote-ref-28)
29. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-29)
30. Based on incremental costs for 60W equivalent (dominant bulb) from “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. [↑](#footnote-ref-30)
31. Ibid. Based on 15W CFL, [↑](#footnote-ref-31)
32. Assumption based on 15 minutes (including portion of travel time) and $20 per hour. [↑](#footnote-ref-32)
33. Assumes CFLs will become baseline in 2020. The measure life has to be further reduced by one year for every install year beyond 2016. [↑](#footnote-ref-33)
34. Assumed rated life of 10,000 hours due to lower switching (10000/5950 = 1.7). [↑](#footnote-ref-34)
35. Based on for 60W EISA equivalent (dominant bulb) from “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. [↑](#footnote-ref-35)
36. Based on lamp life / assumed annual run hours. [↑](#footnote-ref-36)
37. Assumes rated life of incandescent bulb of 1000 hours. [↑](#footnote-ref-37)
38. The manufacturers are simply using a regular incandescent lamp with halogen fill gas rather than Halogen Infrared to meet the standard (as provided by G. Arnold, Optimal Energy and confirmed by N. Horowitz at NRDC) so the lifetime of these EISA qualified bulbs is assumed to be 1000 hours. [↑](#footnote-ref-38)
39. Note, these values have been adjusted by the appropriate In Service Rate – the Time of Sale assumption (0.92) is used for the Residential interior and multi-family in unit, the Direct Install assumption (0.88) for the remaining categories. The discount rate used for these calculations is 5.0%. See ‘MidAtlantic Lighting adjustments and O&M\_042015’ for more information. [↑](#footnote-ref-39)
40. The utilities might consider evaluating what percentage of retail sales end up in commercial locations, and apply the commercial CFL assumptions to that portion. In the absence of such data it is appropriate to use the Residential assumptions for all retail sales since they will represent a significant majority and result in an appropriately conservative estimate. [↑](#footnote-ref-40)
41. Based on ENERGY STAR equivalence table; <http://www.energystar.gov/index.cfm?c=cfls.pr_cfls_lumens> [↑](#footnote-ref-41)
42. A 2006-2008 California Upstream Lighting Evaluation found an average incandescent wattage of 61.7 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program. Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009) [↑](#footnote-ref-42)
43. If the bulb has greater than 500 lumens, this is categorized as general service [↑](#footnote-ref-43)
44. If the bulb has greater than 500 lumens, this is categorized as general service [↑](#footnote-ref-44)
45. The upper bounds for these categories depends on the lower bound of the next higher wattage, which varies by bulb type. [↑](#footnote-ref-45)
46. As above. [↑](#footnote-ref-46)
47. An Illinois evaluation (Energy Efficiency / Demand Response Plan: Plan Year 2 (6/1/2009-5/31/2010) Evaluation Report: Residential Energy Star ® Lighting

    <http://ilsag.org/yahoo_site_admin/assets/docs/ComEd_Res_Lighting_PY2_Evaluation_Report_2010-12-21_Final.12113928.pdf> ) reported 13-17W as the most common specialty CFL wattage (69% of program bulbs). 2009 California data also reported an average CFL wattage of 15.5 Watts (KEMA, Inc, The Cadmus Group, Itron, Inc, PA Consulting Group, Jai J. Mitchell Analytics, Draft Evaluation Report: Upstream Lighting Program, Prepared for the California Public Utilities Commission, Energy Division. December 10, 2009). [↑](#footnote-ref-47)
48. Starting with a first year ISR of 0.80 (based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015) and a lifetime ISR of 0.97 (from Nexus Market Research, RLW Analytics and GDS Associates study; “New England Residential Lighting Markdown Impact Evaluation, January 20, 2009”). Assume that half of the bulbs not installed in year 1 are installed in year 2, and the other half in year 3. Using a discount rate of 5%, this gives a total ISR of 0.8 + 0.085 \* 0.95 + 0.85 \* 0.95^2 = 0.96.

    . [↑](#footnote-ref-48)
49. Assumption is based on the EmPOWER \_EY5 Res Lighting Results Memo\_20Jan2015 DRAFT discussed above, but not adjusted upwards since those people removing bulbs after being installed in Direct Install program are likely to do so because they dislike them, not to use as replacements. Only evaluation we are aware of specifically for Direct Install installation (and persistence) rates is Megdal & Associates, 2003; “2002/2003 Impact Evaluation of LIPA's Clean Energy Initiative REAP Program”, which estimated 81%. [↑](#footnote-ref-49)
50. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 14. [↑](#footnote-ref-50)
51. Multifamily common area lighting assumption is 16.3 hours per day (5950 hours per year) based on Focus on Energy Evaluation, ACES Deemed Savings Desk Review, November 2010. This estimate is consistent with the Common Area “Non-Area Specific” assumption (16.2 hours per day or 5913 annually) from the Cadmus Group Inc., “Massachusetts Multifamily Program Impact Analysis”, July 2012, p 2-4. [↑](#footnote-ref-51)
52. Updated results from Nexus Market Research, “Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs”, Final Report, October 1, 2004, presented in 2005 memo; http://publicservice.vermont.gov/energy/ee\_files/efficiency/eval/marivtfinalresultsmemodelivered.pdf [↑](#footnote-ref-52)
53. For programs where the installation location is unknown (e.g. upstream lighting programs) the assumption is set conservatively to assume an interior residential bulb. [↑](#footnote-ref-53)
54. The value is estimated at 1.12 (calculated as 1 + (0.33 / 2.8)). Based on cooling loads decreasing by 33% of the lighting savings (average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 \* SEER2) + (1.12 \* SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP). [↑](#footnote-ref-54)
55. The value is estimated at 1.10 (calculated as 1 + (0.89\*(0.33 / 2.8)). Based on assumption that 89% of homes have central cooling (based onKEMA Maryland Energy Baseline Study. Feb 2011.). [↑](#footnote-ref-55)
56. Calculated using defaults; 1+ ((0.47/1.67) \* 0.375) = 0.894 [↑](#footnote-ref-56)
57. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-57)
58. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-58)
59. Calculation assumes 59% Heat Pump and 41% Resistance which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey. Average efficiency of heat pump is based on assumption 50% are units from before 2006 and 50% after. [↑](#footnote-ref-59)
60. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-60)
61. The value is estimated at 1.24 (calculated as 1 + (0.66 / 2.8)). See footnote relating to WHFe for details. Note the 66% factor represents the Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load (i.e. consistent with the PJM coincident definition). [↑](#footnote-ref-61)
62. The value is estimated at 1.21 (calculated as 1 + (0.89 \* 0.66 / 2.8)). [↑](#footnote-ref-62)
63. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 16. [↑](#footnote-ref-63)
64. Ibid. [↑](#footnote-ref-64)
65. Consistent with “Lodging Common Area” coincidence factor in Commercial Screw base CFL measure characterization, based on ‘Development of Interior Lighting Hours of Use and Coincidence Factor Values for EmPOWER Maryland Commercial Lighting Program Evaluations, Itron, 2010’. [↑](#footnote-ref-65)
66. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. [↑](#footnote-ref-66)
67. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-67)
68. Negative value because this is an increase in heating consumption due to the efficient lighting. [↑](#footnote-ref-68)
69. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-69)
70. This has been estimated assuming typical efficiencies of existing heating systems weighted by percentage of homes with non-electric heating (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls). [↑](#footnote-ref-70)
71. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-71)
72. Based on “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. [↑](#footnote-ref-72)
73. Assumption based on 15 minutes (including portion of travel time) and $20 per hour. [↑](#footnote-ref-73)
74. The assumed measure life for the specialty bulb measure characterization was reported in "Residential Lighting Measure Life Study", Nexus Market Research, June 4, 2008 (measure life for markdown bulbs). Measure life estimate does not distinguish between equipment life and measure persistence. Measure life includes products that were installed and operated until failure (i.e., equipment life) as well as those that were retired early and permanently removed from service for any reason, be it early failure, breakage, or the respondent not liking the product (i.e., measure persistence). [↑](#footnote-ref-74)
75. Assumed rated life of 10,000 hours due to lower switching (10000/5950 = 1.7). [↑](#footnote-ref-75)
76. Assumed rated life of 8,000 hours due to higher switching and use outside (8000/1643 = 4.9) [↑](#footnote-ref-76)
77. Assuming 1000 hour rated life for incandescent bulb divided by the hours of use assumption. [↑](#footnote-ref-77)
78. Based on “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. [↑](#footnote-ref-78)
79. For text of Energy and Independence and Security Act, see http://www.gpo.gov/fdsys/pkg/PLAW-110publ140/pdf/PLAW-110publ140.pdf [↑](#footnote-ref-79)
80. Base wattage is based upon the post first phase of EISA wattage and wattage bins consistent with ENERGY STAR, v1.1; http://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Lamps%20V1%201\_Specification.pdf [↑](#footnote-ref-80)
81. Based on Nexus Market Research, “Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs”, Final Report, October 1, 2004, p. 42 (Table 4-7). [↑](#footnote-ref-81)
82. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 14. [↑](#footnote-ref-82)
83. Multi family common area lighting assumption is 16.3 hours per day (5950 hours per year) based on Focus on Energy Evaluation, ACES Deemed Savings Desk Review, November 2010. This estimate is consistent with the Common Area “Non-Area Specific” assumption (16.2 hours per day or 5913 annually) from the Cadmus Group Inc., “Massachusetts Multifamily Program Impact Analysis”, July 2012, p 2-4. [↑](#footnote-ref-83)
84. The value is estimated at 1.12 (calculated as 1 + (0.33 / 2.8)). Based on cooling loads decreasing by 33% of the lighting savings (average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 \* SEER2) + (1.12 \* SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP). [↑](#footnote-ref-84)
85. The value is estimated at 1.10 (calculated as 1 + (0.89\*(0.33 / 2.8)). Based on assumption that 89% of homes have central cooling (based on KEMA Maryland Energy Baseline Study. Feb 2011.). [↑](#footnote-ref-85)
86. Calculated using defaults; 1+ ((0.47/1.67) \* 0.375) = 0.894 [↑](#footnote-ref-86)
87. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-87)
88. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-88)
89. Calculation assumes 59% Heat Pump and 41% Resistance which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey. Average efficiency of heat pump is based on assumption 50% are units from before 2006 and 50% after. [↑](#footnote-ref-89)
90. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-90)
91. The value is estimated at 1.24 (calculated as 1 + (0.66 / 2.8)). See footnote relating to WHFe for details. Note the 66% factor represents the Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load (i.e. consistent with the PJM coincident definition). [↑](#footnote-ref-91)
92. The value is estimated at 1.21 (calculated as 1 + (0.89 \* 0.66 / 2.8)). [↑](#footnote-ref-92)
93. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 16. [↑](#footnote-ref-93)
94. Ibid. [↑](#footnote-ref-94)
95. Consistent with “Lodging Common Area” coincidence factor in Commercial Screw base CFL measure characterization, based on ‘Development of Interior Lighting Hours of Use and Coincidence Factor Values for EmPOWER Maryland Commercial Lighting Program Evaluations, Itron, 2010’. [↑](#footnote-ref-95)
96. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-96)
97. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-97)
98. This has been estimated assuming typical efficiencies of existing heating systems weighted by percentage of homes with non-electric heating (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls). [↑](#footnote-ref-98)
99. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-99)
100. ENERGY STAR Qualified Lighting Savings Calculator default incremental cost input for interior fixture (<http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/LightingCalculator.xlsx?b299-55ae&b299-55ae>) [↑](#footnote-ref-100)
101. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007 (http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf ) gives 20 years for an interior fluorescent fixture. [↑](#footnote-ref-101)
102. Based on Northeast Regional Residential Lighting Strategy (RLS) report, prepared by EFG, D&R International, Ecova and Optimal Energy. [↑](#footnote-ref-102)
103. Ibid. [↑](#footnote-ref-103)
104. Based on lamp life / assumed annual run hours. [↑](#footnote-ref-104)
105. Assumes rated life of incandescent bulb of 1000 hours (simplified to 1 year for calculation). [↑](#footnote-ref-105)
106. The manufacturers are simply using a regular incandescent lamp with halogen fill gas rather than Halogen Infrared to meet the standard, so the lifetime of these EISA qualified bulbs is assumed to be 1000 hours. [↑](#footnote-ref-106)
107. Assumes 8000 hours rated life for CFL (8000 hours is the average rated life of ENERGY STAR bulbs (<http://www.energystar.gov/index.cfm?c=cfls.pr_crit_cfls)> [↑](#footnote-ref-107)
108. Note, these values have been adjusted by the appropriate In Service Rate. [↑](#footnote-ref-108)
109. For text of Energy and Independence and Security Act, see http://www.gpo.gov/fdsys/pkg/PLAW-110publ140/pdf/PLAW-110publ140.pdf [↑](#footnote-ref-109)
110. Base wattage is based upon the post first phase of EISA wattage and wattage bins consistent with ENERGY STAR, v1.1; http://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Lamps%20V1%201\_Specification.pdf [↑](#footnote-ref-110)
111. Consistent with Efficiency Vermont and CT Energy Efficiency Fund; based on Nexus Market Research, “Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs”, Final Report, October 1, 2004, p. 42 (Table 4-7). [↑](#footnote-ref-111)
112. Updated results from above study, presented in 2005 memo; http://publicservice.vermont.gov/energy/ee\_files/efficiency/eval/marivtfinalresultsmemodelivered.pdf [↑](#footnote-ref-112)
113. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. [↑](#footnote-ref-113)
114. ENERGY STAR Qualified Lighting Savings Calculator default incremental cost input for exterior fixture (<http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/LightingCalculator.xlsx?b299-55ae&b299-55ae>) [↑](#footnote-ref-114)
115. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007 (http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf ) gives 15 years for an exterior fluorescent fixture. [↑](#footnote-ref-115)
116. Based on Northeast Regional Residential Lighting Strategy (RLS) report, prepared by EFG, D&R International, Ecova and Optimal Energy. [↑](#footnote-ref-116)
117. Ibid. [↑](#footnote-ref-117)
118. Assumes rated life of incandescent bulb of 1000 hours. [↑](#footnote-ref-118)
119. The manufacturers are simply using a regular incandescent lamp with halogen fill gas rather than Halogen Infrared to meet the standard ,so the lifetime of these EISA qualified bulbs is assumed to be 1000 hours. [↑](#footnote-ref-119)
120. Assumes rated life of 8000 hours. [↑](#footnote-ref-120)
121. Note, these values have been adjusted by the appropriate In Service Rate. [↑](#footnote-ref-121)
122. ENERGY STAR specification can be viewed here: https://www.energystar.gov/sites/default/files/asset/document/Luminaires%20V2%200%20Final.pdf [↑](#footnote-ref-122)
123. Based on ENERGY STAR equivalence table; <http://www.energystar.gov/index.cfm?c=cfls.pr_cfls_lumens> [↑](#footnote-ref-123)
124. Baseline wattage based on common 65 Watt BR30 incandescent bulb (e.g. http://www.destinationlighting.com/storeitem.jhtml?iid=16926) [↑](#footnote-ref-124)
125. The upper bounds for these categories depends on the lower bound of the next higher wattage, which varies by bulb type. [↑](#footnote-ref-125)
126. As above. [↑](#footnote-ref-126)
127. Energy Efficient wattage based on 12 Watt LR6 Downlight from LLF Inc. (<http://site4.marketsmartinteractive.com/products.htm>). Adjusted by ratio of lm/w in ENERGY STAR V2.0 compared to ENERGY STAR V1.2 specification. [↑](#footnote-ref-127)
128. Based upon recommendation in NEEP EMV Emerging Tech Research Report. [↑](#footnote-ref-128)
129. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 14. This assumption is consistent with the CFL measures. To date there has not been sufficient data available to provide a separate LED hours assumption, and this should be reviewed in future years. [↑](#footnote-ref-129)
130. Multifamily common area lighting assumption is 16.3 hours per day (5950 hours per year) based on Focus on Energy Evaluation, ACES Deemed Savings Desk Review, November 2010. This estimate is consistent with the Common Area “Non-Area Specific” assumption (16.2 hours per day or 5913 annually) from the Cadmus Group Inc., “Massachusetts Multifamily Program Impact Analysis”, July 2012, p 2-4. [↑](#footnote-ref-130)
131. The value is estimated at 1.12 (calculated as 1 + (0.33 / 2.8)). Based on cooling loads decreasing by 33% of the lighting savings (average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 \* SEER2) + (1.12 \* SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP). [↑](#footnote-ref-131)
132. The value is estimated at 1.10 (calculated as 1 + (0.89\*(0.33 / 2.8)). Based on assumption that 89% of homes have central cooling (based onKEMA Maryland Energy Baseline Study. Feb 2011.). [↑](#footnote-ref-132)
133. Calculated using defaults; 1+ ((0.47/1.67) \* 0.375) = 0.894 [↑](#footnote-ref-133)
134. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-134)
135. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-135)
136. Calculation assumes 59% Heat Pump and 41% Resistance which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey. Average efficiency of heat pump is based on assumption 50% are units from before 2006 and 50% after. [↑](#footnote-ref-136)
137. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-137)
138. The value is estimated at 1.24 (calculated as 1 + (0.66 / 2.8)). See footnote relating to WHFe for details. Note the 66% factor represents the Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load (i.e. consistent with the PJM coincident definition). [↑](#footnote-ref-138)
139. The value is estimated at 1.18 (calculated as 1 + (0.78 \* 0.66 / 2.8)). [↑](#footnote-ref-139)
140. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 16. [↑](#footnote-ref-140)
141. Ibid. [↑](#footnote-ref-141)
142. Consistent with “Lodging Common Area” coincidence factor in Commercial Screw base CFL measure characterization, based on ‘Development of Interior Lighting Hours of Use and Coincidence Factor Values for EmPOWER Maryland Commercial Lighting Program Evaluations, Itron, 2010’. [↑](#footnote-ref-142)
143. Negative value because this is an increase in heating consumption due to the efficient lighting. [↑](#footnote-ref-143)
144. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-144)
145. This has been estimated assuming typical efficiencies of existing heating systems weighted by percentage of homes with non-electric heating (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls). [↑](#footnote-ref-145)
146. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-146)
147. Based on VEIC product review, April 2015. Baseline bulbs available in $3-$5 range, and SSL bulbs available in $20-$60 range. Incremental cost of $36 therefore assumed ($4 for the baseline bulb and $40 for the SSL). Note, this product is likely to fall rapidly in cost, so this should be reviewed frequently. [↑](#footnote-ref-147)
148. The ENERGY STAR Spec for SSL Recessed Downlights requires luminaires to maintain >=70% initial light output for 25,000 hrs in an indoor application for separable luminaires and 50,000 for inseparable luminaires. Measure life is capped at 20 years. [↑](#footnote-ref-148)
149. Based on lamp life / assumed annual run hours. [↑](#footnote-ref-149)
150. Assumes rated life of BR incandescent bulb of 2000 hours, based on product review. Lamp life is therefore 2000/898 = 2.2 years. [↑](#footnote-ref-150)
151. Calculated as 2000/5950 = 0.34 years. [↑](#footnote-ref-151)
152. For text of Energy and Independence and Security Act, see http://www.gpo.gov/fdsys/pkg/PLAW-110publ140/pdf/PLAW-110publ140.pdf [↑](#footnote-ref-152)
153. The upper bounds for these categories depends on the lower bound of the next higher wattage, which varies by bulb type. [↑](#footnote-ref-153)
154. As above. [↑](#footnote-ref-154)
155. First year ISR of 0.9 (EMPOWER MD Lighting Study, EY5). Assume lifetime ISR of 0.99 (2006-2008 California Residential Lighting Evaluations, and used in the Uniform Methods Project). Assume half of bulbs not installed in year one are installed in year two, and the other half in year three. Using a discount rate of 5%, this gives 0.90 + 0.045 \* 0.95 + 0.045 \* 0.95^2 = 0.98 [↑](#footnote-ref-155)
156. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 14. To date there has not been sufficient data available to provide a separate LED hours assumption, and this should be reviewed in future years. [↑](#footnote-ref-156)
157. Multi family common area lighting assumption is 16.3 hours per day (5950 hours per year) based on Focus on Energy Evaluation, ACES Deemed Savings Desk Review, November 2010. This estimate is consistent with the Common Area “Non-Area Specific” assumption (16.2 hours per day or 5913 annually) from the Cadmus Group Inc., “Massachusetts Multifamily Program Impact Analysis”, July 2012, p 2-4. [↑](#footnote-ref-157)
158. Updated results from Nexus Market Research, “Impact Evaluation of the Massachusetts, Rhode Island and Vermont 2003 Residential Lighting Programs”, Final Report, October 1, 2004, presented in 2005 memo; http://publicservice.vermont.gov/energy/ee\_files/efficiency/eval/marivtfinalresultsmemodelivered.pdf [↑](#footnote-ref-158)
159. Based on EmPOWER Maryland 2011Evaluation Report; Chapter 5: Residential Lighting and Appliances. [↑](#footnote-ref-159)
160. The value is estimated at 1.12 (calculated as 1 + (0.33 / 2.8)). Based on cooling loads decreasing by 33% of the lighting savings (average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC), assuming typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 \* SEER2) + (1.12 \* SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP). [↑](#footnote-ref-160)
161. The value is estimated at 1.10 (calculated as 1 + (0.89\*(0.33 / 2.8)). Based on assumption that 89% of homes have central cooling (based onKEMA Maryland Energy Baseline Study. Feb 2011.). [↑](#footnote-ref-161)
162. Calculated using defaults; 1+ ((0.47/1.67) \* 0.375) = 0.894 [↑](#footnote-ref-162)
163. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-163)
164. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-164)
165. Calculation assumes 59% Heat Pump and 41% Resistance which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey. Average efficiency of heat pump is based on assumption 50% are units from before 2006 and 50% after. [↑](#footnote-ref-165)
166. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-166)
167. See ‘ESTAR Integrated Screw SSL Lamp\_032014.xls’ for details. The Minimum Lamp Efficacy Requirements in ENERGY STAR Product Specification for Lamps (Light Bulbs) V2.0 vary by Color Rendering Index (CRI). [↑](#footnote-ref-167)
168. The value is estimated at 1.24 (calculated as 1 + (0.66 / 2.8)). See footnote relating to WHFe for details. Note the 66% factor represents the Residential cooling coincidence factor calculated by dividing average load during the peak hours divided by the maximum cooling load (i.e. consistent with the PJM coincident definition). [↑](#footnote-ref-168)
169. The value is estimated at 1.18 (calculated as 1 + (0.78 \* 0.66 / 2.8)). [↑](#footnote-ref-169)
170. Based on Navigant Consulting “EmPOWER Maryland Evaluation Year 5 (June 1, 2013 – May 31, 2014) Residential Lighting Program: Hours of Use/Metering Study.” April 10, 2015, page 16. [↑](#footnote-ref-170)
171. Ibid. [↑](#footnote-ref-171)
172. Consistent with “Lodging Common Area” coincidence factor in Commercial Screw base CFL measure characterization, based on ‘Development of Interior Lighting Hours of Use and Coincidence Factor Values for EmPOWER Maryland Commercial Lighting Program Evaluations, Itron, 2010’. [↑](#footnote-ref-172)
173. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. [↑](#footnote-ref-173)
174. This means that heating loads increase by 47% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC. [↑](#footnote-ref-174)
175. This has been estimated assuming typical efficiencies of existing heating systems weighted by percentage of homes with non-electric heating (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls). [↑](#footnote-ref-175)
176. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-176)
177. Omindirectional and directional costs based on NEEP 2014-2015 Residential Lighting Strategy Update. Decorative Costs under 15W based on typical costs on 1000bulbs.com. Higher wattage decorative based on VEIC study of units rebated through Efficiency Vermont Retail program. [↑](#footnote-ref-177)
178. The ENERGY STAR Spec V1.1 for Integrated Screw Based SSL bulbs requires lamps to maintain >=70% initial light output for 25,000 hrs in a residential application for omnidirectional and directional bulbs, and 15,000 hrs for decorative bulbs. Lifetime capped at 20 years. [↑](#footnote-ref-178)
179. The ENERGY STAR Spec v2.0 for Integrated Screw Based SSL bulbs requires lamps to maintain >=70% initial light output for 25,000 hrs in a residential application for directional bulbs, and 15,000 hrs for omni directional and decorative bulbs. Lifetime capped at 20 years. [↑](#footnote-ref-179)
180. Assumes incandescent baseline lamp life of 1000 hours. [↑](#footnote-ref-180)
181. Assumed higher lamp life for instances with longer run hours and therefore less switching. [↑](#footnote-ref-181)
182. <http://www.energystar.gov/ia/products/appliances/refrig/NAECA_calculation.xls?c827-f746> [↑](#footnote-ref-182)
183. http://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/43 [↑](#footnote-ref-183)
184. [http://www.energystar.gov/ia/products/appliances/refrig/NAECA\_calculation.xls?c827-f746](http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls?c827-f746) [↑](#footnote-ref-184)
185. http://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/43 [↑](#footnote-ref-185)
186. http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Final%20Version%205.0%20Residential%20Refrigerators%20and%20Freezers%20Specification.pdf [↑](#footnote-ref-186)
187. Volume is based on ENERGY STAR Calculator assumption of 16.14 ft3 average volume, converted to Adjusted volume by multiplying by 1.73. [↑](#footnote-ref-187)
188. Unknown configuration is based upon a weighted average of the different configurations. Data is taken from the DOE Technical Support Document (<http://www1.eere.energy.gov/buildings/appliance_standards/pdfs/refrig_finalrule_tsd.pdf>). Weighting based on 80% Standard v 20% Compact (2007 annual shipments p3-26) and product class market shares from pages 9-17 and 9-24. See ‘Freezer default calcs.xls’ for more details. [↑](#footnote-ref-188)
189. Temperature adjustment factor based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 47) and assuming 78% of refrigerators are in cooled space (based on BGE Energy Use Survey, Report of Findings, December 2005; Mathew Greenwald & Associates) and 22% in un-cooled space. Although this evaluation is based upon refrigerators only it is considered a reasonable estimate of the impact of cycling on freezers and gave exactly the same result as an alternative methodology based on Freezer eShape data. [↑](#footnote-ref-189)
190. Daily load shape adjustment factor also based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 p. 48, (extrapolated by taking the ratio of existing summer to existing annual profile for hours ending 15 through 18, and multiplying by new annual profile). [↑](#footnote-ref-190)
191. Based on review of data from the Northeast Regional ENERGY STAR Consumer Products Initiative. “2009 ENERGY STAR Appliances Practices Report”, submitted by Lockheed Martin, December 2009. [↑](#footnote-ref-191)
192. Energy Star Freezer Calculator; <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/appliance_calculator.xlsx?a8fb-c882&a8fb-c882> [↑](#footnote-ref-192)
193. http://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/43 [↑](#footnote-ref-193)
194. [http://www.energystar.gov/ia/products/appliances/refrig/NAECA\_calculation.xls?c827-f746](http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls?c827-f746) [↑](#footnote-ref-194)
195. http://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/43 [↑](#footnote-ref-195)
196. http://www.energystar.gov/products/specs/sites/products/files/ENERGY%20STAR%20Final%20Version%205.0%20Residential%20Refrigerators%20and%20Freezers%20Specification.pdf [↑](#footnote-ref-196)
197. Volume is based on the ENERGY STAR calculator average assumption of 14.75 ft3 fresh volume and 6.76 ft3 freezer volume. [↑](#footnote-ref-197)
198. Unknown configuration is based upon a weighted average of the different configurations. Data is taken from the 2011 DOE Technical Support Document (<http://www.regulations.gov/#!documentDetail;D=EERE-2008-BT-STD-0012-0128>). Projected product class market shares from pages 9-12 for year 2014. See ‘Refrigerator default calcs.xls’ for more details. [↑](#footnote-ref-198)
199. Temperature adjustment factor based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 47) and assuming 78% of refrigerators are in cooled space (based on BGE Energy Use Survey, Report of Findings, December 2005; Mathew Greenwald & Associates) and 22% in un-cooled space. [↑](#footnote-ref-199)
200. Daily load shape adjustment factor also based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 p. 48, (extrapolated by taking the ratio of existing summer to existing annual profile for hours ending 15 through 18, and multiplying by new annual profile). [↑](#footnote-ref-200)
201. Based on “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. See “Refrigerator Default Calcs.xlsx”. [↑](#footnote-ref-201)
202. Based on Department of Energy, “TECHNICAL REPORT: Analysis of Amended Energy Conservation Standards for Residential Refrigerator-Freezers”, October 2005. [↑](#footnote-ref-202)
203. From ENERGY STAR calculator: <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/appliance_calculator.xlsx?5035-d681&5035-d681> [↑](#footnote-ref-203)
204. Assumed to be 1/3 of the measure life. [↑](#footnote-ref-204)
205. Based on EmPower 2011 Interim Evaluation Report Chapter 5: Lighting and Appliances, Table 15, p33. This suggests an average UEC of 1,146kWh. [↑](#footnote-ref-205)
206. kWh assumptions based on using the NAECA algorithms in each product class and calculating a weighted average of the different configurations. Data for weighting is taken from the 2011 DOE Technical Support Document (<http://www.regulations.gov/#!documentDetail;D=EERE-2008-BT-STD-0012-0128>). Projected product class market shares from pages 9-12 for year 2014. See ‘Refrigerator default calcs.xls’ for more details. [↑](#footnote-ref-206)
207. kWh assumptions based on using the ENERGY STAR algorithms in each product class and calculating a weighted average of the different configurations. [↑](#footnote-ref-207)
208. kWh assumptions based on 25% less than baseline consumption and calculating a weighted average of the different configurations. [↑](#footnote-ref-208)
209. These values are provided in case the utility screening tool does not allow for this mid life baseline adjustment. The values are determined by calculating the Net Present Value of the 12 year annual savings values and finding the equivalent annual savings that produces the same result. The Real Discount Rate of 5.0% is used. [↑](#footnote-ref-209)
210. Temperature adjustment factor based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 47) and assuming 78% of refrigerators are in cooled space (based on BGE Energy Use Survey, Report of Findings, December 2005; Mathew Greenwald & Associates) and 22% in un-cooled space. [↑](#footnote-ref-210)
211. Daily load shape adjustment factor also based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 p. 48, (extrapolated by taking the ratio of existing summer to existing annual profile for hours ending 15 through 18, and multiplying by new annual profile). [↑](#footnote-ref-211)
212. Full ENERGY STAR and baseline costs based on “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. See “Refrigerator Default Calcs.xlsx”. Full CEE Tier 2 cost is based upon incremental cost estimate derived from “TECHNICAL REPORT: Analysis of Amended Energy Conservation Standards for Residential Refrigerator-Freezers”. <http://www1.eere.energy.gov/buildings/appliance_standards/pdfs/refrigerator_report_1.pdf> [↑](#footnote-ref-212)
213. From ENERGY STAR calculator: <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/appliance_calculator.xlsx?5035-d681&5035-d681> [↑](#footnote-ref-213)
214. This measure assumes a mix of primary and secondary refrigerators will be replaced. By definition, the refrigerator in a household’s kitchen that satisfies the majority of the household’s demand for refrigeration is the primary refrigerator. One or more additional refrigerators in the household that satisfy supplemental needs for refrigeration are referred to as secondary refrigerators. [↑](#footnote-ref-214)
215. Note that the hypothetical nature of this measure implies a significant amount of risk and uncertainty in developing the energy and demand impact estimates. [↑](#footnote-ref-215)
216. EmPOWER Maryland Impact Evaluation, Program Year 6 [↑](#footnote-ref-216)
217. The 10 year average annual heating degree day value is calculated for each location, using a balance point of 65 degrees as used in the EmPower Appliance Recycling Evaluation. [↑](#footnote-ref-217)
218. Ibid. [↑](#footnote-ref-218)
219. Based on EmPower DRAFT EY6 Participant Survey Results: Appliance Recycling Program Report [↑](#footnote-ref-219)
220. EmPOWER Maryland Impact Evaluation, Program Year 6

     . [↑](#footnote-ref-220)
221. Temperature adjustment factor based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 47) and assuming 78% of refrigerators are in cooled space (based on BGE Energy Use Survey, Report of Findings, December 2005; Mathew Greenwald & Associates) and 22% in un-cooled space. [↑](#footnote-ref-221)
222. Daily load shape adjustment factor also based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 p. 48, using the average Existing Units Summer Profile for hours ending 15 through 18. [↑](#footnote-ref-222)
223. KEMA “Residential refrigerator recycling ninth year retention study”, 2004. [↑](#footnote-ref-223)
224. The average heating savings from Scott Pigg (Energy Center of Wisconsin), “Electricity Use by New Furnaces: A Wisconsin Field Study”, Technical Report 230-1, October 2003, is 400kWh. An estimate for Mid-Atlantic is provided by multiplying this by the ratio of heating degree days in Baltimore MD compared to Wisconsin (4704 / 7800). [↑](#footnote-ref-224)
225. The average cooling savings from Scott Pigg (Energy Center of Wisconsin), “Electricity Use by New Furnaces: A Wisconsin Field Study”, Technical Report 230-1, October 2003, is 70 to 95kWh. An estimate for Mid-Atlantic is provided by multiplying by the ratio of full load cooling hours in Baltimore compared to Southern Wisconsin (1050/487). Full load hour estimates from: http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/CalculatorConsumerRoomAC.xls. [↑](#footnote-ref-225)
226. See write up in Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, page 38-39. [↑](#footnote-ref-226)
227. Sachs and Smith, April 2003; Saving Energy with Efficient Furnace Air Handlers: A Status Update and Program Recommendations. [↑](#footnote-ref-227)
228. Although the Federal baseline presented does not come in to effect until June 2014, (<http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/41>)

     according to ENERGY STAR Shipment Data the estimated market penetration of ENERGY STAR v2.0 Room AC went from 33% in 2010 to 62% in 2011 and 58% in 2012. The new Federal Standard level is equivalent to ENERGY STAR v2.0 and with the market preparing for the Standard change it is appropriate to use the updated rating from the start of the year. [↑](#footnote-ref-228)
229. VEIC calculated the average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008) to FLH for Central Cooling (provided by AHRI: http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) at 31%. Applying this to the FLH for Central Cooling provided for Baltimore (1050) we get 325 FLH for Room AC. [↑](#footnote-ref-229)
230. Based on maximum capacity average from RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008. [↑](#footnote-ref-230)
231. Minimum Federal Standard for most common Room AC type – 8000-14,999 capacity range with louvered sides. [↑](#footnote-ref-231)
232. Minimum qualifying for ENERGY STAR most common Room AC type – 8000-14,999 capacity range with louvered sides. [↑](#footnote-ref-232)
233. Minimum qualifying for CEE Tier 2 most common Room AC type – 8000-14,999 capacity range with louvered sides. [↑](#footnote-ref-233)
234. Calculated by multiplying the ratio of SSP:PJM for the Central AC measure (0.69:0.66) to the assumption for PJM. [↑](#footnote-ref-234)
235. Consistent with coincidence factors found in:

     RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117\_RLW\_CF%20Res%20RAC.pdf). [↑](#footnote-ref-235)
236. From Lekov, Alex et al. *Evaluation of Energy-Efficiency Standards for Room Air Conditioners in US” Sept 2012. Use average of Efficiency levels 4 and 5 to approximate current energy star standards.*  [↑](#footnote-ref-236)
237. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-237)
238. Typical EER for units with SEER of 14, from the AHRI directory [↑](#footnote-ref-238)
239. The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings). [↑](#footnote-ref-239)
240. Full Load Cooling Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying the EmPower average Maryland full load hours determined for Maryland (542 from the research referenced below) by the ratio of full load hours in Wilmington, DE (1,015) or Washington, DC (1,320) to Baltimore MD (1,050) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) [↑](#footnote-ref-240)
241. Based on average of 5 utilities in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-241)
242. Minimum Federal Standard. [↑](#footnote-ref-242)
243. VEIC estimate based on Department of Energy Federal Standard between 1992 and 2006. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used. [↑](#footnote-ref-243)
244. Based on SEER of 10,0, using formula above to give 9.2 EER. [↑](#footnote-ref-244)
245. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-245)
246. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-246)
247. Costs based upon average cost per ton from “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. Note SEER 17 and 18 are extrapolated from other data points. [↑](#footnote-ref-247)
248. Costs based upon average cost per ton for Equipment and Labor from Itron Measure Cost Study Results Matrix Volume 1 (part of “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014). [↑](#footnote-ref-248)
249. Ibid. [↑](#footnote-ref-249)
250. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-250)
251. Assumed to be one third of effective useful life [↑](#footnote-ref-251)
252. HSPF, SEER and EER refer to Heating Seasonal Performance Factor, Seasonal Energy Efficiency Ratio and Energy Efficiency Ratio, respectively. [↑](#footnote-ref-252)
253. [↑](#footnote-ref-253)
254. nt savings). [↑](#footnote-ref-254)
255. s in Wilmington, DE (1,015) or Washington, DC (1,320) to Baltimore MD (1,050) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) [↑](#footnote-ref-255)
256. Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-256)
257. Minimum Federal Standard [↑](#footnote-ref-257)
258. rd between 1992 and 2006. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used. [↑](#footnote-ref-258)
259. urages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit. [↑](#footnote-ref-259)
260. gs\_calc/ASHP\_Sav\_Calc.xls) [↑](#footnote-ref-260)
261. , table 30, page 48. [↑](#footnote-ref-261)
262. Minimum Federal Standard [↑](#footnote-ref-262)
263. son to adjust the rated HSPF for geographical/climate variances. [↑](#footnote-ref-263)
264. HP SEER rating assumption of 10.0. [↑](#footnote-ref-264)
265. Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF. [↑](#footnote-ref-265)
266. o perform this calculation we are using this formula: (-0.02 \* SEER2) + (1.12 \* SEER) (from Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder). [↑](#footnote-ref-266)
267. From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. [↑](#footnote-ref-267)
268. Estimated by converting the SEER 10 assumption using the algorithm provided. [↑](#footnote-ref-268)
269. e is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit. [↑](#footnote-ref-269)
270. ners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-270)
271. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-271)
272. osts based upon average cost per ton from “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. Note SEER 17 and 18 are extrapolated from other data points. [↑](#footnote-ref-272)
273. [↑](#footnote-ref-273)
274. Ibid. [↑](#footnote-ref-274)
275. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-275)
276. Full Load Cooling Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying the EmPower average Maryland full load hours determined for Maryland (542 from the research referenced below) by the ratio of full load hours in Wilmington, DE (1,015) or Washington, DC (1,320) to Baltimore MD (1,050) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) [↑](#footnote-ref-276)
277. Based on average of 5 utilities in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-277)
278. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-278)
279. Full Load Heating Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying BG&E’s full load hours determined for Baltimore (1195 from the research referenced below) by the ratio of full load hours in Wilmington, DE (2346) or Washington, DC (2061) to Baltimore MD (2172) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/ASHP\_Sav\_Calc.xls) [↑](#footnote-ref-279)
280. Based on average of 5 utilities, two program years, in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-280)
281. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-281)
282. 25 Pascals is the standard assumption for typical pressures experienced in the duct system under normal operating conditions. To convert CFM50 to CFM25 you multiply by 0.64 (inverse of the “Can’t Reach Fifty” factor for CFM25; see Energy Conservatory Blower Door Manual). [↑](#footnote-ref-282)
283. Assumes that for each percent of supply air loss there is one percent annual energy penalty. This assumes supply side leaks are direct losses to the outside and are not recaptured back to the house. This could be adjusted downward to reflect regain of usable energy to the house from duct leaks. For example, during the winter some of the energy lost from supply leaks in a crawlspace will probably be regained back to the house (sometimes 1/2 or more may be regained). More information provided in “Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements” from <http://www.energyconservatory.com/download/dbmanual.pdf> [↑](#footnote-ref-283)
284. Assumes 50% of leaks are in supply ducts. [↑](#footnote-ref-284)
285. Assumes that for each percent of return air loss there is a half percent annual energy penalty. Note that this assumes that return leaks contribute less to energy losses than do supply leaks. This value could be adjusted upward if there was reason to suspect that the return leaks contribute significantly more energy loss than “average” (e.g. pulling return air from a super heated attic), or can be adjusted downward to represent significantly less energy loss (e.g. pulling return air from a moderate temperature crawl space) . More information provided in “Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements” from <http://www.energyconservatory.com/download/dbmanual.pdf> [↑](#footnote-ref-285)
286. Assumes 50% of leaks are in return ducts. [↑](#footnote-ref-286)
287. Full Load Cooling Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying the EmPower average Maryland full load hours determined for Maryland (542 from the research referenced below) by the ratio of full load hours in Wilmington, DE (1,015) or Washington, DC (1,320) to Baltimore MD (1,050) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) [↑](#footnote-ref-287)
288. Based on average of 5 utilities in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-288)
289. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-289)
290. Full Load Heating Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying BG&E’s full load hours determined for Baltimore (1195 from the research referenced below) by the ratio of full load hours in Wilmington, DE (2346) or Washington, DC (2061) to Baltimore MD (2172) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/ASHP\_Sav\_Calc.xls) [↑](#footnote-ref-290)
291. Based on average of 5 utilities, two program years, in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-291)
292. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-292)
293. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-293)
294. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-294)
295. Based on assumption from BG&E billing analysis of furnace program in the '90s, from conversation with Mary Straub; “Evaluation of the High efficiency heating and cooling program, technical report”, June 1995. For other utilities offering this measure, a Heating Degree Day adjustment may be appropriate to this FLHheat assumption. [↑](#footnote-ref-295)
296. Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. [↑](#footnote-ref-296)
297. The equipment efficiency default is based on data provided by GAMA during the federal rule-making process for furnace efficiency standards, suggesting that in 2000, 32% of furnaces purchased in Maryland were condensing units. Assuming an efficiency of 92% for the condensing furnaces and 80% for the non-condensing furnaces gives a weighted average of 83.8%. [↑](#footnote-ref-297)
298. Based on Natural Draft Furnaces requiring 100 CFM per 10,000 BTU, Induced Draft Furnaces requiring 130CFM per 10,000BTU and Condensing Furnaces requiring 150 CFM per 10,000 BTU (rule of thumb from <http://contractingbusiness.com/enewsletters/cb_imp_43580/>). Data provided by GAMA during the federal rule-making process for furnace efficiency standards, suggested that in 2000, 32% of furnaces purchased in Maryland were condensing units. Therefore a weighted average required airflow rate is calculated assuming a 50:50 split of natural v induced draft non-condensing furnaces, as 126 per 10,000BTU or 0.0126/Btu. [↑](#footnote-ref-298)
299. Based on assumption from BG&E billing analysis of furnace program in the '90s, from conversation with Mary Straub; “Evaluation of the High efficiency heating and cooling program, technical report”, June 1995. For other utilities offering this measure, a Heating Degree Day adjustment may be appropriate to this FLHheat assumption. [↑](#footnote-ref-299)
300. Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. [↑](#footnote-ref-300)
301. The equipment efficiency default is based on data provided by GAMA during the federal rule-making process for furnace efficiency standards, suggesting that in 2000, 32% of furnaces purchased in Maryland were condensing units. Assuming an efficiency of 92% for the condensing furnaces and 80% for the non-condensing furnaces gives a weighted average of 83.8%. [↑](#footnote-ref-301)
302. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-302)
303. The Federal Standard does not include an EER requirement, so it is approximated with this formula: (-0.02 \* SEER2) + (1.12 \* SEER) Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. Note this is appropriate for single speed units only. [↑](#footnote-ref-303)
304. VEIC estimate based on Department of Energy Federal Standard between 1992 and 2006. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used. [↑](#footnote-ref-304)
305. Estimated by converting the assumption of existing unit EER efficiency in the Room Air Conditioner Early Replacement measure (7.7EER) in to SEER using the assumption EER≈SEER/1.1. [↑](#footnote-ref-305)
306. Minimum Federal Standard [↑](#footnote-ref-306)
307. For example with a Manual-J calculation or similar modeling. [↑](#footnote-ref-307)
308. Assume COP of 1.0 converted to HSPF by multiplying by 3.412. [↑](#footnote-ref-308)
309. This is estimated based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596, and applying to the existing ASHP SEER rating assumption of 10.0. [↑](#footnote-ref-309)
310. Minimum Federal Standard [↑](#footnote-ref-310)
311. HSPF ratings for Heat Pumps account for the seasonal average efficiency of the units and are based on testing within AHRI climate zone 4 which encompasses all of the Mid Atlantic region. There should therefore be no reason to adjust the rated HSPF for geographical/climate variances. [↑](#footnote-ref-311)
312. For example with a Manual-J calculation or similar modeling. [↑](#footnote-ref-312)
313. This has been estimated assuming that the average efficiency of existing heating systems is likely to include newer more efficient systems. [↑](#footnote-ref-313)
314. Based on converting the SEER 10 to EER using the assumption EER≈SEER/1.1. [↑](#footnote-ref-314)
315. Using the assumption of existing unit EER efficiency in the Room Air Conditioner Early Replacement measure, based on Nexus Market Research Inc, RLW Analytics, December 2005; “Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report.” [↑](#footnote-ref-315)
316. Typical EER for DMSHP units with a SEER of 14 from the AHRI database [↑](#footnote-ref-316)
317. Calculated by multiplying the ratio of SSP:PJM for the Central AC measure (0.69:0.66) to the assumption for PJM. [↑](#footnote-ref-317)
318. Consistent with coincidence factors found in:

     RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117\_RLW\_CF%20Res%20RAC.pdf). [↑](#footnote-ref-318)
319. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-319)
320. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-320)
321. For example with a Manual-J calculation or similar modeling. [↑](#footnote-ref-321)
322. This has been estimated assuming that the average efficiency of existing heating systems is likely to include newer more efficient systems. [↑](#footnote-ref-322)
323. Based upon review of *Ductless Heat Pumps for Residential Customers in Connecticut*,  Swift, Joseph R and Rebecca A. Meyer, The Connecticut Light & Power Company, 2010 ACEEE Summer Study on Energy Efficiency in Buildings (2-292). Also supported by findings in NEEP *Northeast/Mid-Atlantic Air-Source Heat Pump Market Strategies Report,* January 2014 and NEEP *Incremental Cost Study Phase Two Final Report*, January 2013. If existing heating and cooling load is replaced at the end of its life, then a baseline cost should be determined and subtracted from the full install cost. [↑](#footnote-ref-323)
324. Costs based upon average cost per ton for Equipment and Labor from Itron Measure Cost Study Results Matrix Volume 1 (part of “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014). [↑](#footnote-ref-324)
325. Ibid. [↑](#footnote-ref-325)
326. Boiler and Furnace Costs derived from Page E-3 of Appendix E of Residential Furnaces and Boilers Final Rule Technical Support Document:

     <http://www1.eere.energy.gov/buildings/appliance_standards/residential/fb_tsd_0907.html>. Plus $300 labor estimate based on Itron Measure Cost Study Results Matrix Volume 1. [↑](#footnote-ref-326)
327. Ibid. Labor estimated as $500. [↑](#footnote-ref-327)
328. Incremental costs against a SEER 13 mini-split as presented in NEEP *Incremental Cost Study Phase Two Final Report*, January 2013. Results for 1 and 1.5 ton are based upon 21 SEER (most represented) and 18 SEER for 2 ton (only value provided). Values for 2.5 and 3 ton are assumed consistent with the other sizes. [↑](#footnote-ref-328)
329. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     https://neep.org/Assets/uploads/files/emv/emv-library/measure\_life\_GDS%5B1%5D.pdf [↑](#footnote-ref-329)
330. Based on simulation model as described in ODC Delaware Technical Resource Manual, April 30, 2012; http://www.dnrec.delaware.gov/energy/information/otherinfo/Documents/EM-and-V-guidance-documents/DELAWARE\_TRM\_August%202012.pdf [↑](#footnote-ref-330)
331. Based on assumption from BG&E billing analysis of furnace program in the '90s, from conversation with Mary Straub; “Evaluation of the High efficiency heating and cooling program, technical report”, June 1995. For other utilities offering this measure, a Heating Degree Day adjustment may be appropriate to this FLHheat assumption. [↑](#footnote-ref-331)
332. Full load heating hours derived by adjusting FLHheat for Baltimore, MD based on Washington, DC HDD base 60°F: 620 \*2957/3457 = 528 hours. [↑](#footnote-ref-332)
333. Costs derived from Page E-13 of Appendix E of Residential Furnaces and Boilers Final Rule Technical Support Document:

     <http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/fb_fr_tsd/appendix_e.pdf>

     VEIC believes it is reasonable to assume that the cost provided from this study for an 85% unit is appropriate for units in the 85-90% AFUE range and the cost for the 91% unit can be used for 91+% units. This is based on the observation that most of the products available in the 85-90 range are in the lower end of the range, as are those units available above 91% AFUE. [↑](#footnote-ref-333)
334. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-334)
335. Current federal minimum. See http://www.regulations.gov/#!documentDetail;D=EERE-2006-STD-0102-0008. [↑](#footnote-ref-335)
336. Based on simulation model as described in ODC Delaware Technical Resource Manual, April 30, 2012; http://www.dnrec.delaware.gov/energy/information/otherinfo/Documents/EM-and-V-guidance-documents/DELAWARE\_TRM\_August%202012.pdf [↑](#footnote-ref-336)
337. Based on assumption from BG&E billing analysis of furnace program in the '90s, from conversation with Mary Straub; “Evaluation of the High efficiency heating and cooling program, technical report”, June 1995. For other utilities offering this measure, a Heating Degree Day adjustment may be appropriate to this FLHheat assumption. [↑](#footnote-ref-337)
338. Full load heating hours derived by adjusting FLHheat for Baltimore, MD based on Washington, DC HDD base 60°F: 620 \*2957/3457 = 528 hours. [↑](#footnote-ref-338)
339. Costs derived from Page E-3 of Appendix E of Residential Furnaces and Boilers Final Rule Technical Support Document:

     http://www1.eere.energy.gov/buildings/appliance\_standards/residential/fb\_tsd\_0907.html [↑](#footnote-ref-339)
340. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-340)
341. 2007, RLW Analytics, “Validating the Impact of Programmable Thermostats” [↑](#footnote-ref-341)
342. 50.1 MMBtu heating consumption is estimated based on the MD Residential Baseline Database, subtracting Base load from Base + Heat. [↑](#footnote-ref-342)
343. The range of costs observed in VEIC’s review of other utilities TRMs was $35-$40 for the unit, $100 for labor. In the absence of actual program costs, this cost could be used. [↑](#footnote-ref-343)
344. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-344)
345. Minimum Federal Standard for most common Room AC type – 8000-14,999 capacity range with louvered sides. [↑](#footnote-ref-345)
346. Minimum qualifying for ENERGY STAR most common Room AC type – 8000-14,999 capacity range with louvered sides. [↑](#footnote-ref-346)
347. VEIC calculated the average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008) to FLH for Central Cooling (provided by AHRI: http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) at 31%. Applying this to the FLH for Central Cooling provided for Baltimore (1050) we get 325 FLH for Room AC. [↑](#footnote-ref-347)
348. Based on maximum capacity average from RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008. [↑](#footnote-ref-348)
349. Based on Nexus Market Research Inc, RLW Analytics, December 2005; “Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report.” [↑](#footnote-ref-349)
350. Minimum Federal Standard for capacity range. [↑](#footnote-ref-350)
351. Calculated by multiplying the ratio of SSP:PJM for the Central AC measure (0.69:0.66) to the assumption for PJM. [↑](#footnote-ref-351)
352. Consistent with coincidence factors found in:

     RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117\_RLW\_CF%20Res%20RAC.pdf). [↑](#footnote-ref-352)
353. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-353)
354. Based on Connecticut TRM; Connecticut Energy Efficiency Fund; CL&P and UI Program Savings Documentation for 2008 Program Year [↑](#footnote-ref-354)
355. Incremental cost of ENERGY STAR unit over baseline unit; consistent with Time of Sale Room AC measure. [↑](#footnote-ref-355)
356. 69% is the ratio of the Net Present Value (with a 5% discount rate) of the annuity payments from years 4 to 12 of a deferred replacement of a standard efficiency unit. The calculation is done in this way to allow the use of the known ENERGY STAR replacement cost to calculate an appropriate baseline replacement cost. [↑](#footnote-ref-356)
357. VEIC calculated the average ratio of FLH for Room AC (provided in RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008) to FLH for Central Cooling (provided by AHRI: http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) at 31%. Applying this to the FLH for Central Cooling provided for Baltimore (1050) we get 325 FLH for Room AC. [↑](#footnote-ref-357)
358. Based on maximum capacity average from RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008. [↑](#footnote-ref-358)
359. Based on Nexus Market Research Inc, RLW Analytics, December 2005; “Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report.” [↑](#footnote-ref-359)
360. Based on Nexus Market Research Inc, RLW Analytics, December 2005; “Impact, Process, and Market Study of the Connecticut Appliance Retirement Program: Overall Report.” Report states that 63% were replaced with ENERGY STAR units and 13% with non-ENERGY STAR. However this formula assumes all are non-ENERGY STAR since the increment of savings between baseline units and ENERGY STAR would be recorded by the Time of Sale measure when the new unit is purchased. [↑](#footnote-ref-360)
361. Minimum Federal Standard for most common Room AC type – 8000-14,999 capacity range with louvered sides.Note that we assume the replacement is only at federal standard efficiency for the reason explained above. [↑](#footnote-ref-361)
362. Calculated by multiplying the ratio of SSP:PJM for the Central AC measure (0.69:0.66) to the assumption for PJM. [↑](#footnote-ref-362)
363. Consistent with coincidence factors found in:

     RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117\_RLW\_CF%20Res%20RAC.pdf). [↑](#footnote-ref-363)
364. $129 replacement cost is calculated by multiplying the percentage assumed to be replaced – 76% by the assumed cost of a standard efficiency unit of $170 (ENERGY STAR calculator; http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/CalculatorConsumerRoomAC.xls); 0.76 \* 170 = $129.2. [↑](#footnote-ref-364)
365. 3 years of remaining useful life based on Connecticut TRM; Connecticut Energy Efficiency Fund; CL&P and UI Program Savings Documentation for 2008 Program Year [↑](#footnote-ref-365)
366. Determined by calculating the Net Present Value (with a 5% discount rate) of the annuity payments from years 4 to 12 of a deferred replacement of a standard efficiency unit costing multiplied by the 76%, the percentage of units being replaced (i.e. 0.76 \* $170 = $129.2. Baseline cost from ENERGY STAR calculator; <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorConsumerRoomAC.xls>) [↑](#footnote-ref-366)
367. Assumption based on data obtained from the 3E Plus heat loss calculation software provided by the NAIMA (North American Insulation Manufacturer Association) and derived from Table 15 and Table 16 of 2009 ASHRAE Fundamentals Handbook, Chapter 23 Insulation for Mechanical Systems, page 23.17. [↑](#footnote-ref-367)
368. Based on simulation model as described in ODC Delaware Technical Resource Manual, April 30, 2012; http://www.dnrec.delaware.gov/energy/information/otherinfo/Documents/EM-and-V-guidance-documents/DELAWARE\_TRM\_August%202012.pdf [↑](#footnote-ref-368)
369. Based on assumption from BG&E billing analysis of furnace program in the '90s, from conversation with Mary Straub; “Evaluation of the High efficiency heating and cooling program, technical report”, June 1995. For other utilities offering this measure, a Heating Degree Day adjustment may be appropriate to this FLHheat assumption. [↑](#footnote-ref-369)
370. Full load heating hours derived by adjusting FLHheat for Baltimore, MD based on Washington, DC HDD base 60°F: 620 \*2957/3457 = 528 hours. [↑](#footnote-ref-370)
371. Assumes 160°F water temp for a boiler without reset control, 120°F for a boiler with reset control, and 50°F air temperature for pipes in unconditioned basements 40°F for pipes in crawlspaces (Zone 4; NCDC 1881-2010 Normals, average of monthly averages Nov – Apr for zones 1-3 and Nov-March for zones 4 and 5). [↑](#footnote-ref-371)
372. Assumed efficiency of existing boilers. [↑](#footnote-ref-372)
373. Consistent with DEER 2008 Database Technology and Measure Cost Data ([www.deeresources.com](http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf)). [↑](#footnote-ref-373)
374. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     [http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf](http://mn.gov/commerce/energy/images/ElectricFoodService_v03.2.xls) [↑](#footnote-ref-374)
375. Energy savings factor for residential applications taken from an article published by the Energy Solutions Center, a consortium of natural gas utilities, equipment manufacturers and vendors. See: http://www.cleanboiler.org/Eff\_Improve/Efficiency/Boiler\_Reset\_Control.asp [↑](#footnote-ref-375)
376. Based on simulation model as described in ODC Delaware Technical Resource Manual, April 30, 2012; http://www.dnrec.delaware.gov/energy/information/otherinfo/Documents/EM-and-V-guidance-documents/DELAWARE\_TRM\_August%202012.pdf [↑](#footnote-ref-376)
377. Based on assumption from BG&E billing analysis of furnace program in the '90s, from conversation with Mary Straub; “Evaluation of the High efficiency heating and cooling program, technical report”, June 1995. For other utilities offering this measure, a Heating Degree Day adjustment may be appropriate to this FLHheat assumption. [↑](#footnote-ref-377)
378. Full load heating hours derived by adjusting FLHheat for Baltimore, MD based on Washington, DC HDD base 60°F: 620 \*2957/3457 = 528 hours. [↑](#footnote-ref-378)
379. Nexant. Questar DSM Market Characterization Report. August 9, 2006. [↑](#footnote-ref-379)
380. The Brooklyn Union Gas Company d/b/a National Grid NY Case 08-G-1016 High-Efficiency Heating and Water Heating and Controls Gas Energy Efficiency Program Implementation Plan, P 37 https://www.nationalgridus.com/non\_html/eer/nydown/NYC%20Expedited%20Program%20Implementation%20Plan.pdf [↑](#footnote-ref-380)
381. The Federal Standard does not include an EER requirement, so it is approximated with this formula: (-0.02 \* SEER2) + (1.12 \* SEER) Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder. [↑](#footnote-ref-381)
382. Minimum Federal Standard as of 4/1/2015;

     http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf [↑](#footnote-ref-382)
383. Federal Standard from 2004 until 2015, Federal Register Vol. 66, No. 11/1/17/2001, page 4497, [http://www1.eere.energy.gov/buildings/appliance\_standards/residential/pdfs/water\_heater\_fr.pdf](http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf) [↑](#footnote-ref-383)
384. Federal Standard from 2004 until 2015, Federal Register Vol. 66, No. 11/1/17/2001, page 4497 [http://www1.eere.energy.gov/buildings/appliance\_standards/residential/pdfs/water\_heater\_fr.pdf](http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls) [↑](#footnote-ref-384)
385. Full Load Cooling Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying the EmPower average Maryland full load hours determined for Maryland (542 from the research referenced below) by the ratio of full load hours in Wilmington, DE (1,015) or Washington, DC (1,320) to Baltimore MD (1,050) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) [↑](#footnote-ref-385)
386. Based on average of 5 utilities in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-386)
387. Minimum Federal Standard as of 1/1/2015;

     http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf [↑](#footnote-ref-387)
388. As per conversations with David Buss territory manager for Connor Co, the SEER and COP ratings of an ASHP equate most appropriately with the part load EER and COP of a GSHP. [↑](#footnote-ref-388)
389. Based on simulation model as described in ODC Delaware Technical Resource Manual, April 30, 2012; http://www.dnrec.delaware.gov/energy/information/otherinfo/Documents/EM-and-V-guidance-documents/DELAWARE\_TRM\_August%202012.pdf [↑](#footnote-ref-389)
390. Based on assumption from BG&E billing analysis of furnace program in the '90s, from conversation with Mary Straub; “Evaluation of the High efficiency heating and cooling program, technical report”, June 1995. For other utilities offering this measure, a Heating Degree Day adjustment may be appropriate to this FLHheat assumption. [↑](#footnote-ref-390)
391. Full load heating hours derived by adjusting FLHheat for Baltimore, MD based on Washington, DC HDD base 60°F: 620 \*2957/3457 = 528 hours. [↑](#footnote-ref-391)
392. Minimum Federal Standard as of 1/1/2015;

     http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf [↑](#footnote-ref-392)
393. As per conversations with David Buss territory manager for Connor Co, the SEER and COP ratings of an ASHP equate most appropriately with the part load EER and COP of a GSHP. [↑](#footnote-ref-393)
394. Assumes that the desuperheater can provide two thirds of hot water needs for eight months of the year (2/3 \* 2/3 = 44%). Based on input from Doug Dougherty, Geothermal Exchange Organization. [↑](#footnote-ref-394)
395. Minimum Federal Standard as of 4/1/2015;

     http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf [↑](#footnote-ref-395)
396. Federal Standard from 2004 until 2015, Federal Register Vol. 66, No. 11/1/17/2001, page 4497, [http://www1.eere.energy.gov/buildings/appliance\_standards/residential/pdfs/water\_heater\_fr.pdf](http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf) [↑](#footnote-ref-396)
397. Based upon email message from Maureen Hodgins, Research Manager for Water Research Foundation, on August 26, 2014. [↑](#footnote-ref-397)
398. US Energy Information Administration, Residential Energy Consumption Survey 2009; http://www.eia.gov/consumption/residential/data/2009/xls/HC9.10%20Household%20Demographics%20in%20South%20Region.xls [↑](#footnote-ref-398)
399. Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential Retrofit Programs.” April 4, 2014, Appendix E, page 66. [↑](#footnote-ref-399)
400. The Federal Standard does not include an EER requirement, so it is approximated with the conversion formula from Wassmer, M. 2003 thesis refererenced below. [↑](#footnote-ref-400)
401. As per conversations with David Buss territory manager for Connor Co, the EER rating of an ASHP equate most appropriately with the full load EER of a GSHP. [↑](#footnote-ref-401)
402. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-402)
403. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-403)
404. Federal Standard from 2004 until 2015, Federal Register Vol. 66, No. 11/1/17/2001, page 4497 [http://www1.eere.energy.gov/buildings/appliance\_standards/residential/pdfs/water\_heater\_fr.pdf](http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls) [↑](#footnote-ref-404)
405. Based on data provided to VEIC in ‘Results of HomE geothermal and air source heat pump rebate incentives documented by Illinois electric cooperatives’. [↑](#footnote-ref-405)
406. Based upon average cost per ton for Equipment and Labor for SEER 14 ASHP from Itron Measure Cost Study Results Matrix Volume 1 (part of “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014). [↑](#footnote-ref-406)
407. System life of indoor components as per DOE estimate http://energy.gov/energysaver/articles/geothermal-heat-pumps. The ground loop has a much longer life, but the compressor and other mechanical components are the same as an ASHP.

     <http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure_life_GDS%5B1%5D.pdf> [↑](#footnote-ref-407)
408. VEIC analysis looking at average baseline fan (i.e. non-Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM. [↑](#footnote-ref-408)
409. On/off cycling controls may be required of baseline fans larger than 50CFM. [↑](#footnote-ref-409)
410. VEIC analysis looking at average efficient fan (i.e. Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM. [↑](#footnote-ref-410)
411. Bi-level controls may be used by efficient fans larger than 50 CFM [↑](#footnote-ref-411)
412. 50CFM is the closest available fan size to ASHRAE 62.2 Section 4.1 Whole House Ventilation rates based upon typical square footage and bedrooms. [↑](#footnote-ref-412)
413. VEIC analysis looking at average baseline fan (i.e. non-Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM. [↑](#footnote-ref-413)
414. VEIC analysis looking at average efficient fan (i.e. Brushless Permanent Magnet) efficacies at static pressures of 0.1 and 0.25 inches of water column for quiet fans rated for 50 CFM. [↑](#footnote-ref-414)
415. Conservative estimate based upon GDS Associates Measure Life Report “Residential and C&I Lighting and HVAC measures” 25 years for whole-house fans, and 19 for thermostatically-controlled attic fans. [↑](#footnote-ref-415)
416. VEIC analysis using cost data collected from wholesale vendor; [http://www.westsidewholesale.com/](http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117_RLW_CF%20Res%20RAC.pdf). [↑](#footnote-ref-416)
417. <http://www.energystar.gov/products/certified-products/detail/ceiling-fans> [↑](#footnote-ref-417)
418. All fan default assumptions are based upon assumptions provided in the ENERGY STAR Ceiling Fan Savings Calculator;

     <http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?8178-e52c> [↑](#footnote-ref-418)
419. Assuming that the CF for a ceiling fan is the same as Room AC; Calculated by multiplying the ratio of SSP:PJM for the Central AC measure (0.69:0.66) to the assumption for PJM. [↑](#footnote-ref-419)
420. Assuming that the CF for a ceiling fan is the same as Room AC; Consistent with coincidence factors found in:

     RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117\_RLW\_CF%20Res%20RAC.pdf). [↑](#footnote-ref-420)
421. Based on EmPOWER\_EY5 Deemed Savings Recommendations\_20Jan2015 DRAFT. [↑](#footnote-ref-421)
422. Ibid. [↑](#footnote-ref-422)
423. ENERGY STAR Ceiling Fan Savings Calculator

     <http://www.energystar.gov/buildings/sites/default/uploads/files/light_fixture_ceiling_fan_calculator.xlsx?8178-e52c> [↑](#footnote-ref-423)
424. Note, the algorithm and variables are provided as documentation for the deemed savings result provided which should be claimed for all showerhead installations. [↑](#footnote-ref-424)
425. The Energy Policy Act of 1992 (EPAct) established the maximum flow rate for showerheads at 2.5 gallons per minute (gpm). [↑](#footnote-ref-425)
426. US Energy Information Administration, Residential Energy Consumption Survey; http://www.eia.doe.gov/emeu/recs/recs2005/hc2005\_tables/hc3demographics/pdf/tablehc11.3.pdf [↑](#footnote-ref-426)
427. Most commonly quoted value of gallons of water used per person per day (including in U.S. Environmental Protection Agency’s “water sense” documents; http://www.epa.gov/watersense/docs/home\_suppstat508.pdf) [↑](#footnote-ref-427)
428. Estimate based on review of a number of studies:

     a. Pacific Northwest Laboratory; "Energy Savings from Energy-Efficient Showerheads: REMP Case Study Results, Proposed Evaluation Algorithm, and Program Design Implications" http://www.osti.gov/bridge/purl.cover.jsp;jsessionid=80456EF00AAB94DB204E848BAE65F199?purl=/10185385-CEkZMk/native/

     b. East Bay Municipal Utility District; "Water Conservation Market Penetration Study" http://www.ebmud.com/sites/default/files/pdfs/market\_penetration\_study\_0.pdf [↑](#footnote-ref-428)
429. Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential Retrofit Programs.” April 4, 2014, Appendix E, page 66. [↑](#footnote-ref-429)
430. Electric water heater have recovery efficiency of 98%: http://www.ahrinet.org/ARI/util/showdoc.aspx?doc=576 [↑](#footnote-ref-430)
431. Calculated as follows: Assume 9% showers take place during peak hours (based on: http://www.aquacraft.com/Download\_Reports/DISAGGREGATED-HOT\_WATER\_USE.pdf)

     9% \* 7.42 minutes per day (11.6 \* 2.56 / 1.6 / 2.5 = 7.42) = 0.668 minutes

     = 0.668 / 180 (minutes in peak period) = 0.00371 [↑](#footnote-ref-431)
432. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 75%. [↑](#footnote-ref-432)
433. This savings estimate is based upon VEIC analysis of data gathered in audit of DC Water Facilities, MWH Global, “Energy Savings Plan, Prepared for DC Water.” Washington, D.C., 2010. See DC Water Conservation.xlsx for calculations and DC Water Conservation Energy Savings\_Final.doc for write-up. This is believed to be a reasonably proxy for the entire region. [↑](#footnote-ref-433)
434. Navigant Consulting, Ontario Energy Board, “Measures and Assumptions for Demand Side Management (DSM) Planning”, April 2009. [↑](#footnote-ref-434)
435. Consistent with assumptions provided on page C-6 of Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007. (http://neep.org/uploads/EMV%20Forum/EMV%20Studies/measure\_life\_GDS%5B1%5D.pdf) [↑](#footnote-ref-435)
436. Note, the algorithm and variables are provided as documentation for the deemed savings result provided which should be claimed for all faucet aerator installations. [↑](#footnote-ref-436)
437. In 1998, the Department of Energy adopted a maximum flow rate standard of 2.2 gpm at 60 psi for all faucets: 63 Federal Register 13307; March 18, 1998. [↑](#footnote-ref-437)
438. US Energy Information Administration, Residential Energy Consumption Survey; http://www.eia.doe.gov/emeu/recs/recs2005/hc2005\_tables/hc3demographics/pdf/tablehc11.3.pdf [↑](#footnote-ref-438)
439. Most commonly quoted value of gallons of water used per person per day (including in U.S. Environmental Protection Agency’s “water sense” documents; http://www.epa.gov/watersense/docs/home\_suppstat508.pdf) [↑](#footnote-ref-439)
440. Estimate consistent with Ontario Energy Board, "Measures and Assumptions for Demand Side Management Planning." [↑](#footnote-ref-440)
441. Estimate based on East Bay Municipal Utility District; "Water Conservation Market Penetration Study" http://www.ebmud.com/sites/default/files/pdfs/market\_penetration\_study\_0.pdf [↑](#footnote-ref-441)
442. Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential Retrofit Programs.” April 4, 2014, Appendix E, page 66. [↑](#footnote-ref-442)
443. Electric water heater have recovery efficiency of 98%: http://www.ahrinet.org/ARI/util/showdoc.aspx?doc=576 [↑](#footnote-ref-443)
444. Calculated as follows: Assume 13% faucet use takes place during peak hours (based on: http://www.aquacraft.com/Download\_Reports/DISAGGREGATED-HOT\_WATER\_USE.pdf)

     13% \* 3.6 minutes per day (10.9 \* 2.56 / 3.5 / 2.2 = 3.6) = 0.47 minutes

     = 0.47 / 180 (minutes in peak period) = 0.00262 [↑](#footnote-ref-444)
445. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 75%. [↑](#footnote-ref-445)
446. This savings estimate is based upon VEIC analysis of data gathered in audit of DC Water Facilities, MWH Global, “Energy Savings Plan, Prepared for DC Water.” Washington, D.C., 2010. See DC Water Conservation.xlsx for calculations and DC Water Conservation Energy Savings\_Final.doc for write-up. This is believed to be a reasonably proxy for the entire region. [↑](#footnote-ref-446)
447. Navigant Consulting, Ontario Energy Board, “Measures and Assumptions for Demand Side Management (DSM) Planning”, April 2009. [↑](#footnote-ref-447)
448. Conservative estimate based on review of TRM assumptions from other States. [↑](#footnote-ref-448)
449. Assumptions are from Pennsylvania Public Utility Commission Technical Reference Manual (PA TRM) for a poorly insulated 40 gallon tank [↑](#footnote-ref-449)
450. Assumes an R-10 tank wrap is added. [↑](#footnote-ref-450)
451. Assumptions from PA TRM for 40 gallon tank. Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation. Area includes tank sides and top to account for typical wrap coverage. [↑](#footnote-ref-451)
452. Ibid. [↑](#footnote-ref-452)
453. Assumes 125°F water leaving the hot water tank and average temperature of basement of 65°F. [↑](#footnote-ref-453)
454. NREL, National Residential Efficiency Measures Database, <http://www.nrel.gov/ap/retrofits/measures.cfm?gId=6&ctId=40> [↑](#footnote-ref-454)
455. DOE, “Residential Heating Products Final Rule Technical Support Document,” Table 3.2.13, <http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch3.pdf> [↑](#footnote-ref-455)
456. DOE, “Residential Heating Products Final Rule Technical Support Document,” Table 3.2.13, <http://www1.eere.energy.gov/buildings/appliance_standards/residential/pdfs/htgp_finalrule_ch3.pdf> [↑](#footnote-ref-456)
457. Based on VEIC online product review. [↑](#footnote-ref-457)
458. Conservative estimate that assumes the tank wrap is installed on an existing unit with 5 years remaining life. [↑](#footnote-ref-458)
459. Navigant Consulting Inc., April 2009; “Measures and Assumptions for Demand Side Management (DSM) Planning; Appendix C Substantiation Sheets”, p77, presented to the Ontario Energy Board:

     <http://www.oeb.gov.on.ca/OEB/_Documents/EB-2008-0346/Navigant_Appendix_C_substantiation_sheet_20090429.pdf> [↑](#footnote-ref-459)
460. Assumes 130°F water leaving the hot water tank and average temperature of basement of 65°F. [↑](#footnote-ref-460)
461. Electric water heaters have recovery efficiency of 98%: http://www.ahrinet.org/ARI/util/showdoc.aspx?doc=576 [↑](#footnote-ref-461)
462. Review of AHRI Directory suggests range of recovery efficiency ratings for *new* Gas DHW units of 70-87%. Average of *existing* units is estimated at 75% [↑](#footnote-ref-462)
463. Consistent with DEER 2008 Database Technology and Measure Cost Data (www.deeresources.com). [↑](#footnote-ref-463)
464. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-464)
465. The Baseline Energy Factor is based on the Federal Minimum Standard for water heaters sold on or after April 16 2015. This ruling can be found here:

     http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf [↑](#footnote-ref-465)
466. http://www.energystar.gov/index.cfm?c=water\_heat.pr\_crit\_water\_heaters [↑](#footnote-ref-466)
467. The disconnect between rated energy factor and in-situ energy consumption is markedly different for tankless units due to significantly higher contributions to overall household hot water usage from short draws. In tankless units the large burner and unit heat exchanger must fire and heat up for each draw. The additional energy losses incurred when the mass of the unit cools to the surrounding space in-between shorter draws was found to be 9% in a study prepared for Lawrence Berkeley National Laboratory by Davis Energy Group, 2006. “Field and Laboratory Testing of Tankless Gas Water Heater Performance” Due to the similarity (storage) between the other categories and the baseline, this derating factor is applied only to the tankless category. [↑](#footnote-ref-467)
468. Email message from Maureen Hodgins, Research Manager for Water Research Foundation, to TAC/SAG, August 26, 2014 [↑](#footnote-ref-468)
469. US Energy Information Administration, Residential Energy Consumption Survey 2009; http://www.eia.gov/consumption/residential/data/2009/xls/HC9.10%20Household%20Demographics%20in%20South%20Region.xls [↑](#footnote-ref-469)
470. Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential Retrofit Programs.” April 4, 2014, Appendix E, page 66. [↑](#footnote-ref-470)
471. Source for cost info; DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Table 8.2.14 (http://www1.eere.energy.gov/buildings/appliance\_standards/residential/pdfs/htgp\_finalrule\_ch8.pdf) [↑](#footnote-ref-471)
472. Based on ACEEE Life-Cycle Cost analysis; http://www.aceee.org/node/3068#lcc [↑](#footnote-ref-472)
473. Minimum Federal Standard as of 4/1/2015;

     http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf [↑](#footnote-ref-473)
474. Minimum Federal Standard as of 1/1/2015;

     http://www.gpo.gov/fdsys/pkg/CFR-2012-title10-vol3/pdf/CFR-2012-title10-vol3-sec430-32.pdf [↑](#footnote-ref-474)
475. Efficiency based on ENERGY STAR Residential Water Heaters, Final Criteria Analysis: http://www.energystar.gov/ia/partners/prod\_development/new\_specs/downloads/water\_heaters/WaterHeaterDraftCriteriaAnalysis.pdf [↑](#footnote-ref-475)
476. Email message from Maureen Hodgins, Research Manager for Water Research Foundation, to TAC/SAG, August 26, 2014. Describes water usage for a house size of 2.59 people. [↑](#footnote-ref-476)
477. US Energy Information Administration, Residential Energy Consumption Survey 2009; http://www.eia.gov/consumption/residential/data/2009/xls/HC9.10%20Household%20Demographics%20in%20South%20Region.xls [↑](#footnote-ref-477)
478. Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential Retrofit Programs.” April 4, 2014, Appendix E, page 66. [↑](#footnote-ref-478)
479. This algorithm calculates the heat removed from the air by subtracting the HPWH electric consumption from the total water heating energy delivered. This is then adjusted to account for location of the HP unit and the coincidence of the waste heat with cooling requirements, the efficiency of the central cooling and latent cooling demands. [↑](#footnote-ref-479)
480. REMRate determined percentage (33%) of lighting savings that result in reduced cooling loads for several different building configurations in Wilmington, DE, Baltimore, MD and Washington, DC (lighting is used as a proxy for hot water heating since load shapes suggest their seasonal usage patterns are similar). [↑](#footnote-ref-480)
481. A sensible heat ratio (SHR) of 0.75 corresponds to a latent multiplier of 4/3 or 1.33. SHR of 0.75 for typical split system from page 10 of “Controlling Indoor Humidity Using Variable-Speed Compressors and Blowers” by M. A. Andrade and C. W. Bullard, 1999: www.ideals.illinois.edu/bitstream/handle/2142/11894/TR151.pdf [↑](#footnote-ref-481)
482. REMRate determined percentage (47%) of lighting savings that result in increased heating loads (lighting is used as a proxy for hot water heating since load shapes suggest their seasonal usage patterns are similar). [↑](#footnote-ref-482)
483. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-483)
484. Based on a chart showing summer weekday average electrical demand on page 10 of FEMP Study “Field Testing of Pre-Production Prototype Residential Heat Pump Water Heaters” (<http://www1.eere.energy.gov/femp/pdfs/tir_heatpump.pdf>). Using data points from the chart, the average delta kW in heat pump mode during the peak hours compared to resistance mode is 0.17kW. [↑](#footnote-ref-484)
485. This is the additional energy consumption required to replace the heat removed from the home during the heating season by the heat pump water heater. [↑](#footnote-ref-485)
486. Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: ([http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf](http://www.cee1.org/gov/led/led-ace3/ace3led.pdf) ) or by performing duct blaster testing. [↑](#footnote-ref-486)
487. This has been estimated assuming typical efficiencies of existing heating systems weighted by percentage of homes with non-electric heating (based on Energy Information Administration, 2009 Residential Energy Consumption Survey: http://www.eia.gov/consumption/residential/data/2009/xls/HC6.9%20Space%20Heating%20in%20Midwest%20Region.xls). [↑](#footnote-ref-487)
488. Based on KEMA baseline study for Maryland. [↑](#footnote-ref-488)
489. DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Table 8.2.14 [http://www1.eere.energy.gov/buildings/appliance\_standards/residential/pdfs/htgp\_finalrule\_ch8.pdf](http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/LightingCalculator.xlsx) [↑](#footnote-ref-489)
490. DOE, 2010 Residential Heating Products Final Rule Technical Support Document, Page 8-52 [http://www1.eere.energy.gov/buildings/appliance\_standards/residential/pdfs/htgp\_finalrule\_ch8.pdf](http://www.icc.illinois.gov/downloads/public/edocket/303834.pdf) [↑](#footnote-ref-490)
491. Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Mid Atlantic Region. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used. [↑](#footnote-ref-491)
492. The Energy Policy Act of 1992 (EPAct) established the maximum flow rate for showerheads at 2.5 gallons per minute (gpm). [↑](#footnote-ref-492)
493. Average of the following sources: ShowerStart LLC survey; “Identifying, Quantifying and Reducing Behavioral Waste in the Shower: Exploring the Savings Potential of ShowerStart”, City of San Diego Water Department survey; “Water Conservation Program: ShowerStart Pilot Project White Paper”, and PG&E Work Paper PGECODHW113. [↑](#footnote-ref-493)
494. US Energy Information Administration, Residential Energy Consumption Survey; http://www.eia.doe.gov/emeu/recs/recs2005/hc2005\_tables/hc3demographics/pdf/tablehc11.3.pdf [↑](#footnote-ref-494)
495. Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum dated June 2013, directed to Michigan Evaluation Working Group. [↑](#footnote-ref-495)
496. Estimate based on review of a number of studies:

     a. Pacific Northwest Laboratory; "Energy Savings from Energy-Efficient Showerheads: REMP Case Study Results, Proposed Evaluation Algorithm, and Program Design Implications" http://www.osti.gov/bridge/purl.cover.jsp;jsessionid=80456EF00AAB94DB204E848BAE65F199?purl=/10185385-CEkZMk/native/

     b. East Bay Municipal Utility District; "Water Conservation Market Penetration Study" http://www.ebmud.com/sites/default/files/pdfs/market\_penetration\_study\_0.pdf [↑](#footnote-ref-496)
497. Based on “Water and Energy Wasted During Residential Shower Events: Findings from a Pilot Field Study of Hot Water Distribution Systems”, Jim Lutz, Lawrence Berkeley National Laboratory, September 2011. [↑](#footnote-ref-497)
498. Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential Retrofit Programs.” April 4, 2014, Appendix E, page 66. [↑](#footnote-ref-498)
499. Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx> [↑](#footnote-ref-499)
500. 74.6% is the proportion of hot 120F water mixed with 60.1F supply water to give 105F shower water. [↑](#footnote-ref-500)
501. Calculated as follows: Assume 11% showers take place during peak hours (based on: <http://www.aquacraft.com/sites/default/files/pub/DeOreo-%282001%29-Disaggregated-Hot-Water-Use-in-Single-Family-Homes-Using-Flow-Trace-Analysis.pdf>). There are 65 days in the summer peak period, so the percentage of total annual use in peak period is 0.11\*65/365 = 1.96%. The number of hours of recovery during peak periods is therefore assumed to be 1.96% \* 19.4 = 0.38 hours of recovery during peak period, where 19.4 equals the annual electric DHW recovery hours for showerhead use prevented by the device. There are 260 hours in the peak period so the probability you will see savings during the peak period is 0.38/260 = 0.0015 [↑](#footnote-ref-501)
502. Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Mid Attlantic Region.. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used. [↑](#footnote-ref-502)
503. DOE Final Rule discusses Recovery Efficiency with an average around 0.76 for Gas Fired Storage Water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 75%. [↑](#footnote-ref-503)
504. Assumptions based on NY TRM, Pacific Gas and Electric Company Work Paper PGECODHW113, and measure life of low-flow showerhead [↑](#footnote-ref-504)
505. Based on actual cost of the SS-1002CP-SB Ladybug Water-Saving Shower-Head adapter from Evolve showerheads [↑](#footnote-ref-505)
506. Note this algorithm provides savings only from reduction in standby losses. VEIC considered avoided energy from not heating the water to the higher temperature but determined that the potential impact for the three major hot water uses was too small to be characterized; Dishwashers are likely to boost the temperature within the unit (roughly canceling out any savings), faucet and shower use is likely to be at the same temperature so there would need to be more lower temperature hot water being used (cancelling any savings) and clothes washers will only see savings if the water from the tank is taken without any temperature control. [↑](#footnote-ref-506)
507. Assumptions from Pennsylvania TRM. Area values were calculated from average dimensions of several commercially available units, with radius values measured to the center of the insulation. [↑](#footnote-ref-507)
508. Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx> [↑](#footnote-ref-508)
509. Review of AHRI Directory suggests range of recovery efficiency ratings for new Gas DHW units of 70-87%. Average of existing units is estimated at 75%. [↑](#footnote-ref-509)
510. Based on the average clothes washer volume of all units that pass the new Federal Standard on the California Energy Commission (CEC) database of Clothes Washer products accessed on 08/28/2014. [↑](#footnote-ref-510)
511. Weighting percentages are based on available product from the CEC database accessed on 08/28/2014. [↑](#footnote-ref-511)
512. Metered data from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program.” March 21, 2014, page 36. [↑](#footnote-ref-512)
513. The percentage of total energy consumption that is used for the machine, heating the hot water or by the dryer is different depending on the efficiency of the unit. Values are based on a weighted average of top loading and front loading units (based on available product from the CEC Appliance database) and consumption data from Life-Cycle Cost and Payback Period Excel-based analytical tool, available online at: http://www1.eere.energy.gov/buildings/appliance\_standards/residential/docs/rcw\_dfr\_lcc\_standard.xlsm. See “2015 E$ Clothes Washer Analysis.xls” for the calculation. [↑](#footnote-ref-513)
514. Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Mid Atlantic States. [↑](#footnote-ref-514)
515. Default assumption for unknown is based on percentage of homes with electric dryer from EIA Residential Energy Consumption Survey (RECS) 2009 for Mid Atlantic States. [↑](#footnote-ref-515)
516. Note that the baseline savings for all cases (Front, Top and Weighted Average) is based on the weighted average baseline IMEF (as opposed to assuming Front baseline for Front efficient unit). The reasoning is that the support of the program of more efficient units (which are predominately front loading) will result in some participants switching from planned purchase of a top loader to a front loader. [↑](#footnote-ref-516)
517. Metered data from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program.” March 21, 2014, page 36. [↑](#footnote-ref-517)
518. Ibid. [↑](#footnote-ref-518)
519. To account for the different efficiency of electric and Natural Gas hot water heaters (gas water heater: recovery efficiencies ranging from 0.74 to 0.85 (0.78 used), and electric water heater with 0.98 recovery efficiency (<http://www.energystar.gov/ia/partners/bldrs_lenders_raters/downloads/Waste_Water_Heat_Recovery_Guidelines.pdf> ). Therefore a factor of 0.98/0.78 (1.26) is applied. [↑](#footnote-ref-519)
520. Default assumption for unknown fuel is based on percentage of homes with gas DHW from EIA Residential Energy Consumption Survey (RECS) 2009 for Mid Atlantic States. [↑](#footnote-ref-520)
521. Default assumption for unknown is based on percentage of homes with gas dryer from EIA Residential Energy Consumption Survey (RECS) 2009 for Mid Atlantic States. [↑](#footnote-ref-521)
522. Based on relevant specifications as of March 2015. Weighting percentages are based on available product from the CEC database accessed on 08/28/2014. [↑](#footnote-ref-522)
523. This savings estimate is based upon VEIC analysis of data gathered in audit of DC Water Facilities, MWH Global, “Energy Savings Plan, Prepared for DC Water.” Washington, D.C., 2010. See DC Water Conservation.xlsx for calculations and DC Water Conservation Energy Savings\_Final.doc for write-up. This is believed to be a reasonably proxy for the entire region. [↑](#footnote-ref-523)
524. Based on weighted average of top loading and front loading units (based on available product from the CEC Appliance database) and cost data from Life-Cycle Cost and Payback Period Excel-based analytical tool. See ‘2015 Mid Atlantic Early Replacement Clothes Washer Analysis.xls’ for details.. [↑](#footnote-ref-524)
525. Based on DOE Life-Cycle Cost and Payback Period Excel-based analytical tool, available online at: <http://www1.eere.energy.gov/buildings/appliance_standards/residential/docs/rcw_dfr_lcc_standard.xlsm> [↑](#footnote-ref-525)
526. Based on 1/3 of the measure life. [↑](#footnote-ref-526)
527. Based on the average clothes washer volume of all units that pass the new Federal Standard on the California Energy Commission (CEC) database of Clothes Washer products accessed on 08/28/2014**.** [↑](#footnote-ref-527)
528. Weighting percentages are based on available product from the CEC database. [↑](#footnote-ref-528)
529. Existing units efficiencies are based upon an MEF of 1.26 , the 2004 Federal Standard, converted to IMEF using an ENERGY STAR conversion tool. [↑](#footnote-ref-529)
530. For early replacement measures we will always know the configuration of the replaced machine. [↑](#footnote-ref-530)
531. Metered data from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program.” March 21, 2014, page 36. [↑](#footnote-ref-531)
532. The percentage of total energy consumption that is used for the machine, heating the hot water or by the dryer is different depending on the efficiency of the unit. Values are based on a weighted average of top loading and front loading units (based on available product from the CEC Appliance database) and consumption data from Life-Cycle Cost and Payback Period Excel-based analytical tool, available online at: <http://www1.eere.energy.gov/buildings/appliance_standards/residential/docs/rcw_dfr_lcc_standard.xlsm>. [↑](#footnote-ref-532)
533. Metered data from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program.” March 21, 2014, page 36. [↑](#footnote-ref-533)
534. Ibid. [↑](#footnote-ref-534)
535. To account for the different efficiency of electric and Natural Gas hot water heaters (gas water heater: recovery efficiencies ranging from 0.74 to 0.85 (0.78 used), and electric water heater with 0.98 recovery efficiency (<http://www.energystar.gov/ia/partners/bldrs_lenders_raters/downloads/Waste_Water_Heat_Recovery_Guidelines.pdf> ). Therefore a factor of 0.98/0.78 (1.26) is applied. [↑](#footnote-ref-535)
536. Based on relevant specifications as of March 2015. Weighting percentages are based on available product from the CEC database. [↑](#footnote-ref-536)
537. Existing units efficiencies are based upon an WF of 7.93 which was the previous new baseline assumption – converted to IWF using an ENERGY STAR conversion tool copied in to the “2015 Mid Atlantic Early Replacement Clothes Washer Analysis.xls” worksheet. [↑](#footnote-ref-537)
538. For early replacement measures we will always know the configuration of the replaced machine. [↑](#footnote-ref-538)
539. Water Factor is the number of gallons required for each cubic foot of laundry. For ENERGY STAR and CEE Tiers 2 and 3 the average WF of units in the following evaluation are used; Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program.” March 21, 2014, page 36. For baseline and ENERGY STAR Most Efficient the average WF of the post 1/1/2011 units available in each classification is used (based on data pulled from the California Energy Commission Appliance Efficiency Database <http://www.appliances.energy.ca.gov/>) [↑](#footnote-ref-539)
540. This savings estimate is based upon VEIC analysis of data gathered in audit of DC Water Facilities, MWH Global, “Energy Savings Plan, Prepared for DC Water.” Washington, D.C., 2010. See DC Water Conservation.xlsx for calculations and DC Water Conservation Energy Savings\_Final.doc for write-up. This is believed to be a reasonably proxy for the entire region. [↑](#footnote-ref-540)
541. Based on weighted average of top loading and front loading units (based on available product from the CEC Appliance database) and cost data from Life-Cycle Cost and Payback Period Excel-based analytical tool. See ‘2015 Mid Atlantic Early Replacement Clothes Washer Analysis.xls’ for details. [↑](#footnote-ref-541)
542. Based on DOE Life-Cycle Cost and Payback Period Excel-based analytical tool, available online at: <http://www1.eere.energy.gov/buildings/appliance_standards/residential/docs/rcw_dfr_lcc_standard.xlsm> [↑](#footnote-ref-542)
543. Based on 1/3 of the measure life. [↑](#footnote-ref-543)
544. Energy Star Version 3.0 became effective 10/1/12 [↑](#footnote-ref-544)
545. The Federal Standard for Dehumidifiers changed as of October 2012; https://www.federalregister.gov/articles/2010/12/02/2010-29756/energy-conservation-program-for-consumer-products-test-procedures-for-residential-dishwashers#h-11 [↑](#footnote-ref-545)
546. <http://www.energystar.gov/ia/partners/prod_development/revisions/downloads/dehumid/ES_Dehumidifiers_Final_V3.0_Eligibility_Criteria.pdf?d70c-99b0> [↑](#footnote-ref-546)
547. Based on 68 days of 24 hour operation; ENERGY STAR Dehumidifier Calculator <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/appliance_calculator.xlsx?f3f7-6a8b&f3f7-6a8b> [↑](#footnote-ref-547)
548. [↑](#footnote-ref-548)
549. Based on 68 days of 24 hour operation; ENERGY STAR Dehumidifier Calculator <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/appliance_calculator.xlsx?f3f7-6a8b&f3f7-6a8b> [↑](#footnote-ref-549)
550. Assume usage is evenly distributed day vs. night, weekend vs. weekday and is used between April through the end of September (4392 possible hours). 1632 operating hours from ENERGY STAR Dehumidifier Calculator. Coincidence peak during summer peak is therefore 1632/4392 = 37.2% [↑](#footnote-ref-550)
551. Based on available data from the Department of Energy’s Life Cycle Cost analysis spreadsheet:

     <http://www1.eere.energy.gov/buildings/appliance_standards/residential/docs/lcc_dehumidifier.xls> [↑](#footnote-ref-551)
552. ENERGY STAR Dehumidifier Calculator <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/appliance_calculator.xlsx?f3f7-6a8b&f3f7-6a8b> [↑](#footnote-ref-552)
553. ENERGY STAR Appliance Savings Calculator; <http://www.energystar.gov/buildings/sites/default/uploads/files/appliance_calculator.xlsx?7224-046c=&7224-__046ceiling_fan_calculator_xlsx=&a0f2-2e6f&a0f2-2e6f> [↑](#footnote-ref-553)
554. Measured according to the latest ANSI/AHAM AC-1 (AC-1) Standard [↑](#footnote-ref-554)
555. Based on assumptions found in the ENERGY STAR Appliance Savings Calculator;

     Efficiency 1.0 CADR/Watt, 16 hours a day, 365 days a year and 1W standby power. [↑](#footnote-ref-555)
556. Ibid.

     Efficiency 3.0 CADR/Watt, 16 hours a day, 365 days a year and 0.6W standby power. [↑](#footnote-ref-556)
557. Consistent with ENERGY STAR Qualified Room Air Cleaner Calculator; 16 hours a day, 365 days a year. [↑](#footnote-ref-557)
558. Assumes appliance use is equally likely at any hour of the day or night. [↑](#footnote-ref-558)
559. ENERGY STAR Appliance Savings Calculator; EPA research on available models, 2012 [↑](#footnote-ref-559)
560. ENERGY STAR Appliance Savings Calculator; Based on Appliance Magazine, Portrait of the U.S. Appliance Industry 1998. [↑](#footnote-ref-560)
561. Some types of room air cleaners require filter replacement or periodic cleaning, but this is likely to be true for both efficient and baseline units and so no difference in cost is assumed. [↑](#footnote-ref-561)
562. ENERGY STAR Market & Industry Scoping Report. Residential Clothes Dryers. Table 8. November 2011. <http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf> [↑](#footnote-ref-562)
563. Based on ENERGY STAR test procedures. <https://www.energystar.gov/index.cfm?c=clothesdry.pr_crit_clothes_dryers> [↑](#footnote-ref-563)
564. ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis [↑](#footnote-ref-564)
565. Federal standards report CEF for gas clothes dryers in terms of lbs/kWh. To determine gas savings, this number is later converted to therms. [↑](#footnote-ref-565)
566. ENERGY STAR Clothes Dryers Key Product Criteria. <https://www.energystar.gov/index.cfm?c=clothesdry.pr_crit_clothes_dryers> [↑](#footnote-ref-566)
567. Federal standards report CEF for gas clothes dryers in terms of lbs/kWh. To determine gas savings, this number is later converted to therms. [↑](#footnote-ref-567)
568. Ecova, ‘Dryer Field Study’, Northwest Energy Efficiency Alliance (NEEA) 2014. [↑](#footnote-ref-568)
569. %Electric accounts for the fact that some of the savings on gas dryers comes from electricity (motors, controls, etc). 16% was determined using a ratio of the electric to total savings from gas dryers given by ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis. [↑](#footnote-ref-569)
570. Assumes average of 56 minutes per cycle based on Ecova, ‘Dryer Field Study’, Northwest Energy Efficiency Alliance (NEEA) 2014 [↑](#footnote-ref-570)
571. Consistent with coincidence factor of Clothes Washers; Metered data from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program.” March 21, 2014, page 36. [↑](#footnote-ref-571)
572. %Gas accounts for the fact that some of the savings on gas dryers comes from electricity (motors, controls, etc). 84% was determined using a ratio of the gas to total savings from gas dryers given by ENERGY STAR Draft 2 Version 1.0 Clothes Dryers Data and Analysis. [↑](#footnote-ref-572)
573. Based on the difference in installed cost for an efficient dryer ($716) and standard dryer ($564). <http://www.aceee.org/files/proceedings/2012/data/papers/0193-000286.pdf> [↑](#footnote-ref-573)
574. Based on an average estimated range of 12-16 years. ENERGY STAR Market & Industry Scoping Report. Residential Clothes Dryers. November 2011. <http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf> [↑](#footnote-ref-574)
575. http://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/67 [↑](#footnote-ref-575)
576. http://www.energystar.gov/sites/default/files/specs//private/ENERGY%20STAR%20Version%205.2%20Residential%20Dishwasher%20Program%20Requirements.pdf [↑](#footnote-ref-576)
577. http://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Residential%20Dishwasher%20Version%206%200%20Final%20Draft%20Specification\_Final.pdf [↑](#footnote-ref-577)
578. The Federal Standard and ENERGY STAR annual consumption values include electric consumption for both the operation of the machine and for heating the water that is used by the machine. [↑](#footnote-ref-578)
579. ENERGY STAR Dishwasher Calculator, see ‘EnergyStarCalculatorConsumerDishwasher.xls’. [↑](#footnote-ref-579)
580. Ibid. [↑](#footnote-ref-580)
581. Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for South Region, data for the Mid-Atlantic region. [↑](#footnote-ref-581)
582. Assuming one and a half hours per cycle and 140 cycles per year therefore 210 operating hours per year; 140 cycles per year is based on a weighted average of dishwasher usage in Mid-Atlantic region derived from the 2009 RECs data; [http://205.254.135.7/consumption/residential/data/2009/](http://mn.gov/commerce/energy/images/ElectricFoodService_v03.2.xls) [↑](#footnote-ref-582)
583. Based on 8760 end use data for Missouri, provided to VEIC by Ameren for use in the Illinois TRM. The average DW load during peak hours is divided by the peak load. In the absence of a Mid Atlantic specific loadshape this is deemed a reasonable proxy since loads would likely be similar. [↑](#footnote-ref-583)
584. Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for South Region, data for the states of Delaware, Maryland, West Virginia and the District of Columbia. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area then that should be used. [↑](#footnote-ref-584)
585. To account for the different efficiency of electric and Natural Gas hot water heaters (gas water heater: recovery efficiencies ranging from 0.74 to 0.85 (0.78 used), and electric water heater with 0.98 recovery efficiency ([http://www.energystar.gov/ia/partners/bldrs\_lenders\_raters/downloads/Waste\_Water\_Heat\_Recovery\_Guidelines.pdf](http://www.eia.gov/consumption/residential/data/2009/xls/HC7.9%20Air%20Conditioning%20in%20Midwest%20Region.xls)). Therefore a factor of 0.98/0.78 (1.26) is applied. [↑](#footnote-ref-585)
586. Assuming 5 gallons/cycle (maximum allowed) and 140 cycles per year based on a weighted average of dishwasher usage in the Mid-Atlantic Region derived from the 2009 RECs data; [http://205.254.135.7/consumption/residential/data/2009/](http://205.254.135.7/consumption/residential/data/2009/xls/HC7.1%20Air%20Conditioning%20by%20Housing%20Unit%20Type.xls) [↑](#footnote-ref-586)
587. Assuming 4.25 gallons/cycle (maximum allowed) and 140 cycles per year based on a weighted average of dishwasher usage in the Mid-Atlantic Region derived from the 2009 RECs data; [http://205.254.135.7/consumption/residential/data/2009/](http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Consumer_Residential_Refrig_Sav_Calc.xls) [↑](#footnote-ref-587)
588. Assuming 3.50 gallons/cycle (maximum allowed) and 140 cycles per year based on a weighted average of dishwasher usage in the Mid-Atlantic Region derived from the 2009 RECs data; [http://205.254.135.7/consumption/residential/data/2009/](http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/Consumer_Residential_Refrig_Sav_Calc.xls) [↑](#footnote-ref-588)
589. Estimate based on review of Energy Star stakeholder documents [↑](#footnote-ref-589)
590. ENERGY STAR Dishwasher Calculator, see ‘EnergyStarCalculatorConsumerDishwasher.xls’. [↑](#footnote-ref-590)
591. See BPI Building Analyst and Envelope Professional standards, <http://www.bpi.org/standards_approved.aspx> [↑](#footnote-ref-591)
592. N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and exposure of the home to wind, based on methodology developed by Lawrence Berkeley Laboratory (LBL). Since there is minimal stack effect due to low delta T, the height of the building is not included in determining n-factor for cooling savings.

     <http://www.homeenergy.org/archive/hem.dis.anl.gov/eehem/94/940111.html#94011122> [↑](#footnote-ref-592)
593. N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30 year climate normals. For more information see Bruce Harley, CLEAResult “Infiltration Factor Calculations Methodology.doc”. [↑](#footnote-ref-593)
594. Derived by summing the delta between the average outdoor temperature and the base set point of 75 degrees (above which cooling is assumed to be used), each hour of the year. Hourly temperature data obtained from TMY3 data (http://rredc.nrel.gov/solar/old\_data/nsrdb/1991-2005/tmy3/by\_state\_and\_city.html) [↑](#footnote-ref-594)
595. To account for the fact that people do not always operate their air conditioning system when the outside temperature is greater than 75°F. Based on Energy Center of Wisconsin, May 2008 metering study; “Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research”, p31. [↑](#footnote-ref-595)
596. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-596)
597. Derived by calculating the sensible and total loads in each hour. For more information see Bruce Harley, CLEAResult “Infiltration Factor Calculations Methodology.doc”. [↑](#footnote-ref-597)
598. N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30 year climate normals. For more information see Bruce Harley, CLEAResult “Infiltration Factor Calculations Methodology.doc”. [↑](#footnote-ref-598)
599. The 10 year average annual heating degree day value is calculated for each location, using a balance point for heating equipment use of 60 degrees (based on data obtained from <http://academic.udayton.edu/kissock/http/Weather/citylistUS.htm>). The 60 degree balance point is used based on a PRISM evaluation of approximately 600,000 Ohio residential single family customers showing this is the point below which heating is generally used. [↑](#footnote-ref-599)
600. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. [↑](#footnote-ref-600)
601. To convert HSPF to COP, divide the HSPF rating by 3.413. [↑](#footnote-ref-601)
602. Full Load Cooling Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying the EmPower average Maryland full load hours determined for Maryland (542 from the research referenced below) by the ratio of full load hours in Wilmington, DE (1,015) or Washington, DC (1,320) to Baltimore MD (1,050) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) [↑](#footnote-ref-602)
603. Based on average of 5 utilities in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-603)
604. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-604)
605. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-605)
606. N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegisis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9) to the reported wind speeds and outdoor temperatures provided by the NRDC 30 year climate normals. For more information see Bruce Harley, CLEAResult “Infiltration Factor Calculations Methodology.doc”. [↑](#footnote-ref-606)
607. The 10 year average annual heating degree day value is calculated for a number of locations, using a balance point for heating equipment use of 60 degrees (based on data obtained from http://www.engr.udayton.edu/weather/). The 60 degree balance point is used based on a PRISM evaluation of approximately 600,000 Ohio residential single family customers showing this is the point below which heating is generally used. [↑](#footnote-ref-607)
608. Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>) or by performing duct blaster testing. [↑](#footnote-ref-608)
609. The equipment efficiency default is based on data provided by GAMA during the federal rule-making process for furnace efficiency standards, suggesting that in 2000, 32% of furnaces purchased in Maryland were condensing units. Assuming an efficiency of 92% for the condensing furnaces and 80% for the non-condensing furnaces gives a weighted average of 83.8%. The distribution efficiency default is based on assumption that 50% of duct work is inside the envelope, with some leaks and no insulation. VEIC did not have any more specific data to provide any additional defaults. [↑](#footnote-ref-609)
610. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-610)
611. The R-5 assumption for roof assembly is based on J.Neymark & Associates and National Renewable Energy Laboratory, June 2009; “BESTEST-EX Interim Test Procedure” p27. The attic floor and roof should be modeled as a system including solar gains and attic ventilation, and R-5 is the standard assumption for the thermal resistance of the whole attic/roof system. [↑](#footnote-ref-611)
612. Derived by summing the delta between the average outdoor temperature and the base set point of 75 degrees (above which cooling is assumed to be used), each hour of the year. Hourly temperature data obtained from TMY3 data (http://rredc.nrel.gov/solar/) [↑](#footnote-ref-612)
613. To account for the fact that people do not always operate their air conditioning system when the outside temperature is greater than 75°F. Based on Energy Center of Wisconsin, May 2008 metering study; “Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research”, p31. [↑](#footnote-ref-613)
614. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-614)
615. From Illinois TRM, 9 as demonstrated in two years of metering evaluation by Opinion Dynamics [↑](#footnote-ref-615)
616. The 10 year average annual heating degree day value is calculated for a number of locations, using a balance point for heating equipment use of 60 degrees (based on data obtained from <http://academic.udayton.edu/kissock/http/Weather/citylistUS.htm>). The 60 degree balance point is used based on a PRISM evaluation of approximately 600,000 Ohio residential single family customers showing this is the point below which heating is generally used. [↑](#footnote-ref-616)
617. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-617)
618. From Illinois TRM, 9 as demonstrated in two years of metering evaluation by Opinion Dynamics [↑](#footnote-ref-618)
619. Full Load Cooling Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying the EmPower average Maryland full load hours determined for Maryland (542 from the research referenced below) by the ratio of full load hours in Wilmington, DE (1,015) or Washington, DC (1,320) to Baltimore MD (1,050) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) [↑](#footnote-ref-619)
620. Based on average of 5 utilities in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-620)
621. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-621)
622. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-622)
623. The 10 year average annual heating degree day value is calculated for a number of locations, using a balance point for heating equipment use of 60 degrees (based on data obtained from <http://academic.udayton.edu/kissock/http/Weather/citylistUS.htm>). The 60 degree balance point is used based on a PRISM evaluation of approximately 600,000 Ohio residential single family customers showing this is the point below which heating is generally used. [↑](#footnote-ref-623)
624. Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>) or by performing duct blaster testing. [↑](#footnote-ref-624)
625. The equipment efficiency default is based on data provided by GAMA during the Federal rule-making process for furnace efficiency standards, suggesting that in 2000, 32% of furnaces purchased in Maryland were condensing units. Assuming an efficiency of 92% for the condensing furnaces and 80% for the non-condensing furnaces gives a weighted average of 83.8%. The distribution efficiency default is based on assumption that 50% of duct work is inside the envelope, with some leaks and no insulation. VEIC did not have any more specific data to provide any additional defaults. [↑](#footnote-ref-625)
626. From Illinois TRM, 9 as demonstrated in two years of metering evaluation by Opinion Dynamics [↑](#footnote-ref-626)
627. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-627)
628. Based on REMRate modeling of New Jersey baseline existing home moved to Baltimore climate with electric furnace or air source heat pump HSPF 2.0, SEER 10 AC. Ducts installed in un-conditioned basement. Duct leakage set at RESNET/HERS qualitative default. [↑](#footnote-ref-628)
629. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-629)
630. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-630)
631. Alliance to Save Energy Efficiency Windows Collaborative Report, December 2007. [↑](#footnote-ref-631)
632. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007.

     http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-632)
633. When possible, energy savings should be determined through a custom analysis such as building simulation. If that option is not feasible, savings may be estimated using the algorithms in this section [↑](#footnote-ref-633)
634. 1448 ORNL Builders Foundation Handbook, crawl space data from Table 5-5: Initial Effective R-values for Uninsulated Foundation System and Adjacent Soil, 1991, http://www.ornl.gov/sci/roofs+walls/foundation/ORNL\_CON-295.pdf [↑](#footnote-ref-634)
635. ASHRAE, 2001, “Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP),” Table 7.1 [↑](#footnote-ref-635)
636. Derived by summing the delta between the average outdoor temperature and the base set point of 75 degrees (above which cooling is assumed to be used), each hour of the year. Hourly temperature data obtained from TMY3 data (http://rredc.nrel.gov/solar/) [↑](#footnote-ref-636)
637. This factor's source is: Energy Center of Wisconsin, May 2008 metering study; “Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research”, p31. [↑](#footnote-ref-637)
638. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. [↑](#footnote-ref-638)
639. As determined by Illinois Technical Resource Manual [↑](#footnote-ref-639)
640. The 10 year average annual heating degree day value is calculated for a number of locations, using a balance point for heating equipment use of 60 degrees (based on data obtained from <http://academic.udayton.edu/kissock/http/Weather/citylistUS.htm>). The 60 degree balance point is used based on a PRISM evaluation of approximately 600,000 Ohio residential single family customers showing this is the point below which heating is generally used. [↑](#footnote-ref-640)
641. These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Heat Pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate [↑](#footnote-ref-641)
642. Adapted from Table 1, page 24.4, of the 1977 ASHRAE Fundamentals Handbook [↑](#footnote-ref-642)
643. As determined by the Illinois Technical Resource Manual. [↑](#footnote-ref-643)
644. Full Load Cooling Hours assumptions for Wilmington, DE and Washington, DC calculated by multiplying the EmPower average Maryland full load hours determined for Maryland (542 from the research referenced below) by the ratio of full load hours in Wilmington, DE (1,015) or Washington, DC (1,320) to Baltimore MD (1,050) from the ENERGY STAR calculator. (http://www.energystar.gov/ia/business/bulk\_purchasing/bpsavings\_calc/Calc\_CAC.xls) [↑](#footnote-ref-644)
645. Based on average of 5 utilities in Maryland from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Residential HVAC Program.” April 4, 2014, table 30, page 48. [↑](#footnote-ref-645)
646. gs\_calc/ASHP\_Sav\_Calc.xls) [↑](#footnote-ref-646)
647. , table 30, page 48. [↑](#footnote-ref-647)
648. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the Maryland Peak Definition coincidence factor is 0.69. [↑](#footnote-ref-648)
649. Based on BG&E “Development of Residential Load Profiler for Central Air Conditioners and Heat Pumps” research, the PJM Peak Definition coincidence factor is 0.66. [↑](#footnote-ref-649)
650. Based on simulation model as described in ODC Delaware Technical Resource Manual, April 30, 2012; http://www.dnrec.delaware.gov/energy/information/otherinfo/Documents/EM-and-V-guidance-documents/DELAWARE\_TRM\_August%202012.pdf [↑](#footnote-ref-650)
651. Based on assumption from BG&E billing analysis of furnace program in the '90s, from conversation with Mary Straub; “Evaluation of the High efficiency heating and cooling program, technical report”, June 1995. For other utilities offering this measure, a Heating Degree Day adjustment may be appropriate to this FLHheat assumption. [↑](#footnote-ref-651)
652. Full load heating hours derived by adjusting FLHheat for Baltimore, MD based on Washington, DC HDD base 60°F: 620 \*2957/3457 = 528 hours. [↑](#footnote-ref-652)
653. Ideally, the System Efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The Distribution Efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute: (<http://www.bpi.org/files/pdf/DistributionEfficiencyTable-BlueSheet.pdf>) or by performing duct blaster testing. [↑](#footnote-ref-653)
654. The equipment efficiency default is based on data provided by GAMA during the federal rule-making process for furnace efficiency standards, suggesting that in 2000, 32% of furnaces purchased in Maryland were condensing units. Assuming an efficiency of 92% for the condensing furnaces and 80% for the non-condensing furnaces gives a weighted average of 83.8%. The distribution efficiency default is based on assumption that 50% of duct work is inside the envelope, with some leaks and no insulation. VEIC did not have any more specific data to provide any additional defaults. [↑](#footnote-ref-654)
655. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, 2007. [↑](#footnote-ref-655)
656. Based on INTEGRATION OF DEMAND RESPONSE INTO TITLE 20 FOR RESIDENTIAL POOL PUMPS, SCE Design & Engineering; Phase1: Demand Response Potential DR 09.05.10 Report [↑](#footnote-ref-656)
657. All factors are based on data from INTEGRATION OF DEMAND RESPONSE INTO TITLE 20 FOR RESIDENTIAL POOL PUMPS, SCE Design & Engineering; Phase1: Demand Response Potential DR 09.05.10 Report [↑](#footnote-ref-657)
658. Derived from Pool Pump and Demand Response Potential, DR 07.01 Report, SCE Design and Engineering, Table 16 [↑](#footnote-ref-658)
659. Ibid. [↑](#footnote-ref-659)
660. Based on review of Lockheed Martin pump retail price data, July 2009. [↑](#footnote-ref-660)
661. VEIC estimate. [↑](#footnote-ref-661)
662. Based on INTEGRATION OF DEMAND RESPONSE INTO TITLE 20 FOR RESIDENTIAL POOL PUMPS, SCE Design & Engineering; Phase1: Demand Response Potential DR 09.05.10 Report [↑](#footnote-ref-662)
663. All factors are based on data from INTEGRATION OF DEMAND RESPONSE INTO TITLE 20 FOR RESIDENTIAL POOL PUMPS, SCE Design & Engineering; Phase1: Demand Response Potential DR 09.05.10 Report [↑](#footnote-ref-663)
664. Derived from Pool Pump and Demand Response Potential, DR 07.01 Report, SCE Design and Engineering, Table 16 [↑](#footnote-ref-664)
665. Ibid. [↑](#footnote-ref-665)
666. Based on review of Lockheed Martin pump retail price data, July 2009. [↑](#footnote-ref-666)
667. VEIC estimate. [↑](#footnote-ref-667)
668. NYSERDA 2011, Advanced Power Strip Research Report, <http://www.nyserda.ny.gov/-/media/Files/EERP/Residential/Energy-Efficient-and-ENERGY-STAR-Products/Power-Management-Research-Report.pdf>. Note that estimates are not based on pre/post metering but on analysis based on frequency and consumption of likely products in active, standby and off modes. This measure should be reviewed frequently to ensure that assumptions continue to be appropriate. [↑](#footnote-ref-668)
669. EmPower 2012 Residential Retrofit evaluation [↑](#footnote-ref-669)
670. NYSERDA 2011, Advanced Power Strip Research Report [↑](#footnote-ref-670)
671. EmPower 2012 Residential Retrofit evaluation [↑](#footnote-ref-671)
672. EmPower EY6 QHEC Survey data. [↑](#footnote-ref-672)
673. EmPower 2012 Residential Retrofit evaluation [↑](#footnote-ref-673)
674. Ibid [↑](#footnote-ref-674)
675. NYSERDA 2011, Advanced Power Strip Research Report [↑](#footnote-ref-675)
676. David Rogers, Power Smart Engineering, October 2008: "Smart Strip electrical savings and usability", p22. Assumes that the unit can only take one surge and then needs to be replaced. [↑](#footnote-ref-676)
677. <http://www.energystar.gov/sites/default/files/Final%20Version%203.0%20AV%20Program%20Requirements%20%28Rev%20Dec-2014%29.pdf> [↑](#footnote-ref-677)
678. Energy Savings from this measure are derived from Energy Star estimates. See ‘RPP Product Analysis 9-23-15.xlsx’ [↑](#footnote-ref-678)
679. The baseline unit energy consumption is based on information provided from a Fraunhofer Center for Sustainable Energy System study, titled Energy Consumption of Consumer Electronics in US Households, 2013, available at: <http://www.ce.org/CorporateSite/media/Government-Media/Green/Energy-Consumption-of-CE-in-U-S-Homes-in-2010.pdf>. [↑](#footnote-ref-679)
680. Due to the high market penetration of ENERGY STAR certified soundbars, a weighted average of the unit energy consumption of both non-ENERGY STAR and ENERGY STAR models was calculated in order to accurately provide savings estimates for the market in 2016. [↑](#footnote-ref-680)
681. Wattage difference between base and efficient sound bars when in sleep mode [↑](#footnote-ref-681)
682. Incremental cost comes from Energy Star characterization. See‘RPP Product Analysis 9-23-15.xlsx’ [↑](#footnote-ref-682)
683. ENERGY STAR assumes a 7 year useful life. [↑](#footnote-ref-683)
684. <http://www.energystar.gov/ia/products/appliances/refrig/NAECA_calculation.xls?c827-f746> [↑](#footnote-ref-684)
685. https://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/43 [↑](#footnote-ref-685)
686. http://www.energystar.gov/ia/partners/product\_specs/program\_reqs/Refrigerators\_and\_Freezers\_Program\_Requirements\_V5.0.pdf [↑](#footnote-ref-686)
687. Savings values come from Energy Star Calculations. See ‘RPP Product Analysis 9-23-15.xlsx’ [↑](#footnote-ref-687)
688. The weighted average unit energy savings is calculated using the market share of upright and chest freezers. The assumed market share, as presented in the table above, comes from 2011 NIA-Frz-2008 Shipments data. [↑](#footnote-ref-688)
689. Temperature adjustment factor based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 (p. 47) and assuming 78% of refrigerators are in cooled space (based on BGE Energy Use Survey, Report of Findings, December 2005; Mathew Greenwald & Associates) and 22% in un-cooled space. Although this evaluation is based upon refrigerators only it is considered a reasonable estimate of the impact of cycling on freezers and gave exactly the same result as an alternative methodology based on Freezer eShape data. [↑](#footnote-ref-689)
690. Daily load shape adjustment factor also based on Blasnik, Michael, "Measurement and Verification of Residential Refrigerator Energy Use, Final Report, 2003-2004 Metering Study", July 29, 2004 p. 48, (extrapolated by taking the ratio of existing summer to existing annual profile for hours ending 15 through 18, and multiplying by new annual profile). [↑](#footnote-ref-690)
691. Based on the Freezer TSD Life-Cycle Cost and Payback Analysis found in Table 8.2.7 Standard-Size Freezers: Average Consumer Cost in 2014, available at: <http://www.regulations.gov/contentStreamer?documentId=EERE-2008-BT-STD-0012-0128&disposition=attachment&contentType=pdf> [↑](#footnote-ref-691)
692. ENERGY STAR assumes 11 years based on Appliance Magazine U.S. Appliance Industry: Market Value, Life Expectancy & Replacement Picture for 2005-2012, 2011. [↑](#footnote-ref-692)
693. ENERGY STAR Market & Industry Scoping Report. Residential Clothes Dryers. Table 8. November 2011. <http://www.energystar.gov/ia/products/downloads/ENERGY_STAR_Scoping_Report_Residential_Clothes_Dryers.pdf> [↑](#footnote-ref-693)
694. Baseline energy consumption is based on a modified 2015 Federal Standard (available at: <http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/36>). The goal of the translation is to account for the use of the amended DOE test procedure 10 CFR 430, Subpart B, Appendix D2 which assesses energy efficiency as a result of clothes dryer automatic cycle termination controls. The DOE 2015 standard CEF values are based on the DOE Appendix D1 test. ENERGY STAR is requiring an updated DOE test, published in Appendix D2. On average, clothes dryers use more energy when tested under Appendix D2, and so the translation adjusts the D1 Federal standard to refelec the estimated average energy efficiency preforemance of minimally-compliant 2015 models under D2. The translation values (-16.6% for the electric standard and -13.9% for the gas dryers) are based on DOE testing published in their NOPR test proceduce in January 2013. Performance requirements for ENERGY STAR certified clothes dryers can be found in the ENERGY STAR specifications (V 1.0) (available at: <http://www.energystar.gov/sites/default/files/specs//ENERGY%20STAR%20Final%20Version%201%200%20Clothes%20Dryers%20Program%20Requirements.pdf>). Calculations assume 283 cycles per year and an 8.45 lb load for standard sized dryers (≥ 4.4 cu-ft capacity).

     [↑](#footnote-ref-694)
695. Savings values come from Energy Star Calculations. See ‘RPP Product Analysis 9-23-15.xlsx’ [↑](#footnote-ref-695)
696. Assumes average of 56 minutes per cycle based on Ecova, ‘Dryer Field Study’, Northwest Energy Efficiency Alliance (NEEA) 2014 [↑](#footnote-ref-696)
697. Consistent with coincidence factor of Clothes Washers; Metered data from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program.” March 21, 2014, page 36. [↑](#footnote-ref-697)
698. Savings values come from Energy Star Calculations. See ‘RPP Product Analysis 9-23-15.xlsx’ [↑](#footnote-ref-698)
699. Based on the Dryer TSD Life-Cycle Cost and Payback Analysis “2011-03-18\_TSD\_Chapter\_8\_Life-Cycle\_Cost\_and\_Payback\_Period\_Analysis.pdf” Table 8.2.12 Vented Dryer, Gas and Table 8.2.9 Vented Dryer, Electric, Standard: Consumer Produces Costs, Installation Costs, and Total Installed Costs in 2014, available at: <http://www.regulations.gov/contentStreamer?documentId=EERE-2007-BT-STD-0010-0053&attachmentNumber=9&disposition=attachment&contentType=pdf> [↑](#footnote-ref-699)
700. Based on Appliances Magazine (Appliance Magazine. US Appliance Industry: Market Value, Life Expectancy & Replacement Picture). Please note that this report provides slightly different average life expectancies for gas and electric. To minimize confusion, ENERGY STAR uses 12 years for both product types. [↑](#footnote-ref-700)
701. <http://www.energystar.gov/sites/default/files/specs//private/Room_Air_Cleaners_Final_V1.2_Specification.pdf> [↑](#footnote-ref-701)
702. Baseline and ENERGY STAR energy consumptions are calculated by taking a weighted average of five product category sub types: 51-100 CADR, 101-150 CADR, 151-200 CADR, 201-250 CADR, and >250 CADR. Wattages for all five product sub types are derived from AHAM data. Duty cycle assumes 16 hours per day, 365 days per year based on filter replacement instructions. [↑](#footnote-ref-702)
703. Consistent with ENERGY STAR Qualified Room Air Cleaner Calculator; 16 hours a day, 365 days a year. [↑](#footnote-ref-703)
704. Assumes appliance use is equally likely at any hour of the day or night. [↑](#footnote-ref-704)
705. EPA assumption [↑](#footnote-ref-705)
706. ENERGY STAR assumption based on Lawrence Berkeley National Laboratory 2008 Status Report: Savings Estimates for the ENERGY STAR Voluntary Labeling Program, available at: <http://enduse.lbl.gov/Info/LBNL-56380(2008).pdf> [↑](#footnote-ref-706)
707. Some types of room air cleaners require filter replacement or periodic cleaning, but this is likely to be true for both efficient and baseline units and so no difference in cost is assumed. [↑](#footnote-ref-707)
708. <http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/41> [↑](#footnote-ref-708)
709. <http://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Version%204.0%20Room%20Air%20Conditioners%20Program%20Requirements.pdf> [↑](#footnote-ref-709)
710. Baseline energy consumption is based on the federal standard for room air conditioners, available at: <http://www1.eere.energy.gov/buildings/appliance_standards/product.aspx/productid/41>. The unit energy savings are calculated by taking a market share weighted average of the unit energy consumption of all product subtypes listed in the ENERGY STAR specification. See ‘RPP Product Analysis 9-23-15.xlsx’ [↑](#footnote-ref-710)
711. Calculated by multiplying the ratio of SSP:PJM for the Central AC measure (0.69:0.66) to the assumption for PJM. [↑](#footnote-ref-711)
712. Consistent with coincidence factors found in:

     RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008 (http://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/117\_RLW\_CF%20Res%20RAC.pdf). [↑](#footnote-ref-712)
713. Based on Room Air Conditioner TSD Life-Cycle Cost and Payback Analysis “2011-04-18\_TSD\_Chapter\_8\_Life-Cycle\_Cost\_and\_Payback\_Period\_Analysis.pdf”, available at: <http://www.regulations.gov/#!documentDetail;D=EERE-2007-BT-STD-0010-0053>. To calculated an average incremental cost, a weighted average was created based on the market share of each product subtype. [↑](#footnote-ref-713)
714. Based on Appliances Magazine – Market Research - The U.S. Appliance Industry: Market Value, Life Expectancy & Replacement Picture 2013 (Dec. 2013). [↑](#footnote-ref-714)
715. For text of Energy and Independence and Security Act, see http://www.gpo.gov/fdsys/pkg/PLAW-110publ140/pdf/PLAW-110publ140.pdf [↑](#footnote-ref-715)
716. Base wattage is based upon the post first phase of EISA wattage and wattage bins consistent with ENERGY STAR, v1.1; http://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Lamps%20V1%201\_Specification.pdf. [↑](#footnote-ref-716)
717. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-717)
718. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-718)
719. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zone heat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-719)
720. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-720)
721. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-721)
722. Based on incremental costs for 60W equivalent (dominant bulb) from “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. [↑](#footnote-ref-722)
723. Measure life calculated by building type as “10,000/HOURS” where 10,000 is the median lifetime of General Purpose Replacement, CFL-type ENERGY STAR Certified Light Bulbs (“ENERGY STAR Certified Light Bulbs,” Accessed on April 13, 2015, <http://www.energystar.gov/productfinder/product/certified-light-bulbs/results> [↑](#footnote-ref-723)
724. Based on for 60W EISA equivalent (dominant bulb) from “2010-2012 WA017 Ex Ante Measure Cost Study Draft Report”, Itron, February 28, 2014. [↑](#footnote-ref-724)
725. Itron, Inc. 2014. A Study of Non-Energy Impacts for the State of Maryland REVIEW DRAFT. [↑](#footnote-ref-725)
726. The manufacturers are simply using a regular incandescent lamp with halogen fill gas rather than Halogen Infrared to meet the standard (as provided by G. Arnold, Optimal Energy and confirmed by N. Horowitz at NRDC), so the lifetime of these EISA qualified bulbs is assumed to be 1,000 hours. [↑](#footnote-ref-726)
727. Note, these values have been adjusted by the appropriate In Service Rate (1.00). See ‘MidAtlantic Lighting Adjustments and O&M\_042015.xls’ for more information. The discount rate used for these calculations is 5.0%. [↑](#footnote-ref-727)
728. Consortium for Energy Efficiency. Janurary 2015. CEE Commercial Lighting Initiative Specification for T8 Replacement Lamps. <http://library.cee1.org/sites/default/files/library/12035/CEE\_T8\_Replacement\_Lamp\_Spec\_Jan2015\_Updated03242015.pdf> [↑](#footnote-ref-728)
729. For retrofits replacing T12s prior to July 1, 2017, two baselines are needed to account for the change in baseline technology over the measure lifetime due to federal standards as discussed elsewhere in this measure. [↑](#footnote-ref-729)
730. Code of Federal Regulations, Energy Conservation Program for Consumer Products, title 10, sec. 430.32(m) and (n) (2014). [↑](#footnote-ref-730)
731. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-731)
732. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-732)
733. As discussed in the Measure Life section below, this adjusted measure life represents a simplification calculated as the lifetime measure savings accounting for the shift in baseline after June 2017 divided by the first year savings relative to the T12 baseline. [↑](#footnote-ref-733)
734. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zone heat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-734)
735. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-735)
736. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-736)
737. Efficiency Vermont Technical Reference Manual 2014-85b, May 2014. [↑](#footnote-ref-737)
738. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, <http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>. [↑](#footnote-ref-738)
739. Derivation of the adjusted measure life is presented in Mid-Atlantic\_TRM\_V6\_T12\_Meas\_Life\_Adjustment\_4.18.2016.xlsx. [↑](#footnote-ref-739)
740. 42 U.S.C. §6291(30)(B) (2014) [↑](#footnote-ref-740)
741. Unless otherwise noted, all table values adapted from Efficiency Vermont Technical Reference Manual 2013-82.5, August 2013. [↑](#footnote-ref-741)
742. While the retrofit example assumes a baseline T12 system for calculating the first year annual savings, the baseline component values for the retrofit scenario reflect a standard T8 system because it is assumed that standard T12 components will no longer be sold in 2017 (when T12 lamps installed in 2012 are expected to fail assuming 3,500 annual operating hours and 20,000 lamp life) when relamping/reballasting is necessary due to federal standards. [↑](#footnote-ref-742)
743. Based on lamp life divided by / assumed annual operatingrun hours. [↑](#footnote-ref-743)
744. Assumes baseline lamp with rated life of 20,000 hours operated for 3,500 hours annually. [↑](#footnote-ref-744)
745. Assumes baseline ballast with rated life of 70,000 hours operated for 3,500 hours annually. [↑](#footnote-ref-745)
746. Assumes efficient lamp with rated life of 30,000 hours operated for 3,500 hours annually. [↑](#footnote-ref-746)
747. Assumes efficient ballast with rated life of 70,000 hours operated for 3,500 hours annually. [↑](#footnote-ref-747)
748. Based on lamp life divided by/ assumed annual operating hours. [↑](#footnote-ref-748)
749. Assumes baseline lamp with rated life of 20,000 hours operated for 3,500 hours annually. [↑](#footnote-ref-749)
750. Assumes baseline ballast with rated life of 70,000 hours operated for 3,500 hours annually. [↑](#footnote-ref-750)
751. Assumes efficient lamp with rated life of 30,000 hours operated for 3,500 hours annually. [↑](#footnote-ref-751)
752. Assumes efficient ballast with rated life of 70,000 hours operated for 3,500 hours annually. [↑](#footnote-ref-752)
753. Note, these values have been adjusted by the appropriate In Service Rate (1.0) and assume a 5% discount rate. Additionally, the retrofit example assumes the ballast must be replaced at the time the existing T12 lamps fail. See “MidAtlantic Lighting Adjustments and O&M\_042015.xls” for calculations. [↑](#footnote-ref-753)
754. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-754)
755. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-755)
756. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zoneheat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-756)
757. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-757)
758. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-758)
759. Efficiency Vermont Technical Reference Manual 2009-55, December 2008. [↑](#footnote-ref-759)
760. 'Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-760)
761. Assumes a fluorescent illuminated exit sign. Wattage consistent with ENERGY STAR assumptions. See http://www.energystar.gov/ia/business/small\_business/led\_exitsigns\_techsheet.pdf. [↑](#footnote-ref-761)
762. Assumes operation 24 hours per day, 365 days per year. [↑](#footnote-ref-762)
763. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-763)
764. Efficiency Vermont Technical Reference Manual 2009-55, December 2008. [↑](#footnote-ref-764)
765. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zoneheat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-765)
766. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-766)
767. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-767)
768. Represents the full installed cost of an LED exit sign. LED exit signs can typically be purchased for ~$25 (see http://www.exitlightco.com/Exit\_Signs and “http://www.simplyexitsigns.com”). Assuming replacing exit sign requires 15 minutes of a common building laborer's time in Washington D.C. (RSMeans Electrical Cost Data 2008), the total installed cost would be approximately $35. [↑](#footnote-ref-768)
769. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, <http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>. Measure life in source study is reduced by ~50% assuming existing equipment is at one half of its useful life. [↑](#footnote-ref-769)
770. Represents the full installed cost of a replacement fluorescent lamp. Replacement lamps can typically be purchased for ~$5 (based on a review of online retailers performed 3/14/2013 including “<http://www.exitlightco.com/>” and “http://www.1000bulbs.com/”). Assuming lamp replacement requires 15 minutes of a common building laborer's time in Washington D.C. (RSMeans Electrical Cost Data 2008), the total installed cost would be approximately $12. [↑](#footnote-ref-770)
771. Assumes rated life of fluorescent replacement lamp is 10,000 hours. Assuming annual exit sign operating hours of 8,760, estimated lamp life is 1.14 years. [↑](#footnote-ref-771)
772. Note, these values have been adjusted by the appropriate In Service Rate. [↑](#footnote-ref-772)
773. ENERGY STAR specification can be viewed here: https://www.energystar.gov/sites/default/files/asset/document/Luminaires%20V2%200%20Final.pdf [↑](#footnote-ref-773)
774. Based on ENERGY STAR equivalence table; <http://www.energystar.gov/index.cfm?c=cfls.pr_cfls_lumens> [↑](#footnote-ref-774)
775. Baseline wattage based on common 65 Watt BR30 incandescent bulb (e.g. http://www.destinationlighting.com/storeitem.jhtml?iid=16926) [↑](#footnote-ref-775)
776. The upper bounds for these categories depends on the lower bound of the next higher wattage, which varies by bulb type. [↑](#footnote-ref-776)
777. As above. [↑](#footnote-ref-777)
778. Energy Efficient wattage based on 12 Watt LR6 Downlight from LLF Inc. (<http://site4.marketsmartinteractive.com/products.htm>) Adjusted by ratio of lm/w in ENERGY STAR V2.0 compared to ENERGY STAR V1.2 specification. [↑](#footnote-ref-778)
779. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-779)
780. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-780)
781. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zone heat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-781)
782. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-782)
783. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-783)
784. Based on VEIC product review, April 2015. Baseline bulbs available in $3-$5 range, and SSL bulbs available in $20-$60 range. Incremental cost of $36 therefore assumed ($4 for the baseline bulb and $40 for the SSL). Note, this product is likely to fall rapidly in cost, so this should be reviewed frequently. Product review, November 2012 and March 2014 suggests incremental cost estimate is still appropriate and wide range of costs available. [↑](#footnote-ref-784)
785. The ENERGY STAR specification for solid state recessed downlights requires luminaires to maintain >=70% initial light output for 25,000 hours in an indoor application for separable luminaires and 50,000 for inseparable luminaires. Measure life is therefore assumed to be 14.2 years for downlights featuring inseparable components (calculated as 50,000 hours divided by an approximate 3,500 annual operating hours) and 7.1 years for downlights with replaceable parts (25,000/3,500). [↑](#footnote-ref-785)
786. Itron, Inc. 2014. A Study of Non-Energy Impacts for the State of Maryland REVIEW DRAFT. [↑](#footnote-ref-786)
787. Assumes rated life of BR incandescent bulb of 2,000 hours, based on product review. Lamp life is therefore 2,000/3,500 = 0.57 years. [↑](#footnote-ref-787)
788. Analysis assumes a discount rate of 5%. [↑](#footnote-ref-788)
789. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-789)
790. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zoneheat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-790)
791. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-791)
792. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-792)
793. Assumes delamping a single fixture requires 15 minutes of a common building laborer's time in Washington D.C.; Adapted from RSMeans Electrical Cost Data 2008. [↑](#footnote-ref-793)
794. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-794)
795. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-795)
796. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-796)
797. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-797)
798. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-798)
799. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zoneheat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-799)
800. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-800)
801. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-801)
802. Northeast Energy Efficiency Partnerships Incremental Cost Study Report, Navigant, 2011. Sensors costs assume the simple average of cost for those sensors using only passive infrared technology and those using both passive infrared and ultrasonic technology. [↑](#footnote-ref-802)
803. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-803)
804. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-804)
805. Williams, A., B. Atkinson, K. Garesi, E. Page, and F. Rubinstein. 2012. “Lighting Controls in Commercial Buildings.” The Journal of the Illuminating Engineering Society of North America 8 (3): 161-180. [↑](#footnote-ref-805)
806. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-806)
807. As a conservative assumption, the peak demand savings algorithm assumes the same annual savings factor (SVG) as the energy savings equation. It is probable that higher than average availability of daylight coincides with summer peak periods. This factor is a candidate for future study as increased accuracy will likely lead to increased peak demand savings estimates. [↑](#footnote-ref-807)
808. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zoneheat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-808)
809. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-809)
810. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-810)
811. Northeast Energy Efficiency Partnerships Incremental Cost Study Report, Navigant, 2011. Assumes the simple average of cost of all photosensors types. Source does not differentiate costs between fixture and remote-mounted sensors. [↑](#footnote-ref-811)
812. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-812)
813. Energy code lighting power density requirements can generally be satisfied by using one of two methods. The Building Area Method simply applies a blanket LPD requirement to the entire building based on the building type. Broadly speaking, as long as the total connected lighting wattage divided by the total floor space does not exceed the LPD requirement, the code is satisfied. The second method, the Space-by-Space Method, provides LPD requirements by space type based on the function of the particular space (e.g., “Hospital – Operating Room”, “Library – Reading Room”). LPD requirements must be satisfied for each individual space in the building. This method usually allows a higher total connected wattage as compared to the Building Area Method. [↑](#footnote-ref-813)
814. If the Space-by-Space Method is used, the total energy savings will be the sum of the energy savings for each individual space type. [↑](#footnote-ref-814)
815. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-815)
816. IECC 2015, Table C405.4.2(1); IECC 2012, Table C405.5.2(1). Note that the Delaware energy code may also be satisfied by meeting the requirements of ASHRAE 90.1-2010, Table 9.5.1. As the IECC 2012 requirements are less stringent they are presented here. [↑](#footnote-ref-816)
817. IECC 2012, Table C405.5.2(2). Note that the Delaware energy code may also be satisfied by meeting the requirements of ASHRAE 90.1-2010, Table 9.5.1. As the IECC 2012 requirements are less stringent they are presented here. [↑](#footnote-ref-817)
818. IECC 2015, Table C405.4.2(2). [↑](#footnote-ref-818)
819. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zone heat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-819)
820. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-820)
821. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-821)
822. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, <http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf>. Assumes Advanced Lighting Design lifetime will be consistent with that of the “Fluorescent Fixture” measure from the reference document. This measure life assumes that the most common implementation of this measure will be for new construction or major renovation scenarios where new fixtures are installed. In such cases, adopting the fixture lifetime for the LPD reduction measure seems most appropriate. [↑](#footnote-ref-822)
823. DesignLights Consortium Qualified Products List <http://www.designlights.org/solidstate.about.QualifiedProductsList\_Publicv2.php> [↑](#footnote-ref-823)
824. Baseline and efficient fixtures have been grouped into wattage categories based on typical applications. The typical baseline equipment in each group was weighted based on personal communication with Kyle Hemmi, CLEAResult on Sept. 18. 2012. Weighting reflects implementation program data from Texas, Nevada, Rocky Mountain, and Southwest Regions. When adequate program data is collected from the implementation of this measure in the Mid-Atlantic region, these weightings should be updated accordingly. Baseline fixture wattage assumptions developed from multiple TRMs including: Arkansas TRM Version 2.0, Volume 2: Deemed Savings, Frontier Associates, LLC, 2012; Massachusetts Technical Reference Manual for Estimating Savings from Energy Efficiency Measures, 2012 Program Year – Plan Version, Massachusetts Electric and Gas Energy

     Efficiency Program Administrators, 2011, and 2012 Statewide Customized Offering Procedures Manual for Business - Appendix B Table of Standard Fixture Wattages and Sample Lighting Table, Southern California Edison et al., 2012. As the total wattage assumptions for like fixtures typically do not vary by more than a few watts between sources, the values from the Arkansas document have been adopted here. Efficient fixture wattage estimated assuming mean delivered lumen equivalence between the baseline and efficient case. Baseline initial lamp lumen output was reduced by estimates of lamp lumen depreciation and optical efficiency. Efficient wattage and lumen information was collected from appropriate product categories listed in the DesignLights Consortium Qualified Products List – Updated 11/21/2012. Analysis presented in the “Mid Atlantic C&I LED Lighting Analysis.xlsx” supporting workbook. [↑](#footnote-ref-824)
825. Efficiency Vermont Technical Reference Manual 2009-55, December 2008; based on 5 years of metering on 235 outdoor circuits in New Jersey. [↑](#footnote-ref-825)
826. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-826)
827. It is assumed that efficient outdoor area lighting, when functioning properly, will never result in coincident peak demand savings. [↑](#footnote-ref-827)
828. Efficiency Maine Technical Reference User Manual No.2010-1, 2010. [↑](#footnote-ref-828)
829. The median rated lifetime for applicable products on the DesignLights Consortium Qualified Products List – Updated 4/3/2015 <https://www.designlights.org/resources/file/NEEPDLCQPL> is 50,000 hours for both luminaires and retrofit kits. Assuming average annual operating hours of 3,338 (Efficiency Vermont TRM User Manual No. 2014-85b; based on 5 years of metering on 235 outdoor circuits in New Jersey), the estimated measure life is 15 years. [↑](#footnote-ref-829)
830. Component information for the <175W HID and 176-250W HID categories adopted from Efficiency Vermont TRM User Manual No. 2012-77a. The remaining categories are based on a review of pricing for available products from http://1000bulbs.com. Accessed on 11/22/2012. NPV O&M Savings calculated assuming a 5% discount rate; detailed calculation presented in the “Mid Atlantic C&I LED Lighting Analysis.xlsx” workbook. [↑](#footnote-ref-830)
831. DesignLights Consortium Qualified Products List <http://www.designlights.org/QPL> [↑](#footnote-ref-831)
832. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-832)
833. Because of the comparatively high cost of LED equipment, it is likely that the ISR will be near 1.0. Additionally, it may be inappropriate to assume the “Equipment” category ISR from the EmPOWER Maryland DRAFT 2010 Interim Evaluation Report, Chapter 2: Commercial and Industrial Prescriptive, Navigant Consulting, 2010. [↑](#footnote-ref-833)
834. Wattage of illustrative LED luminaire developed by averaging the wattage for all DesignLights Consortium qualified high-bay products from the DesignLights Consortium Qualified Products List <http://www.designlights.org/QPL> delivering between 90% and 100% of the baseline mean system lumens. [↑](#footnote-ref-834)
835. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zone heat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-835)
836. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-836)
837. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-837)
838. Efficiency Vermont Technical Reference User Manual No. 2013-82.5, August 2013. [↑](#footnote-ref-838)
839. The median rated lifetime for applicable products on the DesignLights Consortium Qualified Products List – Updated 4/3/2015 <https://www.designlights.org/resources/file/NEEPDLCQPL> is 50,000 hours for both luminaires and retrofit kits. Assuming average annual operating hours of 4,116 for a typical warehouse lighting application, the estimated measure life is 12 years. [↑](#footnote-ref-839)
840. DesignLights Consortium Qualified Products List <http://www.designlights.org/QPL> [↑](#footnote-ref-840)
841. The lighting hours of use tables in Appendix D are primarily based on fluorescent lamp operating hours. It is assumed that, for general ambient lighting applications, LED operating hours will be similar to fluorescent operating hour; however, LED operating hours are a potential candidate for future study. [↑](#footnote-ref-841)
842. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-842)
843. Because of the comparatively high cost of LED equipment, it is likely that the ISR will be near 1.0. Additionally, it may be inappropriate to assume the “Equipment” category ISR from the EmPOWER Maryland DRAFT 2010 Interim Evaluation Report, Chapter 2: Commercial and Industrial Prescriptive, Navigant Consulting, 2010. [↑](#footnote-ref-843)
844. Wattage of illustrative LED luminaire developed by averaging the wattage for all DesignLights Consortium qualified high-bay products from the DesignLights Consortium Qualified Products List <http://www.designlights.org/QPL> delivering between 80% and 100% of the baseline mean system lumens. [↑](#footnote-ref-844)
845. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zone heat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-845)
846. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-846)
847. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-847)
848. Efficiency Vermont Technical Reference User Manual No. 2013-82.5, August 2013. [↑](#footnote-ref-848)
849. The median rated lifetime for applicable products on the DesignLights Consortium Qualified Products List – Updated 4/3/2015 <https://www.designlights.org/resources/file/NEEPDLCQPL> is 50,000 hours for both luminaires and retrofit kits. Assuming average annual operating hours of 3,500 for a typical commercial lighting application, the estimated measure life is 14 years. [↑](#footnote-ref-849)
850. DesignLights Consortium Qualified Products List <http://www.designlights.org/solidstate.about.QualifiedProductsList\_Publicv2.php> [↑](#footnote-ref-850)
851. Baseline and efficient fixtures have been grouped into wattage categories based on typical applications. The typical baseline equipment in each group were weightings based on personal communication with Kyle Hemmi, CLEAResult on Sept. 18. 2012. Weighting reflects implementation program data from Texas, Nevada, Rocky Mountain, and Southwest Regions. When adequate program data is collected from the implementation of this measure in the Mid-Atlantic region, these weightings should be updated accordingly. Baseline fixture wattage assumptions developed from multiple TRMs including: Arkansas TRM Version 2.0, Volume 2: Deemed Savings, Frontier Associates, LLC, 2012; Massachusetts Technical Reference Manual for Estimating Savings from Energy Efficiency Measures, 2012 Program Year – Plan Version, Massachusetts Electric and Gas Energy

     Efficiency Program Administrators, 2011, and 2012 Statewide Customized Offering Procedures Manual for Business - Appendix B Table of Standard Fixture Wattages and Sample Lighting Table, Southern California Edison et al., 2012. As the total wattage assumptions for like fixture typically do not vary by more than a few watts between sources, the values from the Arkansas document have been adopted here. Efficient fixture wattage estimated assuming mean delivered lumen equivalence between the baseline and efficient case. Baseline initial lamp lumen output was reduced by estimates of lamp lumen depreciation and optical efficiency. Efficient wattage and lumen information was collected from appropriate product categories listed in the DesignLights Consortium Qualified Products List – Updated 11/21/2012. Analysis presented in the “Mid Atlantic C&I LED Lighting Analysis.xlsx” supporting workbook. [↑](#footnote-ref-851)
852. Efficiency Vermont Technical Reference Manual 2009-55, December 2008; based on 5 years of metering on 235 outdoor circuits in New Jersey. Parking garages typically require artificial illumination 24 hours per day. [↑](#footnote-ref-852)
853. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-853)
854. Because of the comparatively high cost of LED equipment, it is likely that the ISR will be near 1.0. Additionally, it may be inappropriate to assume the “Equipment” category ISR from the EmPOWER Maryland DRAFT 2010 Interim Evaluation Report, Chapter 2: Commercial and Industrial Prescriptive, Navigant Consulting, 2010. [↑](#footnote-ref-854)
855. It is assumed that efficient canopy lighting, when functioning properly, will never result in coincident peak demand savings. Parking garages typically require artificial illumination 24 hours per day and will therefore exhibit 100% peak coincidence. [↑](#footnote-ref-855)
856. Efficiency Maine Technical Reference User Manual No.2010-1, 2010. [↑](#footnote-ref-856)
857. The average rated lifetime for applicable products on the DesignLights Consortium Qualified Products List – Updated 3/13/2015 <http://www.designlights.org/solidstate.about.QualifiedProductsList_Publicv2.php> is 79,863 for parking garage luminaires (62,500 for retrofit kits) and 69,844 for canopy luminaires (80,000 for retrofit kits). For the purposes of this characterization, it is assumed the typical equipment will operate for 70,000 hours. Assuming average annual operating hours of 3,338 for canopy applications (Efficiency Vermont Technical Reference Manual 2009-55, December 2008; based on 5 years of metering on 235 outdoor circuits in New Jersey), the estimated measure life is 21 years. Assuming average annual operating hours of 8,760 for parking garage applications, the estimated measure life is 8 years. [↑](#footnote-ref-857)
858. Component information for the <175W HID and 176-250W HID categories adopted from Efficiency Vermont TRM User Manual No. 2012-77a. The remaining category is based on a review of pricing for available products from http://1000bulbs.com. Accessed on 11/22/2012. NPV O&M Savings calculated assuming a 5% discount rate; detailed calculation presented in the “Mid Atlantic C&I LED Lighting Analysis.xlsx” workbook. [↑](#footnote-ref-858)
859. For text of Energy and Independence and Security Act, see http://www.gpo.gov/fdsys/pkg/PLAW-110publ140/pdf/PLAW-110publ140.pdf [↑](#footnote-ref-859)
860. Based on ENERGY STAR Version 1.1 ENERGY STAR Product Specification for Lamps equivalence table; http://1.usa.gov/1RjFnX4 [↑](#footnote-ref-860)
861. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-861)
862. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-862)
863. The upper bounds for these categories depends on the lower bound of the next higher wattage, which varies by bulb type. [↑](#footnote-ref-863)
864. As above. [↑](#footnote-ref-864)
865. See ‘ESTAR Integrated Screw SSL Lamp\_032014.xls’ for details. The Minimum Lamp Efficacy Requirements in ENERGY STAR Product Specification for Lamps (Light Bulbs) V2.0 vary by Color Rendering Index (CRI). [↑](#footnote-ref-865)
866. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zone heat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-866)
867. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-867)
868. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-868)
869. Omnidirectional and directional costs based on NEEP 2014-2015 Residential Lighting Strategy Update. Decorative Costs under 15W based on typical costs on 1000bulbs.com. Higher wattage decorative based on VEIC study of units rebated through Efficiency Vermont Retail program. There is currently not sufficient data available to differentiate the incremental costs between ENERGY STAR V1.1 and V2.0 lamps. [↑](#footnote-ref-869)
870. The v1.1 ENERGY STAR Product Specification for Lamps (Light Bulbs) requires rated life of 25,000 hours for solid-state omnidirectional and directional lamps, and 15,000 hours for solid-state decorative lamps. Measure lifetimes assume 3,500 average annual operating hours. [↑](#footnote-ref-870)
871. The v2.0 ENERGY STAR Product Specification for Lamps (Light Bulbs) requires rated life of 15,000 hours for solid-state omnidirectional and decorative lamps, and 25,000 hours for solid-state directional lamps. Measure lifetimes assume 3,500 average annual operating hours. [↑](#footnote-ref-871)
872. [↑](#footnote-ref-872)
873. Assumes incandescent baseline lamp life of 1000 hours. [↑](#footnote-ref-873)
874. Pacific Gas & Electric. May 2007. LED Refrigeration Case Lighting Workpaper 053007 rev1. Values normalized on a per linear foot basis. [↑](#footnote-ref-874)
875. Pacific Gas & Electric. May 2007. LED Refrigeration Case Lighting Workpaper 053007 rev1. Values normalized on a per linear foot basis. [↑](#footnote-ref-875)
876. Theobald, M. A., Emerging Technologies Program: Application Assessment Report #0608, LED Supermarket Case Lighting Grocery Store, Northern California, Pacific Gas and Electric Company, January 2006. Assumes refrigerated case lighting typically operates 17 hours per day, 365 days per year. [↑](#footnote-ref-876)
877. New York Department of Public Service. 2014. The New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs – Residential, Multi-family, and Commercial/Industrial Measures Version 2. [↑](#footnote-ref-877)
878. New York Department of Public Service. 2014. The New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs – Residential, Multi-family, and Commercial/Industrial Measures Version 2. [↑](#footnote-ref-878)
879. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. [↑](#footnote-ref-879)
880. Navigant. May 2014. Incremental Cost Study Phase Three Final Report. Prepared for NEEP Regional Evaluation, Measurement & Verification Forum [↑](#footnote-ref-880)
881. The median rated lifetime for applicable products on the DesignLights Consortium Qualified Products List – Updated 4/3/2015 <https://www.designlights.org/resources/file/NEEPDLCQPL> is 50,000 hours. Assuming average annual operating hours of 6,205, the estimated measure life is 8 years. [↑](#footnote-ref-881)
882. DesignLights Consortium Qualified Products List <https://www.designlights.org/qpl> [↑](#footnote-ref-882)
883. Efficiency Vermont TRM User Manual No. 2014-85b; baseline are based on analysis of actual Efficiency Vermont installations of LED lighting. Exterior LED flood and spot luminaires are an evolving technology that may replace any number of baseline lamp and fixture types. It is recommended that programs track existing and new lamps and/or luminaire types, wattages, and lumen output in such way that baseline assumptions can be refined for future use. [↑](#footnote-ref-883)
884. Source does not specify an upper lumen range for LED luminaires. Based on a review of manufacturer product catalogs, 15,000 lumens is the approximate initial lumen output of a 175W MH lamp. [↑](#footnote-ref-884)
885. Efficiency Vermont TRM User Manual No. 2014-85b; based on 5 years of metering on 235 outdoor circuits in New Jersey. [↑](#footnote-ref-885)
886. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-886)
887. It is assumed that efficient outdoor area lighting, when functioning properly, will never result in coincident peak demand savings. [↑](#footnote-ref-887)
888. Efficiency Vermont TRM User Manual No. 2014-85b. [↑](#footnote-ref-888)
889. The median rated lifetime for applicable products on the DesignLights Consortium Qualified Products List – Updated 4/3/2015 <https://www.designlights.org/resources/file/NEEPDLCQPL> is 50,000 hours for Architectural Flood and Spot Luminaires and 100,000 hours for Landscape/Accent Flood and Spot Luminaires. Assuming average annual operating hours of 3,338 (Efficiency Vermont TRM User Manual No. 2014-85b; based on 5 years of metering on 235 outdoor circuits in New Jersey), the estimated measure life is 15 years for Architectural Flood and Spot Luminaires and 30 years for Landscape/Accent Flood and Spot Luminaires. By convention, measure life of C&I LED lighting is capped at 15 years. [↑](#footnote-ref-889)
890. Exterior LED flood and spot luminaires are an evolving technology that may replace any number of baseline lamp and fixture types. It is recommended that programs track existing and new lamps and/or luminaire types, wattages, lumen output, and costs in such way that generalized prescriptive O&M values can be developed for future use. [↑](#footnote-ref-890)
891. Underwriters Laboratories (UL) Standard 1598 [↑](#footnote-ref-891)
892. DesignLights Consortium Qualified Products List <http://www.designlights.org/QPL> [↑](#footnote-ref-892)
893. California Technical Forum. February 2015. T8 LED Tube Lamp Replacement Abstract Revision # 0; Note that the “Delta Watts” values, presented on a per lamp basis, implicitly, and conservatively, assume no savings for reduced or eliminated ballast energy consumption. [↑](#footnote-ref-893)
894. The lighting hours of use tables in Appendix D are primarily based on fluorescent lamp operating hours. It is assumed that, for general ambient lighting applications, LED operating hours will be similar to fluorescent operating hours; however, LED operating hours are a potential candidate for future study. [↑](#footnote-ref-894)
895. Site-specific annual operating hours should be collected following best-practice data collection techniques as appropriate. In most cases, it should not be assumed that the lighting hours of operation are identical to the reported operating hours for the business. Any use of site-specific annual operating hours information will be subject to regulatory approval and potential measurement and verification adjustment. [↑](#footnote-ref-895)
896. Because of LED linear replacement lamps have not been specifically evaluated in the Mid-Atlantic region an initial ISR of 1.0 is assumed. However, costs of these products continue to drop rapidly increasing the probability that participants may purchase additional stock to be installed at a later date. This factor should be considered for future evaluation work. [↑](#footnote-ref-896)
897. HVAC-Lighting interaction impacts adapted from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions. Typical aspect ratio for perimeter zones. Heating factor applies to perimeter zone heat, therefore it must be adjusted to account for lighting in core zones. [↑](#footnote-ref-897)
898. Fraction of lighting heat that contributes to space heating. Based on 0.23 factor for Washington DC (from 1993 ASHRAE Journal: Calculating Lighting and HVAC Interactions). [↑](#footnote-ref-898)
899. Typical heating system efficiency of 75%, consistent with current federal standards for fossil fuel-fired systems. [↑](#footnote-ref-899)
900. Based on a review of incremental cost estimates from California Technical Forum. February 2015. T8 LED Tube Lamp Replacement Abstract Revision # 0, Efficiency Vermont TRM User Manual No. 2014-85b, and online wholesalers. As this measure is a retrofit-type, incremental costs assume the full cost of replacement of the lamps and (removal of) the ballast(s). [↑](#footnote-ref-900)
901. The median rated lifetime for applicable products on the DesignLights Consortium Qualified Products List – Updated 4/3/2015 <https://www.designlights.org/resources/file/NEEPDLCQPL> is 50,000 hours. Assuming average annual operating hours of 3,500 for a typical commercial lighting application, the estimated measure life is 14 years. [↑](#footnote-ref-901)
902. Fluorescent LED replacement lamps luminaires are an evolving technology that may replace any number of baseline lamp types. It is recommended that programs track existing and new lamps types, wattages, lumen output, and costs in such way that generalized prescriptive O&M values can be developed for future use. [↑](#footnote-ref-902)
903. Commercial energy code baseline requirements for Washington, D.C. and Delaware are currently consistent with IECC 2012 (Delaware currently uses ASHRAE 90.1-2010, but the HVAC system requirements are consistent with IECC 2012), whereas Maryland’s baseline requirements are consistent with IECC 2015. [↑](#footnote-ref-903)
904. Heating mode efficiencies for heat pumps >=65,000 Btu/h are provided at the 47°F db/43° wb outdoor air rating condition. [↑](#footnote-ref-904)
905. Code of Federal Regulations, Energy Efficiency Program for Certain Commercial and Industrial Equipment, title 10, sec. 431.96 (2016). [↑](#footnote-ref-905)
906. Replacement unit shall be factory labeled as follows: “MANUFACTURED FOR REPLACEMENT APPLICATIONS ONLY: NOT TO BE INSTALLED IN NEW CONSTRUCTION PROJECTS.” Replacement efficiencies apply only to units with existing sleeves less than 16 inches (406 mm) in height and less than 42 inches (1067 mm) in width. [↑](#footnote-ref-906)
907. “Cap” = The rated cooling capacity of the project in Btu/h. If the unit’s capacity is less than 7,000 Btu/h, use 7,000 Btu/h in the calculation. If the unit’s capacity is greater than 15,000 Btu/h, use 15,000 Btu/h in the calculations. [↑](#footnote-ref-907)
908. Federal standard as presented for this equipment type is effective January 1, 2017. This standard is consistent with IECC 2015 and ASHRAE 90.1-2013 requirements and is recommended as a consistent regional baseline. [↑](#footnote-ref-908)
909. Replacement unit shall be factory labeled as follows: “MANUFACTURED FOR REPLACEMENT APPLICATIONS ONLY: NOT TO BE INSTALLED IN NEW CONSTRUCTION PROJECTS.” Replacement efficiencies apply only to units with existing sleeves less than 16 inches (406 mm) in height and less than 42 inches (1067 mm) in width. [↑](#footnote-ref-909)
910. “Cap” = The rated cooling capacity of the project in Btu/h. If the unit’s capacity is less than 7,000 Btu/h, use 7,000 Btu/h in the calculation. If the unit’s capacity is greater than 15,000 Btu/h, use 15,000 Btu/h in the calculations. [↑](#footnote-ref-910)
911. The two equations are provided to show how savings are determined during the initial phase of the measure (i.e., efficient unit relative to existing equipment) and the remaining phase (i.e., efficient unit relative to new baseline unit). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new baseline to efficient savings)/(existing to efficient savings). The remaining measure life should be determined on a site-specific basis. [↑](#footnote-ref-911)
912. The two equations are provided to show how savings are determined during the initial phase of the measure (i.e., efficient unit relative to existing equipment) and the remaining phase (i.e., efficient unit relative to new baseline unit). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new baseline to efficient savings)/(existing to efficient savings). The remaining measure life should be determined on a site-specific basis. [↑](#footnote-ref-912)
913. From U.S. DOE. 2013. *The Uniform Methods Project: Methods for Determining Energy Efficiency Savings for Specific Measures*: “Although the EFLH is calculated with reference to a peak kW derived from EER, it is acceptable to use these EFLH with SEER or IEER. Some inconsistency occurs in using full-load hours with efficiency ratings measured at part loading, but errors in calculation are thought to be small relative to the expense and complexity of developing hours-of-use estimates precisely consistent with SEER and IEER.” [↑](#footnote-ref-913)
914. Full load cooling hours estimated by adjusting the “Mid-Atlantic” hours from “C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011” by the full-load cooling hour estimates from the ENERGY STAR Central AC Calculator, 2013. For scaling purposes, the analysis assumes the initial Mid-Atlantic values are consistent with Baltimore, MD as suggested by the KEMA study. Because the ENERGY STAR calculator does not provide full load hours estimates for all cities of interest, a second scaling was performed using cooling degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-914)
915. HOURS estimates developed from data presented in "New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs", TecMarket Works, October 15, 2010, adjusted to Mid-Atlantic region using heating degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-915)
916. C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. Final values are presented in Metoyer, Jarred, “Report Revision Memo,” KEMA, August 2011 [↑](#footnote-ref-916)
917. C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. Final values are presented in Metoyer, Jarred, “Report Revision Memo,” KEMA, August 2011 [↑](#footnote-ref-917)
918. Assumes baseline unit with 13 SEER converted to EER using the following estimate: EER = SEER/1.2 [↑](#footnote-ref-918)
919. Navigant. May 2014. Incremental Cost Study Phase Three Final Report. Prepared for NEEP Regional Evaluation, Measurement & Verification Forum. In all cases, incremental costs are presented relative to the baseline efficiencies presented in the Baseline Efficiencies by System Type and Unit Capacity table for the relevant size categories. [↑](#footnote-ref-919)
920. Incremental costs for water- and evaporatively-cooled ACs, PTACs, and PTHPs will be addressed in subsequent versions of the TRM. In the interim, incremental costs for these equipment types should be determined on a site-specific basis. [↑](#footnote-ref-920)
921. CEE efficiency tiers as presented in Consortium for Energy Efficiency. 2016. CEE Commercial Unitary Air-Conditioning and Heat Pumps Specification, Effective January 12, 2016. [↑](#footnote-ref-921)
922. Ibid. In the absence of better data, incremental costs for air-source heat pumps >=65,000 Btu/h are assumed equal to the air-cooled unitary AC equipment for the corresponding size categories. Therefore, the CEE tiers presented here reflect cooling mode requirements for AC equipment. [↑](#footnote-ref-922)
923. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, http://www.ctsavesenergy.org/files/Measure%20Life%20Report%202007.pdf [↑](#footnote-ref-923)
924. To enable improvements to this measure characterization in the future, the existing equipment types should be tracked by the program to ensure that this measure characterizes the appropriate baseline conditions. [↑](#footnote-ref-924)
925. Federal standards. http://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/41 [↑](#footnote-ref-925)
926. Federal Standard as of January 1, 2015. http://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/75 [↑](#footnote-ref-926)
927. Ibid [↑](#footnote-ref-927)
928. Ibid [↑](#footnote-ref-928)
929. Federal Standards for gas boilers [↑](#footnote-ref-929)
930. Federal standards for air source heat pumps [↑](#footnote-ref-930)
931. Electric heat has a COP of 1.0. Converted into HSPF units this is approximately 3.41. [↑](#footnote-ref-931)
932. This will be negative if the baseline has non-electric heat. This is because some electricity from the DMSHP is now assumed to be used for space heating. There us a corresponding savings in fossil fuel heat. [↑](#footnote-ref-932)
933. Federal standard for typical window AC sizes with louvered sides. [↑](#footnote-ref-933)
934. If unknown, assume the baseline is a gas furnace, with no electrical savings [↑](#footnote-ref-934)
935. Full load cooling hours estimated by adjusting the “Mid-Atlantic” hours from “C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011” by the full-load cooling hour estimates from the ENERGY STAR Central AC Calculator, 2013. For scaling purposes, the analysis assumes the initial Mid-Atlantic values are consistent with Baltimore, MD as suggested by the KEMA study. Because the ENERGY STAR calculator does not provide full load hours estimates for all cities of interest, a second scaling was performed using cooling degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-935)
936. HOURS estimates developed from data presented in "New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs", TecMarket Works, October 15, 2010, adjusted to Mid-Atlantic region using heating degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory [↑](#footnote-ref-936)
937. Federal standard for typical window AC sizes with louvered sides. [↑](#footnote-ref-937)
938. C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. Final values are presented in Metoyer, Jarred, “Report Revision Memo,” KEMA, August 2011 [↑](#footnote-ref-938)
939. C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. Final values are presented in Metoyer, Jarred, “Report Revision Memo,” KEMA, August 2011 [↑](#footnote-ref-939)
940. Federal standard for gas boilers. [↑](#footnote-ref-940)
941. Navigant, Inc. Incremental Cost Study Phase 2. January 16, 2013. Table 16. [↑](#footnote-ref-941)
942. Energy Star Calculator. <http://www.energystar.gov/ia/business/bulk_purchasing/bpsavings_calc/CalculatorConsumerRoomAC.xls>) [↑](#footnote-ref-942)
943. Energy Star Calculator. 46% added to value to reflect labor, based on ratio of equipment to labor cost for measure EffFurn-cond-90AFUE in DEER database. <http://www.energystar.gov/buildings/sites/default/uploads/files/Furnace_Calculator.xls?8178-e52c> [↑](#footnote-ref-943)
944. If existing case is electric resistance heat, assume project replaces existing functional baseboard. [↑](#footnote-ref-944)
945. GDS Associates, Inc. (2007). *Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures*. Prepared for The New England State Program Working Group; Page 1-3, Table 1. [↑](#footnote-ref-945)
946. Unless otherwise noted, savings characterization and associated parameters adopted from Del Balso, R., and K. Monsef. “Investigation into the Reliability of Energy Efficiency/Demand Side Management Savings Estimates for Variable Frequency Drives in Commercial Applications”, University of Colorado, Department of Civil, Environmental and Architectural Engineering, 2013. [↑](#footnote-ref-946)
947. Del Balso, R., and K. Monsef, 2013 notes that the default HVAC interactive effects factor presented in the paper, 15.7%, “should not be used for actual program implementation, but such a factor should be developed and used based on a more complete set of energy modeling results for a given jurisdiction.” A value of zero should be assumed, essentially omitting interactive effects, until a jurisdiction-specific analysis can be performed. [↑](#footnote-ref-947)
948. UI and CL&P Program Saving Documentation for 2009 Program Year, Table 1.1.1; HVAC - Variable Frequency Drives – Pumps. [↑](#footnote-ref-948)
949. United Illuminating Company and Connecticut Light & Power Company. 2012. Connecticut Program Savings Document – 8th Edition for 2013 Program Year. Orange, CT. [↑](#footnote-ref-949)
950. United Illuminating Company and Connecticut Light & Power Company. 2012. Connecticut Program Savings Document – 8th Edition for 2013 Program Year. Orange, CT; energy and demand savings constants were derived using a temperature bin spreadsheet and typical heating, cooling, and fan load profiles. Note, these values have been adjusted from the source data for remove the embedded load factor. [↑](#footnote-ref-950)
951. Navigant. 2013. Incremental Cost Study Phase Two Final Report. Burlington, MA. [↑](#footnote-ref-951)
952. The Incremental Cost Study does not provide labor cost estimates for units 100 hp and above. Labor cost estimates derived from RSMeans Mechanical Cost Data 2010. US average labor costs for 100 hp and 200 hp units adjusted to the Mid-Atlantic region using population weighted (2010 Census) “Location Factors” from RSMeans. [↑](#footnote-ref-952)
953. Ibid. [↑](#footnote-ref-953)
954. Navigant. 2013. Incremental Cost Study Phase Two Final Report. Burlington, MA. [↑](#footnote-ref-954)
955. The two equations are provided to show how savings are determined during the initial phase of the measure (i.e., efficient unit relative to existing equipment) and the remaining phase (i.e., efficient unit relative to new baseline unit). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new baseline to efficient savings)/(existing to efficient savings). The remaining measure life should be determined on a site-specific basis. [↑](#footnote-ref-955)
956. Integrated Part Load Value (IPLV) is an HVAC industry standard single-number metric for reporting part-load performance. [↑](#footnote-ref-956)
957. Baseline efficiencies based on International Energy Conservation Code 2012, Table C403.2.3(7) Minimum Efficiency Requirements: Water Chilling Packages and International Energy Conservation Code 2015, Table C403.2.3(7) Water Chilling Packages - Efficiency Requirements [↑](#footnote-ref-957)
958. Baseline efficiencies based on International Energy Conservation Code 2012, Table C403.2.3(7) Minimum Efficiency Requirements: Water Chilling Packages and International Energy Conservation Code 2015, Table C403.2.3(7) Water Chilling Packages - Efficiency Requirements [↑](#footnote-ref-958)
959. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. Combined with full load hour assumptions used for efficiency measures to account for diversity of equipment usage within the peak period hours. [↑](#footnote-ref-959)
960. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. [↑](#footnote-ref-960)
961. Navigant. 2013. Incremental Cost Study Phase Two Final Report. Burlington, MA. Table values adapted from published values to align with baseline code requirements (“Path A”) by interpolating or extrapolating from nearest pair of published efficiency values. “N/A” indicates either an efficiency value below baseline requirements or a gap in the published data from the source document. [↑](#footnote-ref-961)
962. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, "http://www.ctsavesenergy.org/files/Measure Life Report 2007.pdf" [↑](#footnote-ref-962)
963. Baseline efficiencies based on International Energy Conservation Code 2012, Table C403.2.3(7) Minimum Efficiency Requirements: Water Chilling Packages. [↑](#footnote-ref-963)
964. Baseline efficiencies based on International Energy Conservation Code 2015, Table C403.2.3(7) Water Chilling Package - Efficiency Requirements. [↑](#footnote-ref-964)
965. HOURS estimates developed from data presented in "New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs", TecMarket Works, October 15, 2010, adjusted to Mid-Atlantic region using cooling degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-965)
966. HOURS estimates developed from data presented in "New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs", TecMarket Works, October 15, 2010, adjusted to Mid-Atlantic region using heating degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-966)
967. Baseline efficiencies based on the Energy Independence and Security Act of 2007 and the International Energy Conservation Code 2009, Table 503.2.3(5) Boilers, Gas- and Oil-Fired, Minimum Efficiency Requirements. [↑](#footnote-ref-967)
968. Focus on Energy Evaluation. Business Programs: Measure Life Study. August 25, 2009. [↑](#footnote-ref-968)
969. Baseline efficiencies based on current federal standards: http://www1.eere.energy.gov/buildings/appliance\_standards/pdfs/74fr36312.pdf. [↑](#footnote-ref-969)
970. For units <300 kBtu/h, costs derived from Page E-13 of Appendix E of Residential Furnaces and Boilers Final Rule Technical Support Document:

     http://www1.eere.energy.gov/buildings/appliance\_standards/residential/fb\_tsd\_0907.html

     VEIC believes it is reasonable to assume that the cost provided from this study for an 85% unit is appropriate for units in the 85-90% AFUE range and the cost for the 91% unit can be used for 91+% units. This is based on the observation that most of the products available in the 85-90 range are in the lower end of the range, as are those units available above 91% AFUE. For units >= 300 kBtu/h costs adopted from the Northeast Energy Efficiency Partnerships Incremental Cost Study Report, Navigant, 2011. [↑](#footnote-ref-970)
971. HOURS estimates developed from data presented in "New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs", TecMarket Works, October 15, 2010, adjusted to Mid-Atlantic region using heating degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-971)
972. Energy and Demand Savings come from the ECM furnace fan motor. These motors are also available as a separate retrofit on an existing furnace. [↑](#footnote-ref-972)
973. Deemed savings from ECM Furnace Impact Assessment Report. Prepared by PA Consulting for the Wisconsin Public Service Commission 2009. Based on in depth engineering analysis and interviews taking into account the latest research on behavioral aspects of furnace fan use. [↑](#footnote-ref-973)
974. Efficiency Vermont Technical Reference User Manual No. 2010-67a. Measure Number I-A-6-a. [↑](#footnote-ref-974)
975. HOURS estimates developed from data presented in "New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs", TecMarket Works, October 15, 2010, adjusted to Mid-Atlantic region using heating degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-975)
976. Baseline efficiencies based on International Energy Conservation Code 2012, Table C403.2.3(4) Warm Air Furnaces and Combination Warm Air Furnaces/Air-Conditioning Units, Warm Air Duct Furnaces and Unit Heaters, Minimum Efficiency Requirements and International Energy Conservation Code 2015, Table C403.2.3(4) Warm Air Furnaces and Combination Warm Air Furnaces/Air-Conditioning Units, Warm Air Duct Furnaces and Unit Heaters, Minimum Efficiency Requirements. Review of GAMA shipment data indicates a more suitable market baseline is 80% AFUE. Further, pending federal standards, 10 CFR 430.32(e)(1)(i), scheduled to take effect in November 2015 will raise the baseline for non-weatherized gas furnaces to 80% AFUE. The baseline unit is non-condensing. [↑](#footnote-ref-976)
977. Costs derived from Page E-3 of Appendix E of Residential Furnaces and Boilers Final Rule Technical Support Document:

     http://www1.eere.energy.gov/buildings/appliance\_standards/residential/fb\_tsd\_0907.html [↑](#footnote-ref-977)
978. Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, "http://www.ctsavesenergy.org/files/Measure Life Report 2007.pdf" [↑](#footnote-ref-978)
979. HOURS estimates developed from data presented in "New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs", TecMarket Works, October 15, 2010, adjusted to Mid-Atlantic region using heating degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory [↑](#footnote-ref-979)
980. kWh/ton savings from "New York Standard Approach for Estimating Energy Savings from Energy Efficiency Programs", TecMarket Works, October 15, 2010, scaled based on enthalpy data from New York City and Mid-Atlantic cities from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-980)
981. Demand savings are assumed to be zero because economizer will typically not be operating during the peak period. [↑](#footnote-ref-981)
982. Navigant. 2013. Incremental Cost Study Phase Two Final Report. Burlington, MA. [↑](#footnote-ref-982)
983. General agreement among sources; Recommended value from Focus on Energy Evaluation. Business Programs: Measure Life Study. August 25, 2009. [↑](#footnote-ref-983)
984. kWh/ton savings from NY Standard Approach Model, with scaling factors based on enthalpy data from NYC and Mid-Atlantic cities. [↑](#footnote-ref-984)
985. California Public Utilities Commission. *HVAC Impact Evaluation Final Report.* January 28, 2014. [↑](#footnote-ref-985)
986. Energy Center of Wisconsin, May 2008; “Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research.” [↑](#footnote-ref-986)
987. Full load cooling hours estimated by adjusting the “Mid-Atlantic” hours from “C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011” by the full-load cooling hour estimates from the ENERGY STAR Central AC Calculator, 2013. For scaling purposes, the analysis assumes the initial Mid-Atlantic values are consistent with Baltimore, MD as suggested by the KEMA study. Because the ENERGY STAR calculator does not provide full load hours estimates for all cities of interest, a second scaling was performed using cooling degree day estimates from Typical Meteorological Year 3 (TMY3) data published by the National Renewable Energy Laboratory. [↑](#footnote-ref-987)
988. Energy Center of Wisconsin, May 2008; “Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research.” [↑](#footnote-ref-988)
989. C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. Final values are presented in Metoyer, Jarred, “Report Revision Memo,” KEMA, August 2011 [↑](#footnote-ref-989)
990. C&I Unitary HVAC Load Shape Project Final Report, KEMA, 2011. Final values are presented in Metoyer, Jarred, “Report Revision Memo,” KEMA, August 2011 [↑](#footnote-ref-990)
991. Illinois Statewide Technical Reference Manual for Energy Efficiency Version 4.0 Final February 24 2015 [↑](#footnote-ref-991)
992. GDS Associates, Inc. (2007). *Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures*. Prepared for The New England State Program Working Group; Page 1-3, Table 1. [↑](#footnote-ref-992)
993. ENERGY STAR Program Requirements Product Specification for Commercial Refrigerators

     and Freezers Eligibility Criteria Version 2.1, ENERGY STAR, January 2008. [↑](#footnote-ref-993)
994. Code of Federal Regulations, Energy Efficiency Program for Certain Commercial and Industrial Equipment, title 10, sec. 431.66 (2013). [↑](#footnote-ref-994)
995. ENERGY STAR Program Requirements Product Specification for Commercial Refrigerators

     and Freezers Eligibility Criteria Version 2.1, ENERGY STAR, January 2008. [↑](#footnote-ref-995)
996. Efficiency Vermont Technical Reference User Manual No. 2013-82.5, August 2013; Derived from Washington Electric Coop data by West Hill Energy Consultants. [↑](#footnote-ref-996)
997. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. Combined with full load hour assumptions used for efficiency measures to account for diversity of equipment usage within the peak period hours. [↑](#footnote-ref-997)
998. Unit Energy Savings (UES) Measures and Supporting Documentation, ComFreezer\_v3\_0.xlsm, October 2012, Northwest Power & Conservation Council, Regional Technical Forum [↑](#footnote-ref-998)
999. 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, “Effective/Remaining Useful Life Values”, California Public Utilities Commission, December 16, 2008. [↑](#footnote-ref-999)
1000. ENERGY STAR Program Requirements Product Specification for Commercial Refrigerators

      and Freezers Eligibility Criteria Version 2.1, ENERGY STAR, January 2008. [↑](#footnote-ref-1000)
1001. Code of Federal Regulations, Energy Efficiency Program for Certain Commercial and Industrial Equipment, title 10, sec. 431.66 (2013). [↑](#footnote-ref-1001)
1002. ENERGY STAR Program Requirements Product Specification for Commercial Refrigerators

      and Freezers Eligibility Criteria Version 2.1, ENERGY STAR, January 2008. [↑](#footnote-ref-1002)
1003. Efficiency Vermont Technical Reference User Manual No. 2013-82.5, August 2013; Derived from Washington Electric Coop data by West Hill Energy Consultants. [↑](#footnote-ref-1003)
1004. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. Combined with full load hour assumptions used for efficiency measures to account for diversity of equipment usage within the peak period hours. [↑](#footnote-ref-1004)
1005. Unit Energy Savings (UES) Measures and Supporting Documentation, ComRefrigerator\_v3.xlsm, October 2012, Northwest Power & Conservation Council, Regional Technical Forum. [↑](#footnote-ref-1005)
1006. 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, “Effective/Remaining Useful Life Values”, California Public Utilities Commission, December 16, 2008. [↑](#footnote-ref-1006)
1007. Davis Energy Group, Analysis of Standard Options for Open Case Refrigerators and Freezers, May 11, 2004. Accessed on 7/7/10 < http://www.energy.ca.gov/appliances/2003rulemaking/documents/case\_studies/CASE\_Open\_Case\_Refrig.pdf> [↑](#footnote-ref-1007)
1008. Kuiken et al, Focus on Energy Evaluation, Business Programs: Deemed Savings Manual V1.0, KEMA, March 22, 2010. [↑](#footnote-ref-1008)
1009. Effects of the Low Emissivity Shields on Performance and Power Use of a Refrigerated Display Case, Southern California Edison, August 8, 1997. Accessed on 7/7/10. <http://www.sce.com/NR/rdonlyres/2AAEFF0B-4CE5-49A5-8E2C-3CE23B81F266/0/AluminumShield\_Report.pdf>; Characterization assumes covers are deployed for six hours daily. [↑](#footnote-ref-1009)
1010. Assumed that the continuous covers are deployed at night; therefore no demand savings occur during the peak period. [↑](#footnote-ref-1010)
1011. 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, “Cost Values and Summary Documentation”, California Public Utilities Commission, December 16, 2008 <http://deeresources.com/deer0911planning/downloads/DEER2008\_Costs\_ValuesAndDocumentation\_080530Rev1.zip> [↑](#footnote-ref-1011)
1012. 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, “Effective/Remaining Useful Life Values”, California Public Utilities Commission, December 16, 2008. [↑](#footnote-ref-1012)
1013. Cadmus. 2015. *Commercial Refrigeration Loadshape Project.* Lexington, MA. [↑](#footnote-ref-1013)
1014. Ibid. [↑](#footnote-ref-1014)
1015. Ibid. [↑](#footnote-ref-1015)
1016. Ibid. Coincidence factors developed by dividing the PJM Summer Peak Savings for ASDH Controls from Table 52 by the product of the average wattage of ASDH per connected door (0.13 kW) and the Waste Heat Factor for Demand. [↑](#footnote-ref-1016)
1017. Ibid. [↑](#footnote-ref-1017)
1018. Navigant. 2015. *Incremental Cost Study Phase Four, Final Report*. Burlington, MA. [↑](#footnote-ref-1018)
1019. 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, “Effective/Remaining Useful Life Values”, California Public Utilities Commission, December 16, 2008. [↑](#footnote-ref-1019)
1020. Cadmus. 2015. *Commercial Refrigeration Loadshape Project.* Lexington, MA. [↑](#footnote-ref-1020)
1021. Ibid. [↑](#footnote-ref-1021)
1022. Ibid. [↑](#footnote-ref-1022)
1023. Ibid. [↑](#footnote-ref-1023)
1024. Ibid. [↑](#footnote-ref-1024)
1025. Ibid. Coincidence factors developed by dividing the PJM Peak Savings for EF Motors and Controls from Table 47 by the product of the average ECM wattage per rated horsepower (0.758 kW/hp) and the Waste Heat Factor for Demand. Note: the CF is greater than one because it is calculated relative to the wattage of the post-retrofit ECM motor as opposed to the existing SP motor. [↑](#footnote-ref-1025)
1026. Based on a review of the Maine, Vermont, Illinois, and Wisconsin technical reference manuals, published incremental cost estimates for this measure range from $25 to $245. Assume the median cost of $60. [↑](#footnote-ref-1026)
1027. Energy & Resource Solutions (ERS). 2005. Measure Life Study: prepared for The Massachusetts Joint Utilities [↑](#footnote-ref-1027)
1028. Cadmus. 2015. *Commercial Refrigeration Loadshape Project.* Lexington, MA. [↑](#footnote-ref-1028)
1029. Ibid. [↑](#footnote-ref-1029)
1030. Ibid. [↑](#footnote-ref-1030)
1031. Ibid. [↑](#footnote-ref-1031)
1032. Ibid. [↑](#footnote-ref-1032)
1033. Ibid. Coincidence factors developed by dividing the PJM Peak Savings for EF Motors and Controls from Table 47 by the product of the average baseline motor wattage per rated horsepower (0.758 kW/hp for ECM and 2.088 kW/hp for SP) and the Waste Heat Factor for Demand. [↑](#footnote-ref-1033)
1034. Navigant. 2015. *Incremental Cost Study Phase Four, Final Report*. Burlington, MA. [↑](#footnote-ref-1034)
1035. Ibid. Navigant’s research revealed that ON/OFF controls are typically only found in refrigeration management systems. These systems have capabilities beyond evaporator fan control, including controls for the compressor cycle, defrost cycle, door heaters, outdoor air economizer, and more. The cost of these systems is highly variable depending on capability and falls in the approximate range of $500 - $1,700. [↑](#footnote-ref-1035)
1036. Energy & Resource Solutions (ERS). 2005. Measure Life Study: prepared for The Massachusetts Joint Utilities [↑](#footnote-ref-1036)
1037. Assumes an office with 25 employees; According to 2003 ASHRAE Handbook: HVAC Applications, Office typically uses 1.0 gal/person per day.

      Assumes an 80F temperature rise based on a typical hot water holding tank temperature setpoint of 140F and 60F supply water. Actual supply water temperature will vary by season and source.

      Water heating requirement equation adopted from FEMP Federal Technology Alert: Commercial Heat Pump Water Heater, 2000. [↑](#footnote-ref-1037)
1038. Assumes an elementary school with 300 students; According to 2003 ASHRAE Handbook: HVAC Applications, Elementary School typically uses 0.6 gal/person per day of operation. Assumes 37 weeks of operation.

      Assumes an 80F temperature rise based on a typical hot water holding tank temperature setpoint of 140F and 60F supply water. Actual supply water temperature will vary by season and source.

      Water heating requirement equation adopted from FEMP Federal Technology Alert: Commercial Heat Pump Water Heater, 2000. [↑](#footnote-ref-1038)
1039. Efficiencies based on ENERGY STAR Residential Water Heaters, Final Criteria Analysis: http://www.energystar.gov/ia/partners/prod\_development/new\_specs/downloads/water\_heaters/WaterHeaterDraftCriteriaAnalysis.pdf [↑](#footnote-ref-1039)
1040. Ibid. [↑](#footnote-ref-1040)
1041. Calculated from Itron eShapes, which is 8760 hourly data by end use for Upstate New York. [↑](#footnote-ref-1041)
1042. Ibid. [↑](#footnote-ref-1042)
1043. Ibid. [↑](#footnote-ref-1043)
1044. Ibid. [↑](#footnote-ref-1044)
1045. Cost based on ENERGY STAR Residential Water Heaters, Final Criteria Analysis: http://www.energystar.gov/ia/partners/prod\_development/new\_specs/downloads/water\_heaters/WaterHeaterDraftCriteriaAnalysis.pdf [↑](#footnote-ref-1045)
1046. Vermont Energy Investment Corporation “Residential Heat Pump Water Heaters: Energy Efficiency Potential and Industry Status” November 2005. [↑](#footnote-ref-1046)
1047. Measures and Assumptions for DSM Planning (2009). Navigant Consulting. Prepared for the Ontario Energy Board. This factor is a candidate for future improvement through evaluation. [↑](#footnote-ref-1047)
1048. Engineering judgment; assumes typical supply water temperature of 70°F and a hot water storage tank temperature of 140°F. [↑](#footnote-ref-1048)
1049. Federal Standards. http://www1.eere.energy.gov/buildings/appliance\_standards/product.aspx/productid/51 [↑](#footnote-ref-1049)
1050. IECC 2006. Performance requirement for electric resistance water heaters. [↑](#footnote-ref-1050)
1051. Hours estimates based on *PG&E savings estimates, algorithms, sources* (2005). Food Service Pre-Rinse Spray Valves [↑](#footnote-ref-1051)
1052. 2008 Database for Energy-Efficiency Resources (DEER), Version 2008.2.05, “Effective/Remaining Useful Life Values”, California Public Utilities Commission, December 16, 2008. [↑](#footnote-ref-1052)
1053. U.S. EPA. 2015. ENERGY STAR® Program Requirements Product Specification for Clothes Washers Eligibility Criteria Version 7.1 [↑](#footnote-ref-1053)
1054. U.S. EPA. 2016. Savings Calculator for ENERGY STAR Qualified Appliances. Accessed March 7, 2016. http://www.energystar.gov/sites/default/files/asset/document/appliance\_calculator.xlsx [↑](#footnote-ref-1054)
1055. Ibid [↑](#footnote-ref-1055)
1056. Ibid [↑](#footnote-ref-1056)
1057. Ibid [↑](#footnote-ref-1057)
1058. Based on the average commercial clothes washer volume of all units meeting ENERGY STAR V7.1 criteria listed in the ENERGY STAR database of certified products accessed on 03/07/2016. https://www.energystar.gov/productfinder/product/certified-commercial-clothes-washers/results [↑](#footnote-ref-1058)
1059. U.S. EPA. 2016. Savings Calculator for ENERGY STAR Qualified Appliances. Accessed March 7, 2016. http://www.energystar.gov/sites/default/files/asset/document/appliance\_calculator.xlsx [↑](#footnote-ref-1059)
1060. Ibid [↑](#footnote-ref-1060)
1061. Ibid [↑](#footnote-ref-1061)
1062. Ibid [↑](#footnote-ref-1062)
1063. Metered data from Navigant Consulting “EmPOWER Maryland Draft Final Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Appliance Rebate Program.” March 21, 2014, page 36. This data applies to residential applications. In the absence of metered data specific to multifamily common area and commercial laundromat applications, this coincidence value is used as a proxy given consistency with the PJM peak definition; however, this value is likely conservatively low for commercial applications and is a candidate for update should more applicable data become available. [↑](#footnote-ref-1063)
1064. Ibid. [↑](#footnote-ref-1064)
1065. U.S. EPA. 2016. Savings Calculator for ENERGY STAR Qualified Appliances. Accessed March 7, 2016. http://www.energystar.gov/sites/default/files/asset/document/appliance\_calculator.xlsx [↑](#footnote-ref-1065)
1066. This savings estimate is based upon VEIC analysis of data gathered in audit of DC Water Facilities, MWH Global, “Energy Savings Plan, Prepared for DC Water.” Washington, D.C., 2010. See DC Water Conservation.xlsx for calculations and DC Water Conservation Energy Savings\_Final.doc for write-up. This is believed to be a reasonably proxy for the entire region. [↑](#footnote-ref-1066)
1067. U.S. EPA. 2016. Savings Calculator for ENERGY STAR Qualified Appliances. Accessed March 7, 2016. http://www.energystar.gov/sites/default/files/asset/document/appliance\_calculator.xlsx [↑](#footnote-ref-1067)
1068. Ibid [↑](#footnote-ref-1068)
1069. Most advanced power strips have one or more uncontrolled plugs that can be used for devices where a constant power connection is desired such as fax machines and wireless routers. [↑](#footnote-ref-1069)
1070. Energy & Resource Solutions. 2013. Emerging Technologies Research Report; Advanced Power Strips for Office Environments prepared for the Regional Evaluation, Measurement, and Verification Forum facilitated by the Northeast Energy Efficiency Partnerships.” Assumes savings consistent with the 20W threshold setting for the field research site (of two) demonstrating higher energy savings. ERS noted that the 20 W threshold may be unreliable due to possible inaccuracy of the threshold setting in currently available units. It is assumed that future technology improvements will reduce the significance of this issue. Further, savings from the site with higher average savings was adopted (26.9 kWh versus 4.7 kWh) acknowledging that investigations of APS savings in other jurisdictions have found significantly higher savings. For example, Northwest Power and Conservation Council, Regional Technical Forum. 2011. “Smart Power Strip Energy Savings Evaluation” found average savings of 145 kWh. [↑](#footnote-ref-1070)
1071. NYSERDA Measure Characterization for Advanced Power Strips [↑](#footnote-ref-1071)
1072. David Rogers, Power Smart Engineering, "Smart Strip Electrical Savings and Usability," October 2008 [↑](#footnote-ref-1072)
1073. Standard fryers measures >12 inches and < 18 inches wide, and have shortening capacities > 25 pounds and < 65 pounds. Large vat fryers measure > 18 inches and < 24 inches wide, and have shortening capacities > 50 pounds. [↑](#footnote-ref-1073)
1074. US EPA. December 2015. ENERGY STAR® Program Requirements Product Specification for Commercial Fryers Eligibility Criteria Version 3.0 [↑](#footnote-ref-1074)
1075. Unless otherwise noted, all default assumptions are from US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1075)
1076. No specific study of commercial kitchen equipment coincident peak demand savings is available. In the absence of this information, a simple average demand value is used: Annual energy savings divided by the total annual hours of operation. [↑](#footnote-ref-1076)
1077. Unless otherwise noted, all default assumption from US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1077)
1078. US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1078)
1079. Navigant. 2015. *Incremental Cost Study Phase Four Final Report*. Burlington, MA. [↑](#footnote-ref-1079)
1080. US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1080)
1081. US EPA. August 2003. ENERGY STAR® Program Requirements Product Specification for Commercial Steam Cookers Eligibility Criteria Version 1.2 [↑](#footnote-ref-1081)
1082. Unless otherwise noted, all default assumption from US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1082)
1083. No specific study of commercial kitchen equipment coincident peak demand savings is available. In the absence of this information, a simple average demand value is used: Annual energy savings divided by the total annual hours of operation. [↑](#footnote-ref-1083)
1084. Unless otherwise noted, all default assumption from US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1084)
1085. Unless otherwise noted, all default assumption from US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1085)
1086. US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1086)
1087. Ibid. [↑](#footnote-ref-1087)
1088. US EPA. April 2011. ENERGY STAR® Program Requirements Product Specification for Commercial Hot Food Holding Cabinets Eligibility Criteria Version 2.0. [↑](#footnote-ref-1088)
1089. Unless otherwise noted, all default assumption from US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1089)
1090. No specific study of commercial kitchen equipment coincident peak demand savings is available. In the absence of this information, a simple average demand value is used: Annual energy savings divided by the total annual hours of operation. [↑](#footnote-ref-1090)
1091. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1091)
1092. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1092)
1093. US EPA. January 2011. ENERGY STAR® Program Requirements Product Specification for Commercial Griddles Eligibility Criteria Version 1.2. [↑](#footnote-ref-1093)
1094. Unless otherwise noted, all default assumption from US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1094)
1095. No specific study of commercial kitchen equipment coincident peak demand savings is available. In the absence of this information, a simple average demand value is used: Annual energy savings divided by the total annual hours of operation. [↑](#footnote-ref-1095)
1096. Unless otherwise noted, all default assumption from US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1096)
1097. US EPA. February 2015. Savings Calculator for ENERGY STAR Certified Commercial Kitchen Equipment. <http://www.energystar.gov/buildings/sites/default/uploads/files/commercial\_kitchen\_equipment\_calculator.xlsx> [↑](#footnote-ref-1097)
1098. Ibid [↑](#footnote-ref-1098)
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1113. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014 [↑](#footnote-ref-1113)
1114. To encourage the use of building type-specific values, the assumed lighting operating hours for unknown building types have been set equal to the lowest value from the table. [↑](#footnote-ref-1114)
1115. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014 [↑](#footnote-ref-1115)
1116. C&I Lighting Load Shape Project FINAL Report, KEMA, 2011 [↑](#footnote-ref-1116)
1117. To encourage the use of building type-specific values, the assumed lighting coincidence factors for unknown building types have been set equal to the lowest values from the table. [↑](#footnote-ref-1117)
1118. EmPOWER Maryland DRAFT Final Impact Evaluation Report Evaluation Year 4 (June 1, 2012 – May 31, 2013) Commercial & Industrial Prescriptive & Small Business Programs, Navigant, March 31, 2014. Values for Washington, D.C. and Delaware assume values from Maryland, Pepco and Maryland, DPL, respectively. [↑](#footnote-ref-1118)
1119. Waste Heat Factors for “NoAC/ElecRes” estimated as at difference between “AC/ElecRes” and “AC/NonElec” plus one. [↑](#footnote-ref-1119)